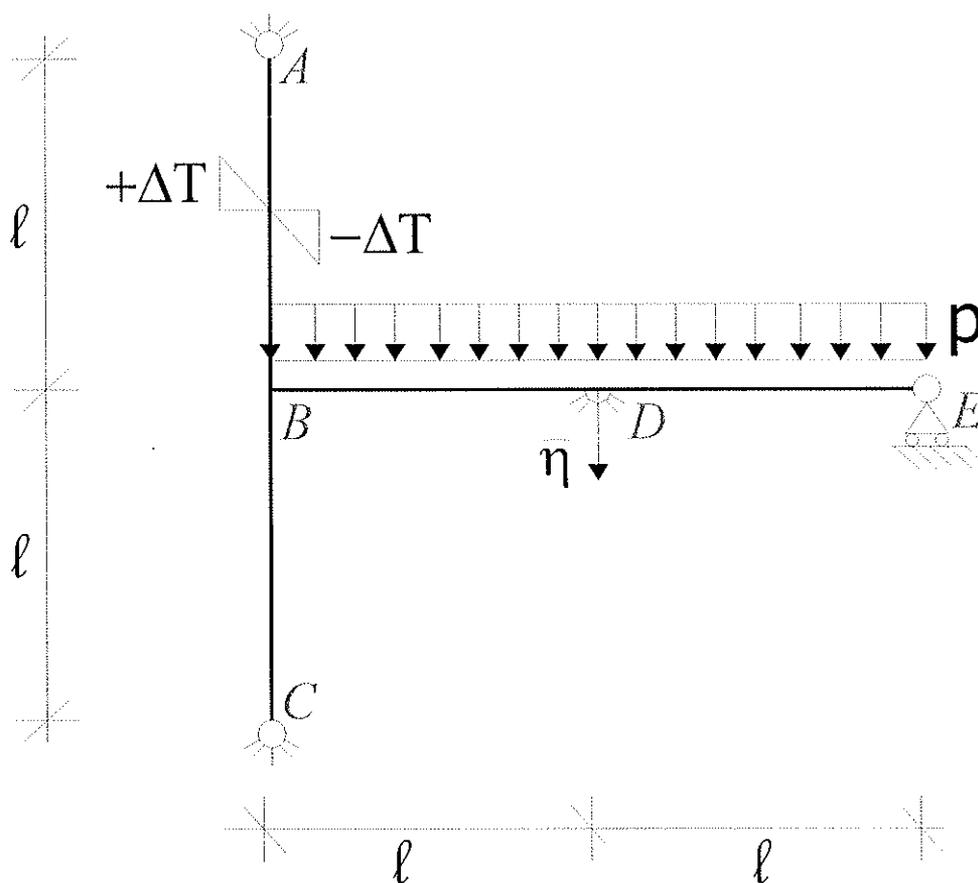


Esame di
FONDAMENTI di PROGETTAZIONE STRUTTURALE
(Corso di laurea in Ingegneria Civile N.O.)

PROVA SCRITTA
 (Fila 1)

24 – 11 – 2008



$$\frac{\alpha \Delta T}{h} = \frac{5 p \ell^2}{4 E J} ; \quad \bar{\eta} = \frac{p \ell^4}{36 E J} \quad E J = \cos t \quad E A \rightarrow \infty$$

Si richiedono:

- Grafico del Momento flettente (con il valore e la posizione dei massimi)
- Grafico del Taglio
- Grafico dell' Azione assiale
- Deformata qualitativa con posizione dei flessi
- Reazione vincolare verticale nel punto D (valore e verso)

SOLUZIONE

Sistema risolvente

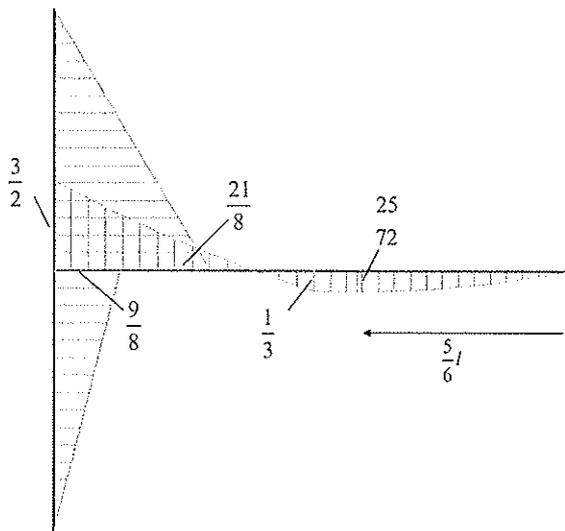
$$\begin{cases} \frac{10EJ}{\ell} \varphi_B + \frac{2EJ}{\ell} \varphi_D + \frac{7}{2} p\ell^2 = 0 \\ \frac{2EJ}{\ell} \varphi_B + \frac{7EJ}{\ell} \varphi_D - \frac{1}{8} p\ell^2 = 0 \end{cases}$$

Soluzioni

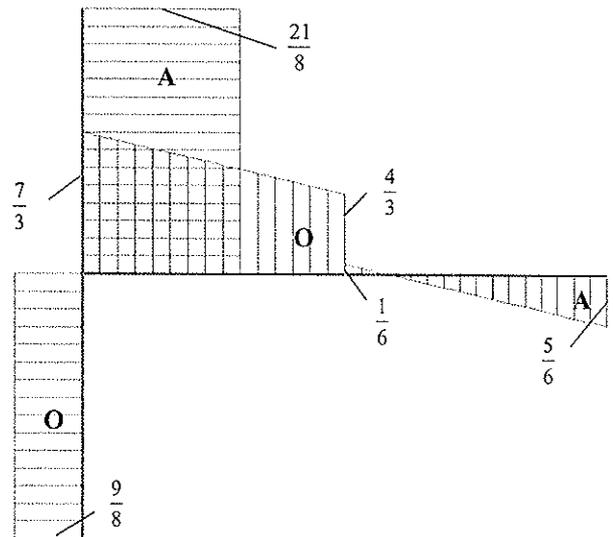
$$\begin{cases} \varphi_B = -\frac{3p\ell^3}{8EJ} \\ \varphi_D = \frac{p\ell^3}{8EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata

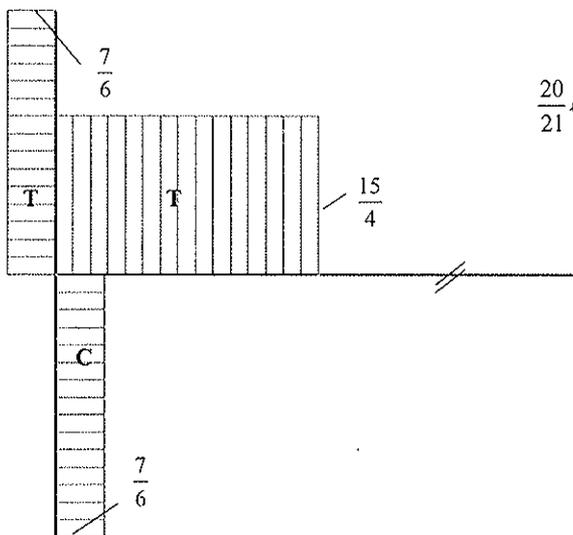
$M (\cdot p\ell^2)$



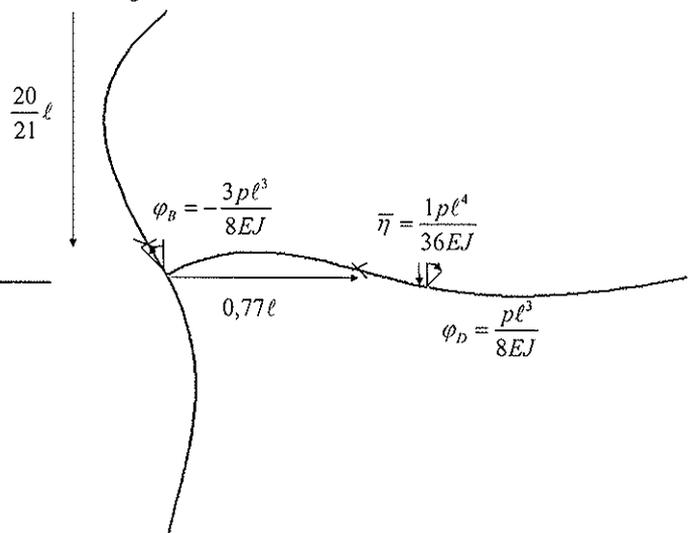
$T (\cdot p\ell)$



$N (\cdot p\ell)$



Deformata

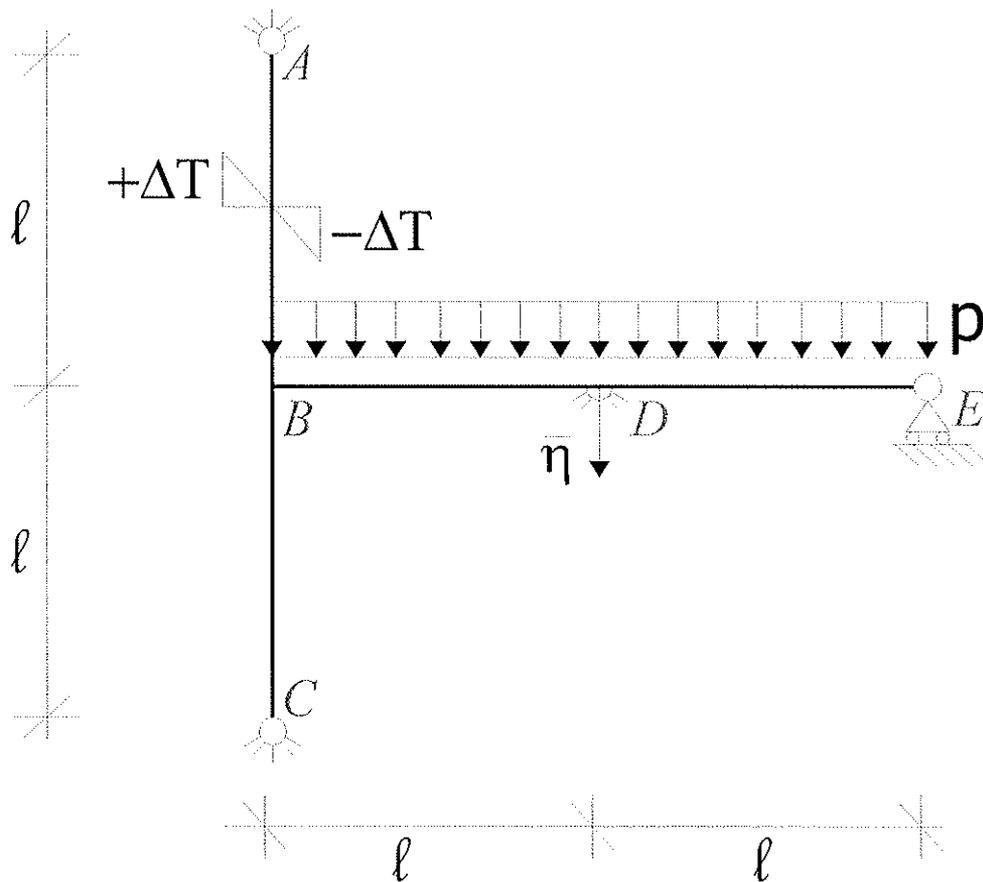


La reazione vincolare verticale nel punto D vale $V_D = \frac{7}{6} p\ell$ verso il basso

Esame di
FONDAMENTI di PROGETTAZIONE STRUTTURALE
(Corso di laurea in Ingegneria Civile N.O.)

PROVA SCRITTA
(Fila 2)

24 – 11 – 2008



$$\frac{\alpha \Delta T}{h} = \frac{p \ell^2}{8EJ}; \quad \bar{\eta} = \frac{p \ell^4}{4EJ} \quad EJ = \text{cost} \quad EA \rightarrow \infty$$

Si richiedono:

- Grafico del Momento flettente (con il valore e la posizione dei massimi)
- Grafico del Taglio
- Grafico dell' Azione assiale
- Deformata qualitativa con posizione dei flessi
- Reazione vincolare verticale nel punto D (valore e verso)

SOLUZIONE

Sistema risolvente

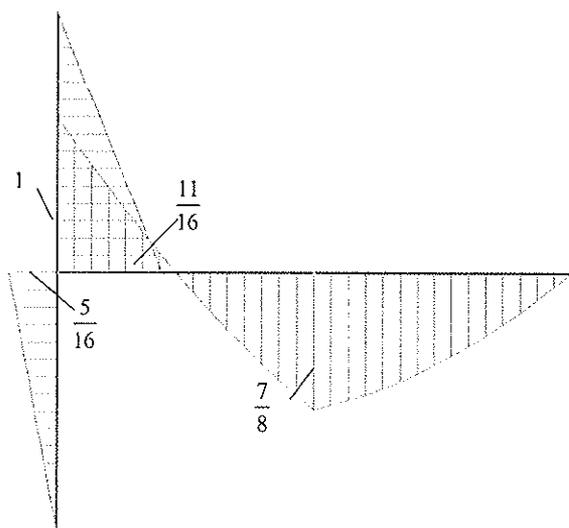
$$\begin{cases} \frac{10EJ}{\ell} \varphi_B + \frac{2EJ}{\ell} \varphi_D - \frac{29}{24} p\ell^2 = 0 \\ \frac{2EJ}{\ell} \varphi_B + \frac{7EJ}{\ell} \varphi_D - \frac{19}{24} p\ell^2 = 0 \end{cases}$$

Soluzioni

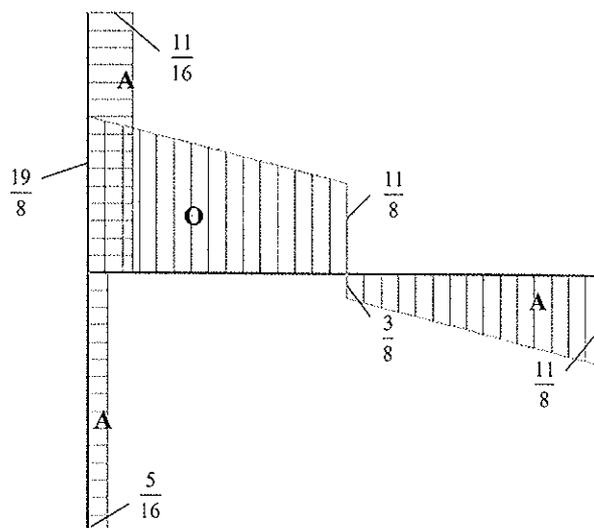
$$\begin{cases} \varphi_B = \frac{5p\ell^3}{48EJ} \\ \varphi_D = \frac{p\ell^3}{12EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata

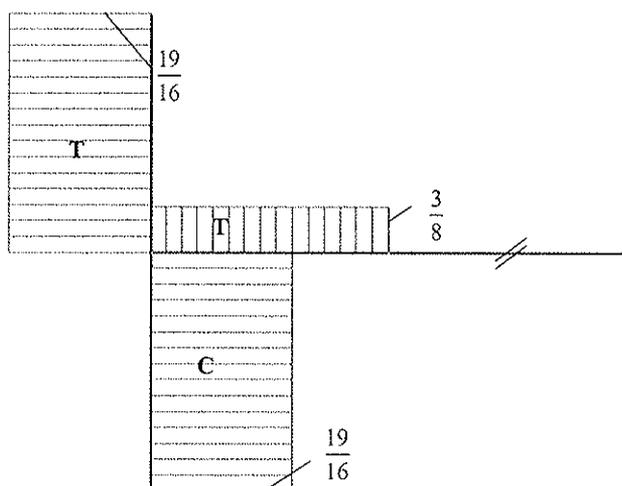
$M (\cdot p\ell^2)$



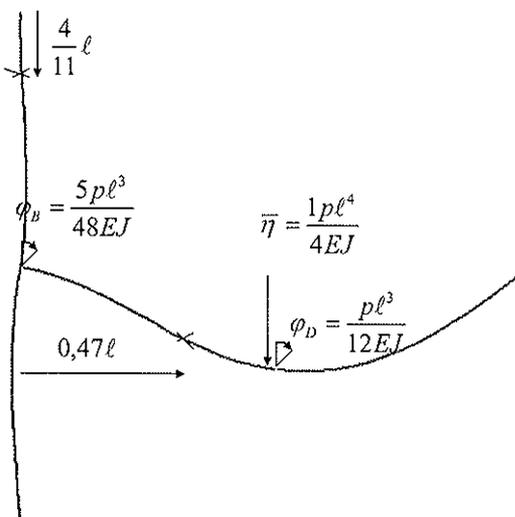
$T (\cdot p\ell)$



$N (\cdot p\ell)$

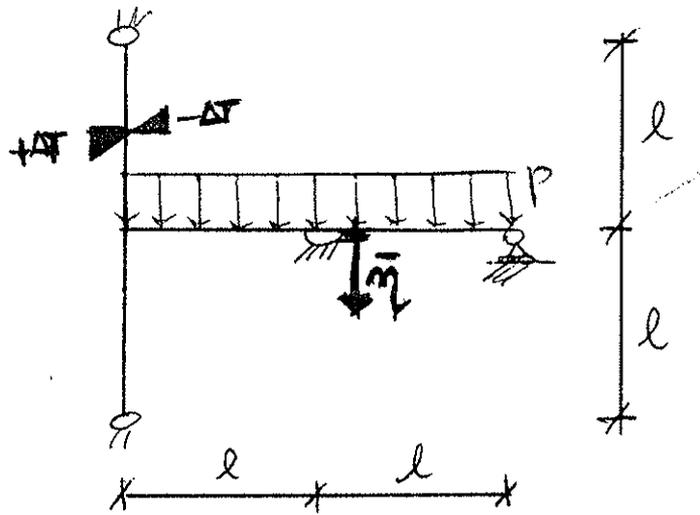


Deformata



La reazione vincolare verticale nel punto D vale $V_D = \frac{7}{4} p\ell$ verso il basso

Tema esame del 24/11/08



FILA 1

$$\bullet \frac{\alpha \Delta T}{l} = \frac{5}{4} \frac{p l^2}{EJ}$$

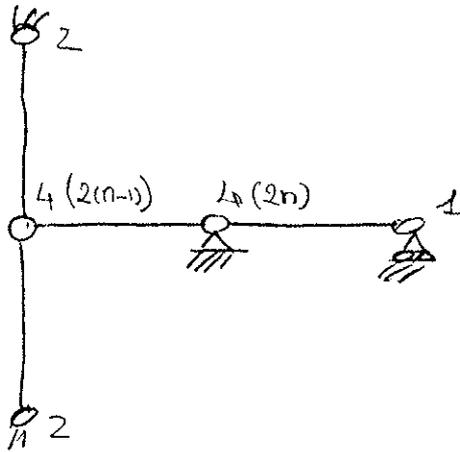
$$\bullet \bar{M} = \frac{1}{36} \frac{p l^3}{EJ}$$

FILA 2

$$\bullet \frac{\alpha \Delta T}{l} = \frac{1}{8} \frac{p l^2}{EJ}$$

$$\bullet \bar{M} = \frac{1}{4} \frac{p l^4}{EJ}$$

Si tratta di una struttura IPERSTATICA a NODI FISSI



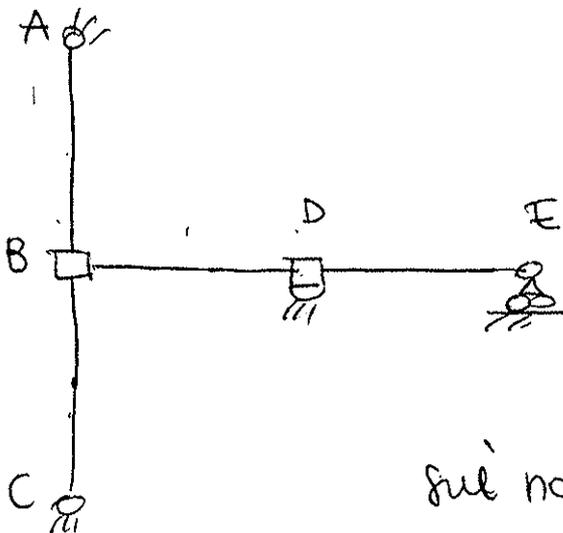
$$L = 3 \times 4 = 12$$

$$V = 2 + 2 + 1 + 4 + 4 = 13$$

IPERSTANCA

Non c'è alcun cinetismo

↳ NODI FISSI

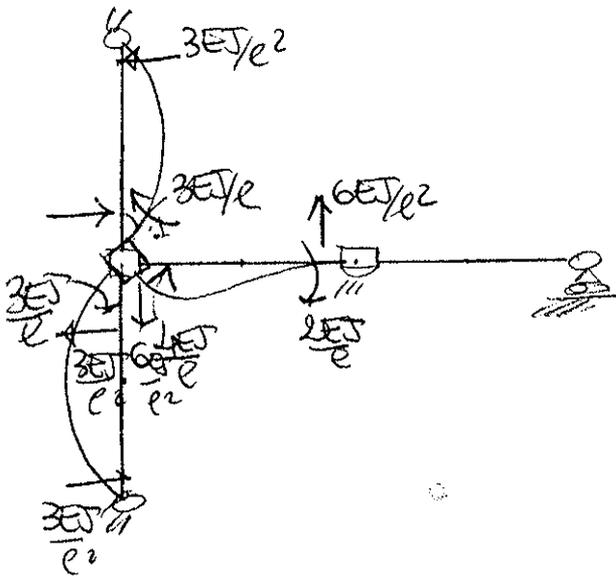


$$\left\{ \begin{aligned} M_{BB} \varphi_B + M_{BD} \varphi_D + M_{B0} &= 0 \\ M_{DB} \varphi_B + M_{DD} \varphi_D + M_{D0} &= 0 \end{aligned} \right.$$

sui nodi



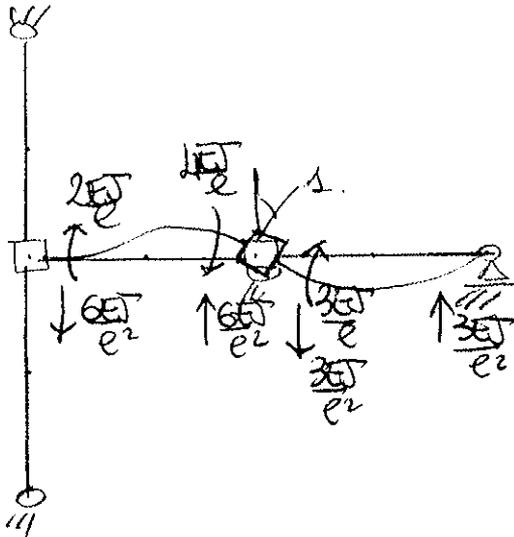
$$\varphi_B = 1$$



$$u_{BB} = \frac{10EJ}{e}$$

$$M_{DB} = \frac{2EJ}{e}$$

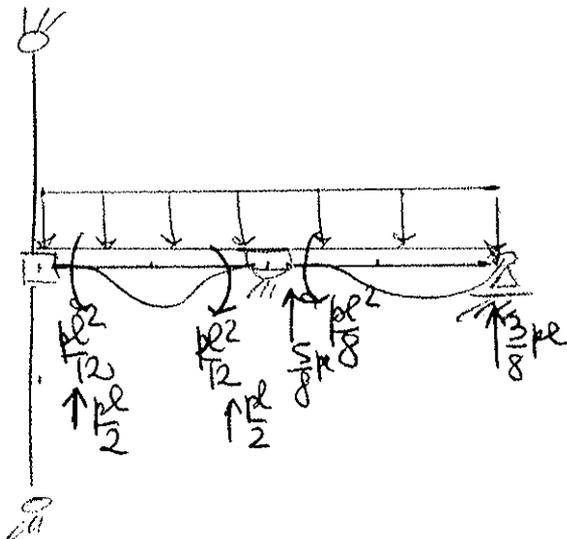
$$\varphi_D = 1$$



$$u_{DD} = \frac{7EJ}{e}$$

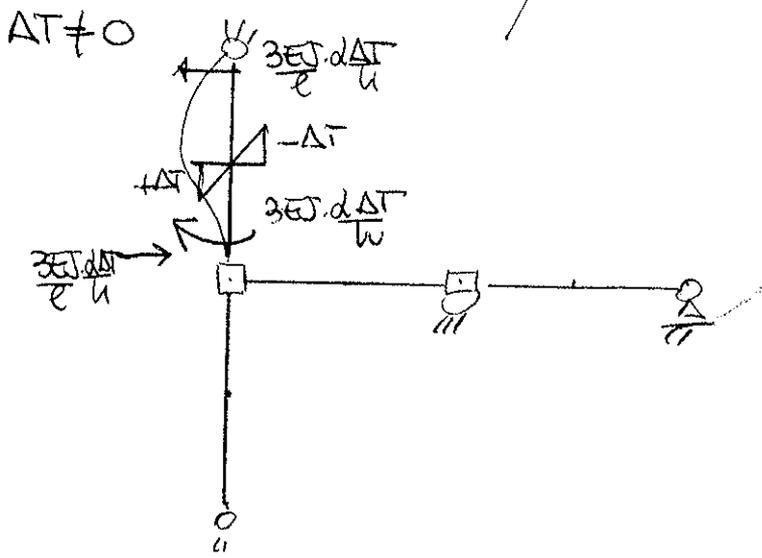
$$M_{BD} = \frac{2EJ}{e}$$

$$p \neq 0$$



$$u_{B0}^{p \neq 0} = -\frac{pe^2}{12}$$

$$u_{D0}^{p \neq 0} = \frac{pe^2}{12} - \frac{pe^2}{8} = \frac{2-3pe^2}{24} = -\frac{pe^2}{24}$$

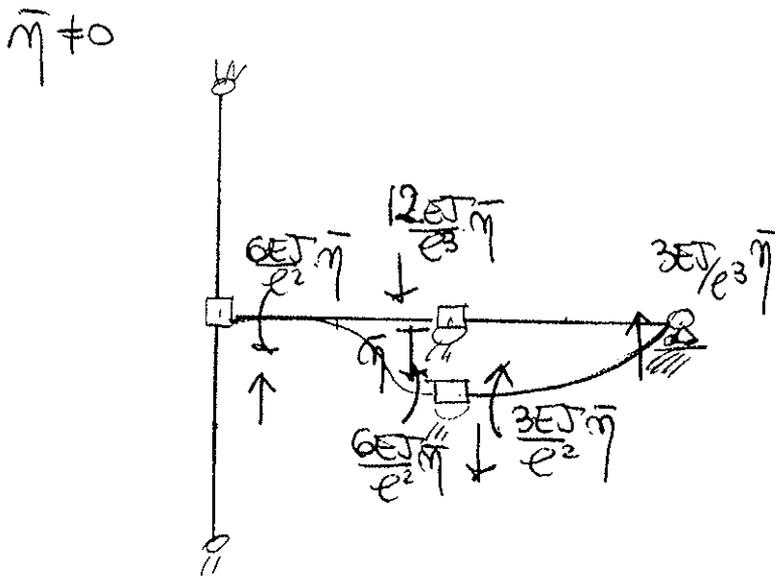


$$\Delta T \neq 0$$

$$\mu_{B0} = \frac{3EJ \cdot d\Delta T}{w}$$

$$\Delta T \neq 0$$

$$\mu_{D0} = 0$$



$$\bar{\eta} \neq 0$$

$$\mu_{B0} = -\frac{6EJ}{e^2} \bar{\eta}$$

$$\bar{\eta} \neq 0$$

$$\mu_{D0} = \left(\frac{3EJ}{e^2} - \frac{6EJ}{e^2} \right) \bar{\eta}$$

$$= -\frac{3EJ}{e^2} \bar{\eta}$$

Il sistema risolvibile diventa

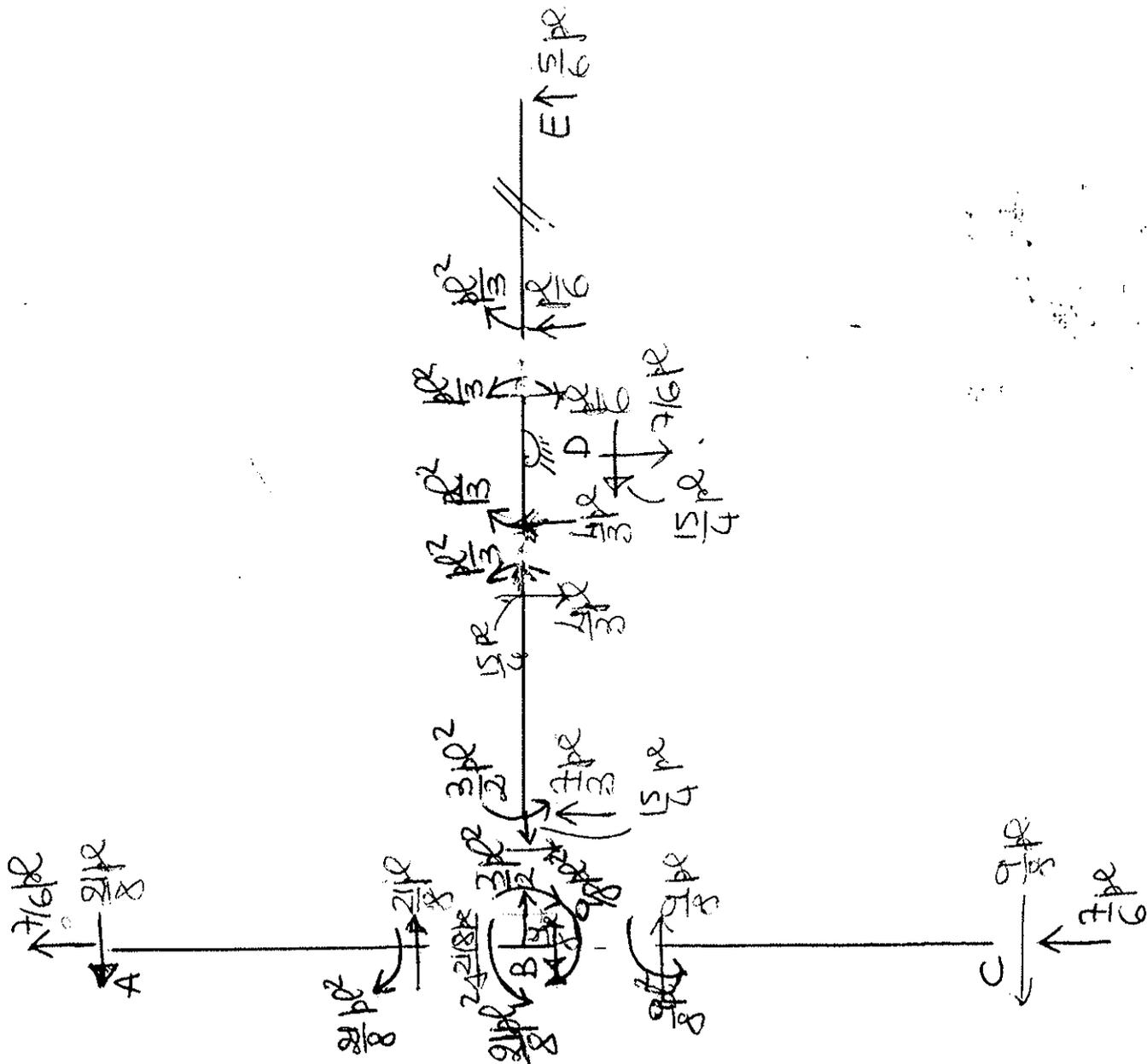
$$\begin{cases} \frac{10EJ}{e} \varphi_B + \frac{2EJ}{e} \varphi_D - \frac{pl^2}{12} + 3EJ \frac{d\Delta T}{w} - \frac{6EJ}{e^2} \bar{\eta} = 0 \\ \frac{2EJ}{e} \varphi_B + \frac{7EJ}{e} \varphi_D - \frac{pl^2}{24} - \frac{3EJ}{e^2} \bar{\eta} = 0 \end{cases}$$

$$\begin{cases} \frac{10EJ}{e} \varphi_B + \frac{2EJ}{e} \varphi_D - \frac{pl^2}{12} + 3EJ \frac{5}{4} \frac{pl^2}{EJ} - \frac{6EJ}{e^2} \cdot \frac{1}{36} \frac{pl^4}{EJ} = 0 \\ \frac{2EJ}{e} \varphi_B + \frac{7EJ}{e} \varphi_D - \frac{pl^2}{24} - \frac{3EJ}{e^2} \cdot \frac{1}{36} \frac{pl^4}{EJ} \end{cases}$$

$$\begin{cases} \frac{10EJ}{e} \varphi_B + \frac{2EJ}{e} \varphi_D + \frac{7}{2} pl^2 = 0 \\ \frac{2EJ}{e} \varphi_B + \frac{7EJ}{e} \varphi_D - \frac{1}{8} pl^2 = 0 \end{cases}$$

$$\begin{cases} \frac{10EJ}{e} \varphi_B + \frac{2EJ}{e} \varphi_D + \frac{7}{2} pl^2 = 0 \\ \frac{2EJ}{e} \varphi_B + \frac{7EJ}{e} \varphi_D - \frac{1}{8} pl^2 = 0 \end{cases}$$

$$\begin{cases} \varphi_B = -\frac{3}{8} \frac{pl^3}{EJ} \\ \varphi_D = \frac{pl^3}{8EJ} \end{cases}$$



$$T_A = \frac{3EJ}{e^2} \cdot \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{3EJ}{e} \cdot \frac{5}{4} \frac{pl^2}{EJ} = -\frac{9}{8} pl + \frac{15}{4} pl = \frac{-9+30}{8} = \frac{21}{8} pl$$

$\overline{\text{SMA}}$
 $T_B = T_A$

$$T_C = \frac{3EJ}{e^2} \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) = -\frac{9}{8} pl$$

$\overline{\text{SMA}}$
 $M_B = \frac{3EJ}{e} \cdot \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{3EJ}{e} \cdot \frac{5}{4} \frac{pl^2}{EJ} = \frac{21}{8} pl^2$

$\overline{\text{SMA}}$
 $M_B^{\text{SMA}} = \frac{3EJ}{e} \cdot \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) = -\frac{9}{8} pl^2$

$$M_B^{\text{dx}} = \frac{4EJ}{e} \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{2EJ}{e} \cdot \frac{pl^3}{8EJ} - \frac{pl^2}{12} - \frac{6EJ}{e^2} \cdot \frac{1}{36} \frac{pl^4}{EJ} =$$

$$= -\frac{3}{2} pl^2 + \frac{pl^2}{4} - \frac{pl^2}{12} - \frac{1}{6} pl^2 = \frac{-18+3-1-2}{12} = -\frac{18}{12} pl^2$$

Equilibrio al nodo $\frac{21}{8} pl^2 - \frac{3}{2} pl^2 - \frac{9}{8} pl^2 = 0$

$$T_B^{\text{dx}} = \frac{6EJ}{e^2} \cdot \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{6EJ}{e^2} \cdot \left(\frac{pl^3}{8EJ}\right) - \frac{12EJ}{e^3} \cdot \frac{1}{36} \frac{pl^4}{EJ} = \left(-\frac{9}{4} + \frac{3}{4} - \frac{1}{3} - \frac{1}{2}\right)$$

$$= \frac{-27+9-4-6}{12} = -\frac{28}{12}$$

$$M_D^{\text{dx}} = \frac{2EJ}{e} \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{4EJ}{e} \left(\frac{pl^3}{8EJ}\right) + \frac{pl^2}{12} - \frac{6EJ}{e^2} \left(\frac{1}{36} \frac{pl^4}{EJ}\right) =$$

$$= \left(-\frac{3}{4} + \frac{1}{2} + \frac{1}{12} - \frac{1}{6}\right) pl^2 = \frac{-9+6+1-2}{12} pl^2 = \frac{pl^2}{3}$$

$$M_B^{\text{dx}} = \frac{3EJ}{e} \left(\frac{pl^3}{8EJ}\right) - \frac{pl^2}{8} + \frac{3EJ}{e^2} \left(\frac{1}{36} \frac{pl^4}{EJ}\right) = \left(\frac{3}{8} - \frac{1}{8} + \frac{1}{12}\right) pl^2 = \frac{9-3+2}{24} pl^2 = \frac{pl^2}{3}$$

$$T_D^{\text{dx}} = \frac{6EJ}{e^2} \cdot \left(-\frac{3}{8} \frac{pl^3}{EJ}\right) + \frac{6EJ}{e^2} \left(\frac{pl^3}{8EJ}\right) + \frac{pl}{2} - \frac{12EJ}{e^3} \cdot \frac{1}{36} \frac{pl^4}{EJ} = \frac{9}{4} + \frac{3}{4} + \frac{1}{2} - \frac{1}{3} = \frac{-16}{12} pl - \frac{4}{3}$$

$$T_D^{\text{dx}} = \frac{3EJ}{e^2} \left(\frac{pl^3}{8EJ}\right) - \frac{5}{8} pl + \frac{3EJ}{e^3} \cdot \frac{1}{36} \frac{pl^4}{EJ} = \frac{3}{8} - \frac{5}{8} + \frac{1}{12} = \frac{9-15+2}{24} = -\frac{pl}{6}$$

$$T_E = \frac{3EJ}{e^2} \left(\frac{pl^3}{8EJ}\right) + \frac{3}{8} pl + \frac{3EJ}{e^3} \cdot \frac{1}{36} \frac{pl^4}{EJ} = \left(\frac{3}{8} + \frac{3}{8} + \frac{1}{12}\right) pl = \frac{9+9+2}{24} = \frac{5}{6} pl$$

Trova i tagli determino le azioni annali

$$N_{BD} = N_{DE} = \frac{21}{8}Pl + \frac{9}{8}Pl = \frac{15}{4}Pl$$

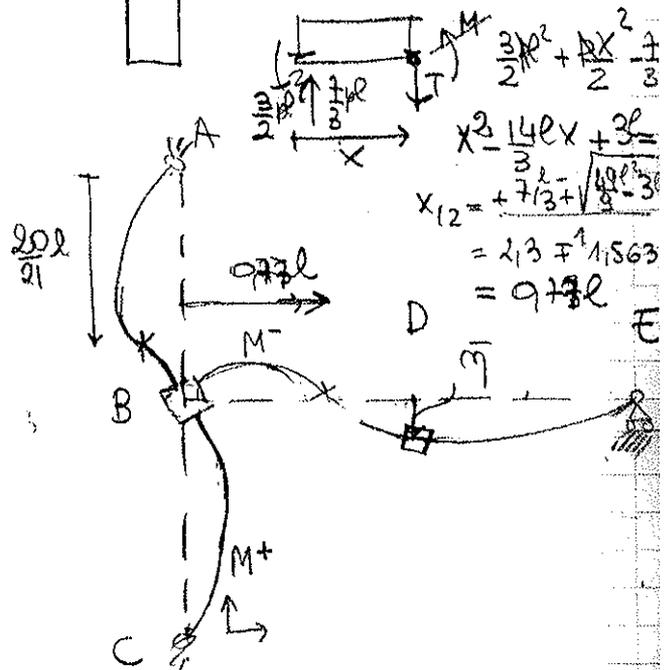
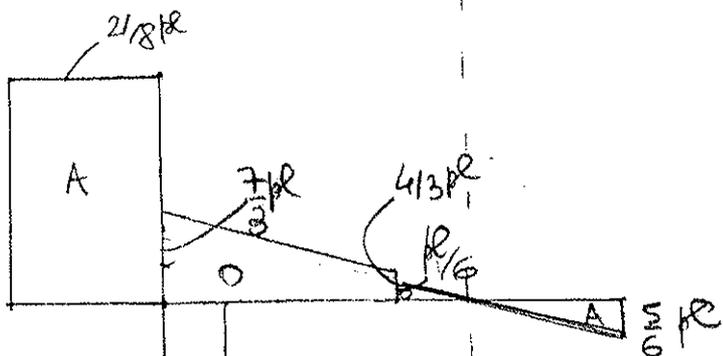
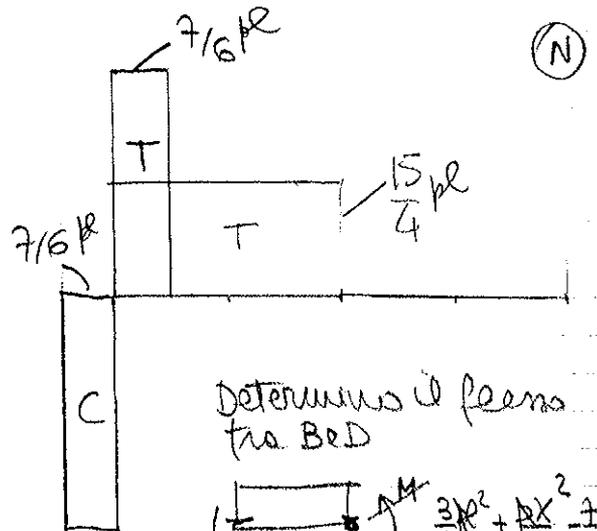
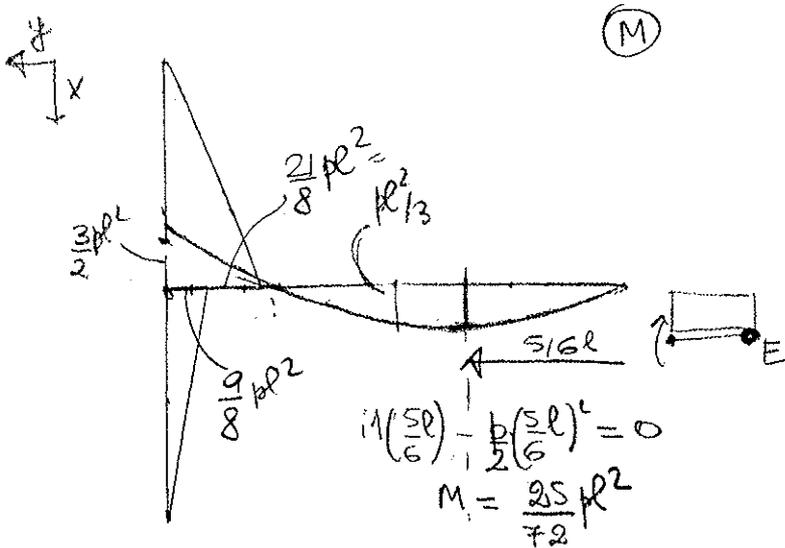
$$N_{AB} = N_{BC} = \frac{7}{3} \cdot \frac{1}{2}Pl = \frac{7}{6}Pl$$

Verifica equilibrio globale

$$\uparrow + \left(\frac{7}{6} + \frac{7}{6} - \frac{7}{6} + \frac{5}{6} - 2 \right) = 0 \quad \text{OK!}$$

$$\rightarrow \left(-\frac{21}{8} - \frac{9}{8} + \frac{15}{4} \right) Pl = 0 \quad \text{OK!}$$

$$\curvearrow + \frac{21}{8}Pl^2 - \frac{9}{8}Pl^2 - 2Pl^2 - \frac{7}{6}Pl^2 + \frac{5}{6}Pl \cdot 2l = \frac{21-9-16+4}{8}Pl^2 = 0 \quad \text{OK!}$$



Determino il punto tra B e D

$$\frac{3}{2}Pl + \frac{Px^2}{2} - \frac{7}{6}Pl$$

$$x^2 - \frac{14}{3}x + \frac{3l}{Pl} = 0$$

$$x_{1,2} = \frac{14l}{3} \pm \sqrt{\frac{196l^2}{9} - \frac{12l^2}{Pl}}$$

$$= 2,3 \neq 1,563$$

$$= 9,79l$$

Det. punto tra A e B

$$M(x) + \frac{21}{8}Plx = 0 \quad M(x) = -\frac{21}{8}Plx$$

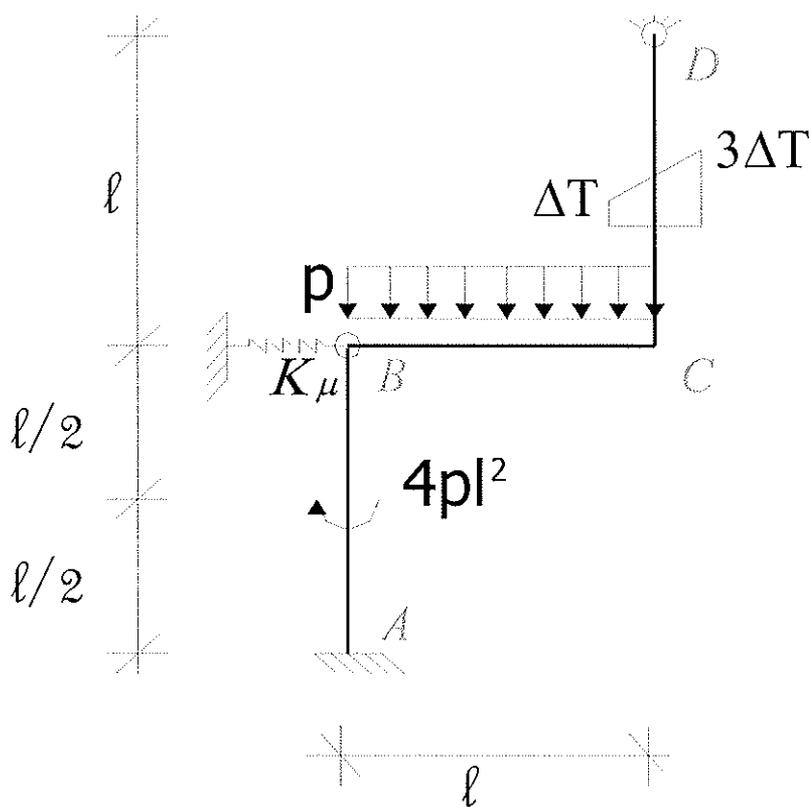
$$y''(x) = -\left[\frac{21}{8}Plx \right] = -\frac{21}{8}Pl$$

$$\frac{21}{8}x - \frac{5}{2}l \geq 0 \quad x \geq \frac{20l}{21}$$

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TECNICA DELLE COSTRUZIONI
(Corso di laurea in Ingegneria Civile e Meccanica V.O.)

PROVA SCRITTA
 (Fila 1)

18 – 12 – 2008



$$h = \frac{l}{12}; \quad \frac{\alpha \Delta T}{h} = \frac{p l^2}{4 E J}; \quad k_{\mu} = \frac{69 E J}{2 l^3} \quad E J = \cos t \quad E A \rightarrow \infty$$

Si richiedono:

- Grafico del Momento flettente (con il valore e la posizione dei massimi)
- Grafico del Taglio
- Grafico dell' Azione assiale
- Deformata qualitativa con posizione dei flessi
- Direzione e verso della forza sulla molla in B

SOLUZIONE

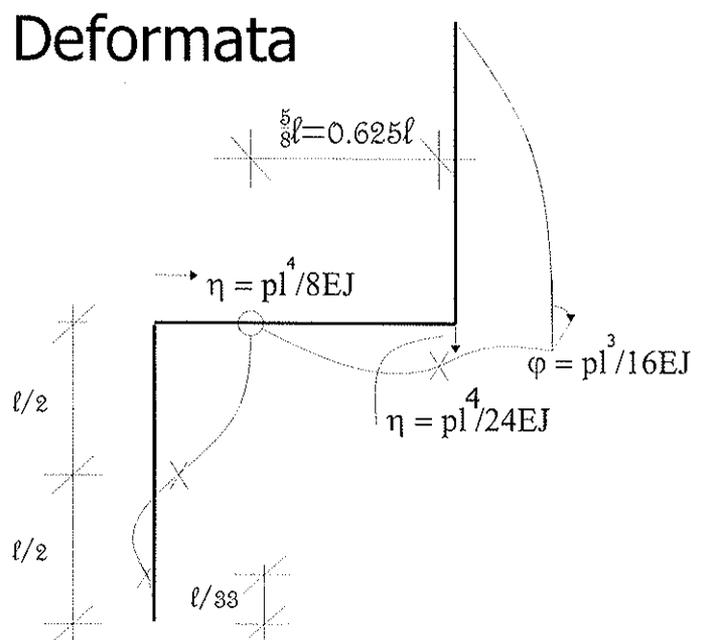
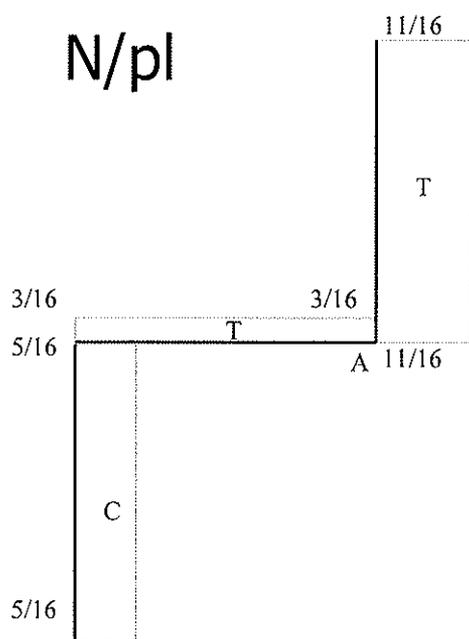
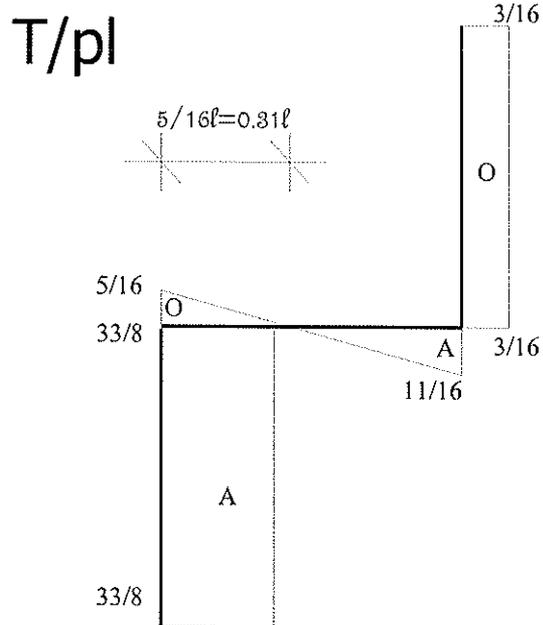
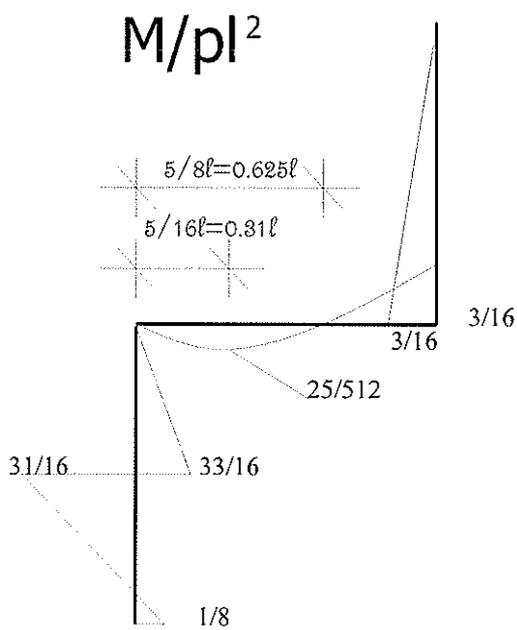
Sistema risolvente

$$\begin{cases} \frac{6EJ}{l} \varphi_c + \frac{3EJ}{l^2} \eta_c - \frac{3}{4} p l^2 = 0 \\ -\frac{3EJ}{l^2} \varphi_c - \frac{81EJ}{2l^3} \eta_c + \frac{21}{4} p l = 0 \end{cases}$$

Soluzioni

$$\begin{cases} \varphi_c = \frac{1 p l^3}{16 E J} \\ \eta_c = \frac{1 p l^4}{8 E J} \end{cases}$$

Diagrammi delle azioni interne e Deformata

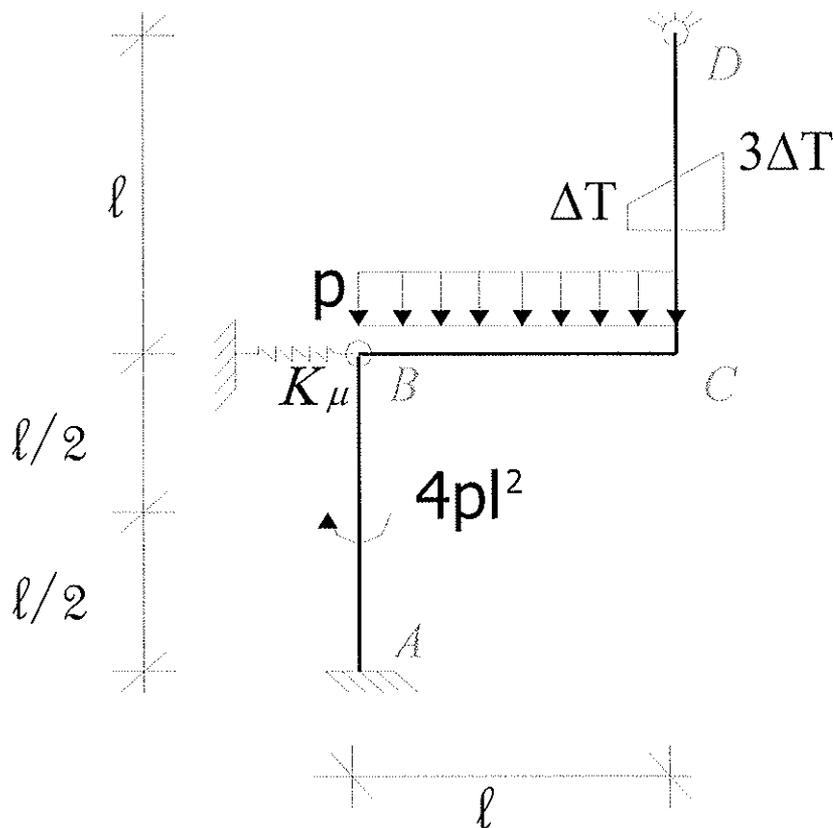


La forza sulla molla vale $\frac{69}{16} pl$ ed è diretta verso destra.

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PROVA SCRITTA
 (Fila 2)

18 – 12 – 2008



$$h = \frac{l}{8}; \quad \frac{\alpha \Delta T}{h} = \frac{p l^2}{6 E J}; \quad k_{\mu} = \frac{29 E J}{2 l^3} \quad E J = \text{cost} \quad E A \rightarrow \infty$$

Si richiedono:

- Grafico del Momento flettente (con il valore e la posizione dei massimi)
- Grafico del Taglio
- Grafico dell' Azione assiale
- Deformata qualitativa con posizione dei flessi
- Direzione e verso della forza sulla molla in B

SOLUZIONE

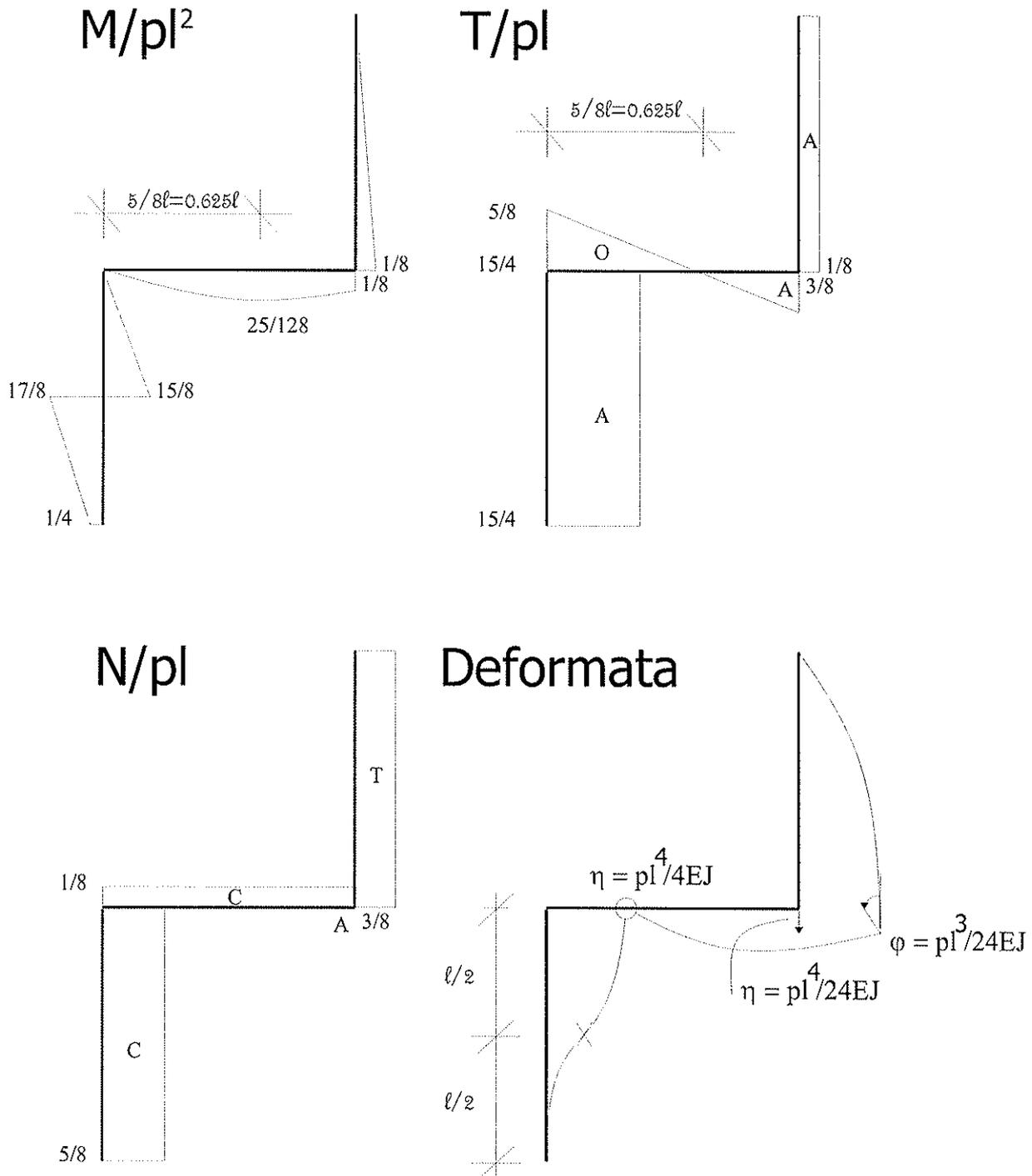
Sistema risolvente

$$\begin{cases} \frac{6EJ}{l} \varphi_c + \frac{3EJ}{l^2} \eta_c - \frac{1}{2} p l^2 = 0 \\ -\frac{3EJ}{l^2} \varphi_c - \frac{81EJ}{2l^3} \eta_c + 5pl = 0 \end{cases}$$

Soluzioni

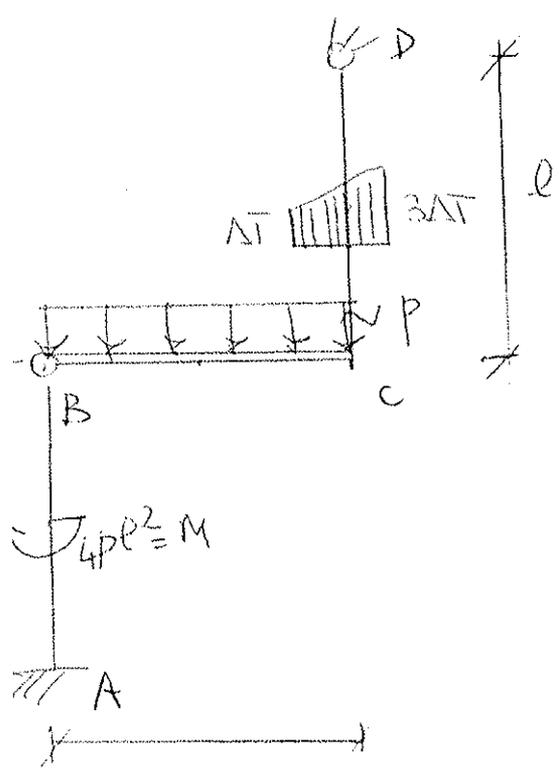
$$\begin{cases} \varphi_c = -\frac{pl^3}{24EJ} \\ \eta_c = \frac{pl^4}{4EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata



La forza sulla molla vale $29/8 pl$ ed è diretta verso destra.

FILA 1



$$\frac{\Delta T}{k} = \frac{1}{4} \frac{pl^2}{EJ} \quad h = \frac{1}{12}$$

$$\frac{\Delta T}{l} = \frac{1}{48} \frac{pl^2}{EJ}$$

$$K = \frac{69}{2} \frac{EJ}{l^3}$$

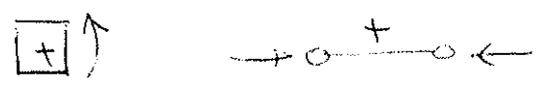
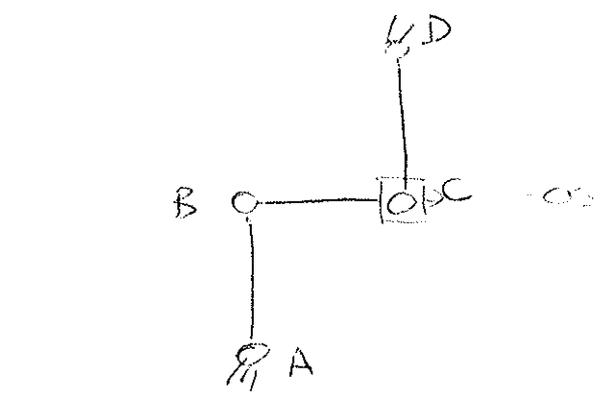
FILA 2

$$\frac{\Delta T}{k} = \frac{1}{6} \frac{pl^2}{EJ} \quad h = \frac{1}{8}$$

$$\frac{\Delta T}{l} = \frac{1}{48} \frac{pl^2}{EJ}$$

$$K = \frac{24}{2} \frac{EJ}{l^3}$$

Si tratta di una struttura
IPERSTATICA a nodi
SPOSTABILI

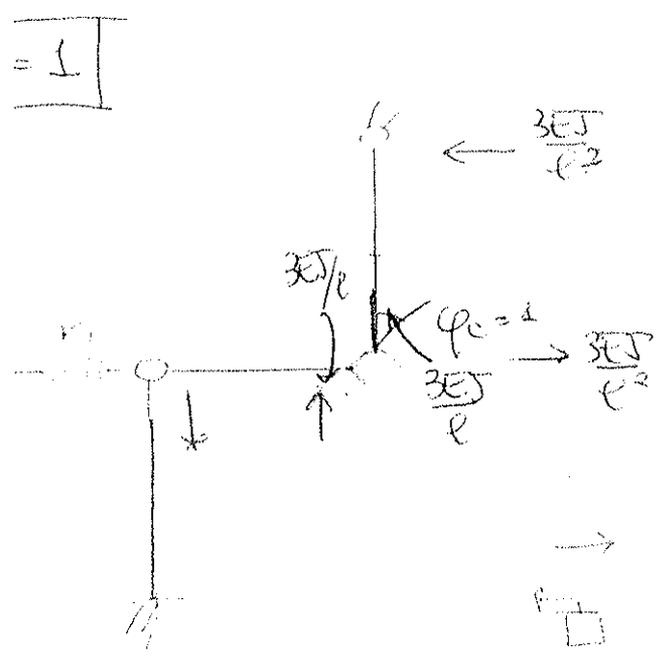


$$u_{cc} \varphi_c + u_{cm} \eta + u_{co} = 0$$

$$h_{cm} \varphi_c + h_{m\eta} \eta + h_{mo} = 0$$

$$u_{cc} = \frac{6EJ}{l}$$

$$h_{cm} = -\frac{3EJ}{l^2}$$



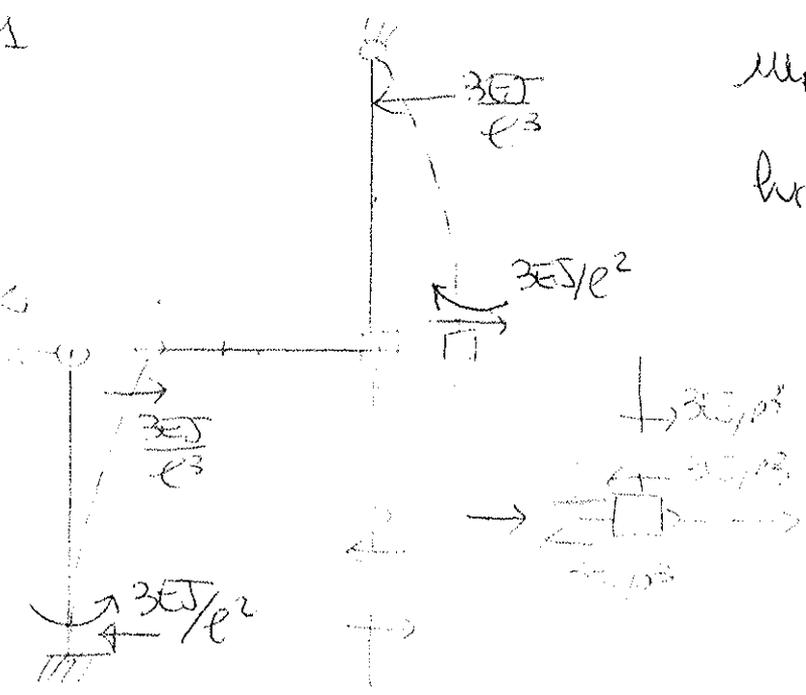
1

2

3

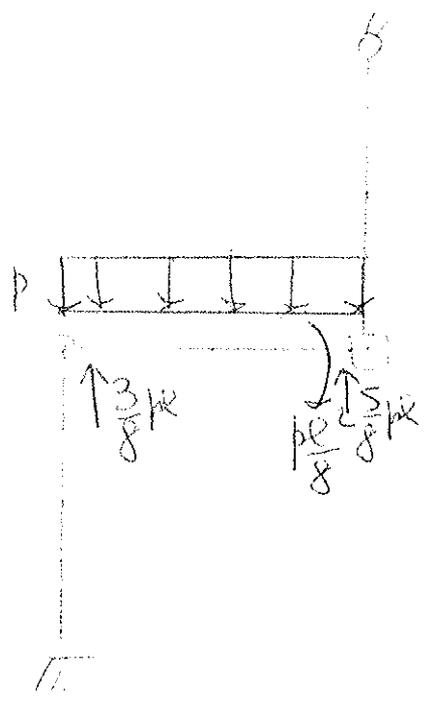
$$M_{eq} = 3EJ/l^2$$

$$l_{mp} = \left[\frac{3EJ}{e^3} \cdot 2 + kv \right]$$

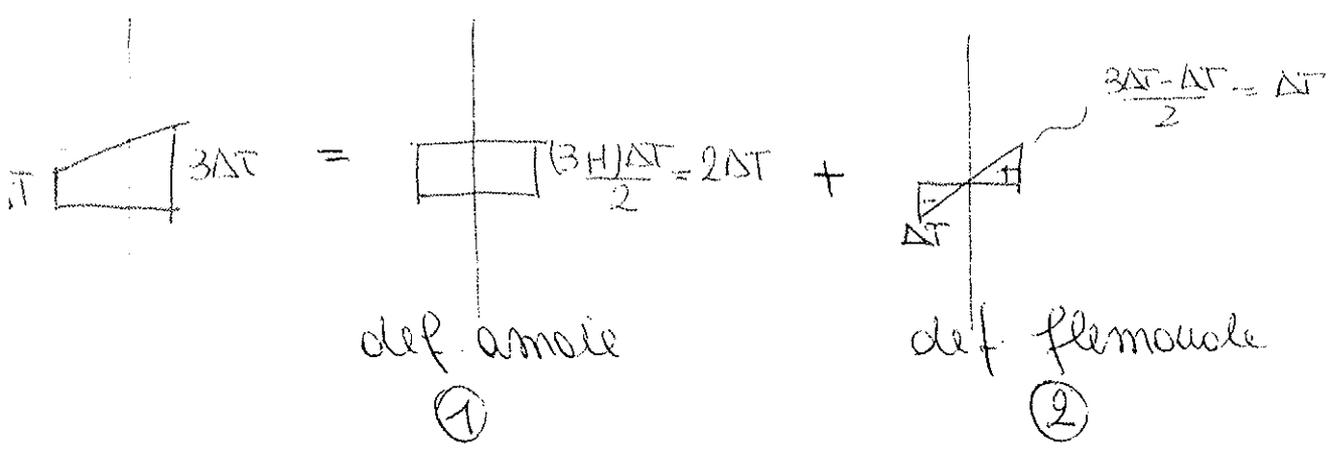


$$M_{co}^{p \neq 0} = \frac{pl^2}{8}$$

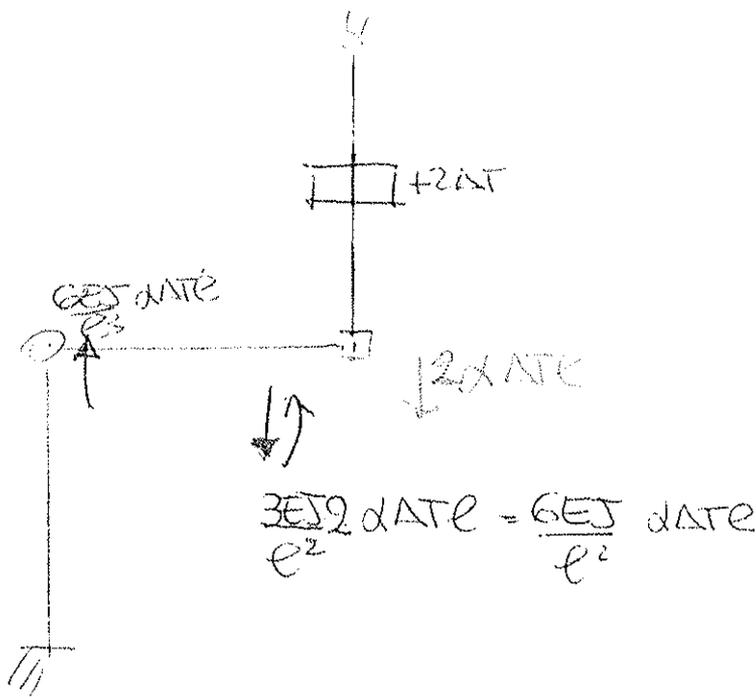
$$l_{mp}^{p \neq 0} = 0$$



$\Gamma \neq 0$ Ci sono 2 contributi



①

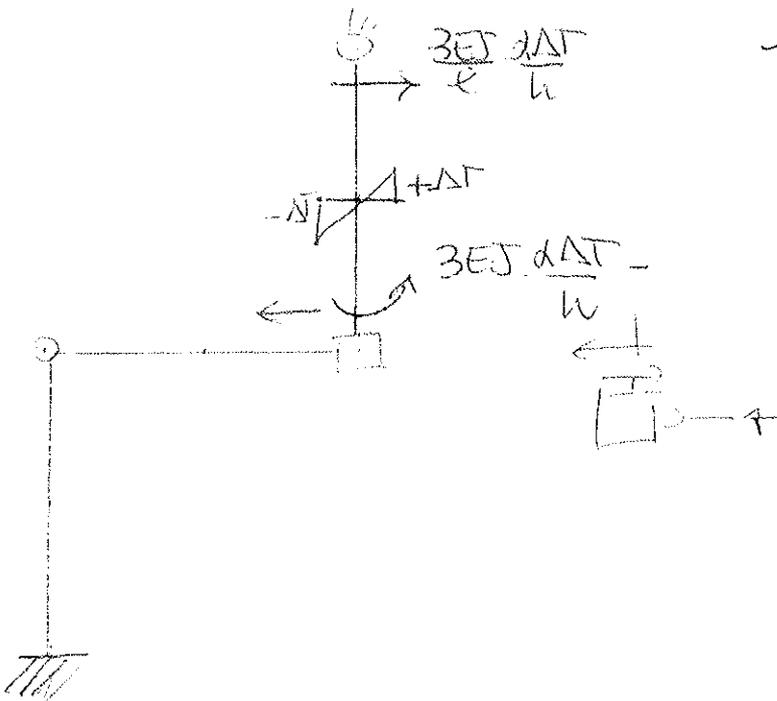


$$\Delta T_1$$

$$M_{CO} = -\frac{6EJ}{l^2} \Delta T l$$

$$l_{mp0} = 0$$

②

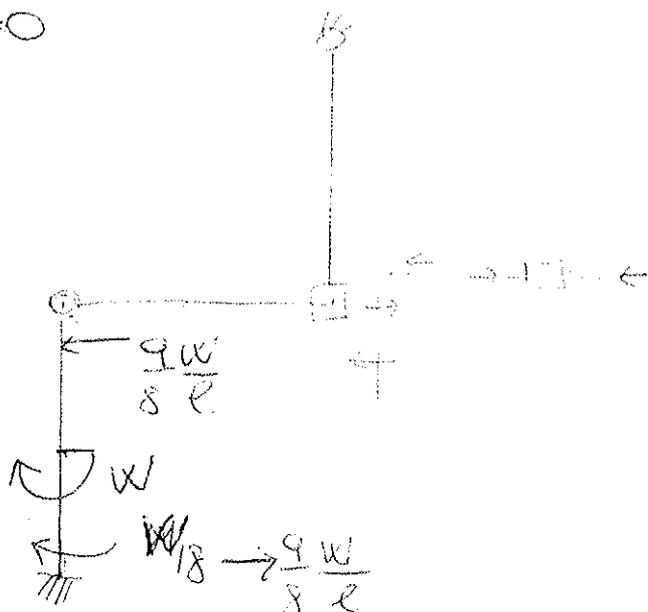


$$\Delta T_2$$

$$M_{CO} = -3EJ \frac{\Delta T}{l}$$

$$l_{mp0} = \frac{3EJ}{e} \frac{\Delta T}{l}$$

$W \neq 0$



$$M_{CO}^W \neq 0$$

$$l_{mp0}^W = \frac{9}{8} \frac{W}{l}$$

(*) Si imposta il sistema per la FILA 1
 (analogamente si procede per la FILA 2)

$$\begin{cases} \frac{6EJ}{e} \varphi_c + \frac{3EJ}{e^2} \eta + \frac{pl^2}{8} - \frac{6EJ}{e^2} \frac{d\Delta T}{e} - 3EJ \frac{d\Delta T}{l} = 0 \\ -\frac{3EJ}{e^2} \varphi_c - \left(\frac{3EJ}{e^3} l + k \right) \eta + \frac{3EJ}{e} \frac{d\Delta T}{l} + \frac{9}{8} \frac{W}{e} = 0 \end{cases}$$

(*)

$$\begin{cases} \frac{6EJ}{e^2} \varphi_c + \frac{3EJ}{e^2} \eta + \frac{pl^2}{8} - \frac{6EJ}{e^2} \frac{1}{48EJ} pl^4 - 3EJ \frac{pl^2}{4EJ} = 0 \\ -\frac{3EJ}{e^2} \varphi_c - \left(\frac{6EJ}{e^3} + \frac{69EJ}{2e^3} \right) \eta + \frac{3EJ}{8} \frac{pl^2}{4EJ} + \frac{9}{8} \frac{pl^2}{e} = 0 \end{cases}$$

$$\begin{cases} \frac{6EJ}{e} \varphi_c + \frac{3EJ}{e^2} \eta = \frac{3}{4} pl^2 = 0 \\ -\frac{3EJ}{e^2} \varphi_c - \frac{81EJ}{2e^3} \eta + \frac{21}{4} pl = 0 \end{cases} \rightarrow \begin{cases} \varphi_c = \frac{1}{16} \frac{pl^3}{EJ} \quad \text{FILA} \\ \eta = \frac{1}{8} \frac{pl^4}{EJ} \end{cases}$$

N.B. Fila 2

$$\begin{cases} \frac{6EJ}{e} \varphi_c + \frac{3EJ}{e^2} \eta - \frac{1}{2} pl^2 = 0 \\ -\frac{3EJ}{e^2} \varphi_c - \frac{41EJ}{2e^3} \eta + 5pl^2 = 0 \end{cases} \begin{cases} \varphi_c = -\frac{pl^3}{24} \\ \eta = \frac{pl^4}{4EJ} \end{cases}$$

APRO la STRUTTURA e valuto i momenti, i tagli e le azioni omologhe (per la FILA 1, analogamente fila 2)

↪

$$M_A = \frac{3EJ}{e^2} \left(\frac{pl^4}{8EJ} \right) - \frac{4pl^2}{8} = -\frac{pl^2}{8}$$

↳

$$T_A = -\frac{3EJ}{e^3} \left(\frac{pl^4}{8EJ} \right) + \frac{9pl^2}{8e} = \frac{33}{8} pl \rightarrow N_{BC} = \frac{69}{16} - \frac{33}{8} = \frac{3}{16} pl$$

↑

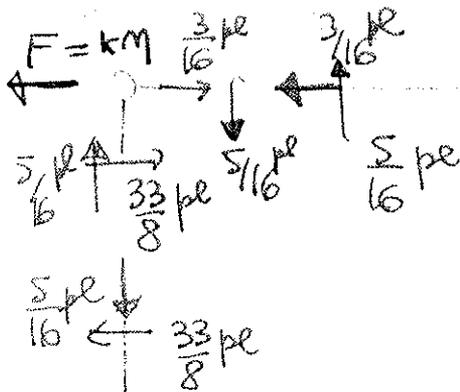
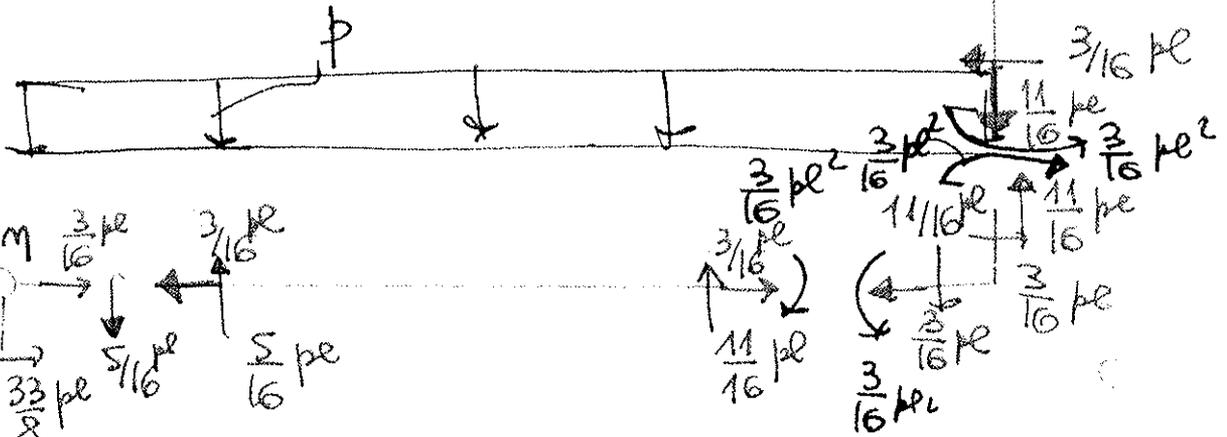
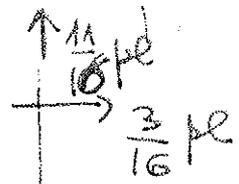
$$T_B = -\frac{3EJ}{e^2} \left(\frac{pl^4}{16EJ} \right) + \frac{3}{8} pl + \frac{6EJ}{e^3} \left(\frac{pl^4}{48EJ} \right) = \frac{5}{16} pl$$

↑

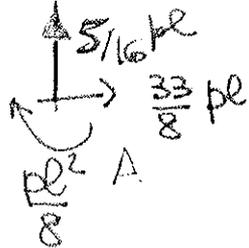
$$T_C = \frac{3EJ}{e^2} \left(\frac{pl^3}{16EJ} \right) + \frac{5}{8} pl - \frac{6}{48} pl = \frac{11}{16} pl \quad \dots \dots \text{u.2 pagine successive}$$

$$F = k \cdot \eta = \frac{69 EJ}{2 l^3} \cdot \frac{1}{8} p l^4 = \frac{69}{16} p l$$

diretta verso dx.



$$\curvearrowleft 4pl^2$$



VERIFICA degli EQUILIBRI

$$\uparrow + \quad \frac{5}{16} pl + \frac{11}{16} pl - pl = 0 \quad \text{ok!}$$

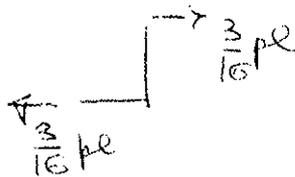
$$\rightarrow + \quad \frac{33}{8} pl - \frac{69}{16} pl + \frac{3}{16} pl = 0 \quad \text{ok!}$$

$$\curvearrowleft + \quad -4pl^2 + \frac{69}{16} pl \cdot l - \frac{pl \cdot l}{2} - \frac{3}{16} pl \cdot 2l - \frac{pl^2}{8} + \frac{11pl^2}{16} = 0$$

$$\frac{-64 + 69 - 8 - 6 - 2 + 11}{16} = 0 \quad \text{ok!}$$

$$T_c = -\frac{3EJ}{l^2} \left(\frac{1}{16} \frac{pl^3}{EJ} \right) - \frac{3EJ}{l^3} \left(\frac{pl^4}{8EJ} \right) + \frac{3EJ}{l} \cdot \left(\frac{1}{4} \frac{pl^2}{EJ} \right) = \frac{3}{16} pl$$

ok! Equilibrio al nodo c

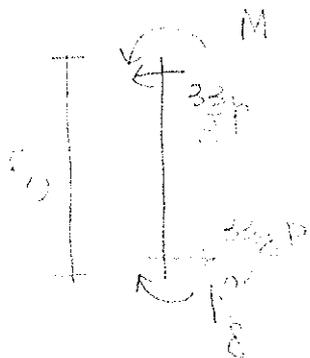


$$T_D = T_c \text{ sopra}$$

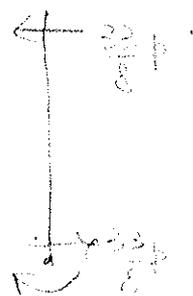
$$M_c^{sx+2} = \frac{3EJ}{l} \cdot \frac{1}{16} \frac{pl^3}{EJ} + \frac{pl^2}{8} - \frac{6EJ}{l^2} \cdot \frac{pl^4}{48EJ} = \frac{3}{16} pl^2$$

$$M_c^{dx+4} = \frac{3EJ}{l} \cdot \frac{1}{16} \frac{pl^3}{EJ} + \frac{3EJ}{l^2} \cdot \frac{pl^4}{8EJ} - \frac{3EJ}{4EJ} \cdot \frac{pl^2}{l} = -\frac{3}{16} pl^2$$

Asta AB (cambio m. dei momenti)



$$M = \frac{pl^2}{8} - \frac{3}{8} \frac{pl^2}{l} = -\frac{51}{16} pl$$

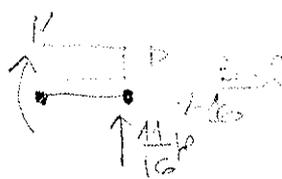
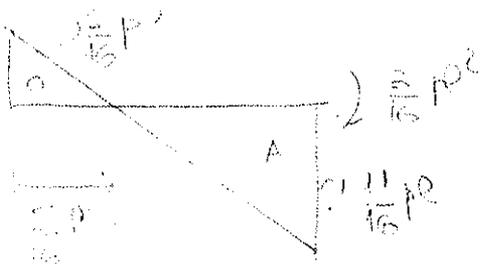


$$M = \frac{33}{16} pl - \frac{31}{16} pl = \frac{2}{16} pl$$

VER. EQ. AL NODO
di mezzo asta AB

$$\begin{aligned} & \left(\frac{33}{16} pl^2 \right) - \left(4pl^2 \right) + \left(\frac{31}{16} pl^2 \right) \\ & \frac{33}{16} + \frac{31}{16} = 4 \end{aligned} \quad \text{ok!}$$

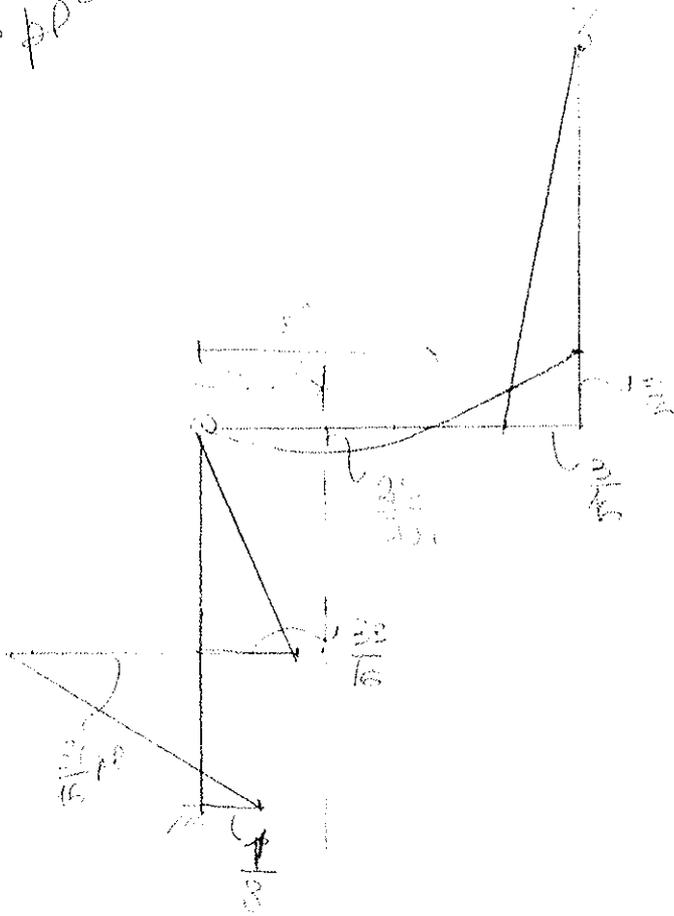
ASTA BC



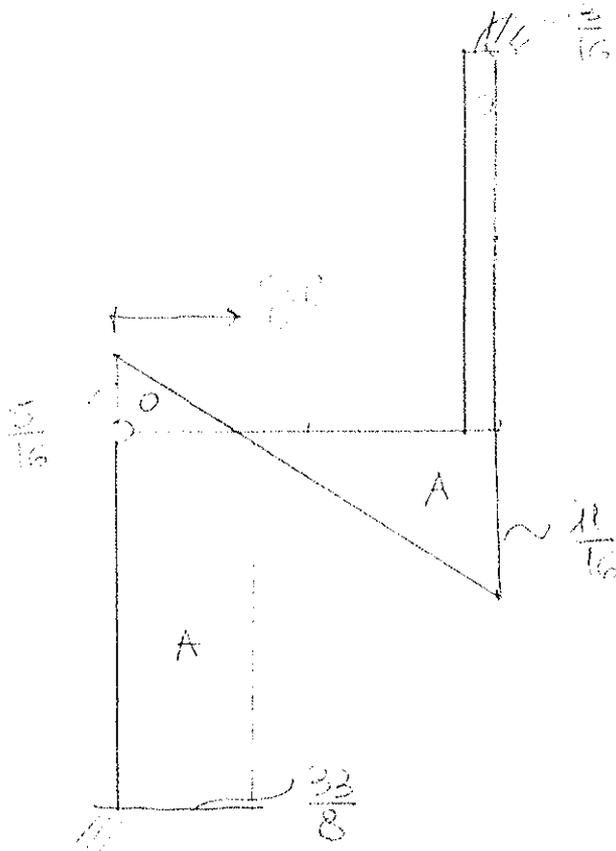
$$\begin{aligned} M &= \frac{11}{16} pl + \frac{pl^2}{16} + pl \cdot \frac{11l}{32} - \frac{11l}{16} \cdot pl \\ M &= \frac{-121}{256} pl^2 - \frac{121}{512} pl^2 - \frac{3}{16} pl^2 \\ &= \frac{212 - 121 - 96}{512} = \frac{25}{512} pl \end{aligned}$$

(*) Calcolo del momento max

$$\boxed{M} \cdot \rho D^2$$

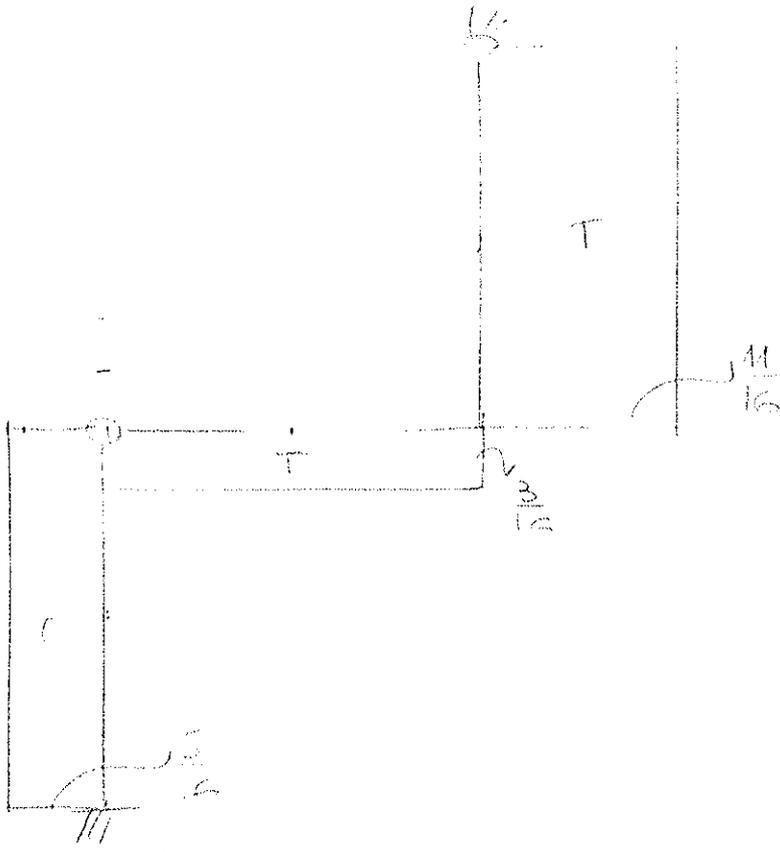


$$\boxed{T} \cdot R$$



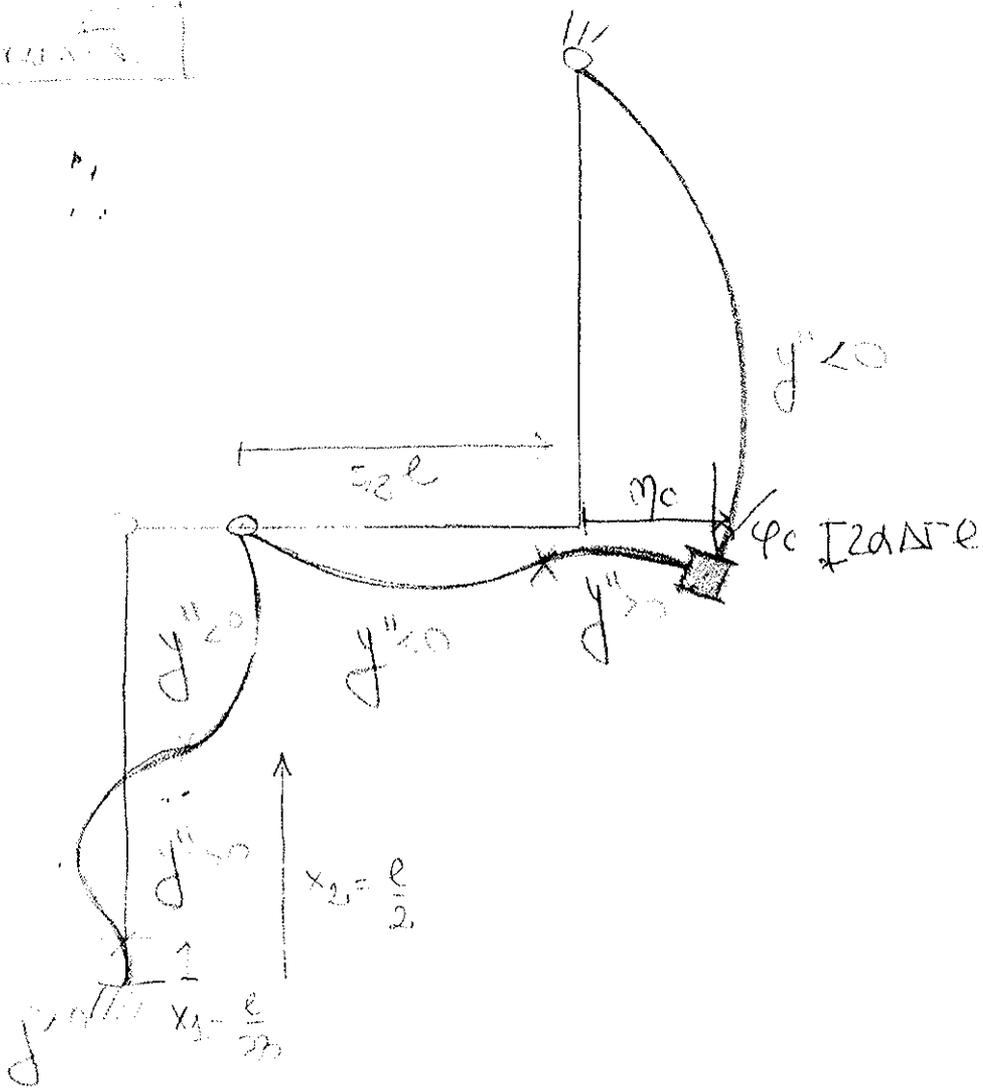
$\frac{44}{16} \rho D^2$
 ρ^2
 ρ^2

N.P.



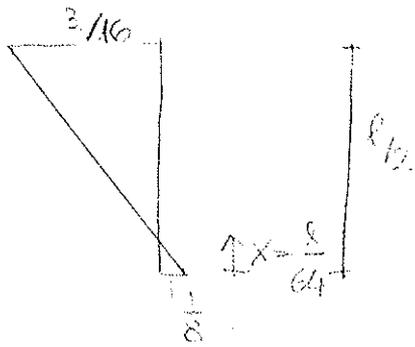
Deflection

P_1
 P_2



Calcolo dei fletti della deformata

AB



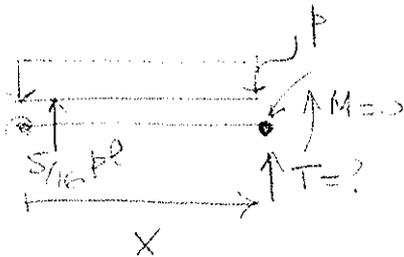
$$\frac{3}{16} : \left(\frac{l}{2} - x\right) = \frac{1}{8} : x$$

$$\frac{3}{8} \left(\frac{l}{2} - x\right) = \frac{3}{16} x$$

$$\left(\frac{3}{16} - \frac{1}{8}\right) = \frac{0}{16} \quad \frac{16}{16}$$

$$x_1 = \frac{l}{3}$$

BC



ROT.

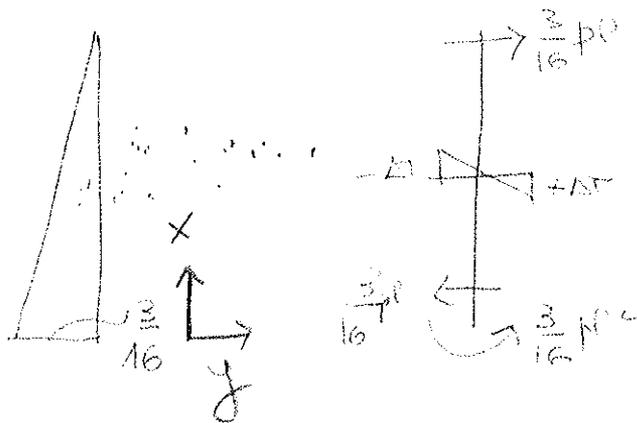
$$\frac{5}{16} p l x - p \frac{x^2}{2} = 0$$

$$x \left(\frac{5}{16} l - \frac{x}{2} \right) = 0$$

$$x = 0$$

$$x = \frac{5}{16} l \cdot 2 = \frac{5l}{8}$$

CD



$$y'''(x) = -\frac{H(x)}{EJ} + \frac{2\alpha \Delta T}{h}$$



Nel nostro caso $y'''(x) = -\frac{1}{EJ} \frac{3}{16} p l (x-l) - 2 \cdot \frac{1}{4} \frac{p l^2}{EJ}$

$\frac{3}{16} p l$ $\frac{3}{16} p l$ $H(x)$

$$H(x) = -\frac{3}{16} p l^2 + \frac{3}{16} p l x = \frac{3}{16} p l (l - x)$$

$$y'''(x) < 0 \quad -\frac{1}{EJ} \frac{3}{16} p l (x-l) - \frac{p l^2}{2EJ} < 0$$

$$\frac{3}{16} x > \frac{3}{16} l - \frac{l}{2} \quad x > \frac{16}{3} \left(\frac{3-8l}{16} \right)$$

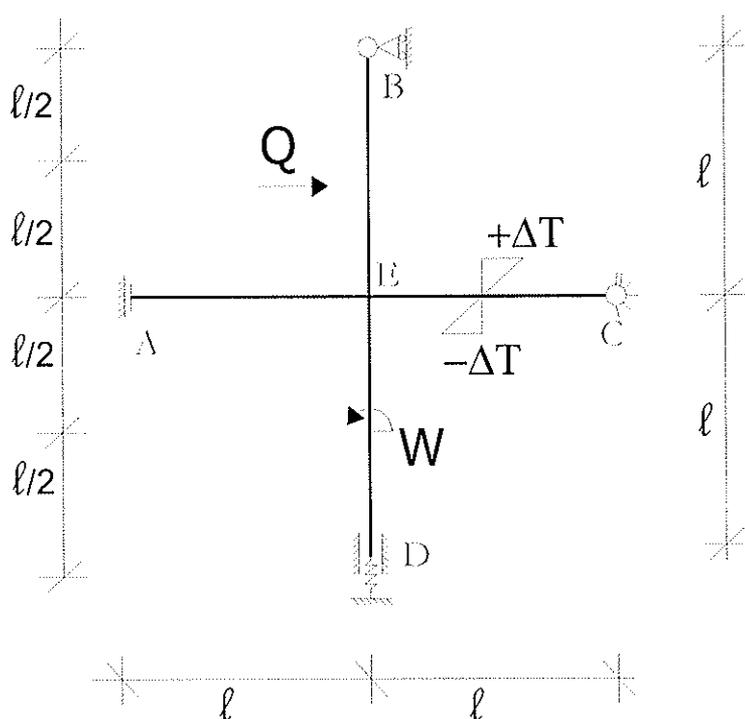
$$x > -\frac{5l}{3}$$

$$\rightarrow \forall x > -\frac{5l}{3} \quad y'''(x) < 0$$

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PROVA SCRITTA

20 – 03 – 2009



$$k_{\eta} = \frac{6EJ}{l^3}$$

$$\frac{\alpha\Delta T}{h} = \frac{pl^2}{EJ}$$

$$W = 4pl^2$$

$$Q = 2pl$$

Si richiedono i grafici di:

- Momento flettente (con il valore e la posizione dei massimi)
- Taglio
- Azione assiale
- Deformata qualitativa con posizione dei flessi

SOLUZIONE

Sistema risolvete

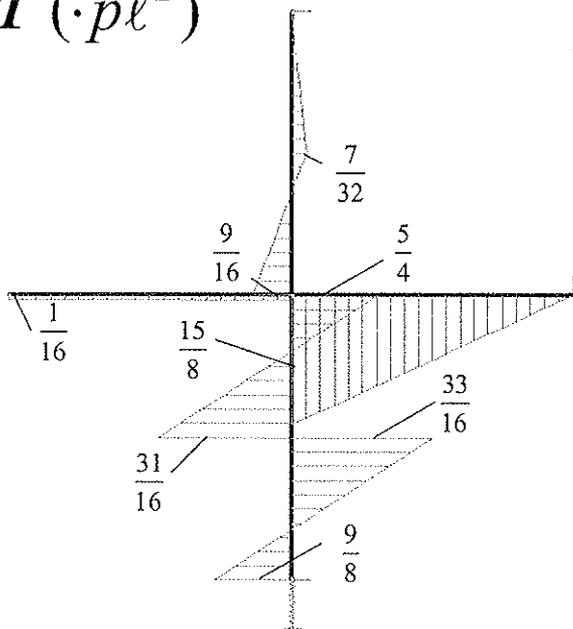
$$\begin{cases} \frac{11EJ}{\ell} \varphi_E + \frac{3EJ}{\ell^2} \eta_D + \frac{13}{8} p \ell^2 = 0 \\ \frac{3EJ}{\ell^2} \varphi_E + \frac{9EJ}{\ell^3} \eta_D + 3p\ell = 0 \end{cases}$$

Soluzioni

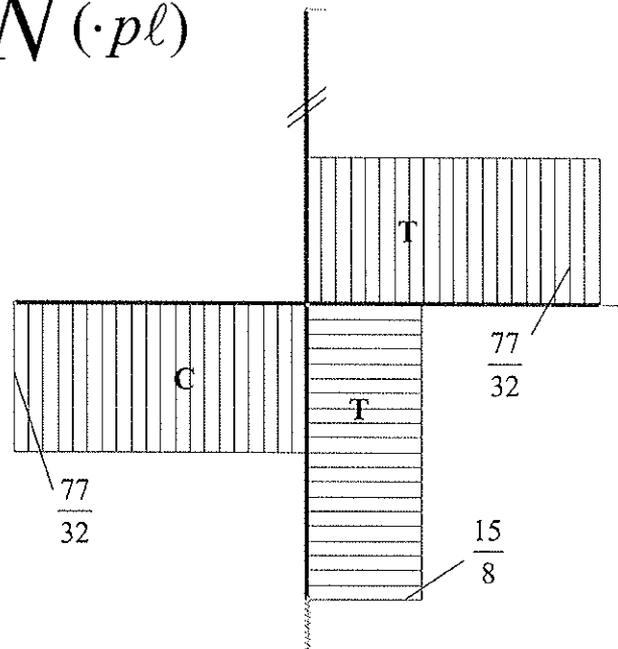
$$\begin{cases} \varphi_E = -\frac{p\ell^3}{16EJ} \\ \eta_D = -\frac{5p\ell^4}{16EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata

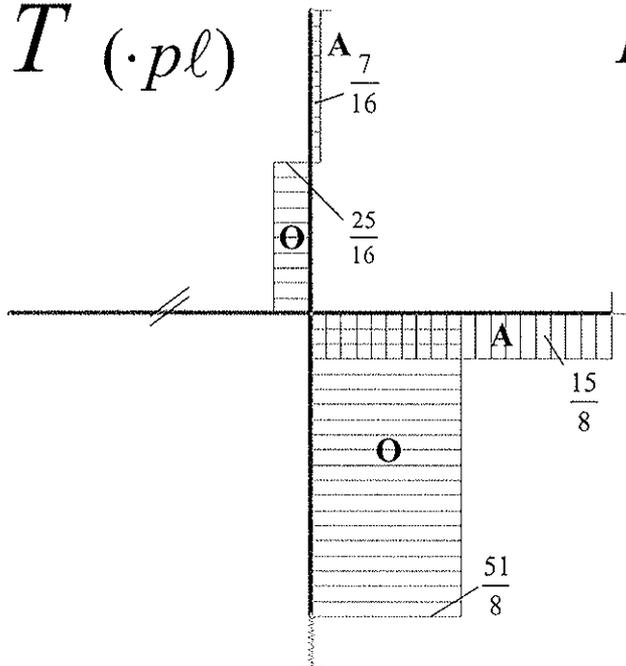
$M (\cdot p\ell^2)$



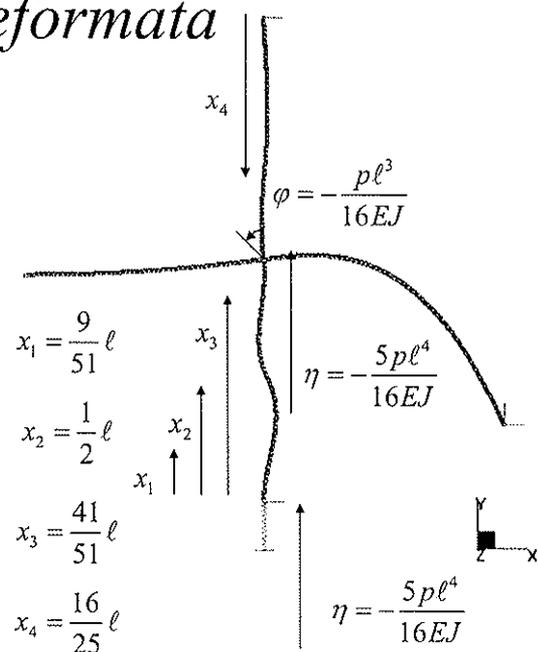
$N (\cdot p\ell)$



$T (\cdot p\ell)$



Deformata



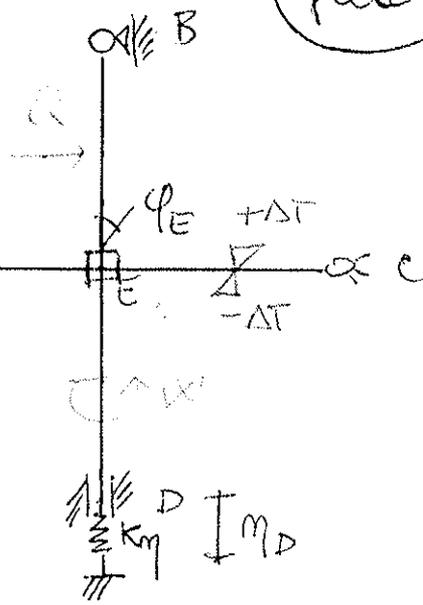
Telaio

file 1

SISTEMA RISOLVENTE

$C_F = 3 \frac{EI}{e}$
 $W = 4 \frac{EI}{e^2}$
 $K_M = \frac{6EI}{e^2}$

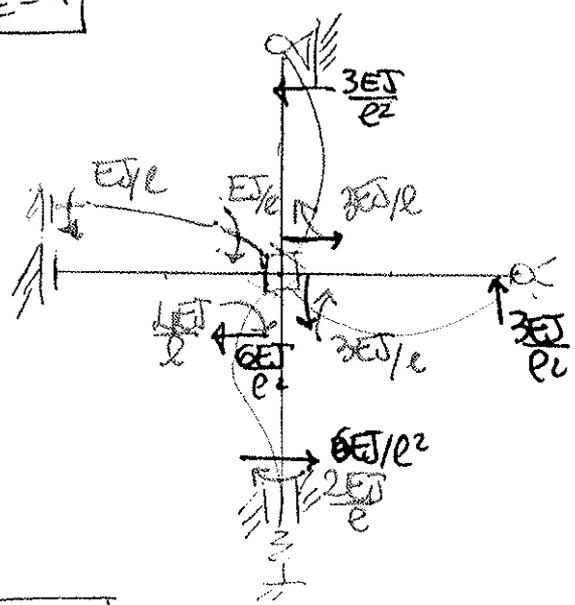
$K_{AF} = \frac{6EI}{e^2}$
 $W = 4 \frac{EI}{e^2}$



$$\begin{cases} u_{EP} \varphi_E + u_{EM} \eta_D + u_{EO} = 0 \\ h_{AP} \varphi_E + h_{DM} \eta_D + h_{EO} = 0 \end{cases}$$

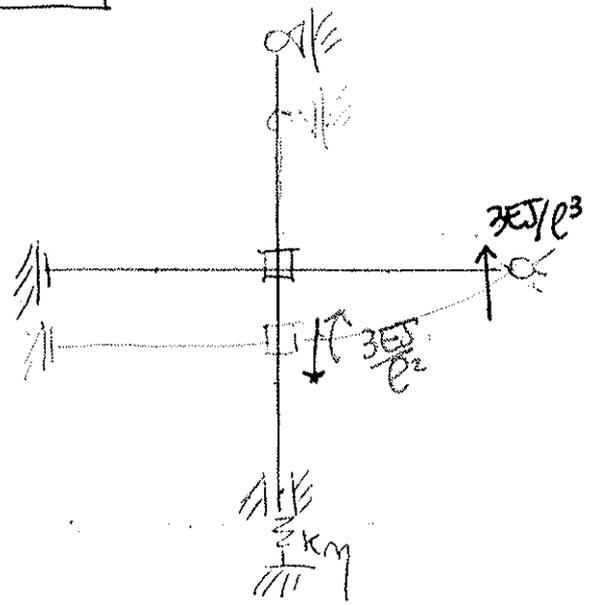
$\boxed{+ \uparrow \leftarrow + \rightarrow}$

$\boxed{\varphi_E = 1}$

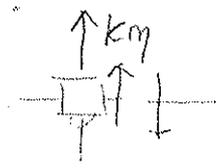


$u_{EP} = 11EI/e$
 $h_{DP} = \frac{3EI}{e^2}$

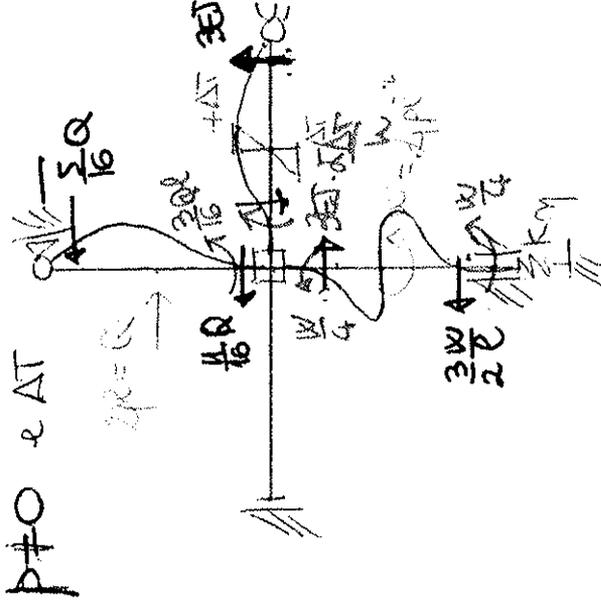
$\boxed{\eta_D = 1}$



$u_{EM} = 3EI/e^2$
 $h_{DM} = \frac{3EI}{e^3} + K_M = \frac{3EI}{e^3} + \frac{6EI}{e^3} = \frac{9EI}{e^3}$



$$\frac{d\Delta T}{\Delta T} = \frac{P \ell^2}{EJ}$$



$$w_{EO} = \left[\frac{3}{16} Q \ell + \frac{W}{4} - 3EJ \cdot \frac{d\Delta T}{\Delta} \right] =$$

$$= - \left[\frac{3}{8} P \ell^2 + \frac{P \ell^2}{8} - 3P \ell^2 \right] = + \frac{13}{8} P \ell^2$$

$$u_{DO} = 3P \ell = 3EJ \cdot \frac{d\Delta T}{\Delta} \cdot \frac{1}{\ell}$$

Systeme nichtlinear

$$\frac{11EJ}{\ell} \varphi_E + \frac{3EJ}{\ell^2} m_D + 13/8 P \ell^2 = 0$$

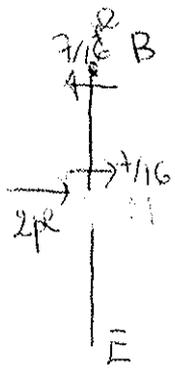
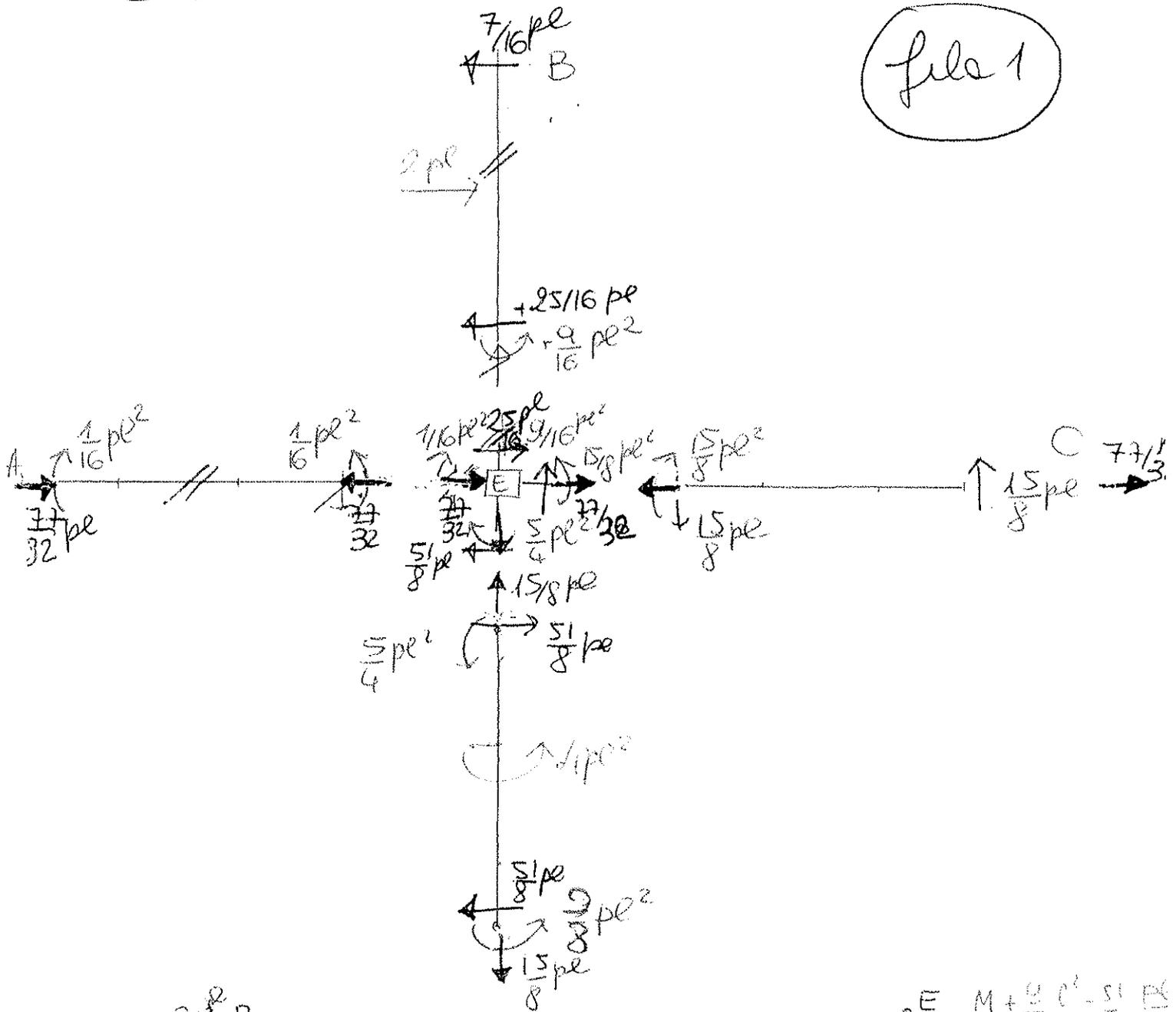
$$\frac{3EJ}{\ell^2} \varphi_E + \frac{9EJ}{\ell^3} m_D + 3P \ell = 0$$

$$\varphi_E = -11/16 P \ell^3 / EJ$$

$$m_D = -5/16 P \ell^4 / EJ$$

D A C

file 1



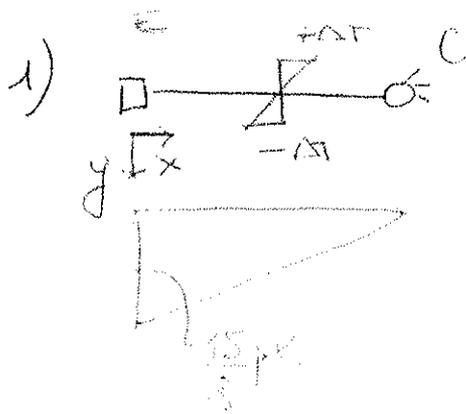
$\uparrow + \frac{7}{16} pl - M = 0 \quad M = \frac{7}{16} pl^2$

$$E \quad M + \frac{5}{8} pl - \frac{5}{8} pl^2$$

$$M = \frac{5}{16} pl^2 - \frac{5}{8} pl^2$$

$$= -\frac{5}{16} pl^2$$

DEFORMATA & FLESSI



$$\left(\begin{array}{c} \uparrow \frac{15pl^2}{8} \\ \uparrow \frac{15pl}{8} \end{array} \right) M(x)$$

$$M(x) + \frac{15pl}{8}x - \frac{15pl^2}{8} = 0 \rightarrow M(x) = \frac{15pl}{8}(l-x)$$

$$y'' = -\frac{M(x)}{EJ} + \frac{2\alpha\Delta T}{H}$$

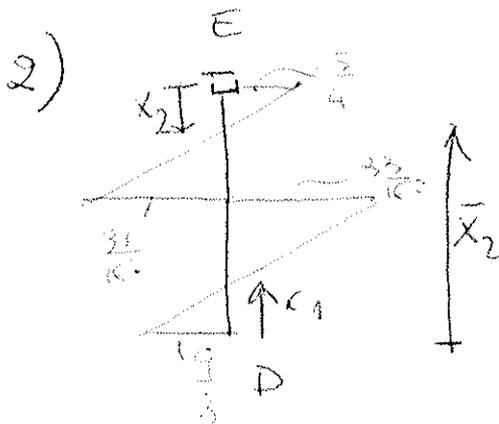
$$y'' = -\frac{15pl(l-x)}{8} \frac{1}{EJ} + \frac{2pl^2}{EJ} \geq 0$$

$$-15pl(l-x) + 16pl^2 \geq 0$$

$$-15pl^2 + 15plx + 16pl^2 \geq 0 \quad -15plx + pl^2 \geq 0$$

$$x \geq -\frac{l}{15}$$

quindi $y'' \geq 0 \quad \forall x$



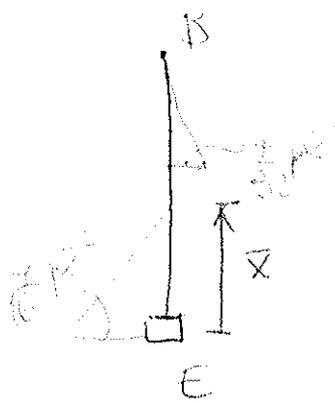
$$\left(\frac{33}{16} + \frac{18}{16} \right) : \frac{l}{2} = \frac{18}{16} \cdot x$$

$$x_1 = \frac{\frac{l}{2} \cdot \frac{18}{16}}{\frac{51}{16}} = \frac{9}{16} \cdot \frac{16}{51} l = \frac{9l}{51}$$

$$x_2 = \frac{\frac{l}{2} \cdot \frac{51}{4}}{\left(\frac{31}{16} + \frac{20}{16} \right)} = \frac{5l}{8} \cdot \frac{16}{51} = \frac{10l}{51}$$

$$\bar{x}_2 = l - \frac{10l}{51} = \frac{41l}{51}$$

3)



$$\left(\frac{F}{32} + \frac{18}{32}\right) \cdot \frac{l}{2} = \frac{9}{16} \cdot x$$

$$x = \frac{9}{16 \cdot 2} / \frac{25}{32} l = \frac{9}{18} \frac{32}{25} \frac{l}{2} = \frac{9}{25} l$$

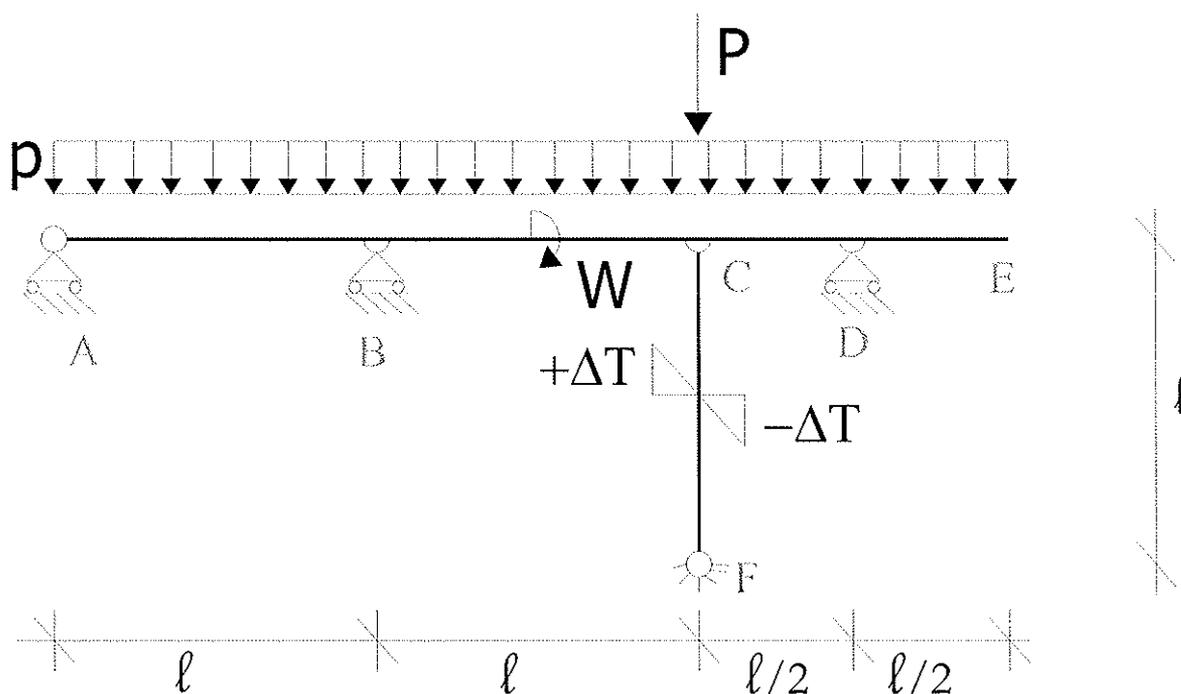
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e

TECNICA DELLE COSTRUZIONI
(Corsi di laurea: Edile – Architettonico, Civile e per l’Ambiente e il Territorio V.O.)

PROVA SCRITTA

22 – 06 – 2009



$$\frac{\alpha \Delta T}{h} = \frac{5 p \ell^2}{4 E J} \quad W = 3 p \ell^2 \quad P = \frac{7 p \ell}{32}$$

Si richiedono i grafici di:

- Momento flettente (con il valore e la posizione dei massimi)
- Taglio
- Azione assiale
- Deformata qualitativa con posizione dei flessi

SOLUZIONE

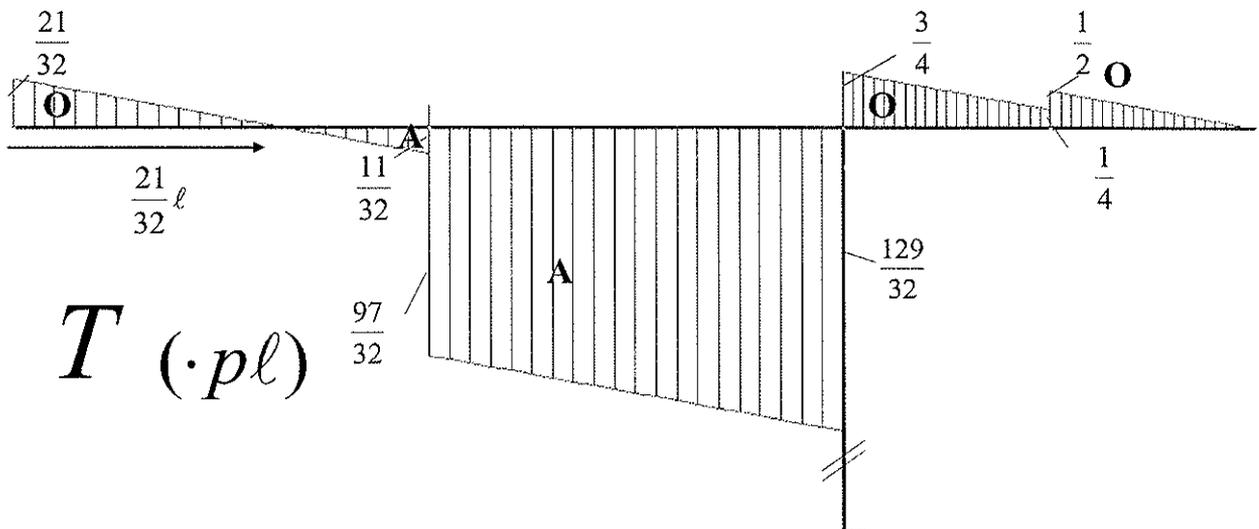
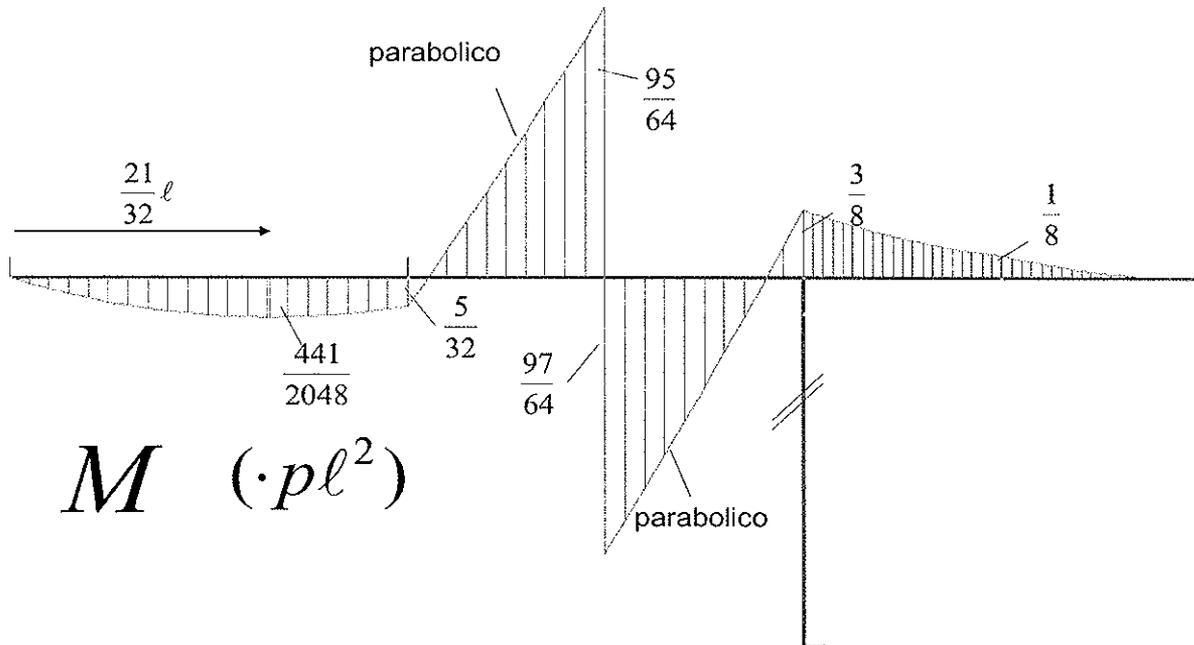
Sistema risolvente

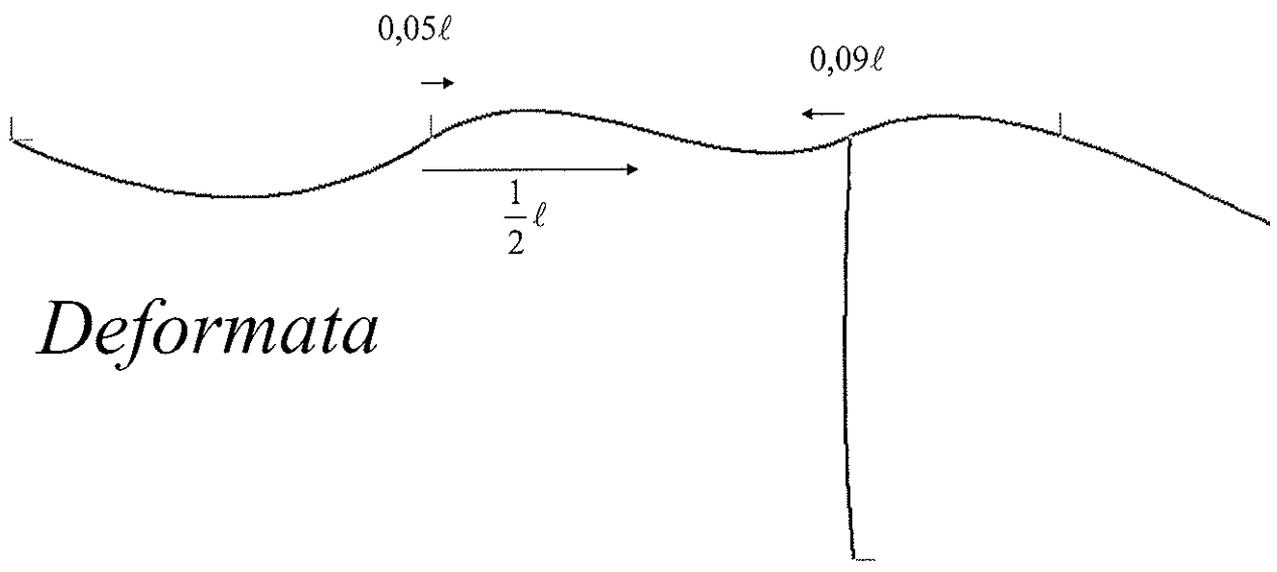
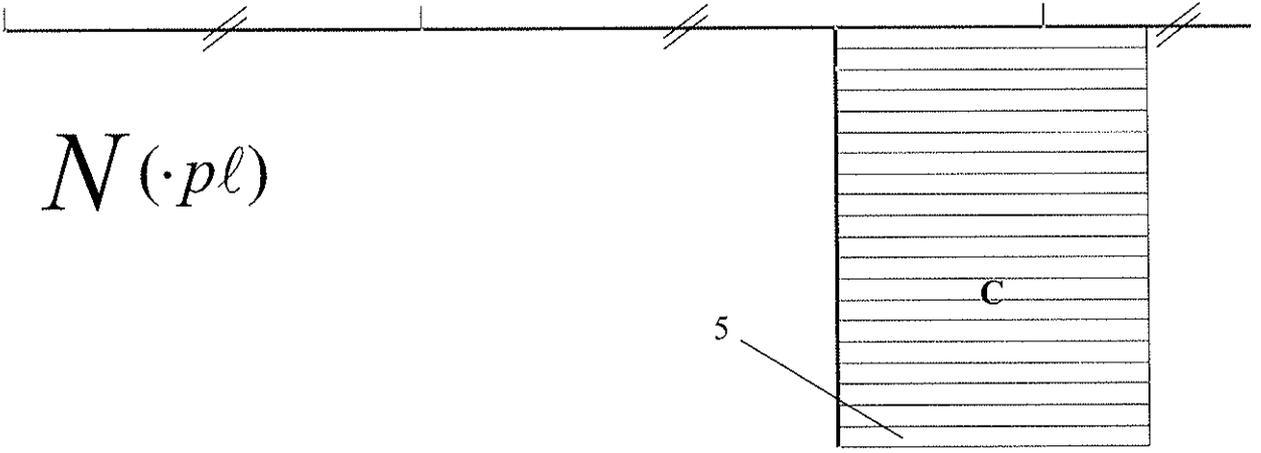
$$\begin{cases} \frac{2l}{3EJ} X_B + \frac{l}{6EJ} X_C - \frac{pl^3}{24EJ} = 0 \\ \frac{l}{6EJ} X_B + \frac{l}{2EJ} X_C + \frac{31pl^3}{192EJ} = 0 \end{cases}$$

Soluzioni

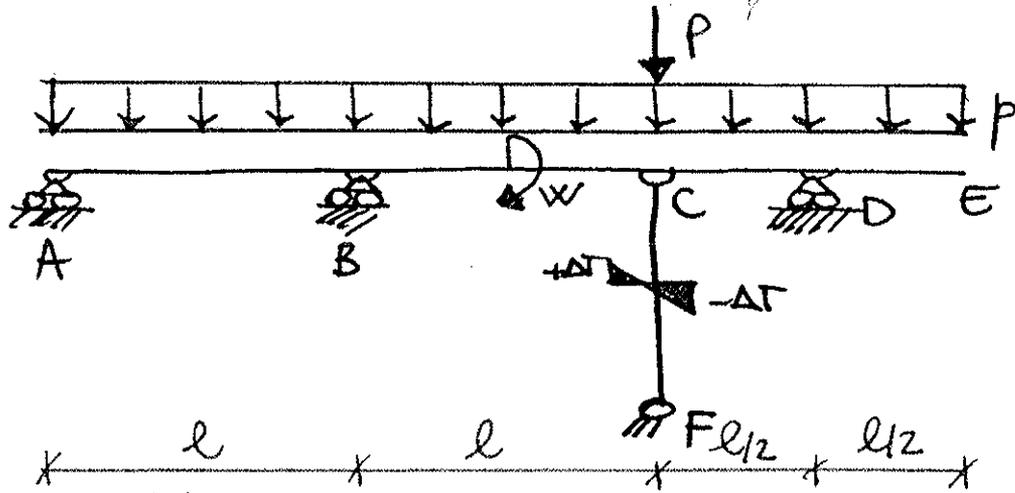
$$\begin{cases} X_B = +\frac{5}{32} pl^2 \\ X_C = -\frac{3}{8} pl^2 \end{cases}$$

Diagrammi delle azioni interne e Deformata





TEMA esame del 22 giugno 2009



$$P = \frac{7Pl}{32}$$

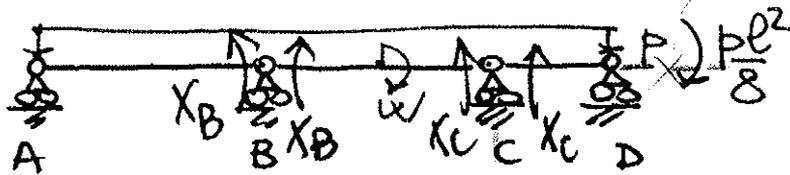
$$W = 3Pl^2$$

$$\frac{\Delta T}{l} = \frac{5}{4} \frac{Pl^2}{EI}$$

$$\begin{cases} X_B \varphi_{BB} + X_C \varphi_{BC} + \varphi_{B0} = 0 \\ X_B \varphi_{CB} + X_C \varphi_{CC} + \varphi_{C0} = 0 \end{cases}$$

si tratta di una trave continua*
 → MTD FORZE

* Mi accorgo dell'appendice isostatica DE e CF quindi riduco la struttura a:

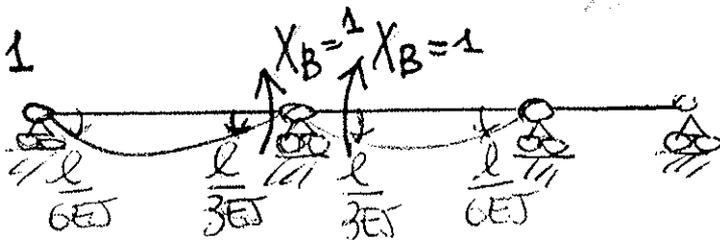


mensole

$$M_C = \frac{Pl}{2} \cdot \frac{l}{4} = \frac{Pl^2}{8}$$

si tratta di un'appendice isostatica → la def. termica non produce alcuna azione interna

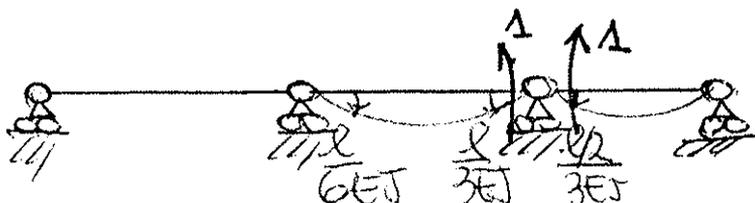
$$X_B = 1$$



$$\varphi_{BB} = \frac{l}{3EI} - \left(-\frac{l}{3EI} \right) = \frac{2l}{3EI}$$

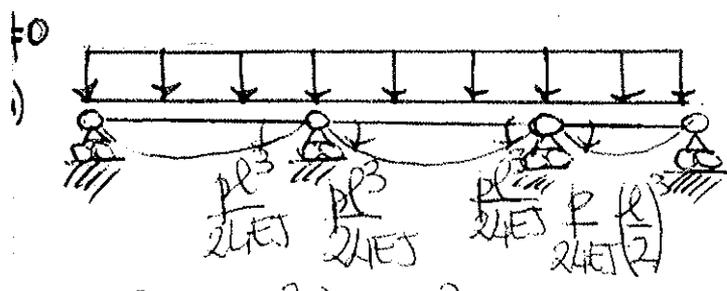
$$\varphi_{CB} = 0 - \left(-\frac{l}{6EI} \right) = \frac{l}{6EI}$$

$$X_C = 1$$



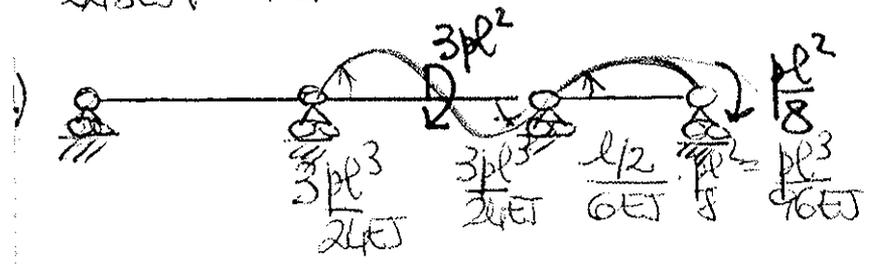
$$\varphi_{BC} = \frac{l}{6EI}$$

$$\begin{aligned} \varphi_{CC} &= \frac{l}{6EI} - \left(-\frac{l}{3EI} \right) \\ &= \frac{1+2}{6} = \frac{l}{2EI} \end{aligned}$$



$$v_0 = \frac{pl^3}{24EI} - \left(-\frac{pl^3}{24EI}\right) = \frac{pl^3}{12EI}$$

$$\theta_0 = \frac{pl^3}{24 \cdot 8EI} - \left(-\frac{pl^3}{24EI}\right) = \frac{1+8pl^3}{24 \cdot 8EI} = \frac{9}{192} \frac{pl^3}{EI}$$



$$v_{B0}^2 = -\frac{3}{24} \frac{pl^3}{EI} = -\frac{pl^3}{8EI}$$

$$v_{C0}^2 = -\frac{pl^3}{96EI} - \left(-\frac{pl^3}{8EI}\right) = \frac{-1+12}{96} \frac{pl^3}{EI} = \frac{11}{96} \frac{pl^3}{EI}$$

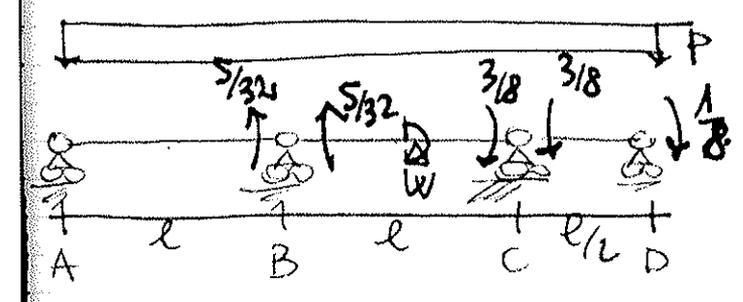
$$v_{B0} = v_{B0}^1 + v_{B0}^2 = \frac{1}{12} - \frac{1}{8} = \frac{2-3}{24} = -\frac{pl^3}{24EI}$$

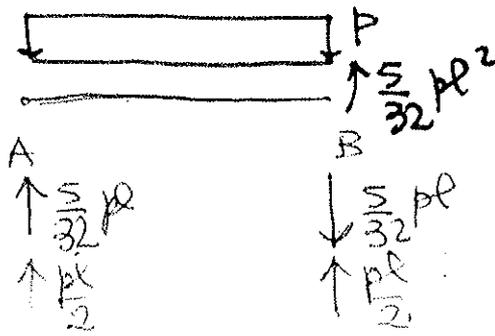
$$v_{C0} = v_{C0}^1 + v_{C0}^2 = \frac{9}{192} + \frac{11}{96} = \frac{9+22}{192} \frac{pl^3}{EI} = \frac{31}{192} \frac{pl^3}{EI}$$

$$\frac{2l}{3EI} X_B + \frac{l}{6EI} X_C - \frac{pl^3}{24EI} = 0$$

$$\frac{l}{6EI} X_B + \frac{l}{2EI} X_C + \frac{31}{192} \frac{pl^3}{EI} = 0$$

$$\left. \begin{aligned} X_B &= \frac{5}{32} pl^2 \\ X_C &= -\frac{3}{8} pl^2 \end{aligned} \right\}$$

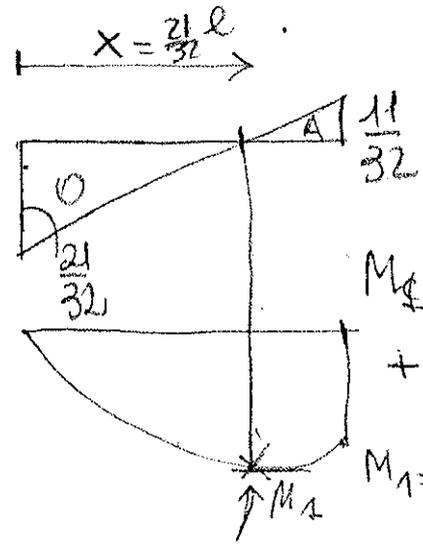




(AB)

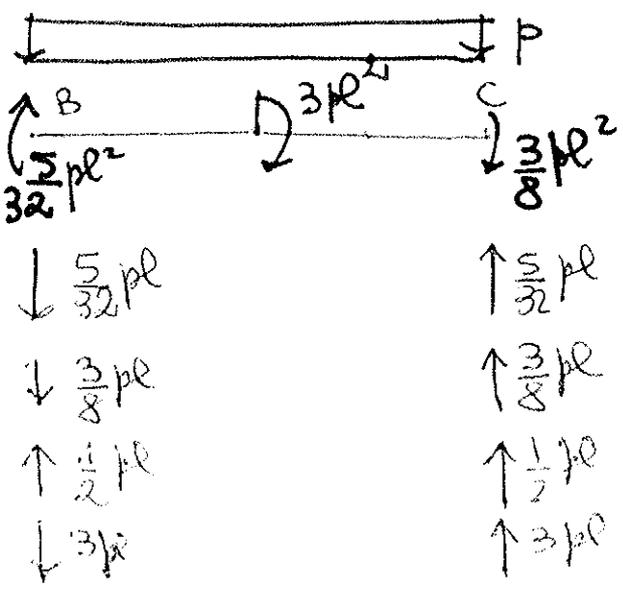
$$\uparrow \frac{1}{2} + \frac{5}{32} = \frac{21}{32} pl$$

$$\uparrow \frac{1}{2} - \frac{5}{32} = \frac{11}{32} pl$$



$$M_2 - \frac{21pl}{32} \cdot \frac{21l}{32} + \frac{21pl}{32} \cdot \frac{1}{2} \cdot \frac{21l}{32} = 0$$

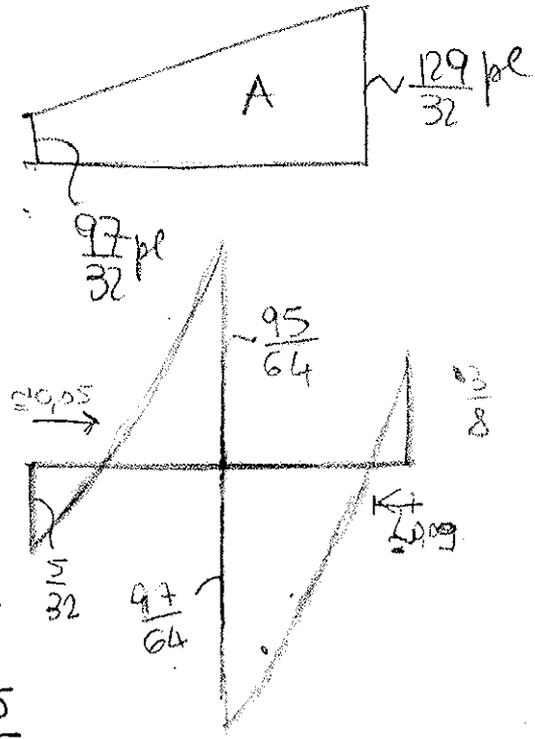
$$M_1 = \left(\frac{21}{32}\right)^2 pl^2 - \frac{1}{2} \left(\frac{21}{32}\right)^2 pl^2 = \frac{1}{2} \left(\frac{21}{32}\right)^2 pl^2$$



(BC)

$$\downarrow 3pl - \frac{1}{2} + \frac{3}{8} + \frac{5}{32} = \frac{96 - 16 + 12 + 5}{32} = \frac{97}{32} pl$$

$$\uparrow 3 + \frac{1}{2} + \frac{3}{8} + \frac{5}{32} = \frac{96 + 16 + 12 + 5}{32} = \frac{129}{32} pl$$



flermi:

$$x_1 = ? \quad \frac{5}{32} pl^2 - \frac{97}{32} plx - \frac{px^2}{2} = 0$$

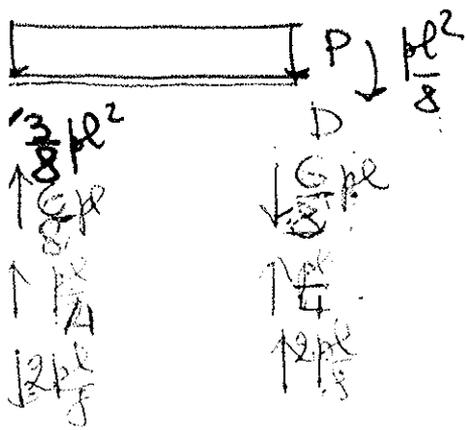
$$16px^2 + 97plx - 5pl^2 = 0$$

$$x_{12} = \frac{-97 \pm \sqrt{97^2 + 320}}{32} < NA$$

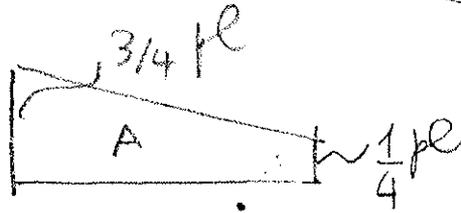
$$x_2 = ? \quad \frac{3}{8} pl^2 - \frac{129}{32} plx + \frac{px^2}{2} = 0$$

$$16px^2 - 129plx + 12pl^2 = 0$$

$$\checkmark - 9,04l$$

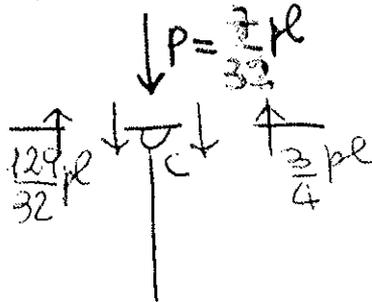


(CD)



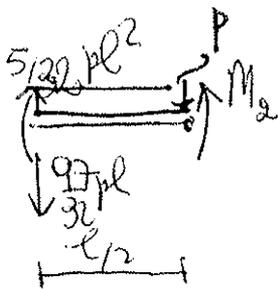
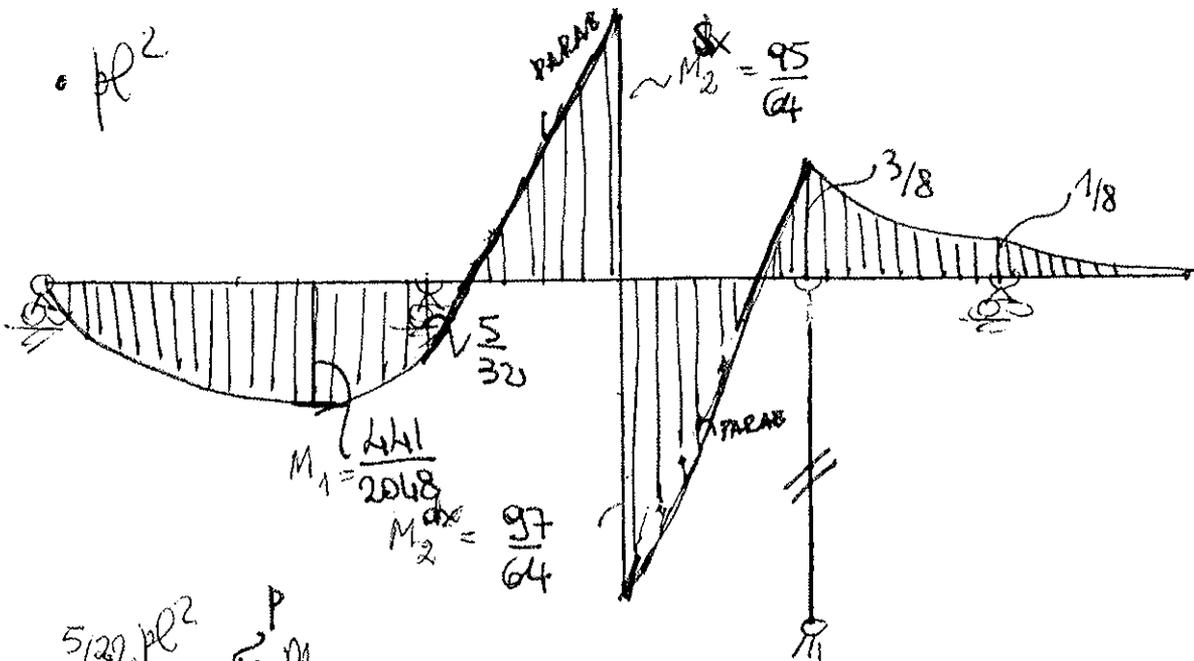
$$\frac{1}{8} + \frac{1}{4} - \frac{1}{3} = \frac{2-6}{8} = -\frac{1}{4} pl$$

Force applied in CF



$$\uparrow \left(\frac{129}{32} + \frac{3}{4} + \frac{7}{32} \right) pl = 5pl$$

1) $\cdot pl^2$

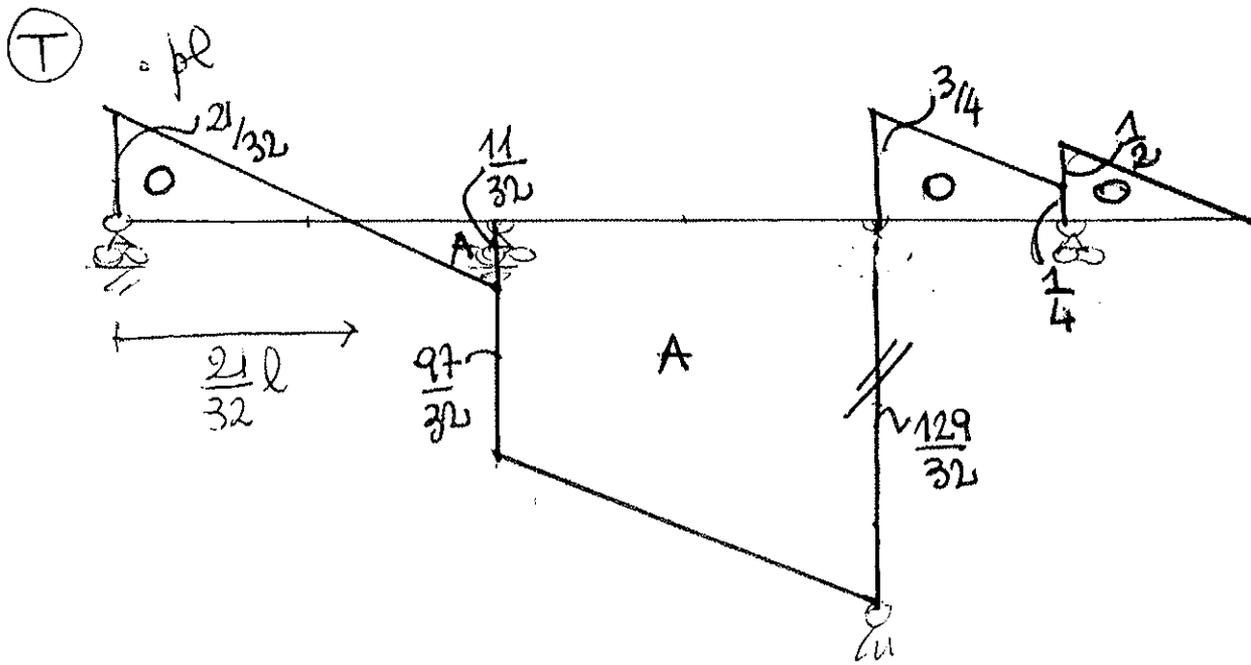


$$M_2^{sx} - \frac{5}{32} pl^2 + \frac{97}{32} \frac{1}{2} pl^2 + \frac{pl^2}{8} = 0$$

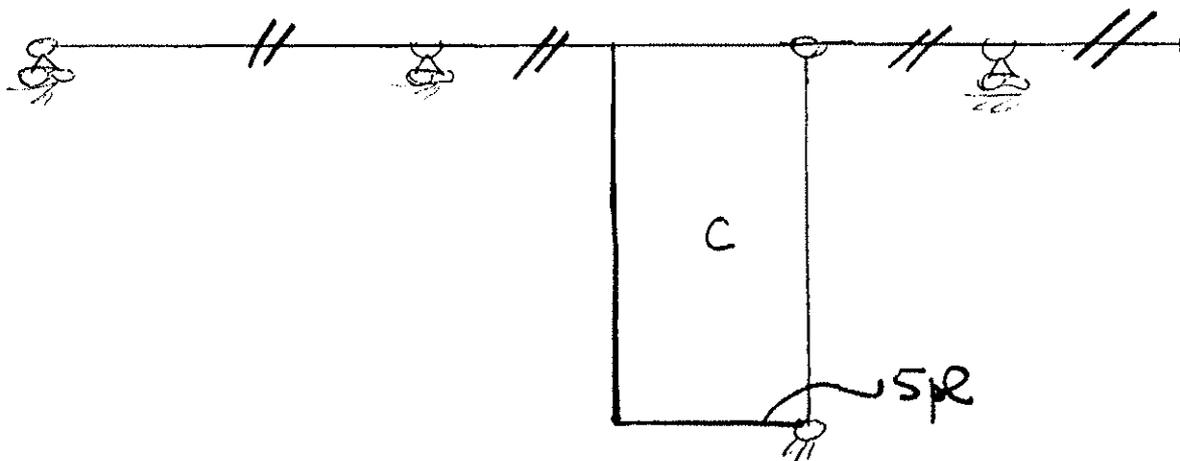
$$M_2^{sx} = \frac{+10 - 97 - 8}{64} = \frac{95}{64} pl^2$$

$$\frac{95}{64} pl^2 + M_2^{dx} - 3pl^2 = 0$$

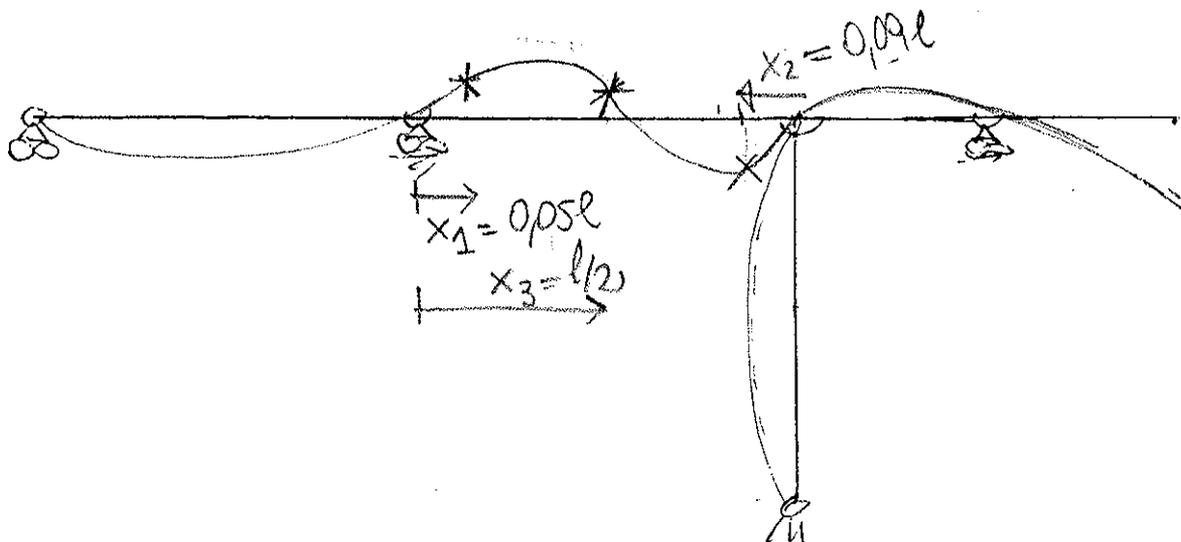
$$\rightarrow M_2^{dx} = \frac{97}{64} pl^2$$

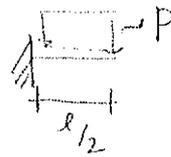
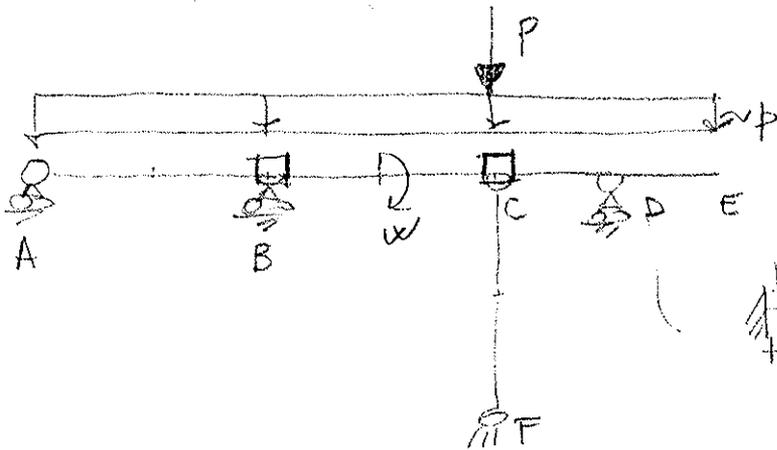


(N) $\cdot p l$



deformata





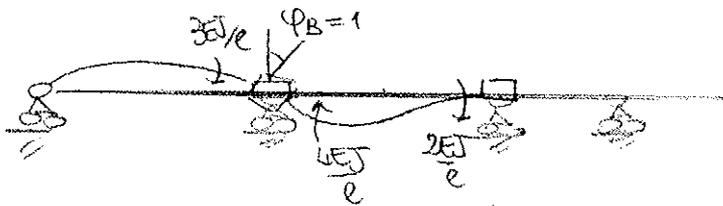
$$H = \frac{Pl}{2} \cdot \frac{1}{2} \cdot \frac{l}{2} = \frac{Pl^2}{8}$$

$$\mathcal{M}_{BB} \varphi_B + \mathcal{M}_{BC} \varphi_C + \mathcal{M}_{B0} = 0$$

[+] ↑

$$\mathcal{M}_{CB} \varphi_B + \mathcal{M}_{CC} \varphi_C + \mathcal{M}_{C0} = 0$$

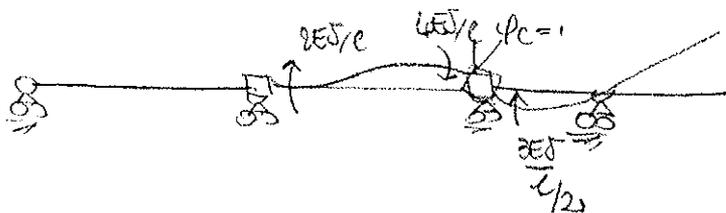
[= 1]



$$\mathcal{M}_{BB} = \frac{7EJ}{l}$$

$$\mathcal{M}_{CB} = \frac{2EJ}{l}$$

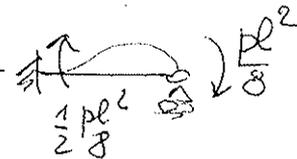
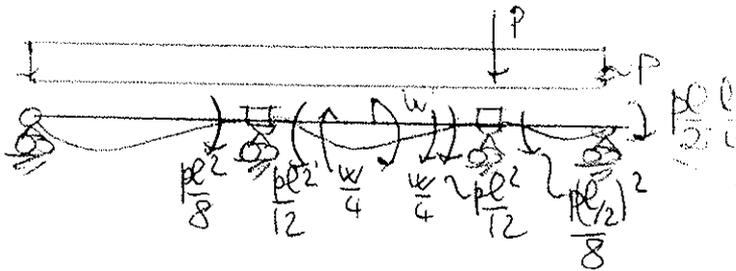
[= 1]



$$\mathcal{M}_{CC} = \frac{10EJ}{l}$$

$$\mathcal{M}_{BC} = \frac{2EJ}{l}$$

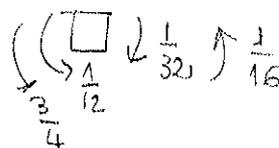
[0]



$$\mathcal{M}_{B0} = \frac{Pl^2}{8} - \frac{Pl^2}{12} + \frac{w}{4} = \frac{1}{8} - \frac{1}{12} + \frac{3}{4} = \frac{3-2+18}{24} = \frac{19}{24} Pl^2$$

$\frac{1}{8} \downarrow \left[\square \right] \frac{1}{12} \uparrow \frac{3}{4}$

$$\mathcal{M}_{C0} = \frac{w}{4} + \frac{Pl^2}{12} - \frac{Pl^2}{32} + \frac{Pl^2}{18} = \frac{3}{4} + \frac{1}{12} - \frac{1}{32} + \frac{1}{18} = \frac{72+8-3+16}{96} = \frac{93}{96} Pl^2$$



Risolvo il sistema

$$\begin{cases} \frac{7EJ}{e} \varphi_B + \frac{2EJ}{e} \varphi_C + \frac{19}{24} pl^2 = 0 \\ \frac{2EJ}{e} \varphi_B + \frac{10EJ}{e} \varphi_C + \frac{83}{96} pl^2 = 0 \end{cases} \quad \begin{cases} \varphi_B = -\frac{3}{32} \frac{pl^3}{EJ} \\ \varphi_C = -\frac{13}{192} \frac{pl^3}{EJ} \end{cases}$$

quindi

$$M_B^{sx} = \frac{3EJ}{e} \left(-\frac{3}{32} \frac{pl^3}{EJ} \right) + \frac{pl^2}{8} = -\frac{5}{32} pl^2 \quad \text{ok! (= punta)}$$

$$M_C^{dx} = \frac{10EJ}{e} \left(-\frac{13}{192} \frac{pl^3}{EJ} \right) - \frac{pl^2}{32} + \frac{pl^2}{16} = \frac{-14+2}{32} = -\frac{12}{32} pl^2 = -\frac{3}{8} pl^2 \quad \text{ok (= punta)}$$



poi si procede come solito prima

l'esercizio, in definitiva, non è altro che una trave continua su quattro appoggi.

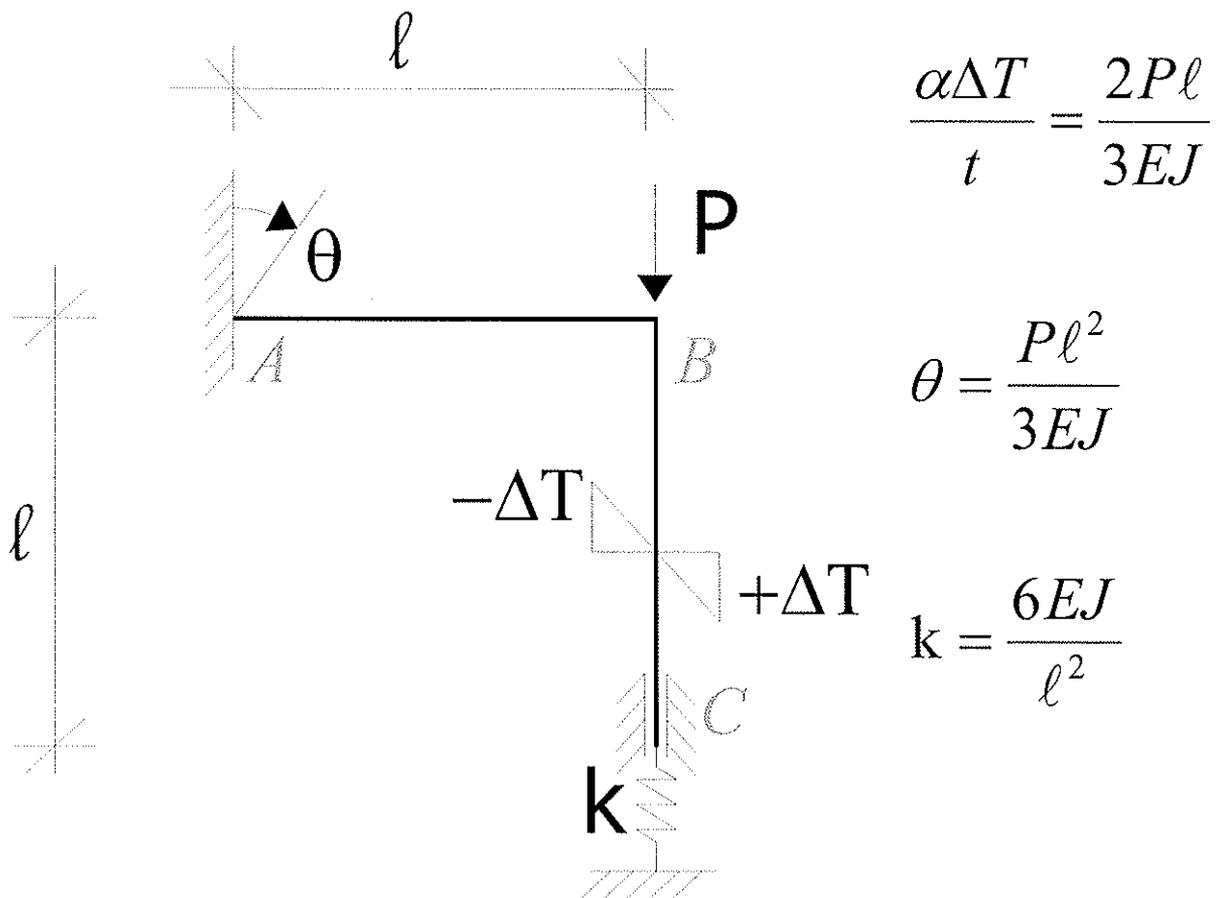
→ È MEGLIO UTILIZZARE IL METODO delle FORZE (+ immediato xché mi permette di calcolare subito i momenti ai nodi evitando, così un parappiò)

Esame di
FONDAMENTI di PROGETTAZIONE STRUTTURALE
(Corso di laurea in Ingegneria Civile N.O.)

APPELLO STRAORDINARIO

PROVA SCRITTA

8 – 07 – 2009



Si richiedono i grafici di:

- Momento flettente (con il valore e la posizione dei massimi)
- Taglio
- Azione assiale
- Deformata qualitativa con posizione dei flessi

SOLUZIONE

Sistema risolvente

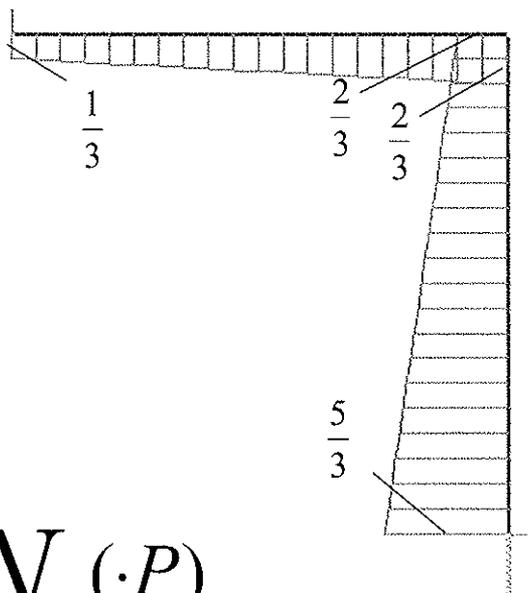
$$\begin{cases} \frac{8EJ}{\ell} \varphi_B - \frac{6EJ}{\ell^2} \eta_C + 2P\ell = 0 \\ -\frac{6EJ}{\ell^2} \varphi_B + \frac{18EJ}{\ell^3} \eta_C - 3P = 0 \end{cases}$$

Soluzioni

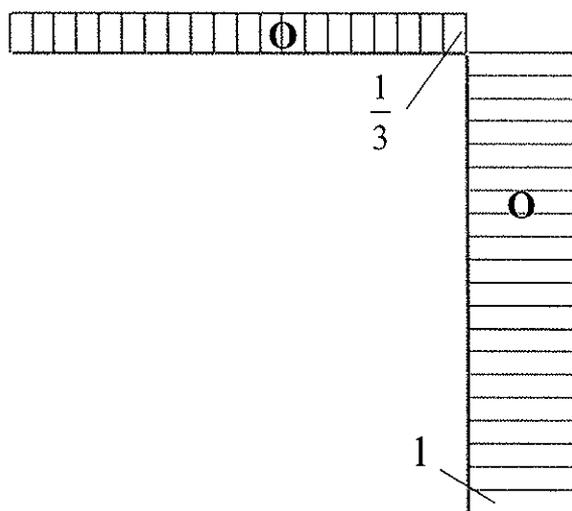
$$\begin{cases} \varphi_B = -\frac{P\ell^2}{6EJ} \\ \eta_C = \frac{P\ell^3}{9EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata

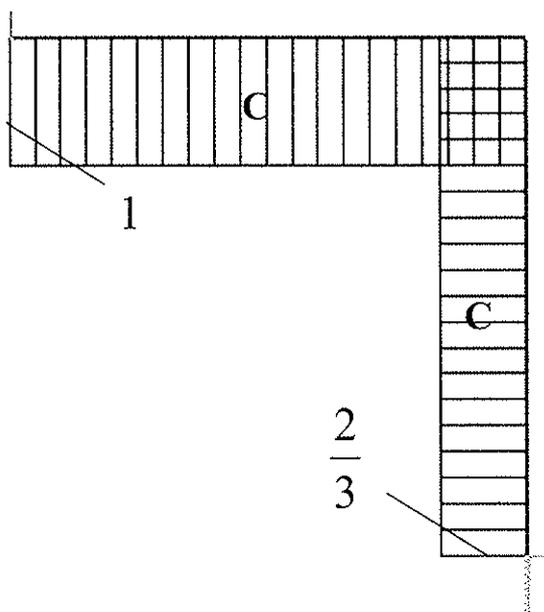
M ($\cdot P\ell$)



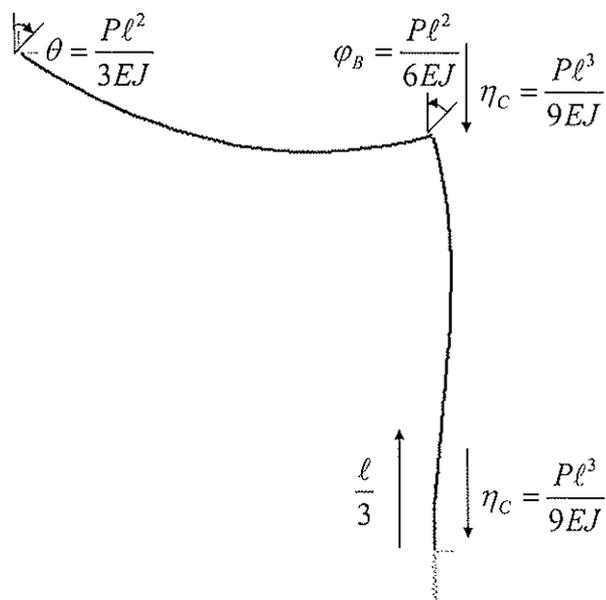
T ($\cdot P$)



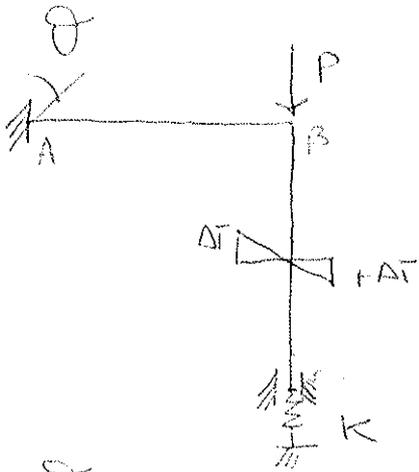
N ($\cdot P$)



Deformata



Timber + Steel Sill für 2011

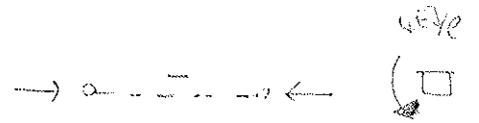
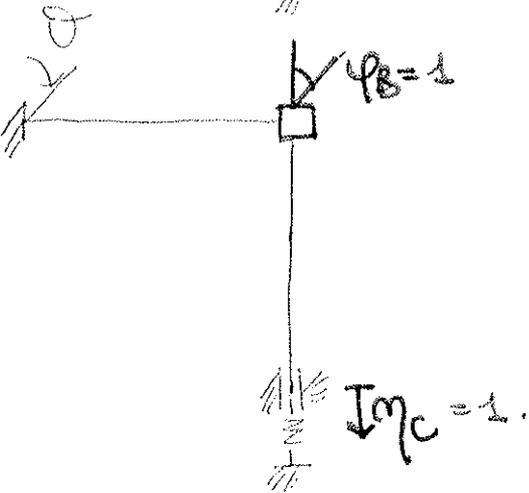


$$\begin{cases} u_{BB} \varphi_B + u_{BC} \eta_C + u_{BC} = 0 \\ l_{cB} \varphi_B + l_{cc} \eta_C + l_{cc} = 0 \end{cases}$$

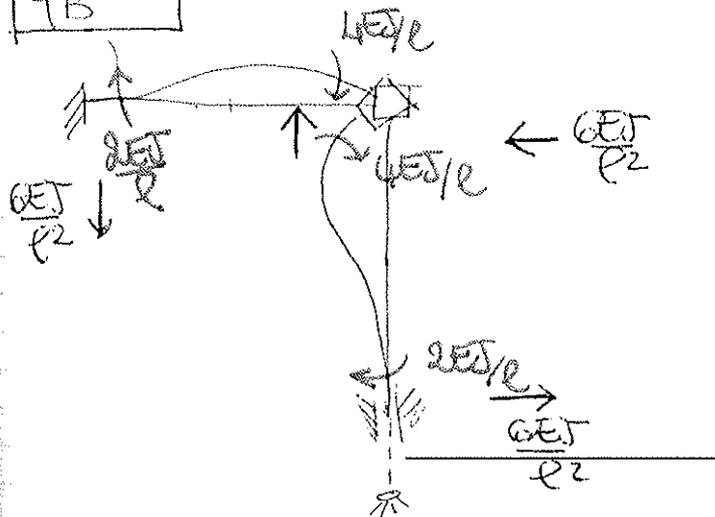
$$\frac{d\Delta T}{dt} = \frac{2_4 P l^2}{3 E J}$$

$$\theta = \frac{P l^2}{3 E J}$$

$$K = \frac{6 E J}{l^3}$$



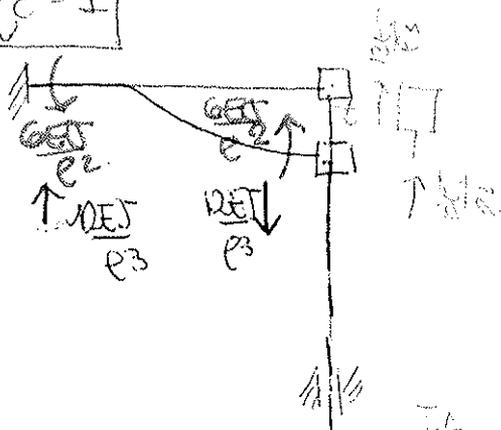
$$\varphi_B = 1$$



$$u_{BB} = \frac{8 E J}{l}$$

$$l_{cB} = -\frac{6 E J}{l^2}$$

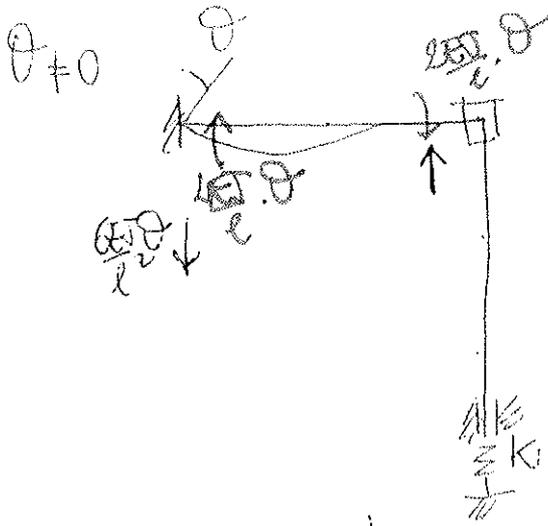
$$\eta_C = 1$$



$$u_{BC} = -\frac{6 E J}{l^2}$$

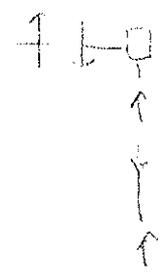
$$l_{cc} = + \frac{12 E J}{l^3} + K$$

$$\uparrow F = K \Delta$$



$$u_{B0} = \frac{2EJ}{l} \cdot \theta$$

$$u_{c0} = -\frac{6EJ}{l^2} \cdot \theta$$



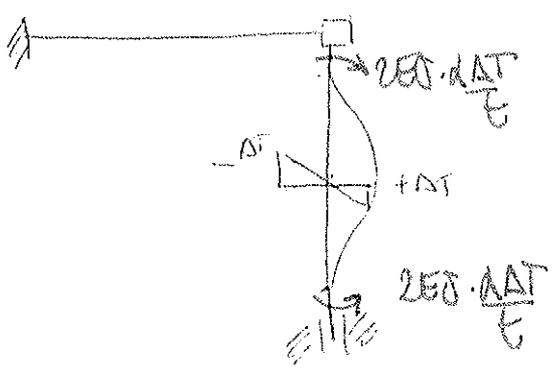
$P \neq 0$



$$u_{B0} = 0$$

$$u_{c0} = -P$$

$\Delta T \neq 0$



$$u_{B0} = +2EJ \cdot \frac{\Delta T}{l}$$

$$u_{c0} = 0$$

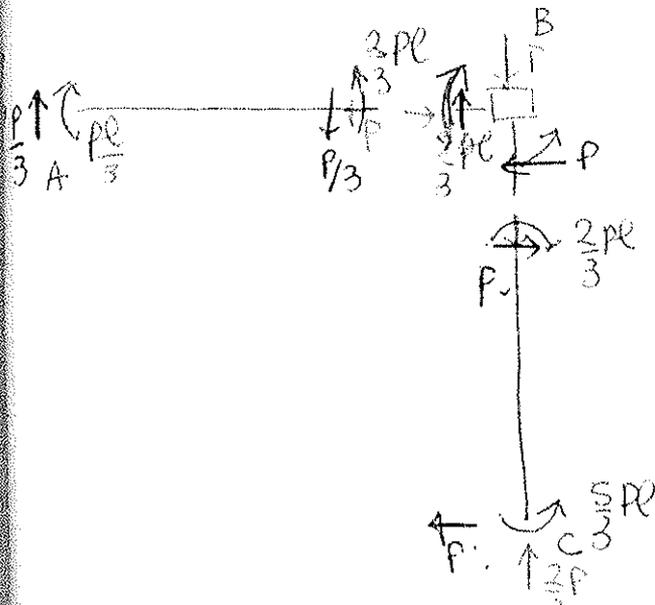
$$\begin{cases} \frac{8EJ}{l} \varphi_B - \frac{6EJ}{l^2} \eta_c + \frac{2EJ}{l} \theta + 2EJ \cdot \frac{\Delta T}{l} = 0 \\ -\frac{6EJ}{l^2} \varphi_B + \left(\frac{12EJ}{l^3} + K \right) \eta_c - \frac{6EJ}{l^2} \theta - P = 0 \end{cases}$$

$$\begin{cases} \frac{8EJ}{l} \varphi_B - \frac{6EJ}{l^2} \eta_c + \frac{2EJ}{l} \cdot \frac{Pl^4}{3EJ} + 2EJ \cdot \frac{2Pl}{3EJ} = 0 \\ -\frac{6EJ}{l^2} \varphi_B + \frac{18EJ}{l^3} \eta_c - \frac{6EJ}{l^2} \cdot \frac{Pl^2}{3EJ} - P = 0 \end{cases}$$

$$\begin{cases} \frac{8EJ}{l} \varphi_B - \frac{6EJ}{l^2} \eta_c + 2Pl = 0 \\ -\frac{6EJ}{l^2} \varphi_B + \frac{18EJ}{l^3} \eta_c - 3P = 0 \end{cases}$$

$$\varphi_B = -\frac{1}{6} \frac{Pl^2}{EJ}$$

$$\eta_c = \frac{1}{9} \frac{Pl^3}{EJ}$$



* Determina il momento flessionale

$$y''(x) = -\frac{M(x)}{EJ} = \frac{2xAT}{l}$$

$$M(x) + \frac{5PE}{3} - Px = 0 \Rightarrow M(x) = P(x - \frac{5}{3}l)$$

$$y''(x) = -P(x - \frac{5l}{3}) \frac{1}{EJ} = -2 \cdot \frac{2PE}{3EJ} \leq 0$$

$$-3Px + 5PE - 4PE \leq 0$$

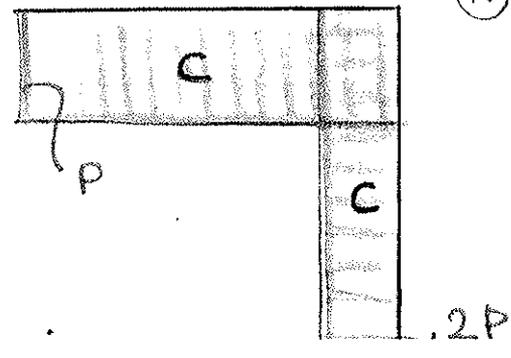
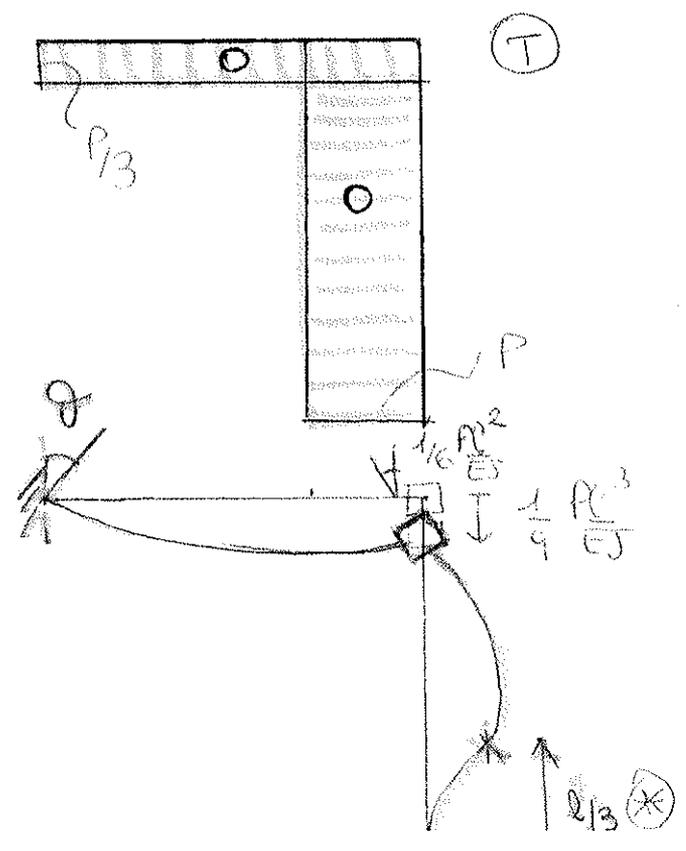
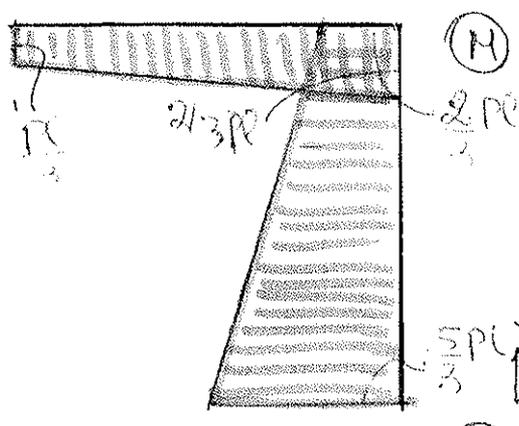
$$-3x \leq -l \quad x \geq \frac{l}{3}$$

$$\rightarrow M_B^{sx} = \frac{2}{AEJ} \left(-\frac{1}{3} \frac{PE^2}{EJ} \right) - \frac{2}{E^2} \left(\frac{1}{9} \frac{PE^3}{EJ} \right) + \frac{2}{3} PE = -\frac{2}{3} PE$$

$$\uparrow M_A = \frac{2EJ}{l} \left(-\frac{1}{3} \frac{PE^2}{EJ} \right) - \frac{2}{l^2} \left(\frac{1}{9} \frac{PE^3}{EJ} \right) + \frac{4}{3} PE = +\frac{PE}{3}$$

$$\uparrow M_C = \frac{2EJ}{l} \left(-\frac{1}{3} \frac{PE^2}{EJ} \right) - \frac{2EJ}{3EJ} \cdot \frac{2PE}{3EJ} = -\frac{5}{3} PE$$

$$\uparrow T_A = \frac{2EJ}{l^2} \left(-\frac{1}{3} \frac{PE^2}{EJ} \right) + \frac{4EJ}{l^3} \left(\frac{1}{9} \frac{PE^3}{EJ} \right) - \frac{2EJ}{l^2} \left(\frac{PE^2}{EJ} \right) = \frac{P}{3}$$

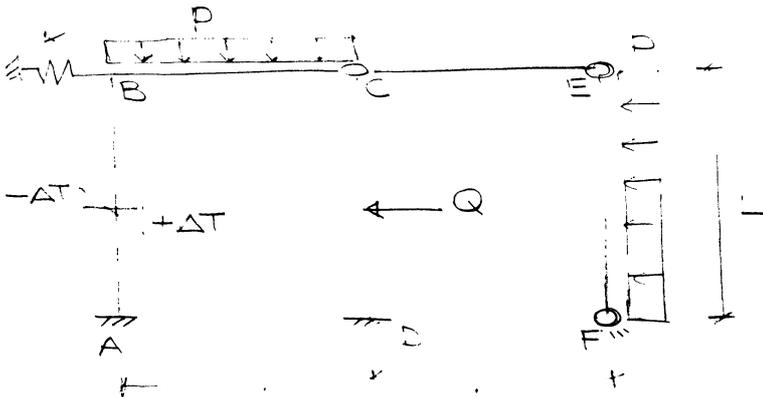


$$K = \frac{3EI}{L^3}$$

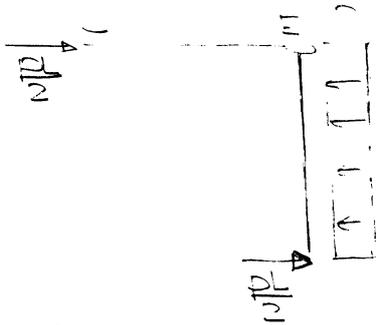
$$Q = \frac{16}{5} DL$$

$$\frac{\Delta \Delta T}{2} = \frac{DL^2}{6EI}$$

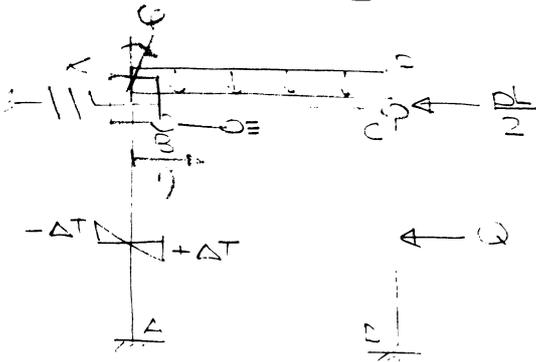
TEMA ESAME 13/07/2009



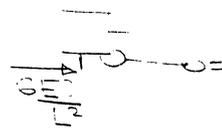
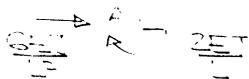
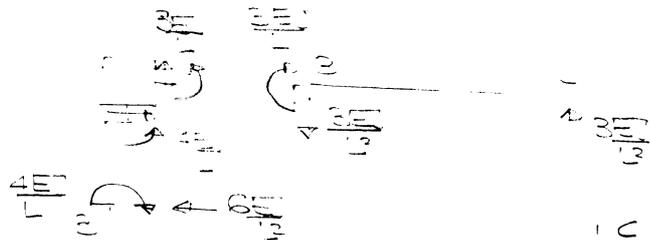
SOLUZIONE APPENDICE COSTATTA CEF



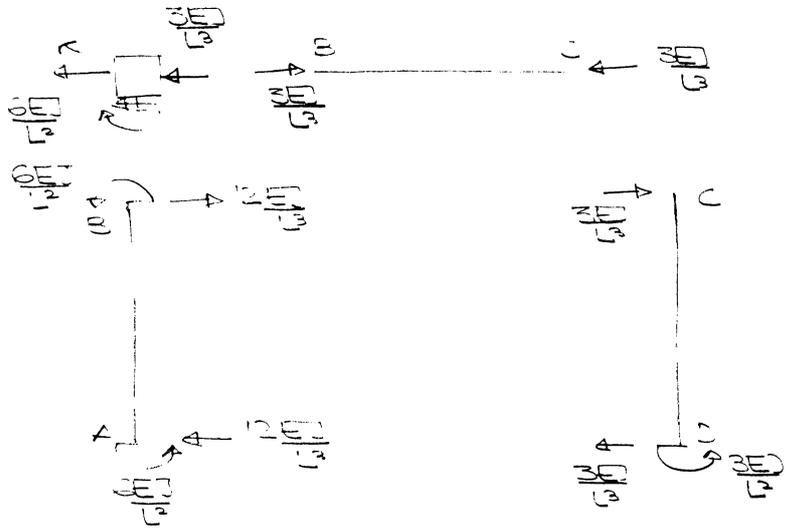
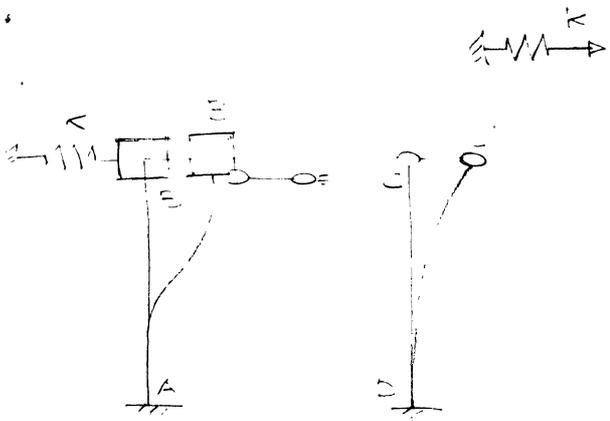
$$\begin{aligned} m_{CB} + m_{BC} + m_{CO} &= 0 \\ n_{CB} + n_{BC} + n_{CO} &= 0 \end{aligned}$$



$$\Delta = 0$$



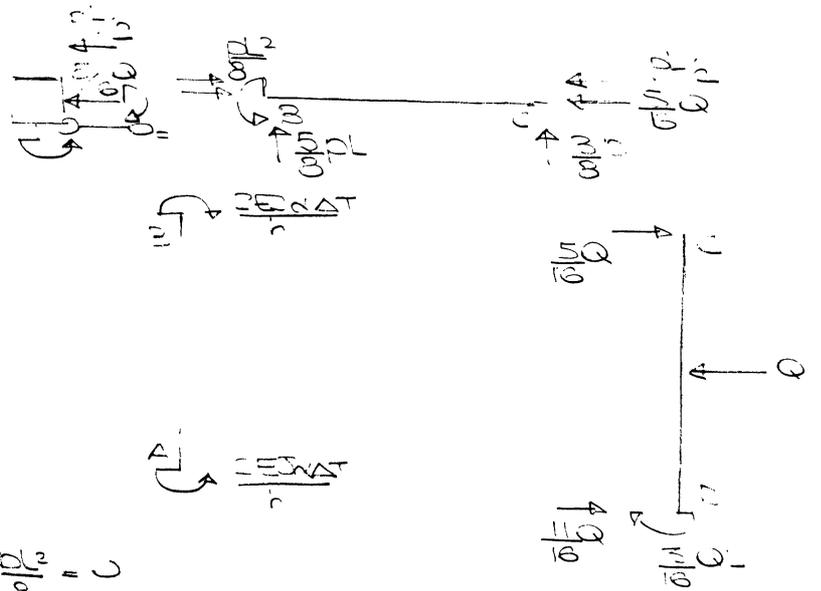
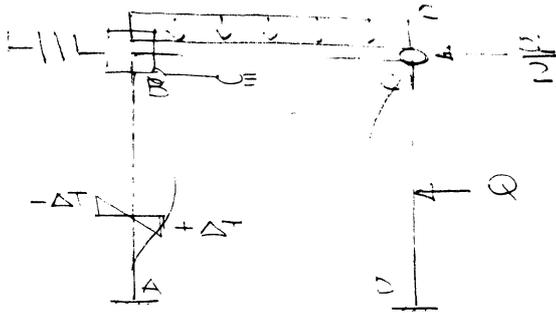
$$\begin{cases} m_{AB} = \frac{4EI}{L} + \frac{3EI}{L} = \frac{7EI}{L} \\ n_{AB} = \frac{6EI}{L^2} \end{cases}$$



$$M_{top} = -6 \frac{EJ}{L^2} h$$

$$h_{top} = -5 \frac{EJ}{L^3} \cdot K = -15 - 3 \frac{EJ}{L^3} = -18 \frac{EJ}{L^3}$$

$P \neq 0 \quad Q \neq 0 \quad \Delta T = 0$



$$M_{top} = \frac{2EJ\alpha\Delta T}{L} - \frac{PQ}{6} = 2 \frac{PQ}{6} - \frac{PQ}{6} = 0$$

$$h_{top} = \frac{EJ}{6} \cdot 0 + \frac{PQ}{2} = -\frac{PQ}{6} \cdot 1 - \frac{PQ}{2} = -\frac{3}{2} PQ$$

$$\frac{7EJ}{L^2} \varphi - 6 \frac{EJ}{L^2} h = 0$$

$$6 \frac{EJ}{L^2} \varphi - 18 \frac{EJ}{L^3} h - \frac{3}{2} PQ = 0$$

$$\varphi = \frac{6h}{7L}$$

$$6 \frac{EJ}{L^2} \varphi - 18 \frac{EJ}{L^3} h = -\frac{3}{2} PQ$$

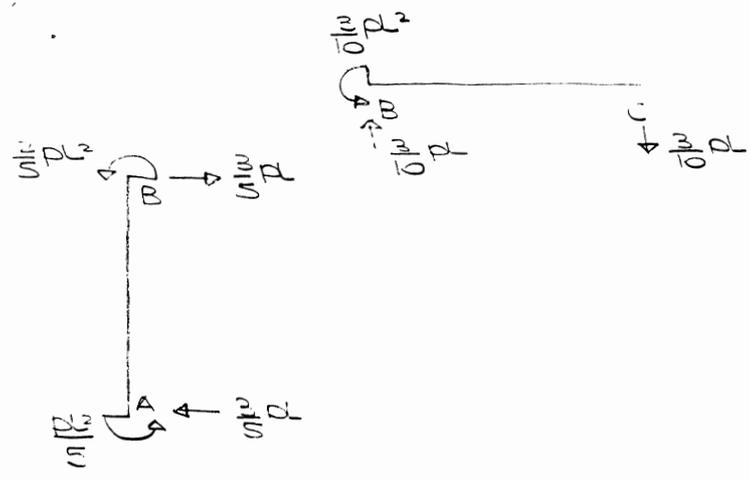
$$\frac{36h}{7} - 18h = \frac{3}{2} PQ$$

$$-\frac{18}{7} \frac{EJ}{L^3} h = \frac{3}{2} PQ$$

$$h = -\frac{7}{60} \frac{PQ}{EJ}$$

$$\varphi = -\frac{PQ}{80} \frac{EJ}{L^2} = -\frac{1}{10} \frac{PQ}{EJ}$$

$$\frac{DL^3}{10EI}$$



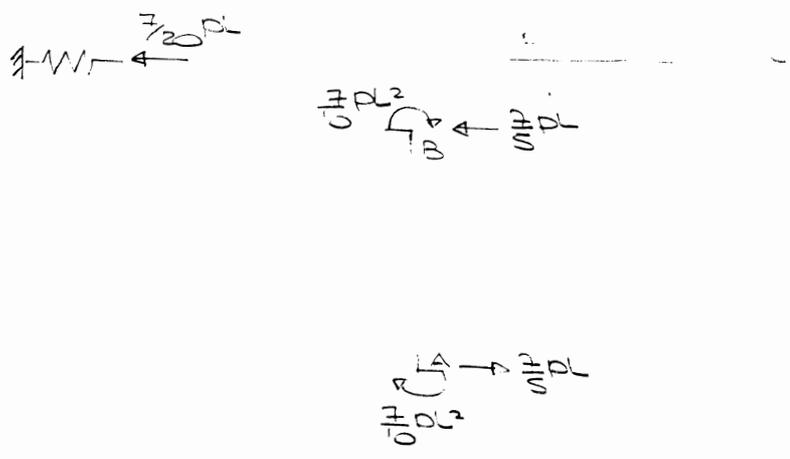
$$\frac{4EI}{L^3} \cdot \frac{1}{10} PL^3 = \frac{1}{10} PL^2$$

$$2 \frac{1}{L} = \frac{1}{10} PL$$

$$6 \frac{1}{L} = \frac{1}{10} PL$$

$$12 \frac{1}{L} = \frac{1}{10} PL$$

$$\eta = - \frac{8}{EI} \frac{PL^3}{L}$$

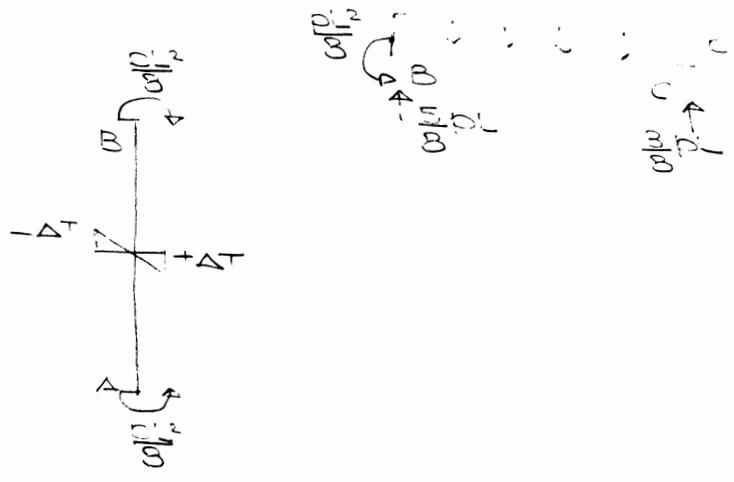


$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

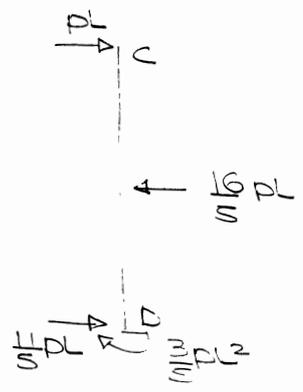
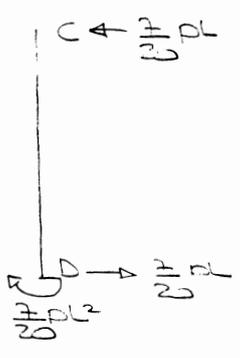
$$\Delta T \neq 0 \quad \omega \neq 0 \quad D \neq 0$$

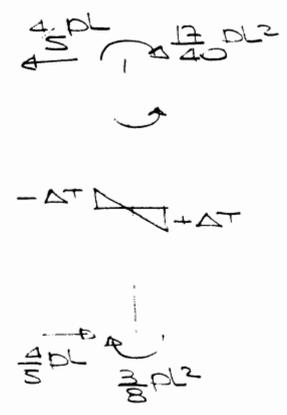
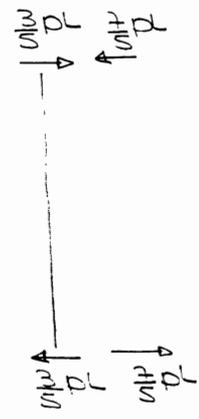


$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

$$\frac{1}{10} PL^2 = \frac{1}{10} PL^2$$

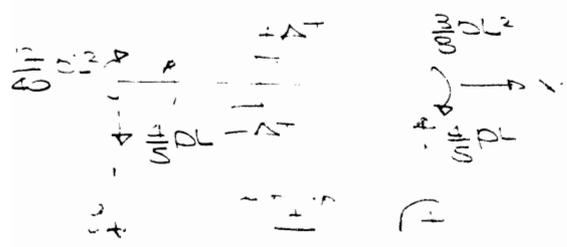




$$\begin{aligned} \uparrow + \sum F_y = 0 & \Rightarrow \frac{17 PL}{40} - \frac{17 PL}{40} = 0 \\ \curvearrowright + \sum M = 0 & \Rightarrow \frac{17 PL^2}{40} - \frac{17 PL^2}{40} = 0 \end{aligned}$$

$\sum M = 0$
 $M - \frac{17 PL^2}{40} + \frac{17 PL}{40} x = 0$
 $\sum F = 0$
 $\frac{17 PL}{40} - \frac{17 PL}{40} = 0$
 $\frac{17 PL}{40} = \frac{17 PL}{40} x \quad x = \frac{17 L}{32}$

total curvature



$$\begin{aligned} M &= \frac{17 PL^2}{40} - \frac{17 PL}{40} x \\ \sum F_y &= 0 = \frac{17 PL}{40} - \frac{17 PL}{40} x \\ \sum M &= 0 = \frac{17 PL^2}{40} - \frac{17 PL}{40} x \\ \frac{17 PL^2}{40} &= \frac{17 PL}{40} x \quad x = \frac{17 L}{32} \end{aligned}$$





In $x = \frac{w}{2P}$ momento es máximo

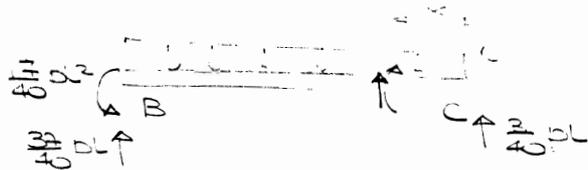
$$M - \frac{w}{2} x^2 + \frac{w}{2} x^2 = 0$$

$$M = \frac{w}{2} x^2 - \frac{w}{2} x^2$$

$$M = 0 \quad \frac{w}{2} x^2 = \frac{w}{2} x^2 \quad x = \frac{w}{2P}$$

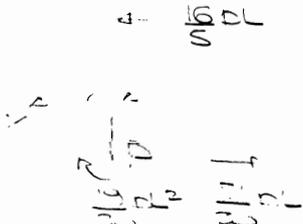
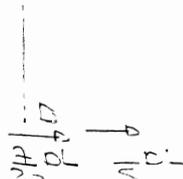
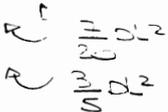
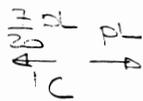
$$M(x) = \frac{w}{2} x^2 - \frac{w}{2} x^2 = \frac{w}{2} \left(\frac{w}{2P} \right)^2 = \frac{w^3}{80} - \frac{w^3}{3200} = \frac{w^3}{3200}$$

$$\frac{dM}{dx} = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right)$$



$$T + \frac{w}{2} L - Px = 0$$

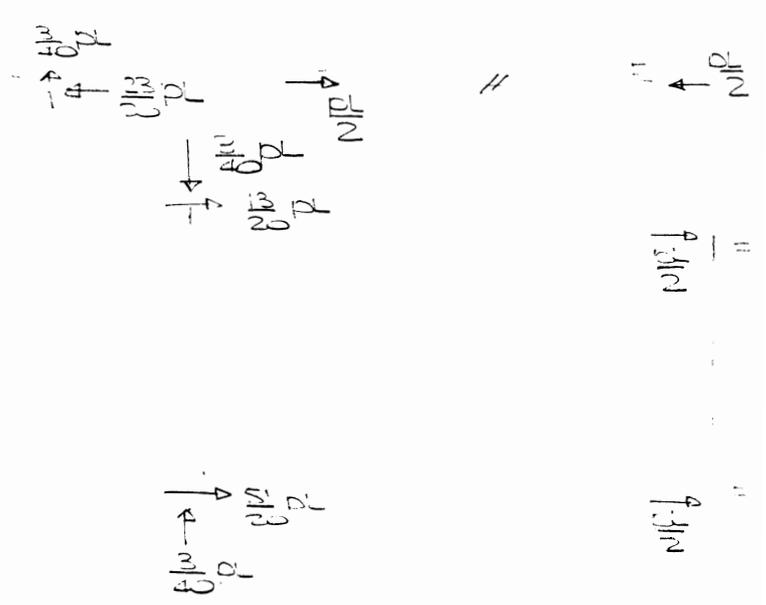
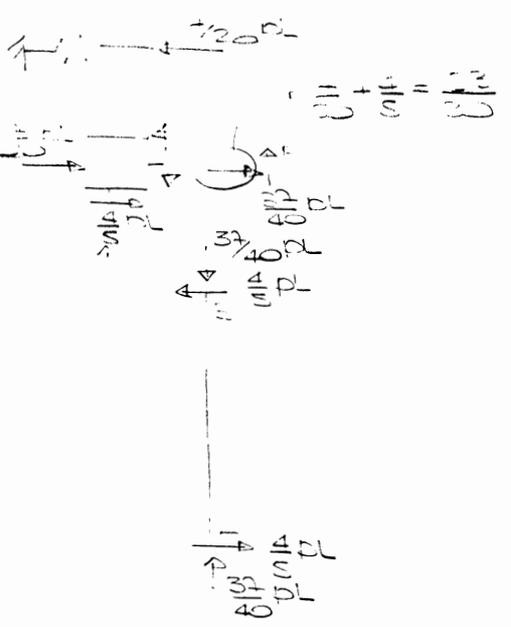
$$T = 0 \quad \frac{w}{2} L = Px \quad x = \frac{wL}{2P}$$



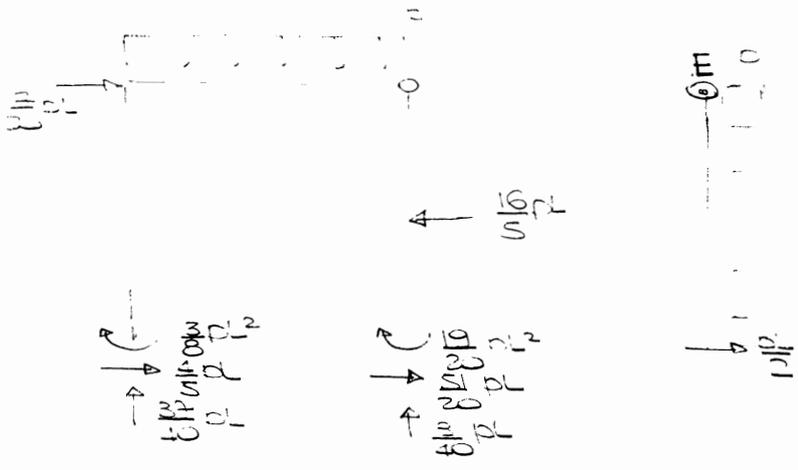
$$\frac{dM}{dx} = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right)$$

$$0 = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right) = \frac{d}{dx} \left(\frac{w}{2} x^2 - \frac{w}{2} x^2 \right)$$

$$x = \frac{wL}{2P}$$



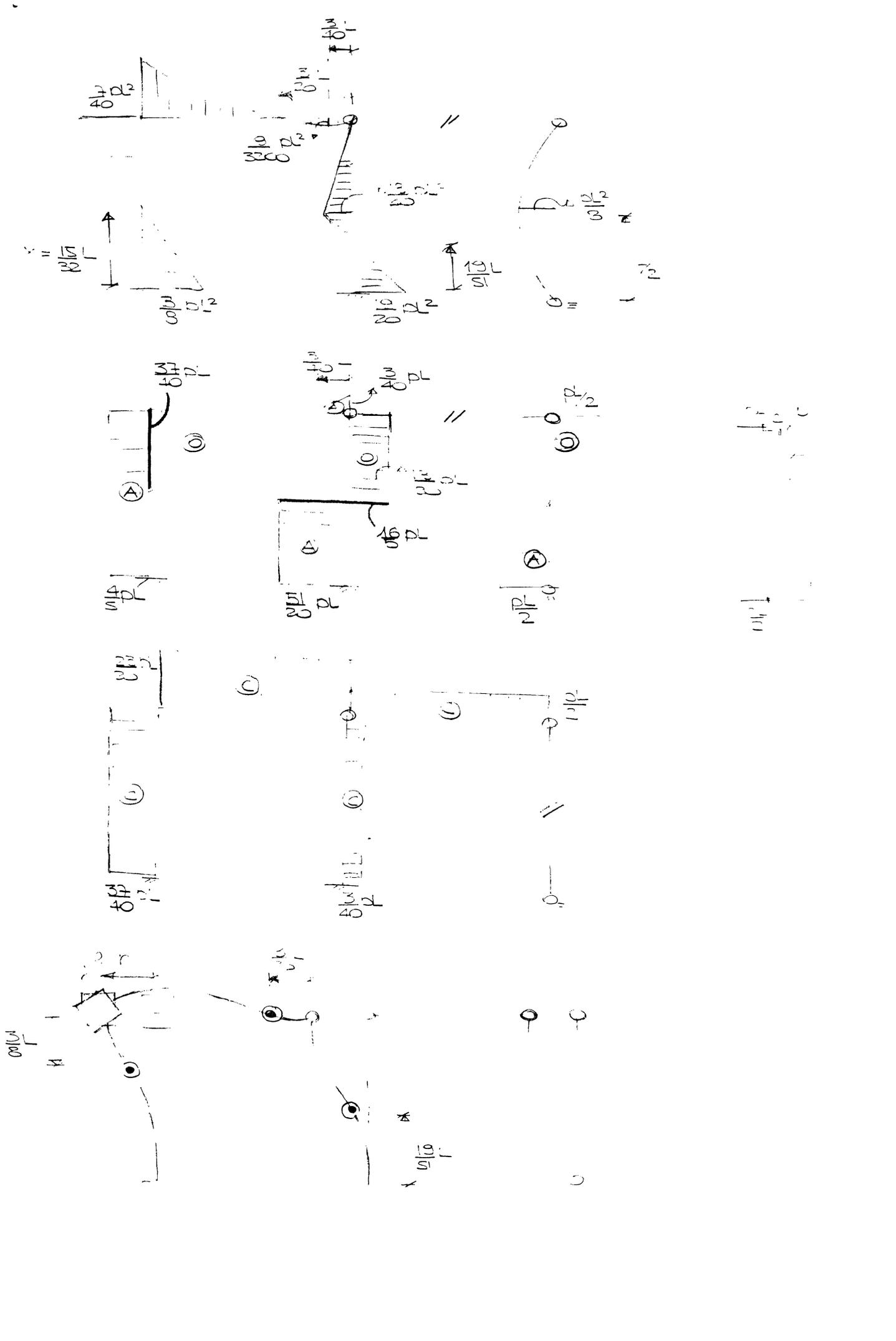
VERIFICA



$$\sum M_x = 0 \quad 2 \cdot 120 + \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 + \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 = \frac{29 - 24 - 5}{0} = 0$$

$$\sum F_y = 0 \quad 1 - \frac{100}{40} - \frac{100}{40} = \frac{40 - 100 - 100}{40} = 0$$

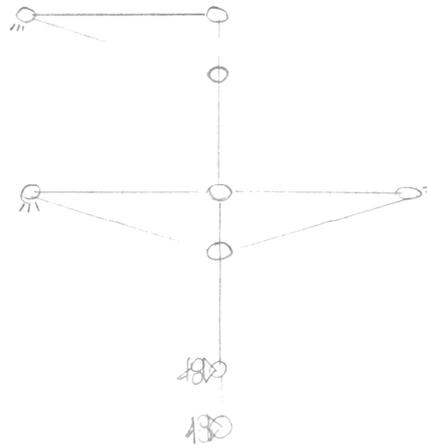
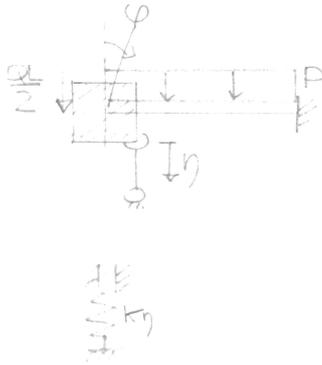
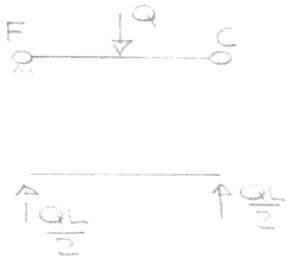
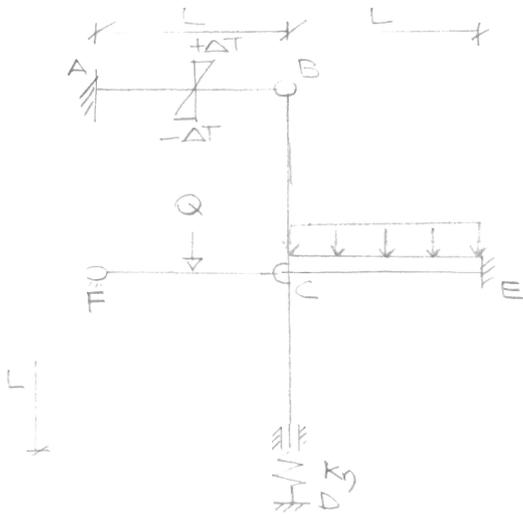
$$\sum M_E = 0 \quad 2 + \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 + \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 + \frac{1}{2} \cdot 100 \cdot 2 - \frac{1}{2} \cdot 100 \cdot 2 = \frac{100 - 100 - 100 + 100 + 100 - 100}{0} = 0$$



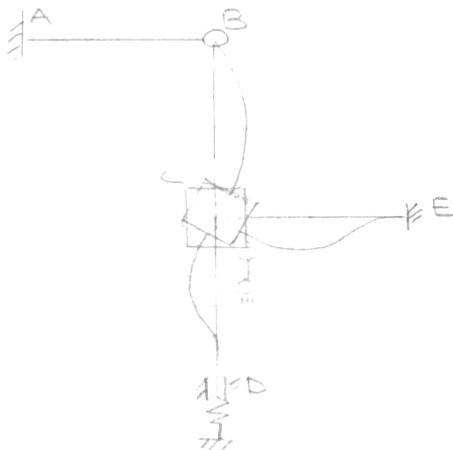
$$Q = PL$$

$$k_{\eta} = \frac{3EI}{L^3}$$

$$\frac{\chi \Delta T}{h} = \frac{1}{2} \frac{PL^2}{EI}$$

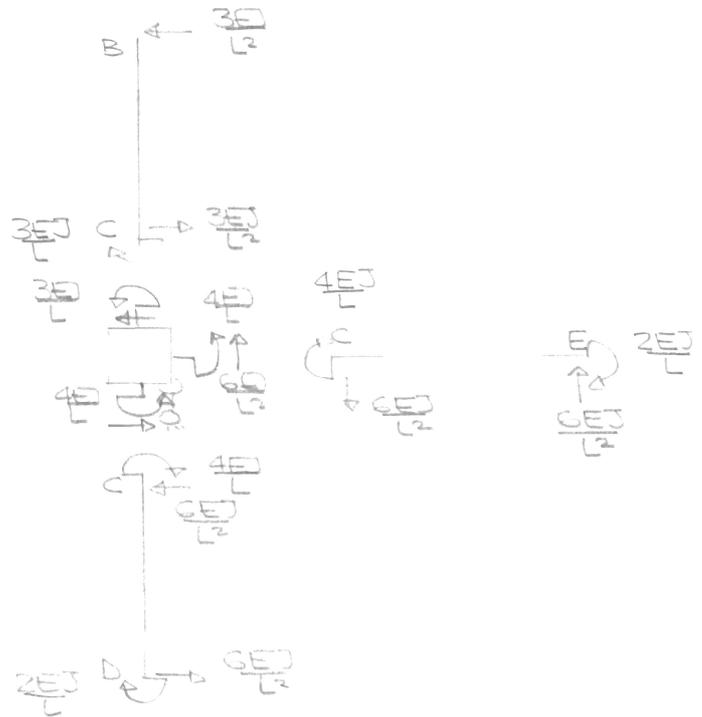


$$\psi = 1 \quad \eta = 0 \quad \rho = 0 \quad \Delta T = 0$$

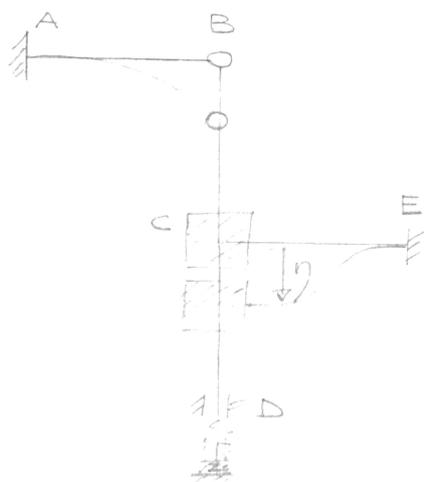


$$M_{cap} = 3 \frac{EI}{L} + 4 \frac{EI}{L} + 4 \frac{EI}{L} = 11 \frac{EI}{L}$$

$$M_{cap} = - \frac{6EI}{L}$$

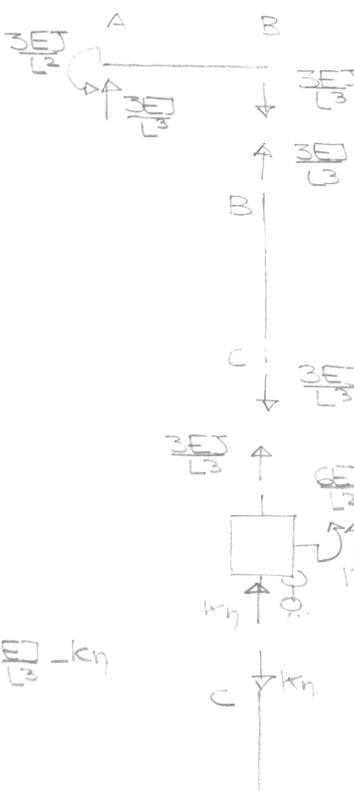


$$\eta = 1 \quad \varphi = Q = p = \Delta T = 0$$



$$m_{c\eta} = \frac{6EI}{L^2}$$

$$h_{c\eta} = -\frac{3EI}{L^3} - \frac{12EI}{L^3} - k_{\eta} = -\frac{15EI}{L^3} - k_{\eta}$$



†



$$\eta = \frac{FL^3}{3EI}$$

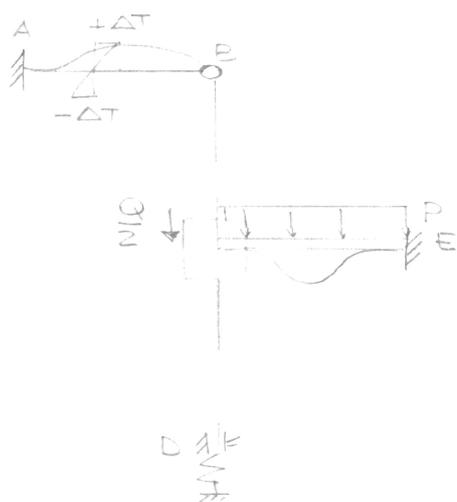
$$F = \frac{3EI}{L^3} \eta$$

$$\eta = 1$$

$$F = \frac{3EI}{L^3}$$

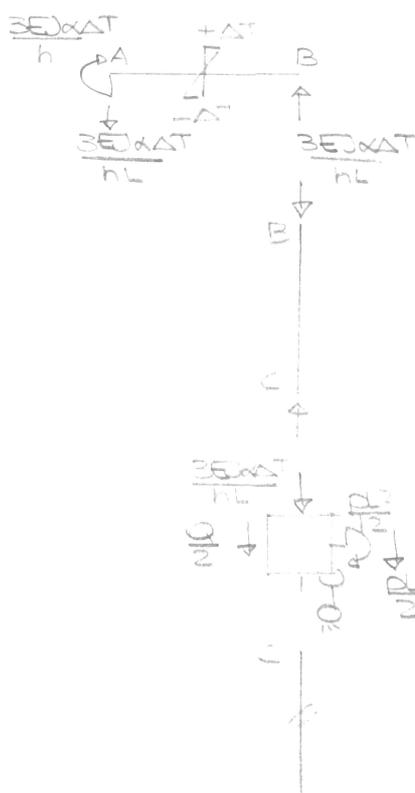


$$\Delta T \neq 0 \quad p \neq 0 \quad Q \neq 0 \quad \eta = \varphi = 0$$



$$m_{c0} = -\frac{pL^2}{2}$$

$$h_{c0} = \frac{pL}{2} + \frac{3EI\alpha\Delta T}{hL} + \frac{pL}{2}$$



$$11 \frac{EJ}{L} \varphi + \frac{6EJ}{L^2} \eta - \frac{PL^2}{12} = 0$$

$$- \frac{6EJ}{L^2} \varphi - (15 \frac{EJ}{L^3} + k\eta) \eta + \frac{P}{2} + \frac{3EJ\alpha\Delta T}{L} + \frac{PL}{2} = 0$$

$$\begin{cases} 11 \frac{EJ}{L} \varphi + \frac{6EJ}{L^2} \eta - \frac{PL^2}{12} = 0 & \times 3 \\ - \frac{6EJ}{L^2} \varphi - 18 \frac{EJ}{L^3} \eta + \frac{P}{2} + \frac{3}{2} PL + \frac{PL}{2} = 0 \end{cases}$$

$$- \frac{6EJ}{L^2} \varphi - 18 \frac{EJ}{L^3} \eta + \frac{5}{2} PL = 0 \quad (+)$$

$$33 \frac{EJ}{L^2} \varphi + 18 \frac{EJ}{L^3} \eta - \frac{PL}{4} = 0$$

$$27 \frac{EJ}{L^2} \varphi + \frac{9}{4} PL = 0 \quad \cancel{27} \frac{EJ}{L^2} \varphi = - \frac{9}{4} PL$$

$$\boxed{\varphi = - \frac{1}{12} \frac{PL^3}{EJ}}$$

$$11 \frac{EJ}{L} \left(- \frac{PL^3}{12EJ} \right) + \frac{6EJ}{L^2} \eta - \frac{PL^2}{12} = 0$$

$$- \frac{11}{12} PL^2 - \frac{PL^2}{12} = - \frac{6EJ}{L^2} \eta$$

$$+ PL^2 = \frac{6EJ}{L^2} \eta$$

$$\boxed{\eta = \frac{1}{6} \frac{PL^2}{EJ}}$$

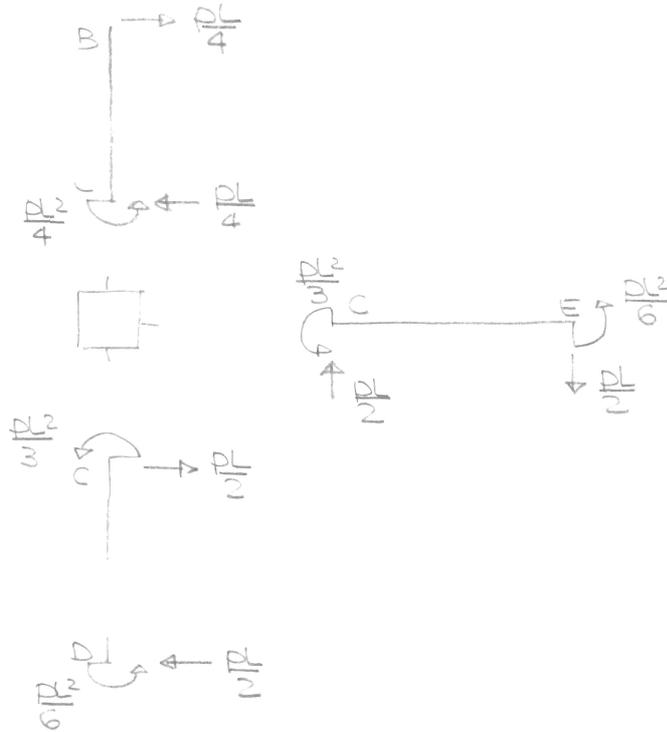
$$\varphi = - \frac{1}{12} \frac{PL^3}{EJ}$$

$$4 \cdot \frac{1}{12 \cdot 3} = \frac{1}{3}$$

$$2 \cdot \frac{1}{12 \cdot 6} = \frac{1}{6}$$

$$6 \cdot \frac{1}{12 \cdot 2} = \frac{1}{2}$$

$$2 \cdot \frac{1}{12 \cdot 4} = \frac{1}{4}$$



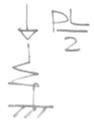
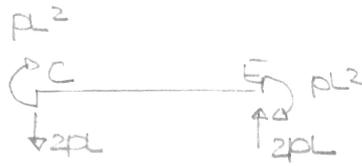
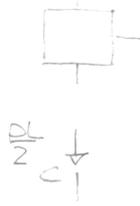
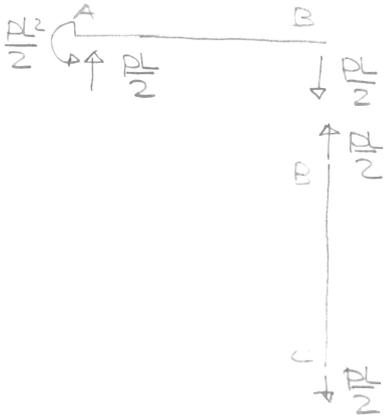
$$\eta = \frac{1}{6}$$

$$\eta = 3 \cdot \frac{1}{6} = \frac{1}{2}$$

$$3 \cdot \frac{1}{6} = \frac{1}{2}$$

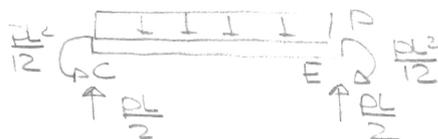
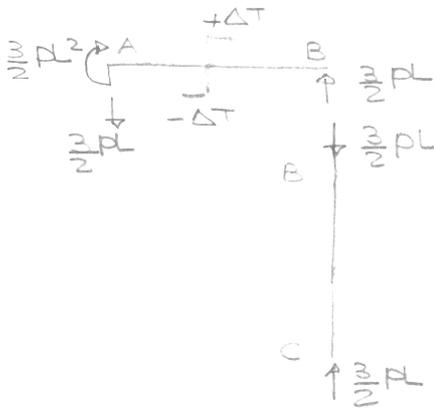
$$6 \cdot \frac{1}{6} = 1$$

$$12 \cdot \frac{1}{6} = 2$$

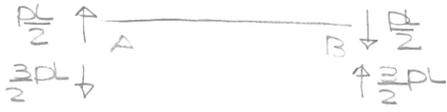


$$\frac{2\Delta T}{h} = \frac{PL^2}{2EI} \quad P \neq 0 \quad Q = PL$$

$$3EI \cdot \frac{PL^2}{2EI} = \frac{3}{2} PL^2$$



ASTA AB



$$\begin{aligned} \curvearrowright \frac{PL}{2} - \frac{1}{2} &= 1 \\ + \downarrow \frac{PL}{2} - \frac{1}{2} &= \\ + \uparrow \frac{PL}{2} - \frac{1}{2} &= 1 \end{aligned}$$



$$M - PL^2 + PLX = 0$$

$$M = PL^2 - PLX$$

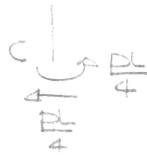
$$y'' = -\frac{M}{EI} = \frac{PLX}{EI} - \frac{PL^2}{EI} + \frac{2 \cdot PL^2}{2EI} \geq 0$$

$$PLX - PL^2 + PL^2 \geq 0$$

$$X \geq 0 \quad (+) \text{ sempre}$$

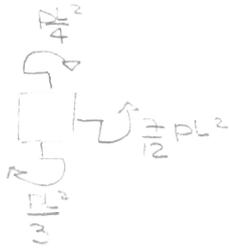
ASTA BC

B

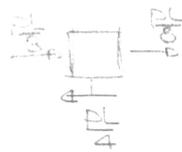
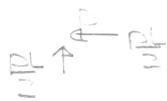
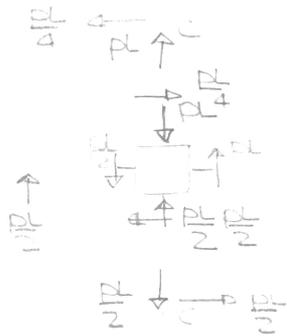


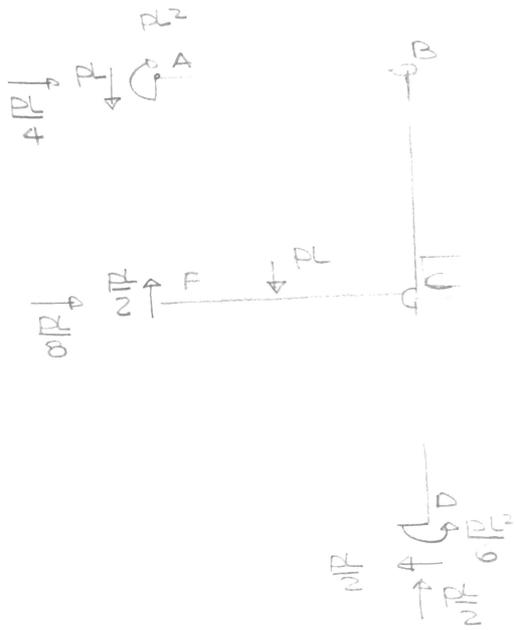


Verif al nodo C



$$\frac{1}{4} + \frac{1}{4} - \frac{1}{2} = \frac{1+1-2}{2} = 0$$





$$\sum F_y = 0$$

$$\downarrow + PL - \frac{PL}{2} - PL + PL = 2PL - \frac{PL}{2} = 0$$

$$\sum F_x = 0$$

$$\rightarrow + \frac{PL}{4} + \frac{PL}{8} + \frac{PL}{8} - \frac{PL}{2} = 0$$

$$\sum M_A = 0$$

$$\uparrow +$$

$$-PL^2 + \frac{PL^2}{8} - \frac{PL^2}{2} - \frac{3}{2}PL^2 - \frac{11}{12}PL^2$$

$$+ 4PL^2 + \frac{PL^2}{8} + \frac{PL^2}{6} + \frac{PL^2}{2} -$$

$$- PL^2 =$$

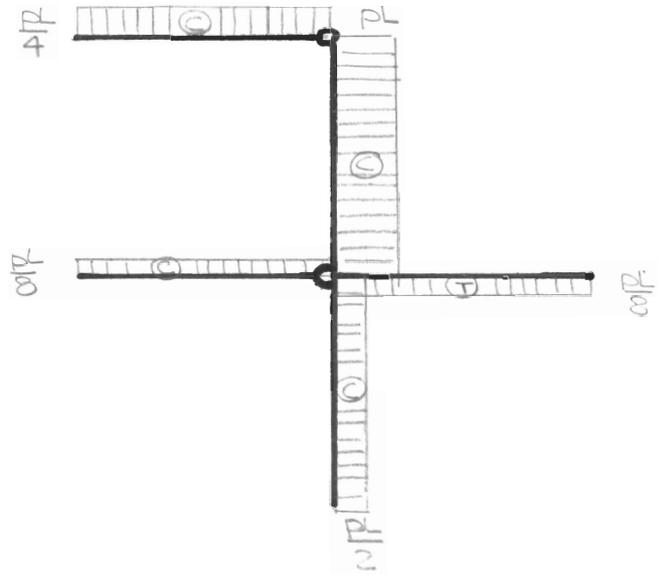
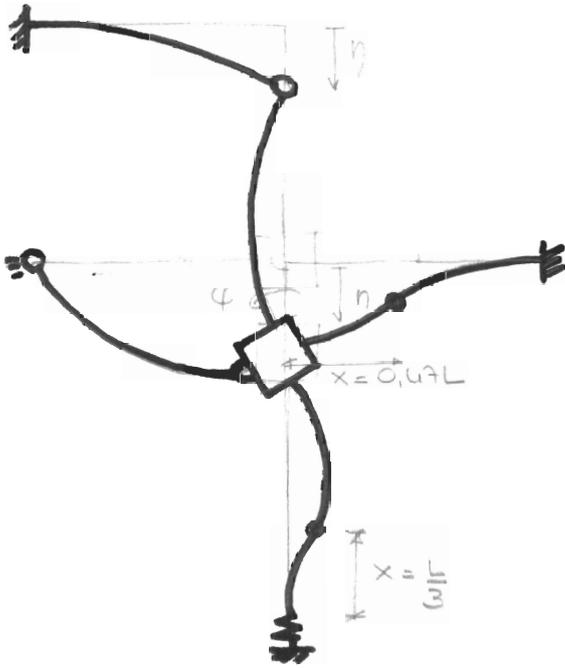
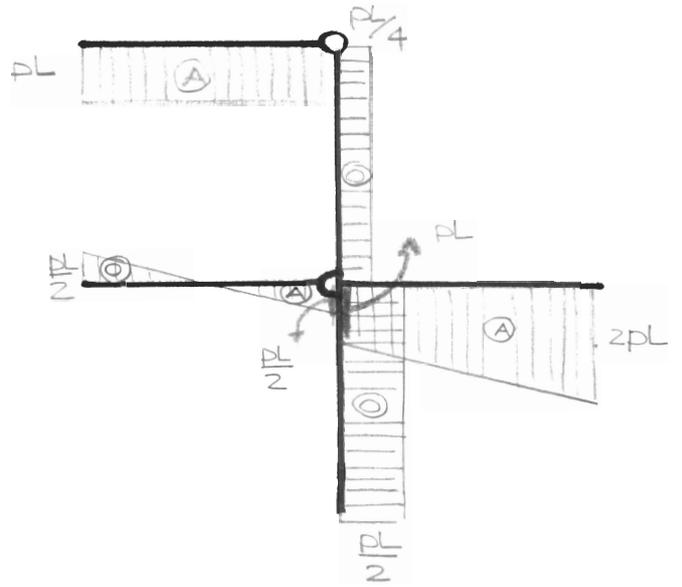
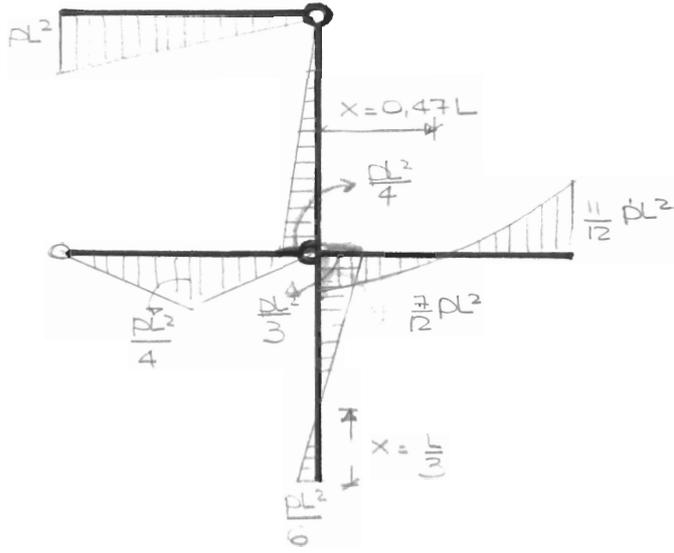
$$2 - \frac{3}{2} + \frac{1}{4} + \frac{1}{6} - \frac{11}{12} =$$

$$= \frac{24 - 18 + 3 + 2 - 11}{12} = 0$$

$$E \curvearrowright \frac{11}{12} PL^2$$

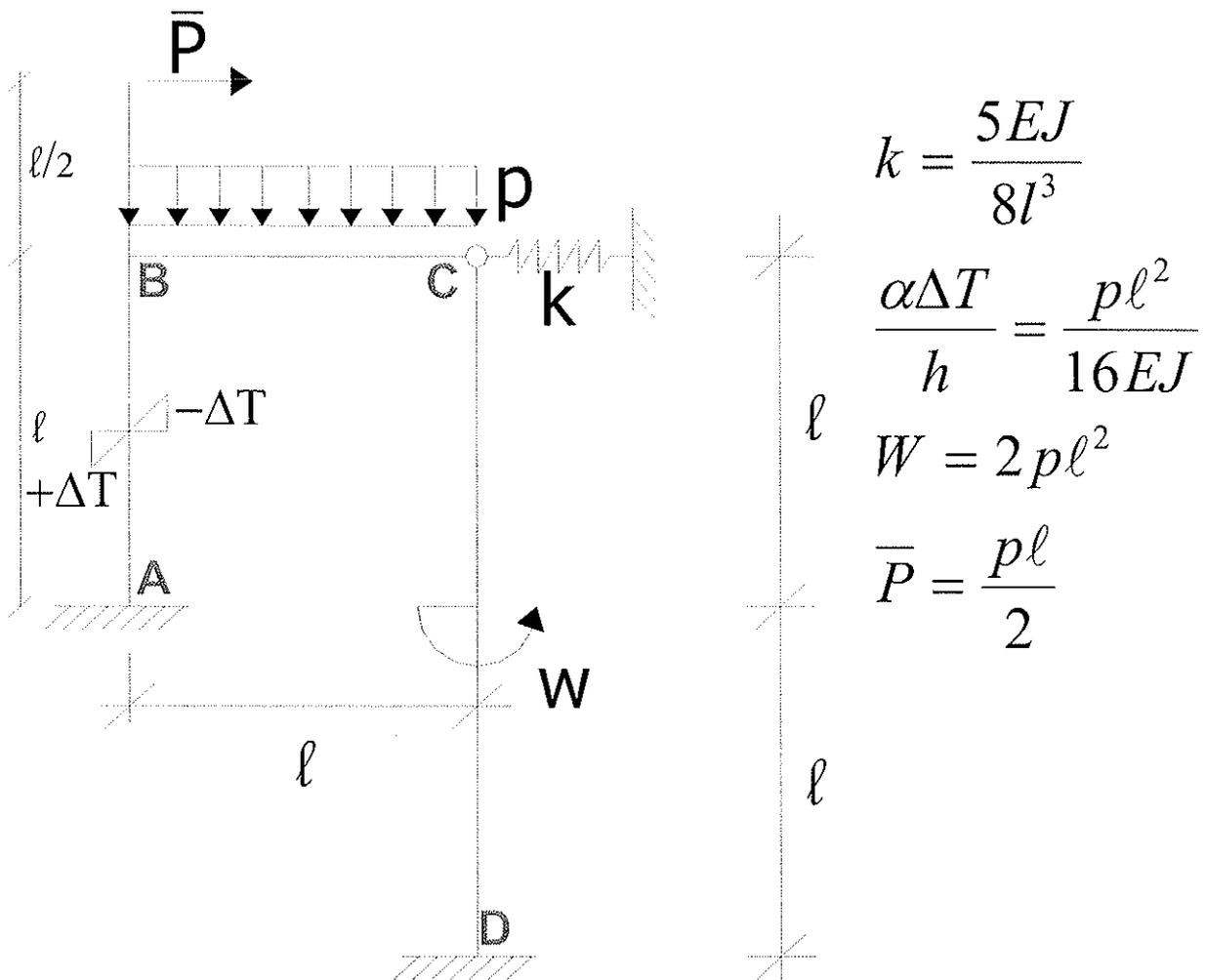
$$\uparrow 2PL$$

$$\rightarrow \frac{PL}{3}$$



Esame di
FONDAMENTI di
PROGETTAZIONE STRUTTURALE
(Corso di laurea in Ingegneria Civile N.O.)

7 – 09 – 2009



Si richiedono i grafici di:

- Momento flettente (con il valore e la posizione dei massimi)
- Taglio
- Azione assiale
- Deformata qualitativa con posizione dei flessi

SOLUZIONE

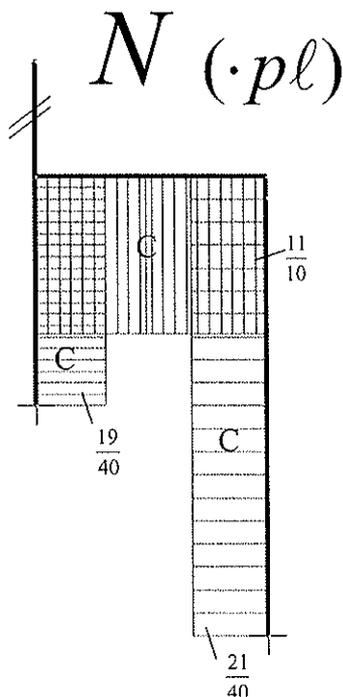
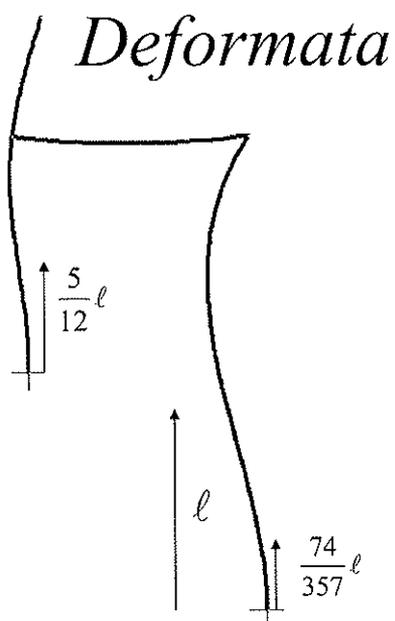
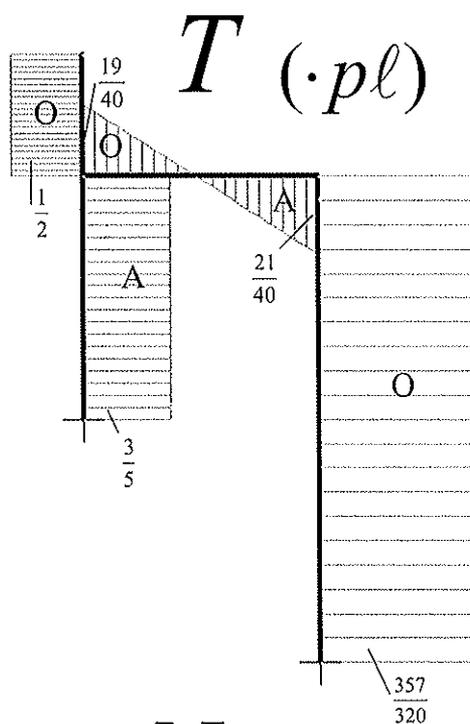
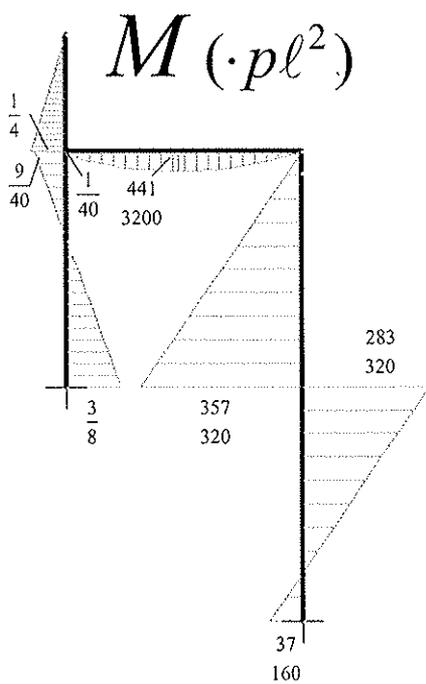
Sistema risolvente

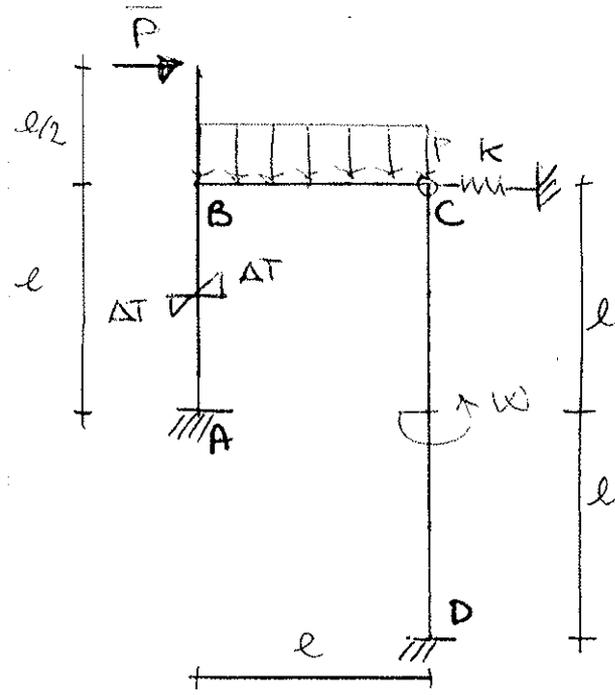
$$\begin{cases} \frac{7EJ}{l} \varphi_B - \frac{6EJ}{l^2} \eta_B - \frac{pl^2}{2} = 0 \\ -\frac{6EJ}{l^2} \varphi_B + \frac{13EJ}{l^3} \eta_B + \frac{5pl}{8} = 0 \end{cases}$$

Soluzioni

$$\begin{cases} \varphi_B = \frac{pl^3}{20EJ} \\ \eta_B = -\frac{pl^4}{40EJ} \end{cases}$$

Diagrammi delle azioni interne e Deformata





$$K = \frac{5}{8} \frac{EI}{e^3}$$

$$\frac{d\Delta\Gamma}{h} = \frac{1}{16} \frac{pe^2}{EI}$$

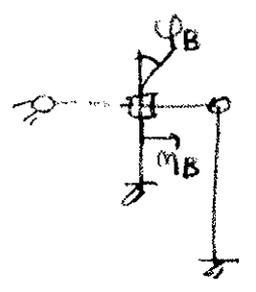
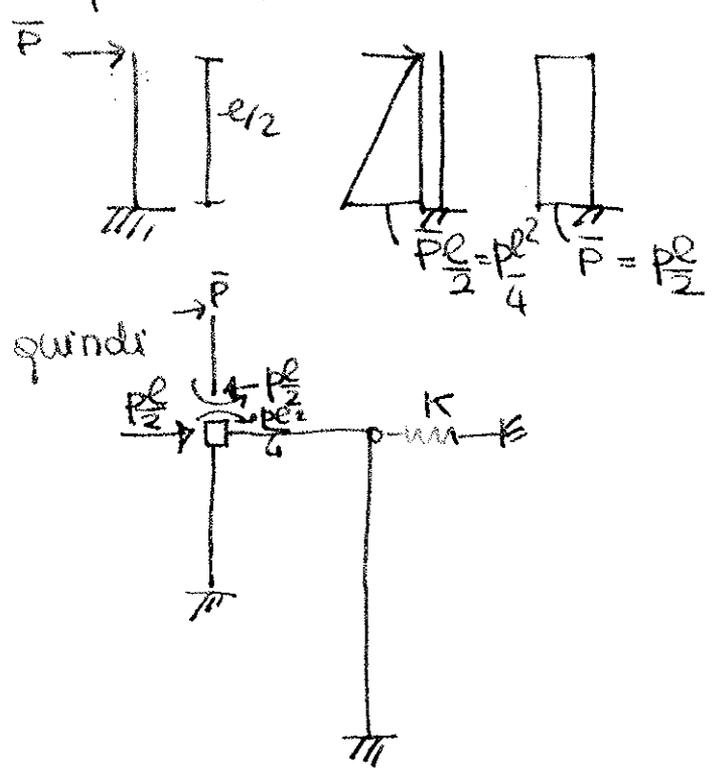
$$\omega = 2pe^2$$

$$\bar{P} = pe/2$$

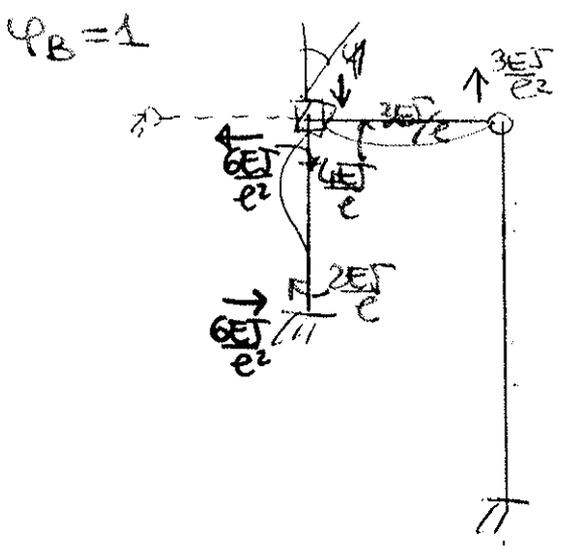
$$u_{BB} \varphi_B + u_{B\eta} \eta + u_{B\omega} = 0$$

$$h_{\eta B} \varphi_B + h_{\eta\eta} \eta + h_{\eta\omega} = 0$$

Tolgo l'appendice isostatica

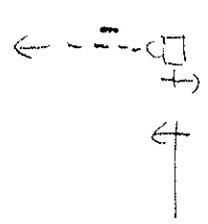


CONVENZIONI

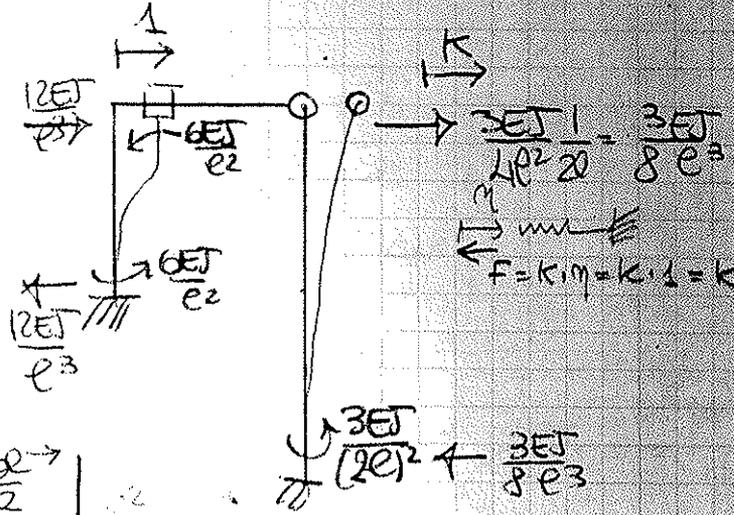
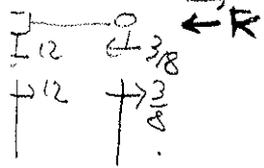


$$M_{BB} = \frac{7EI}{e}$$

$$h_{\eta B} = -\frac{6EI}{e^2}$$



$$M_B = 1$$



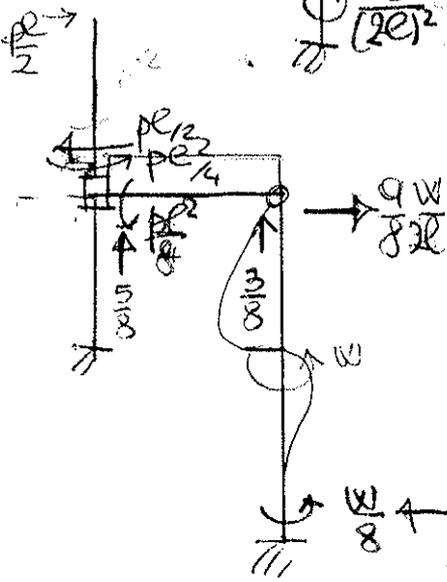
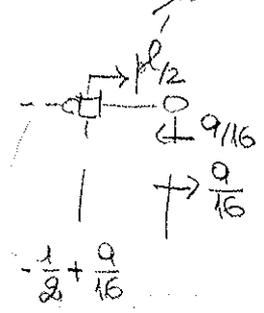
$$m_{B\eta} = -\frac{6EJ}{e^2}$$

$$h_{m\eta} = \frac{12EJ}{e^3} + \frac{3EJ}{8e^3} + k$$

$$= \frac{99EJ}{8e^3} + k =$$

$$= \frac{99}{8} + \frac{5}{8} \frac{EJ}{e^3} = 13 \frac{EJ}{e^3}$$

$$P \neq 0$$



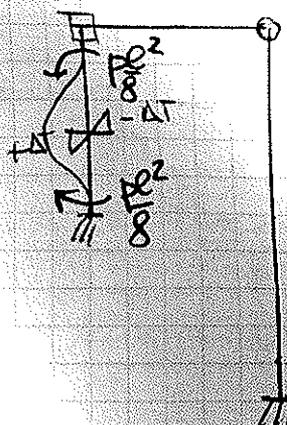
$$u_{B0}^P = -\frac{Pl^2}{8} - \frac{Pl^2}{4} = -\frac{3}{8}Pl^2$$

$$h_{m0}^P = -\frac{Pl}{2} + \frac{9}{82e} = -\frac{P}{2} + \frac{9}{16} \frac{2Pl^2}{e}$$

$$= \frac{10}{16} = \frac{5}{8}Pl$$

$$\frac{W}{8} + \frac{9W}{82e} = \frac{9}{16} \cdot 2Pl = \frac{9}{8}Pl$$

$$\Delta T \neq 0$$



$$h_{m0}^{\Delta T} = 0$$

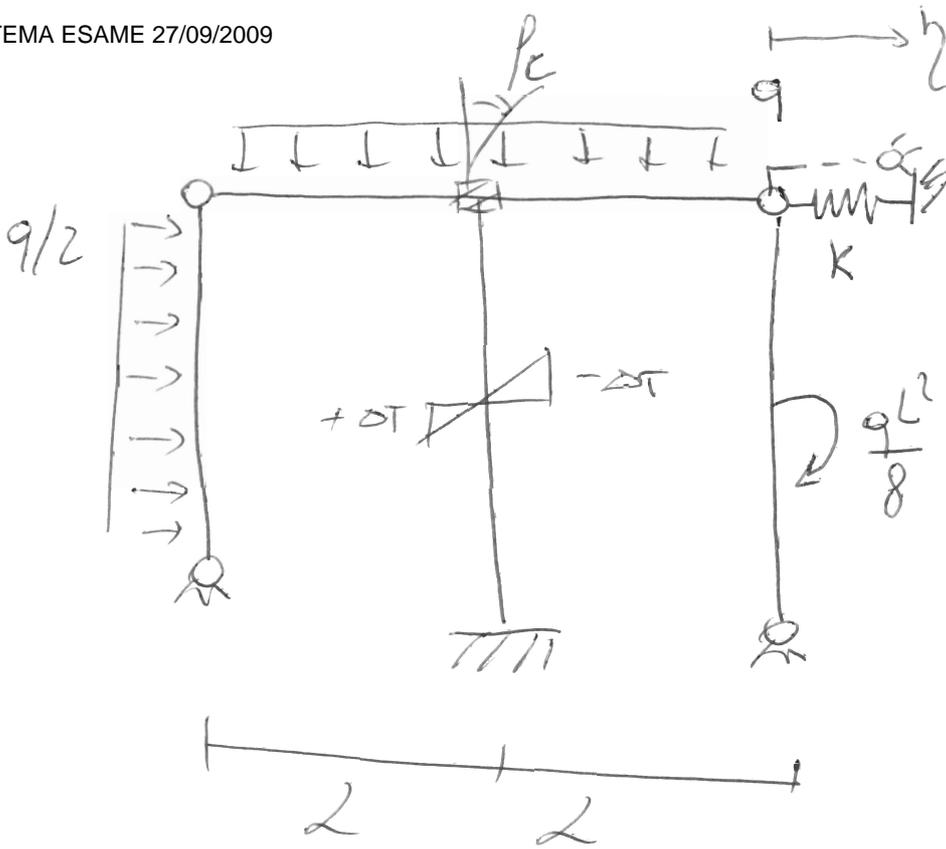
$$u_{B0}^{\Delta T} = -2EJ \frac{\Delta T}{l} = -2EJ \cdot \frac{1}{16} \frac{Pl^2}{EJ} = -\frac{Pl^2}{8}$$

Sistema involuta

$$\left. \begin{aligned} 7EJ \psi_B - \frac{6EJ}{e^2} \eta - \frac{3}{8}Pl^2 - \frac{Pl^2}{8} &= 0 \\ -\frac{6EJ}{e^2} \eta + \frac{13EJ}{e^3} \eta + \frac{5}{8}Pl &= 0 \end{aligned} \right\} \begin{aligned} 7EJ \psi_B - \frac{6EJ}{e^2} \eta - \frac{Pl^2}{2} &= 0 \\ -\frac{6EJ}{e^2} \eta + \frac{13EJ}{e^3} \eta + \frac{5}{8}Pl &= 0 \end{aligned}$$

soluzione del sistema involuta

$$\left. \begin{aligned} \psi_B &= \frac{Pl^3}{20EJ} \\ \eta &= -\frac{Pl^4}{40EJ} \end{aligned} \right\}$$

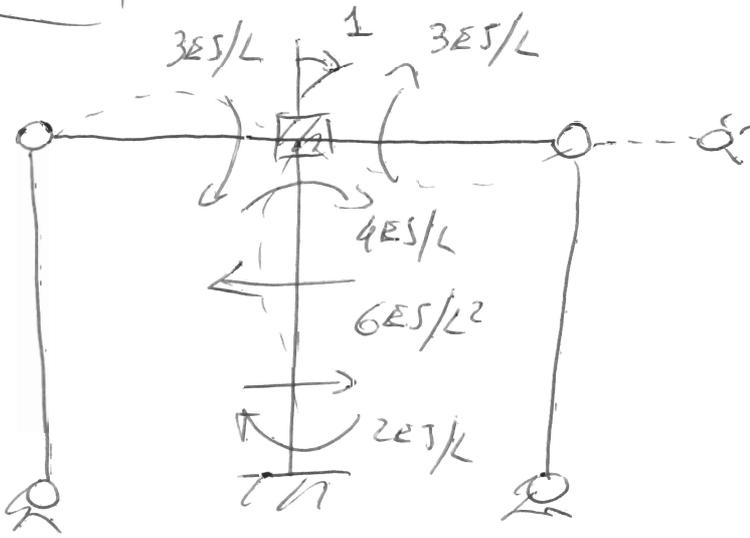


$$k = \frac{24}{5} \frac{EJ}{L^3}$$

$$\frac{\alpha \Delta T}{\epsilon} = 1 \quad \frac{qL^2}{8} \frac{1}{EJ}$$



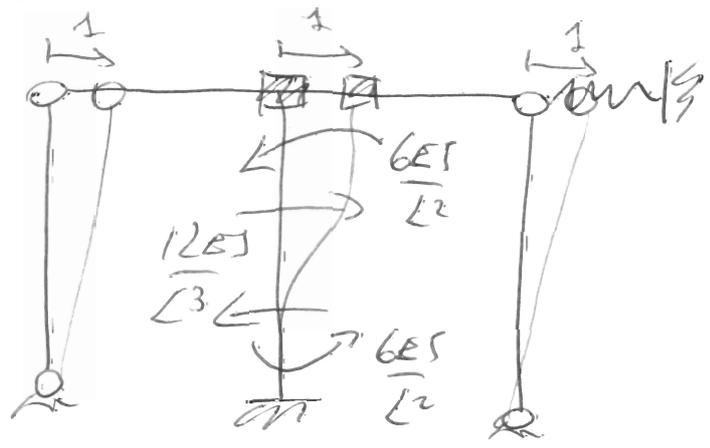
$P_c = 1$



$$m_{cc} = 10 \frac{EJ}{L}$$

$$h_{yc} = \frac{6ET}{L^2}$$

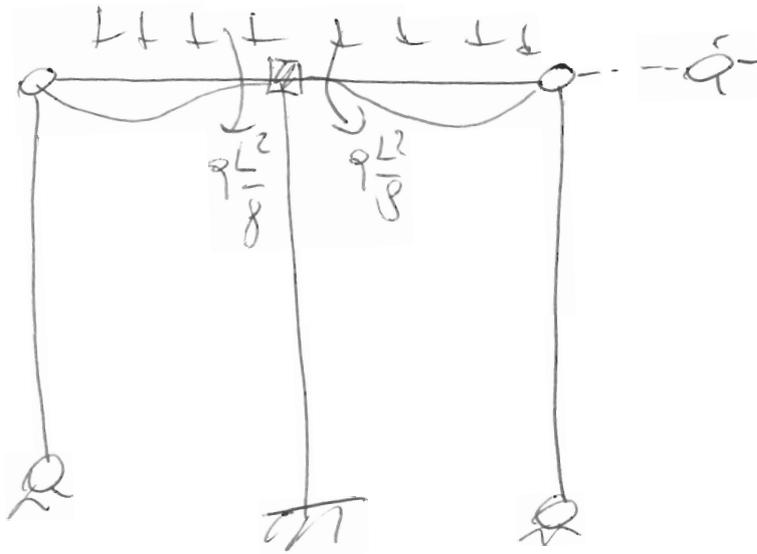
$q = 1$



$$m_{ch} = -\frac{6ET}{L^2}$$

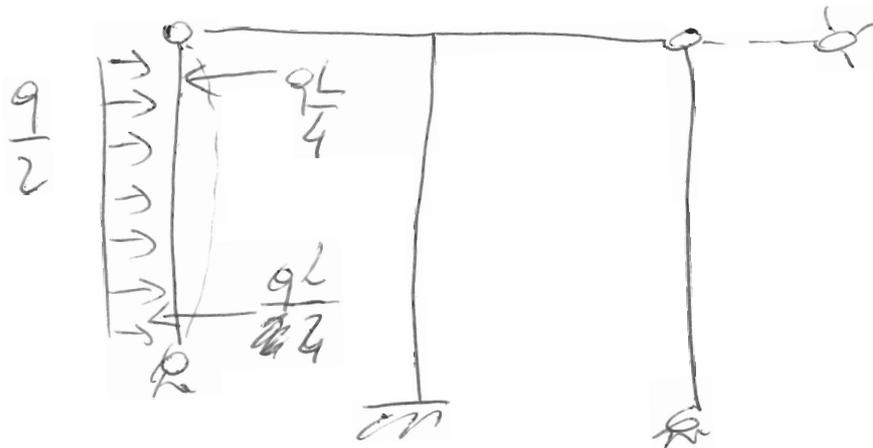
$$h_{yh} = -\frac{12ET}{L^3} - k = -\frac{81}{5} \frac{ET}{L^3}$$

Q#01



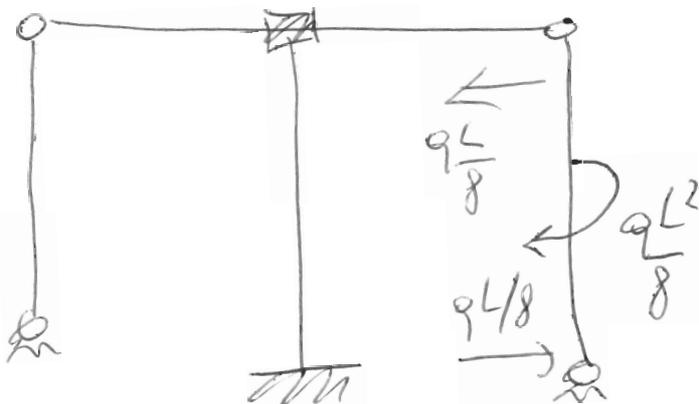
$$M_{B0}^q = 0$$

$$h_{y0}^q = 0$$



$$M_{B0}^q = 0$$

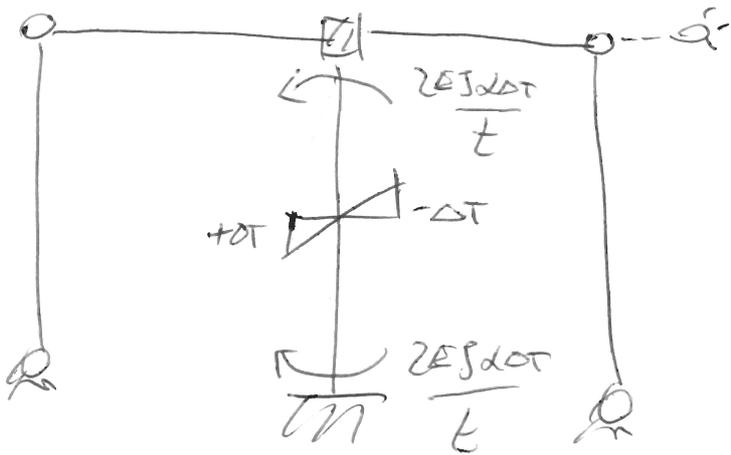
$$h_{y0}^q = \frac{qL}{4}$$



$$M_{B0}^q = 0$$

$$h_{y0}^q = + \frac{qL}{8}$$

$$\Delta T \neq 0$$



$$M_{B0}^{\Delta T} = - \frac{2EJ\delta\Delta T}{t}$$

$$h_{y0}^{\Delta T} = 0$$

$$\left\{ \frac{10 EJ}{L} f_c - \frac{6ET}{L^2} h - \frac{2EJ\delta\Delta T}{t} = 0 \right.$$

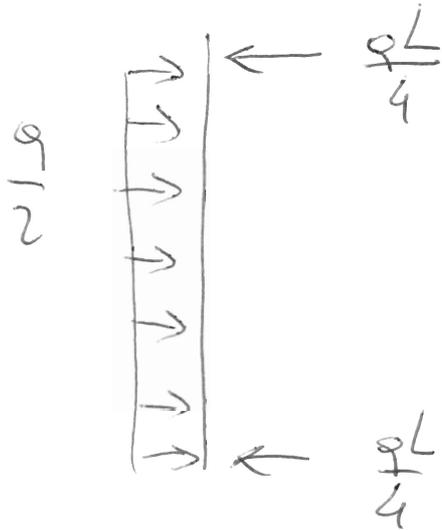
$$\left. \frac{6EJ}{L^2} f_c - \left(12+k\right) \frac{ET}{L^3} h + \frac{qL}{4} + \frac{qL}{8} = 0 \right.$$

$$\left\{ f_c = \frac{11}{40} \frac{qL^3}{EJ} \right.$$

$$\left. h = \frac{1}{8} \frac{qL^4}{EJ} \right.$$

Aziemi interne

aste AB

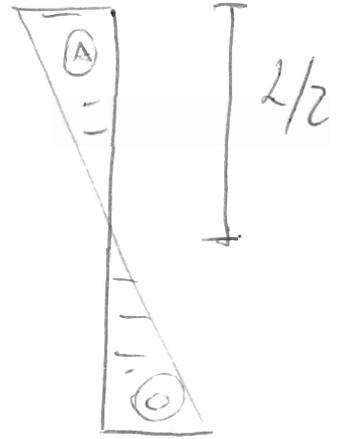


$M(x)$

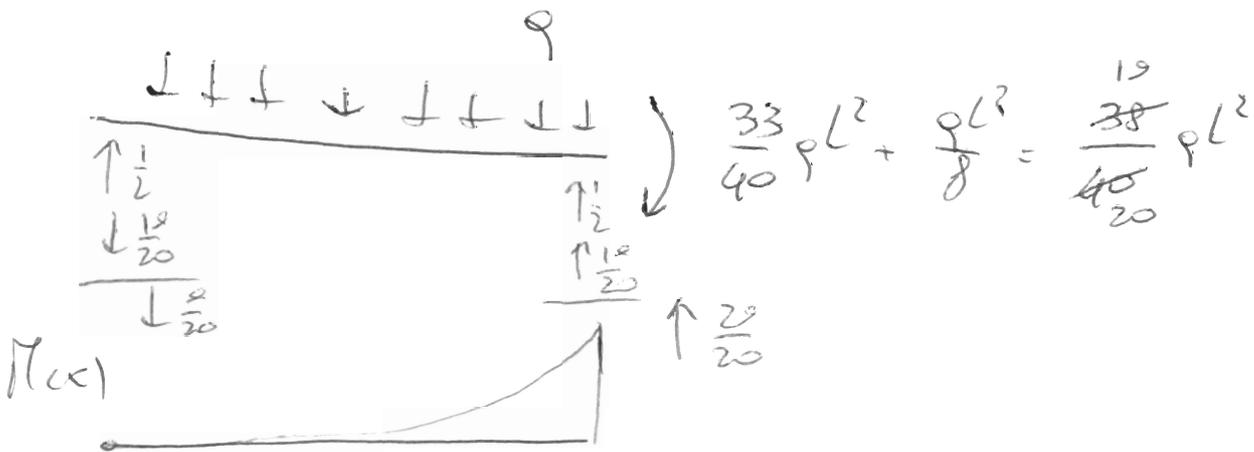


$$\frac{qL^2}{16}$$

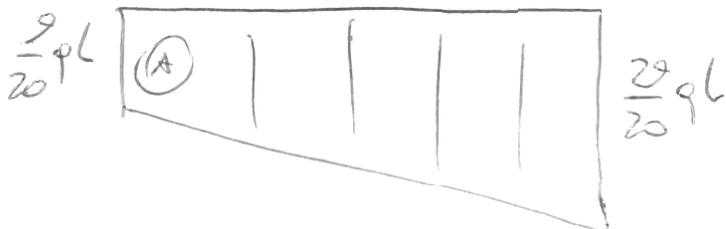
$V(x)$



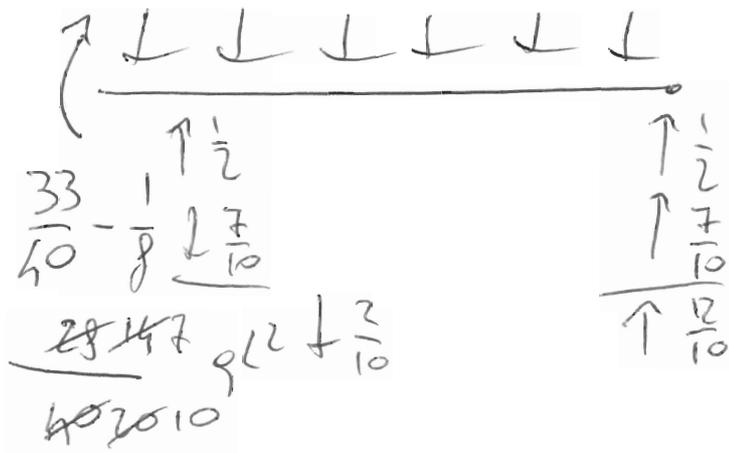
Asta BC



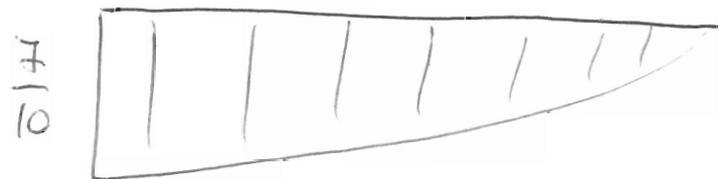
$V(x)$



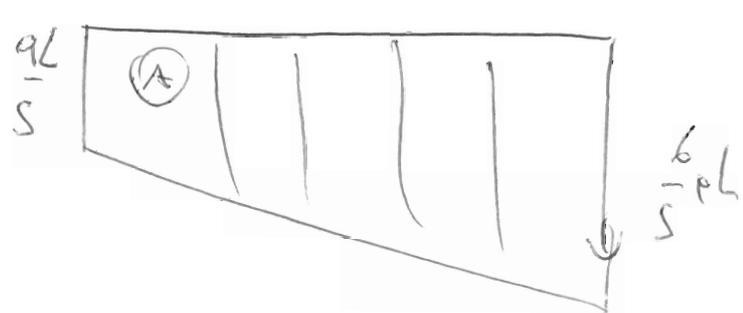
Asta BC



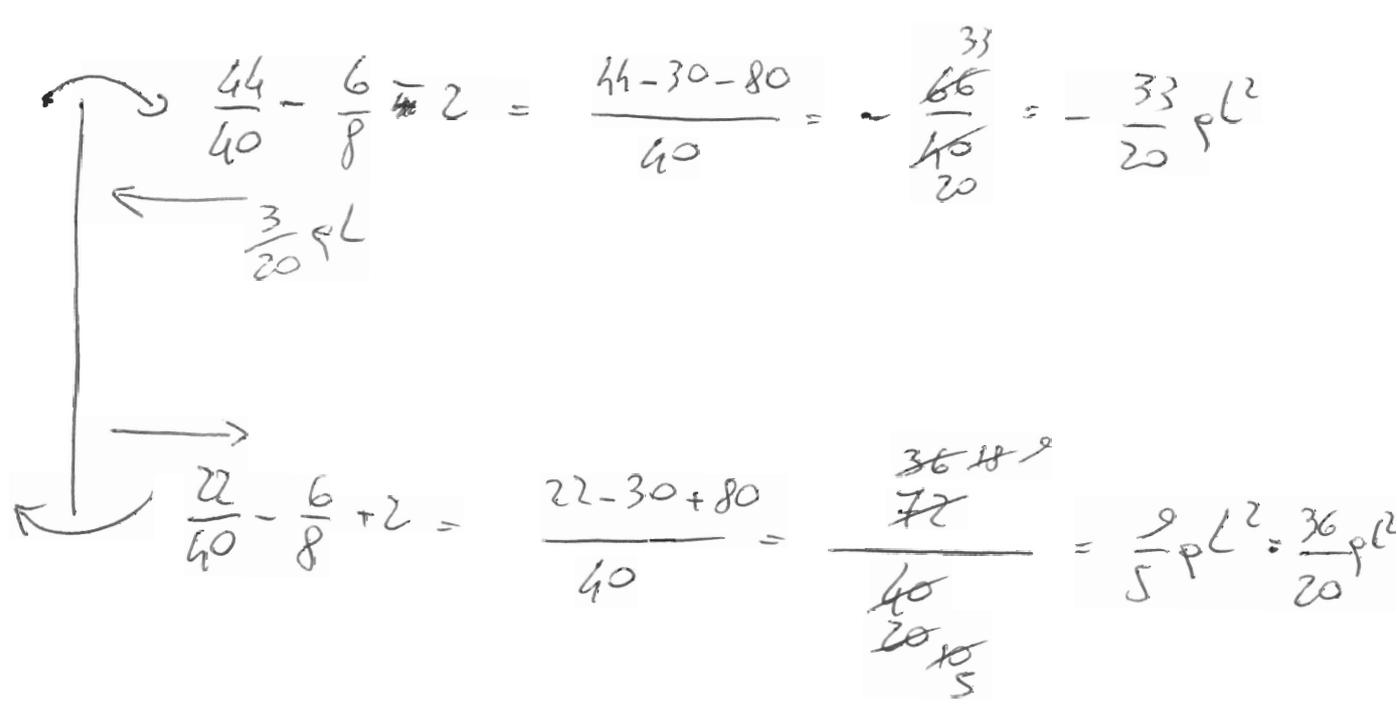
$M(x)$



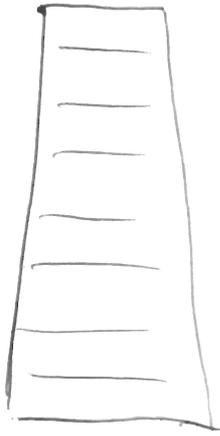
$V(x)$



Asta CD



$\eta(x)$
33/20



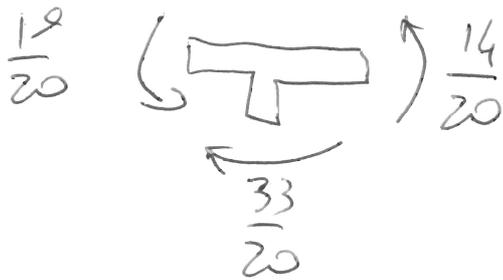
$$\frac{9}{5} qL^2$$

Nodo c

$V(x)$

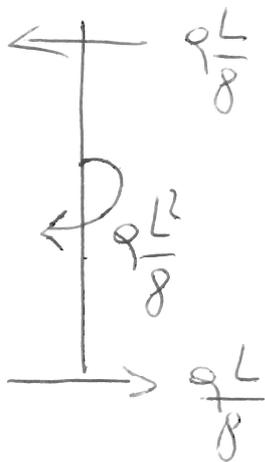


$$\frac{3}{20} qL$$

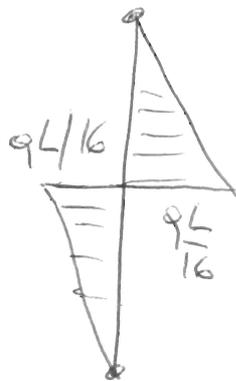


OK

Asta



$\eta(x)$



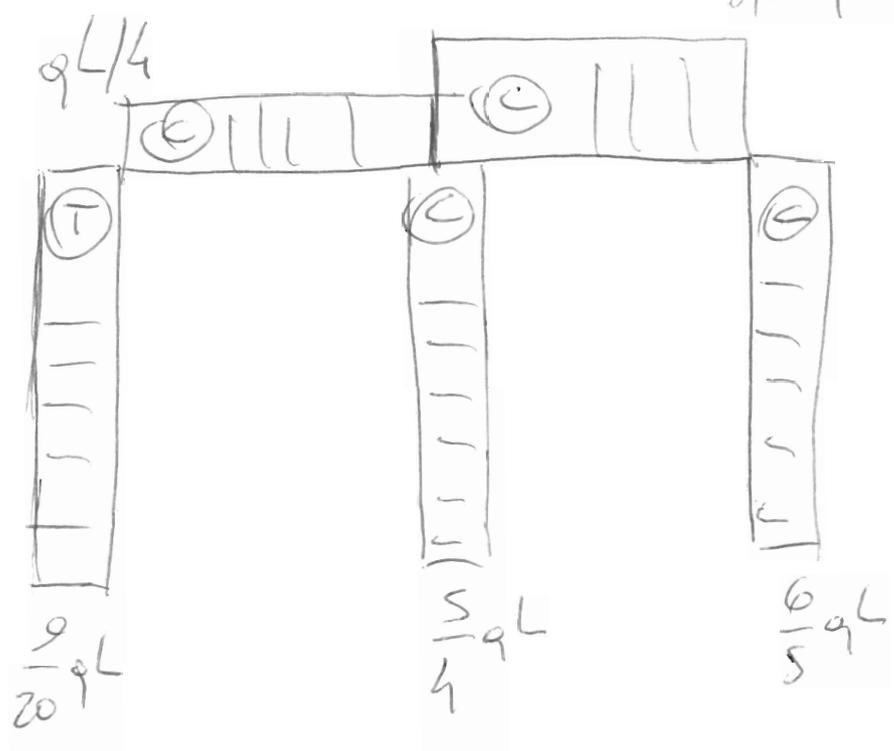
$V(x)$



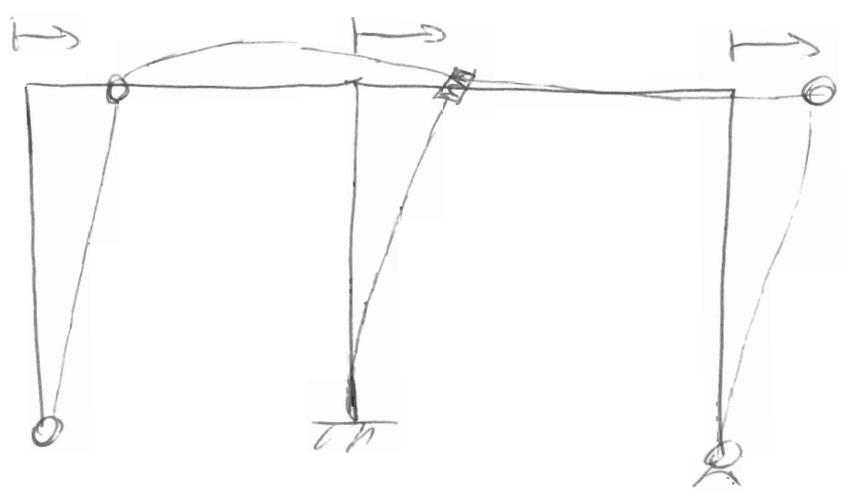
$$\frac{qL}{8}$$

Aziende assiale

8/20 qL



Deformata qual.



FONDAMENTI DI PROGETTAZIONE STRUTTURALE

ESAME 30/11/2009, 3h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:.....

Superato ELEMENTI STRUTTURALI A? Si NO

ESERCIZIO 1 (22 PUNTI)

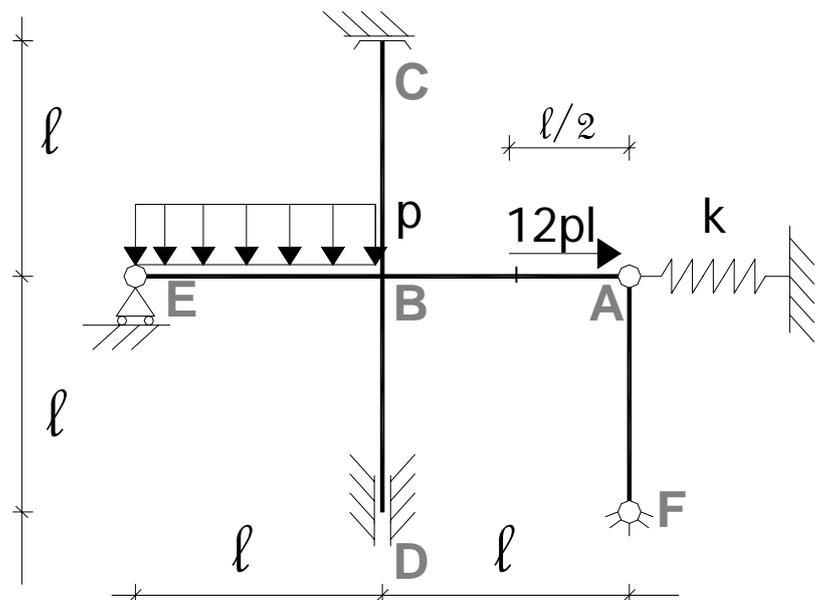
$$k = 4 \frac{EJ}{l^3};$$

$$EA \rightarrow \infty$$

Dato il telaio in figura

Si richiedono:

- 1- Momento flettente (con il valore e la posizione dei massimi)
- 2- Taglio
- 3- Azione assiale
- 4- Deformata qualitativa con posizione dei flessi



ESERCIZIO 2 (10 PUNTI)

Si consideri la trave AC

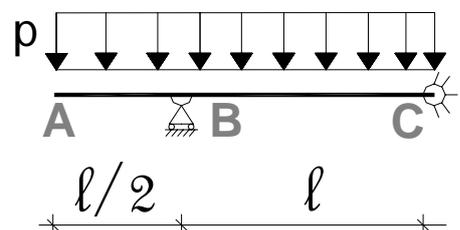
Siano:

$$p = 4000 \text{ kg/ml}$$

$$l = 6 \text{ mt}$$

$$f_{MAX} = \frac{3}{384} \frac{p l^4}{EJ}$$

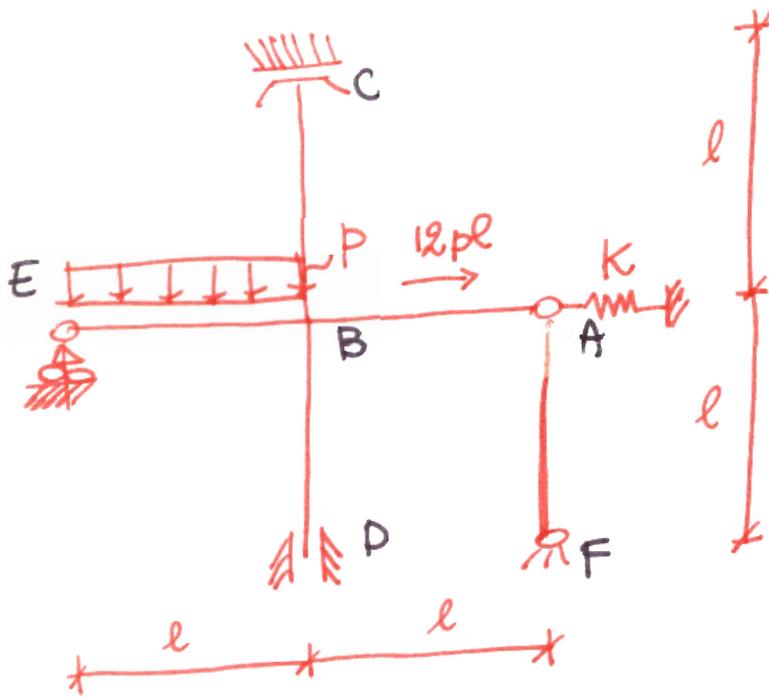
- 1- Si dimensioni la trave AC (2);
- 2- Si verifichi la trave nella sezione più sollecitata (flessione e taglio) (4)
- 3- Si verifichi la trave a deformabilità $f_{max} < l/400$ (1)
- 4- Si dimensioni il pilastro che insiste sull'appoggio B con luce di libera inflessione $h=5\text{mt}$ (3);



N.B. Assumere S275 (Fe430)

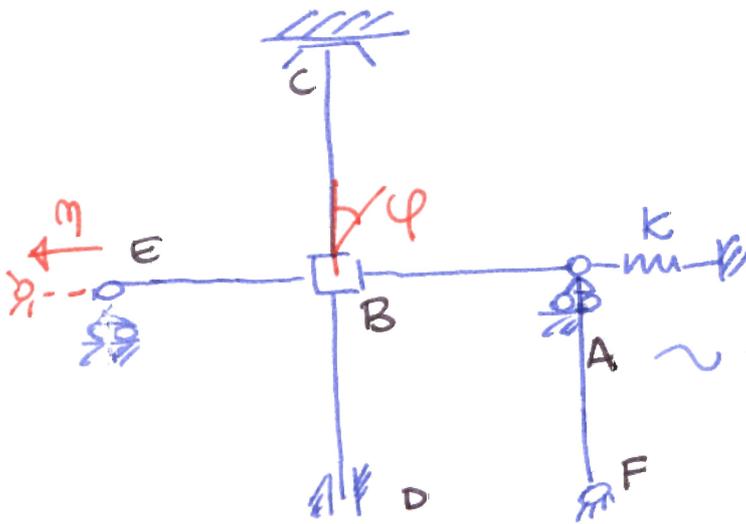
Tema esame 30/11/09

1° APPELLO



$EA \rightarrow \infty$
 $K = \frac{4EJ}{l^3}$

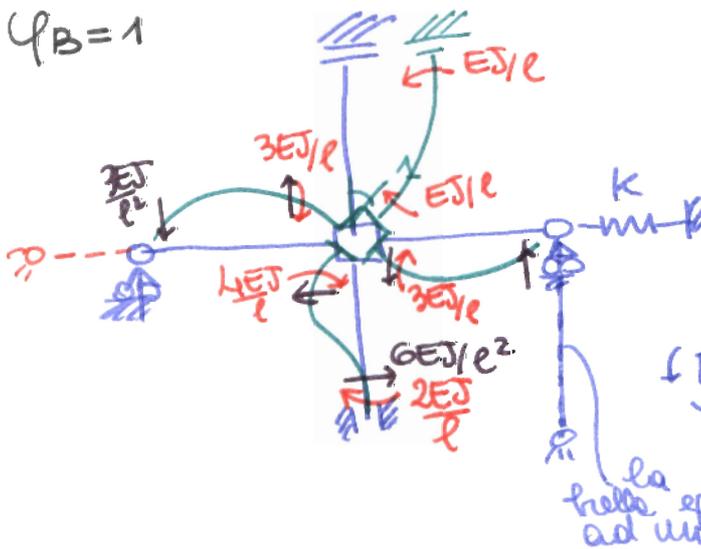
Telaio a NODI SPOSTABILI



$$\begin{cases} M_{BB}\phi_B + M_{B\eta}\eta + M_{B0} = 0 \\ l m_B \phi_B + l m_\eta \eta + l m_0 = 0 \end{cases}$$

la bella AF equivale ad un carrello
 M_{AF} M_{eT} saranno nulli

$\phi_B = 1$

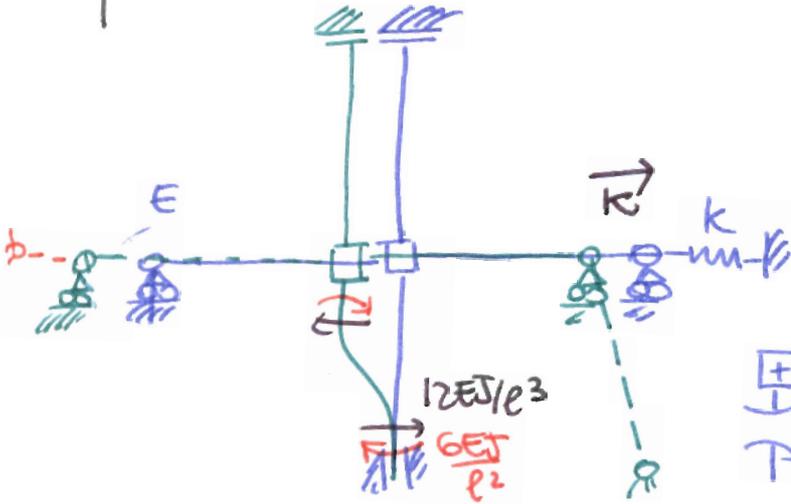


$$M_{BB} = \frac{3EJ}{l} \cdot 2 + \frac{EJ}{l} + \frac{4EJ}{l} = \frac{11EJ}{l}$$

$$l m_B = - \frac{6EJ}{l^2}$$

la bella AF equivale ad un carrello

$$\eta = 1$$

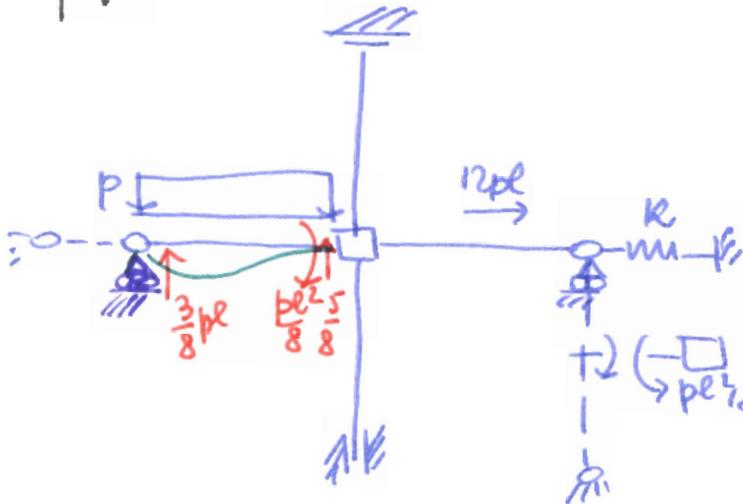


$$M_{B\eta} = \frac{6EJ}{e^2}$$

$$L_{\eta\eta} = -\frac{12EJ}{e^3} \cdot K = -\frac{16EJ}{e^3}$$

N.B. EF. e quilibrio ad un
angolo

$$p \neq 0$$



$$M_{B0} = +\frac{pl^2}{8}$$

$$L_{\eta 0} = -12pl$$

SISTEMA RISOLVENTE

$$\begin{cases} \frac{11EJ}{e} \varphi_B + \frac{6EJ}{e^2} \eta + \frac{pl^2}{8} = 0 \\ \frac{11}{6} \left(-\frac{6EJ}{e^2} \varphi_B - \frac{16EJ}{e^3} \eta - 12pl \right) = 0 \end{cases}$$

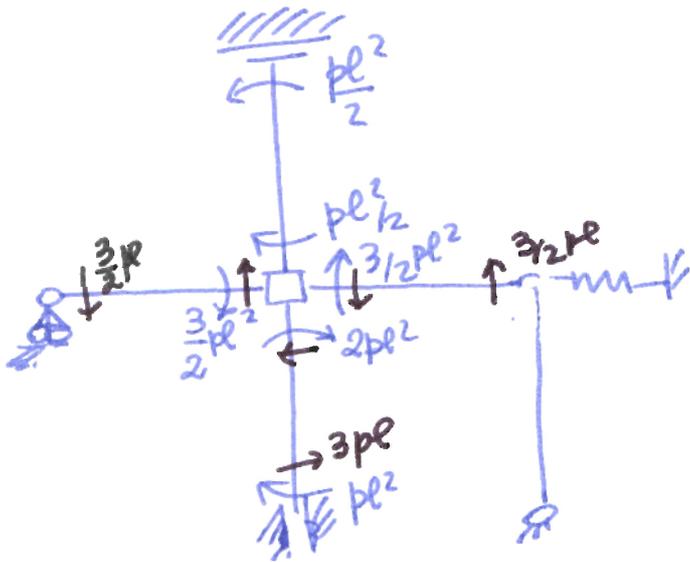
$$(1) + \frac{11}{6}(2) = \varphi_B \left(\frac{11EJ}{e} - \frac{16EJ}{e} \right) + \eta \left(\frac{6EJ}{e^2} - \frac{88EJ}{3e^2} \right) + \left(\frac{1}{8} - 22 \right) pl^2 = 0$$

$$\eta \cdot \left(-\frac{70EJ}{3e^2} \right) = \frac{175pl^2}{8} \rightarrow \eta = -\frac{15}{16} \frac{pl^4}{EJ}$$

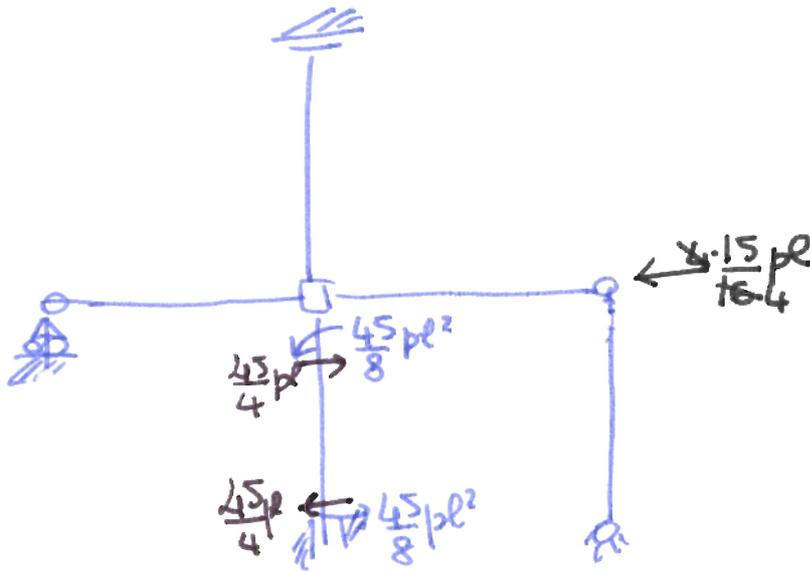
$$-\left(\frac{3}{6} \cdot \left(-\frac{15}{16} \frac{pl^4}{EJ} \right) + \frac{pl^2}{8} \right) \cdot \frac{e}{11EJ} = \varphi_B \rightarrow \varphi_B = \frac{1}{2} \frac{pl^3}{EJ}$$

$$\boxed{\varphi_B = \frac{1}{2} \frac{pl^3}{EJ}}$$

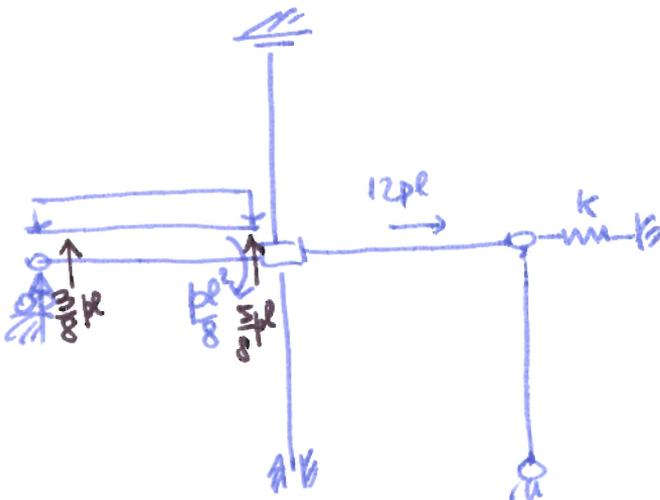
$$\varphi_B = \frac{1}{2} \frac{pl^3}{EI}$$

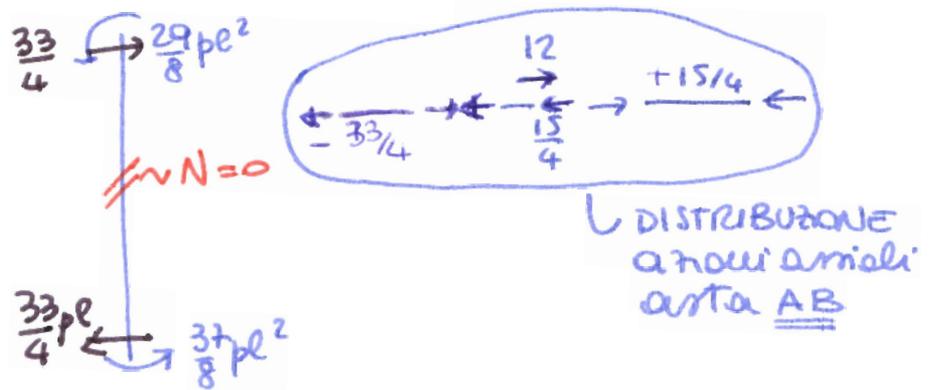
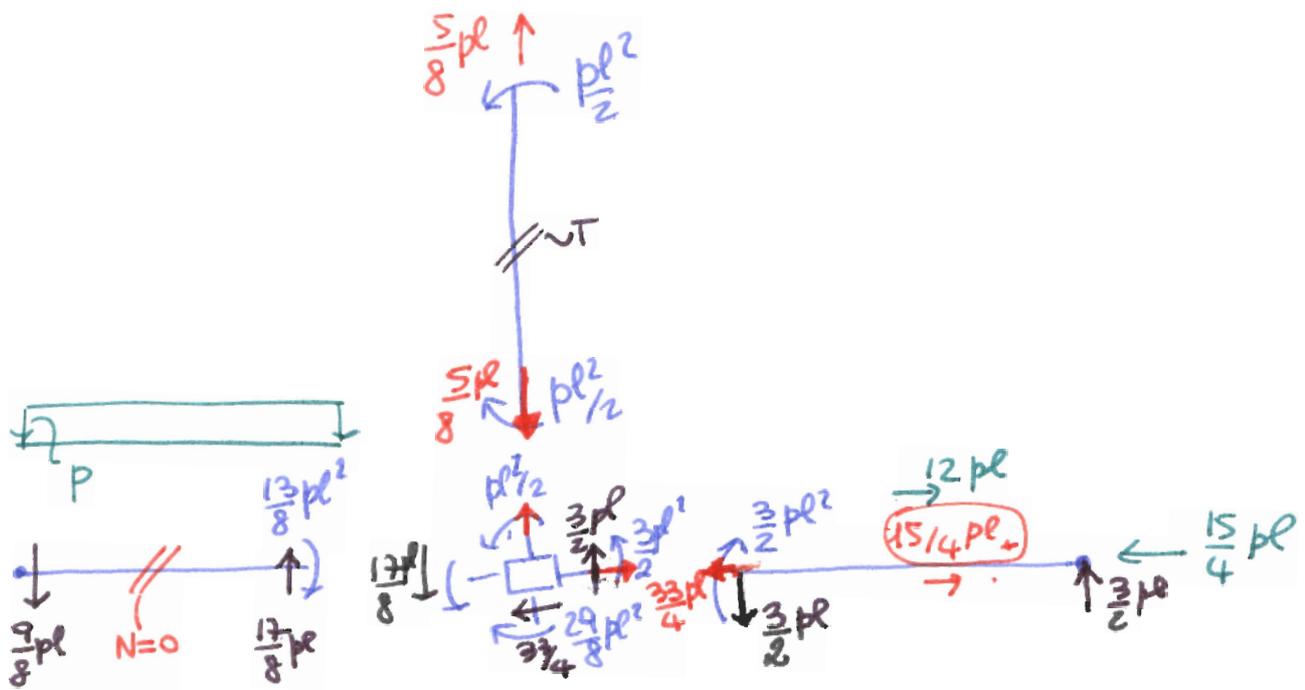


$$\eta = -\frac{15}{16} \frac{pl^4}{EI}$$



$p \neq 0$





Equilibri globali

$$\uparrow + -\frac{9}{8}pl - pl + \frac{3}{2}pl + \frac{5}{8}pl = 0 \quad \underline{\underline{ok!}}$$

$$\rightarrow -\frac{33}{4}pl + 12pl - \frac{15}{4}pl = 0 \quad \underline{\underline{ok!}}$$

$$\curvearrowright + \frac{9}{8}pl \cdot 2e + pl \cdot \frac{3}{2}e - \frac{5}{8}pl^2 + \frac{pl^2}{2} + \frac{37}{8}pl^2 - \frac{33}{4}pl^2 = 0 \quad \underline{\underline{ok!}}$$

Ep. NODO B

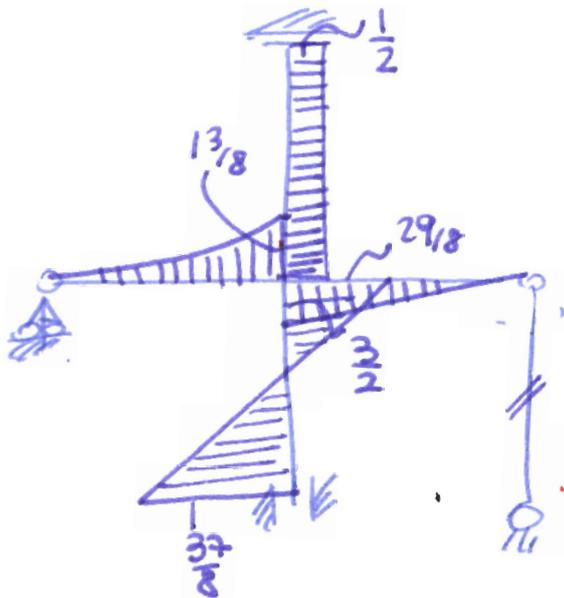
$$\rightarrow \frac{33}{4}pl - \frac{33}{4}pl = 0 \quad \underline{\underline{ok!}}$$

$$\uparrow + \frac{17}{8}pl - \frac{3}{2}pl + \frac{5}{8}pl = 0 \quad \underline{\underline{ok!}}$$

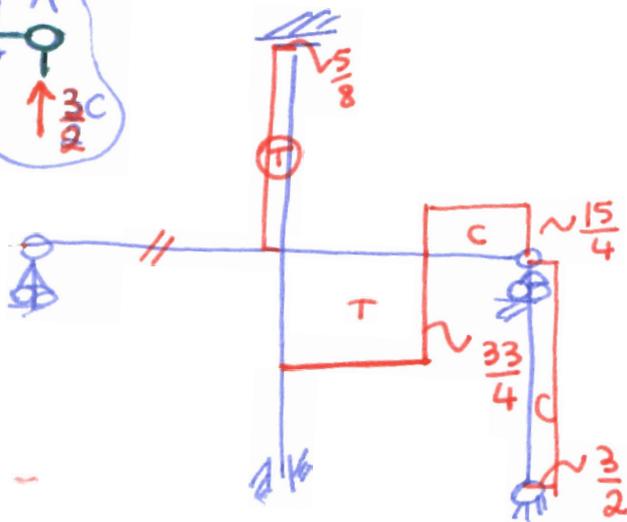
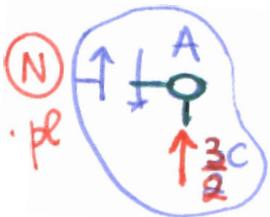
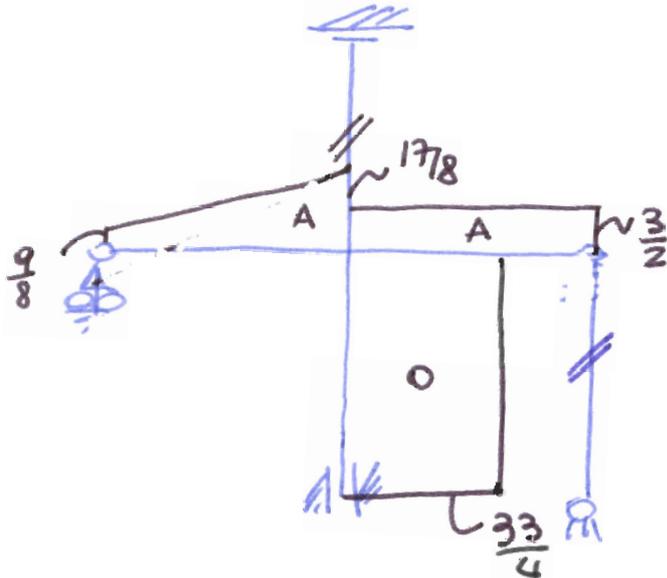
$$\curvearrowright + \frac{3}{2}pl^2 + pl^2 + \frac{13}{8}pl^2 - \frac{29}{8}pl^2 = 0 \quad \underline{\underline{ok!}}$$

Diagrammi

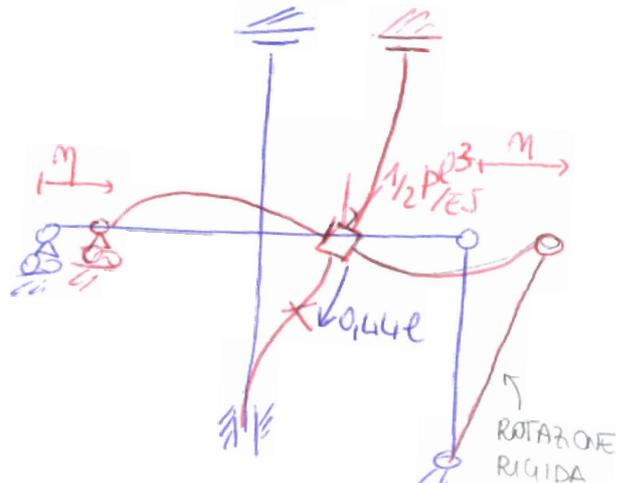
(M)
· pl²



(T)
· pl



Deformata



$$\frac{29}{8} pl^2 \cdot x = \frac{37}{8} pl^2 (l-x)$$

$$\frac{37}{8} pl^2 x = \frac{29}{8} pl^2 (l-x)$$

$$37x + 29x = 29l$$

$$x = \frac{29}{66} l e$$

FONDAMENTI DI PROGETTAZIONE STRUTTURALE

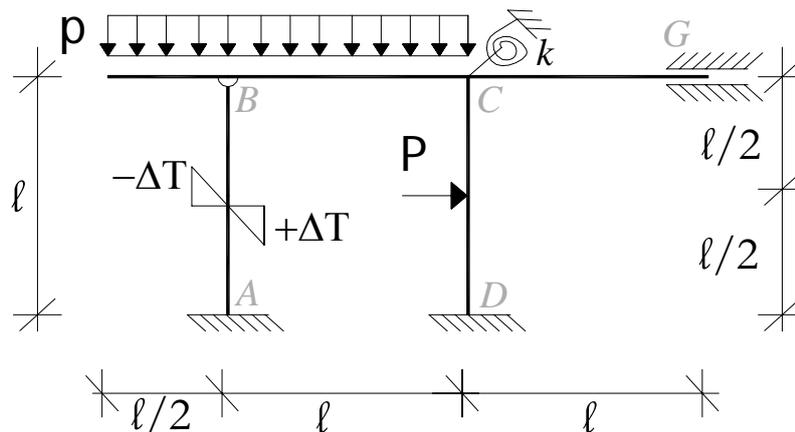
ESAME 17/12/2009, 3h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:

Superato ELEMENTI STRUTTURALI A? Sì NO

ESERCIZIO 1 (20 PUNTI)



$$P = 2pl; \quad k = \frac{2EJ}{l}; \quad \frac{\alpha\Delta T}{h} = \frac{5pl^2}{8EJ}; \quad EA \rightarrow \infty; \quad EJ = \text{cost}$$

Dato il telaio in figura

Si richiedono:

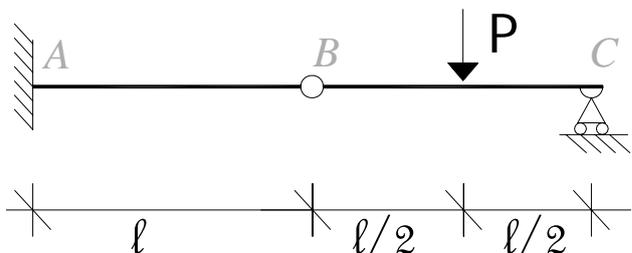
- 1- Momento flettente (con il valore e la posizione dei massimi);
- 2- Taglio;
- 3- Azione assiale;
- 4- Deformata qualitativa con posizione dei flessi.

ESERCIZIO 2 (10 PUNTI)

Le due aste AB e BC (figura 1) sono delle IPE 550. In B si realizza il collegamento disegnato in figura 2.

Siano:

$$P = 16000 \text{ kg}; \quad \ell = 5 \text{ m}$$

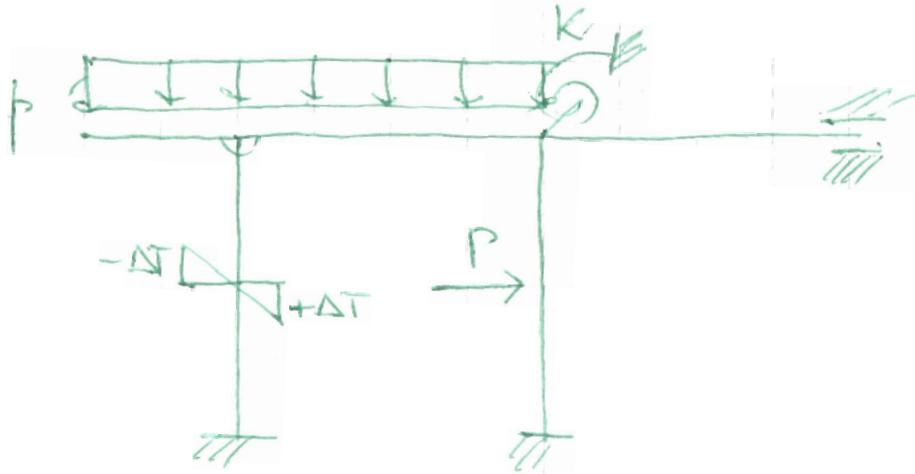


Si richiedono:

- 1- Il tracciamento delle azioni interne della struttura (1);
- 2- Il dimensionamento del diametro dei bulloni e dello spessore (s) dei fazzoletti (4);
- 3- Le verifiche del collegamento (2)
- 4- Si dimensiona il pilastro che insiste sull'appoggio C. Si assuma una luce di libera inflessione $l = 7\text{m}$ (3);

N.B. Assumere S275 (Fe430) e bulloni di classe 6.6

Temp. Exame



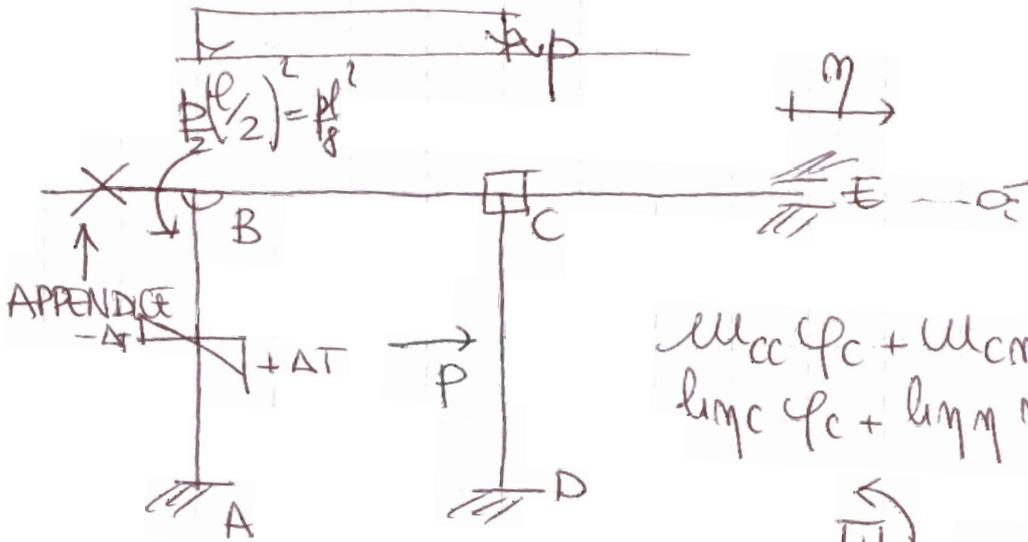
$$P = 2pl$$

$$K = \frac{2ET}{e}$$

$$l \frac{\Delta T}{e} = \frac{5pl^2}{8ET}$$

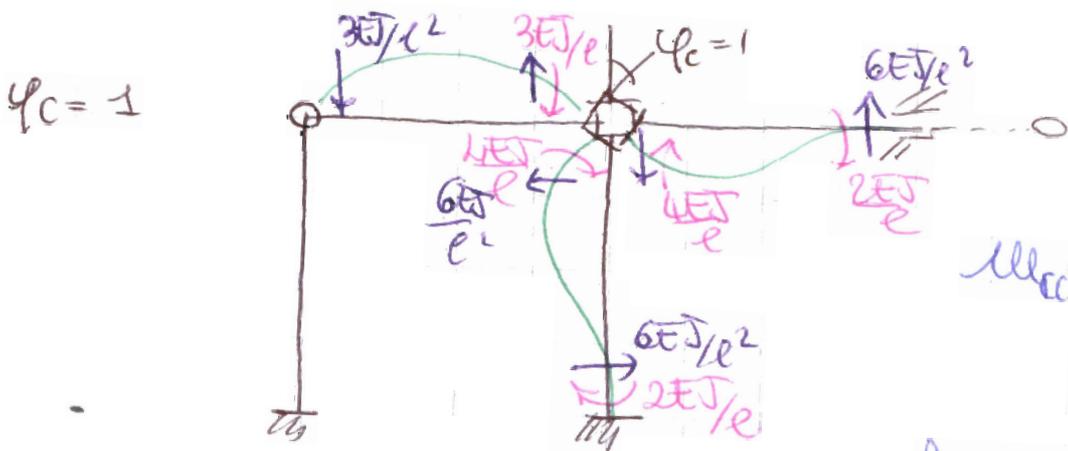
$$EA \rightarrow \infty$$

$$EI \text{ cost}$$



$$M_{cc} \phi_C + M_{cm} \eta + M_{co} = 0$$

$$k_{mc} \phi_C + k_{m\eta} \eta + k_{m\phi} = 0$$

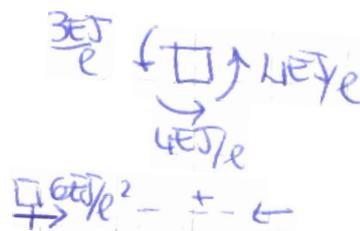


$$M_{cc} = \frac{11EJ}{e} + K$$

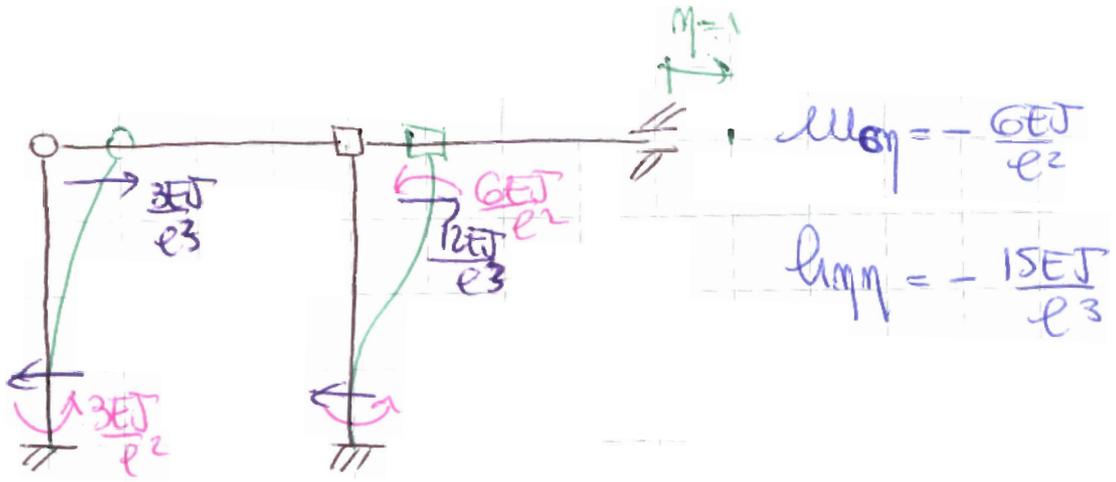
$$= \frac{13EJ}{e}$$

$$k_{mc} = \frac{6EJ}{e^2}$$

Nodo C



$$\eta = 1$$

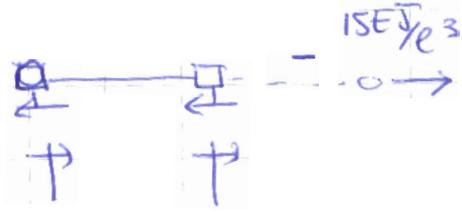


$$M_{\eta} = -\frac{6EJ}{e^2}$$

$$L_{\eta} = -\frac{15EJ}{e^3}$$

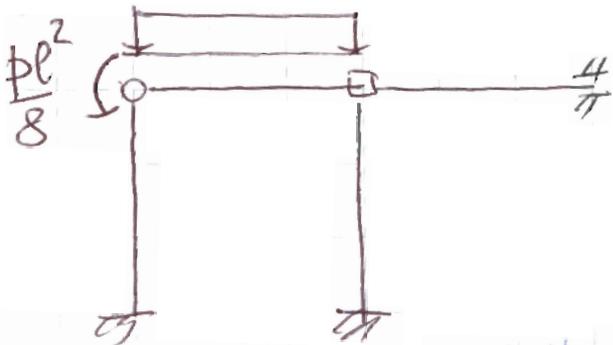
NODO C

$$\frac{6EJ}{e^2}$$



Carichi esterni

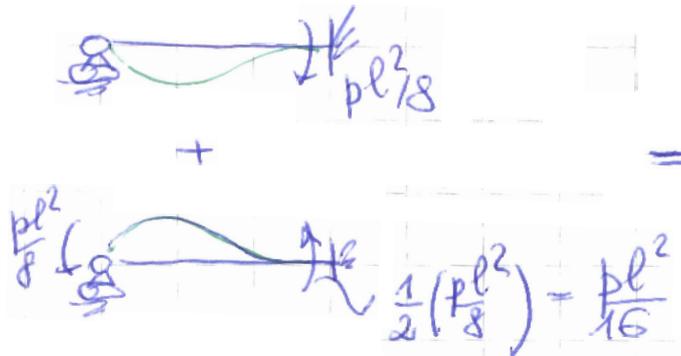
① p distribuito $\neq 0$



$$L_{\eta} = 0$$

$$M_{\eta} = +\frac{1}{16} pl^2$$

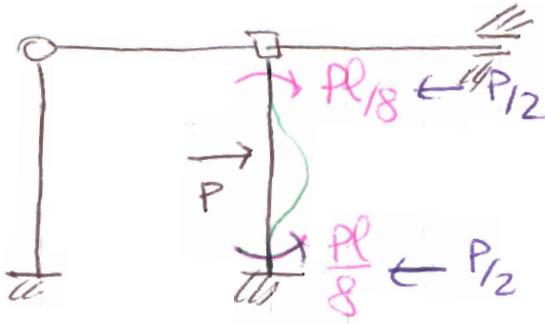
è somma di due contributi



NODO C

$$\frac{1}{8} \left(\frac{pl^2}{8} \right) = \frac{1}{16} pl^2$$

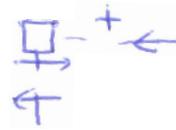
② $P \neq 0$



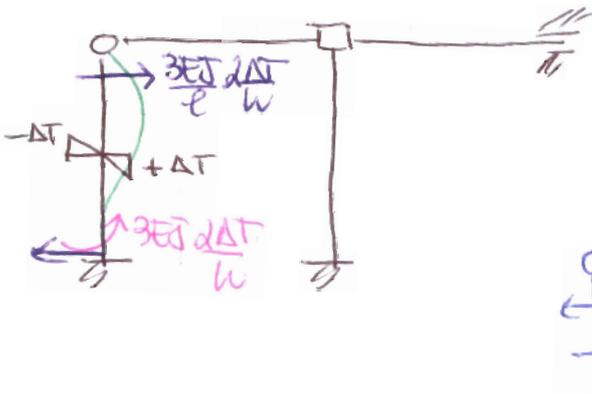
$$\Delta u_{co}^{(2)} = + \frac{Pl}{8} = \frac{1}{8} \frac{2pl}{2} = \frac{pl^2}{4}$$

$$\Delta \eta_0^{(2)} = + \frac{P}{2} = \frac{2pl}{2} = pl$$

Nodo c



③ $\Delta T \neq 0$



$$\Delta u_{co}^{(3)} = 0$$

$$\Delta \eta_0^{(3)} = - \frac{3EJ}{l} \frac{\Delta T}{w} = - \frac{15}{8} pl$$

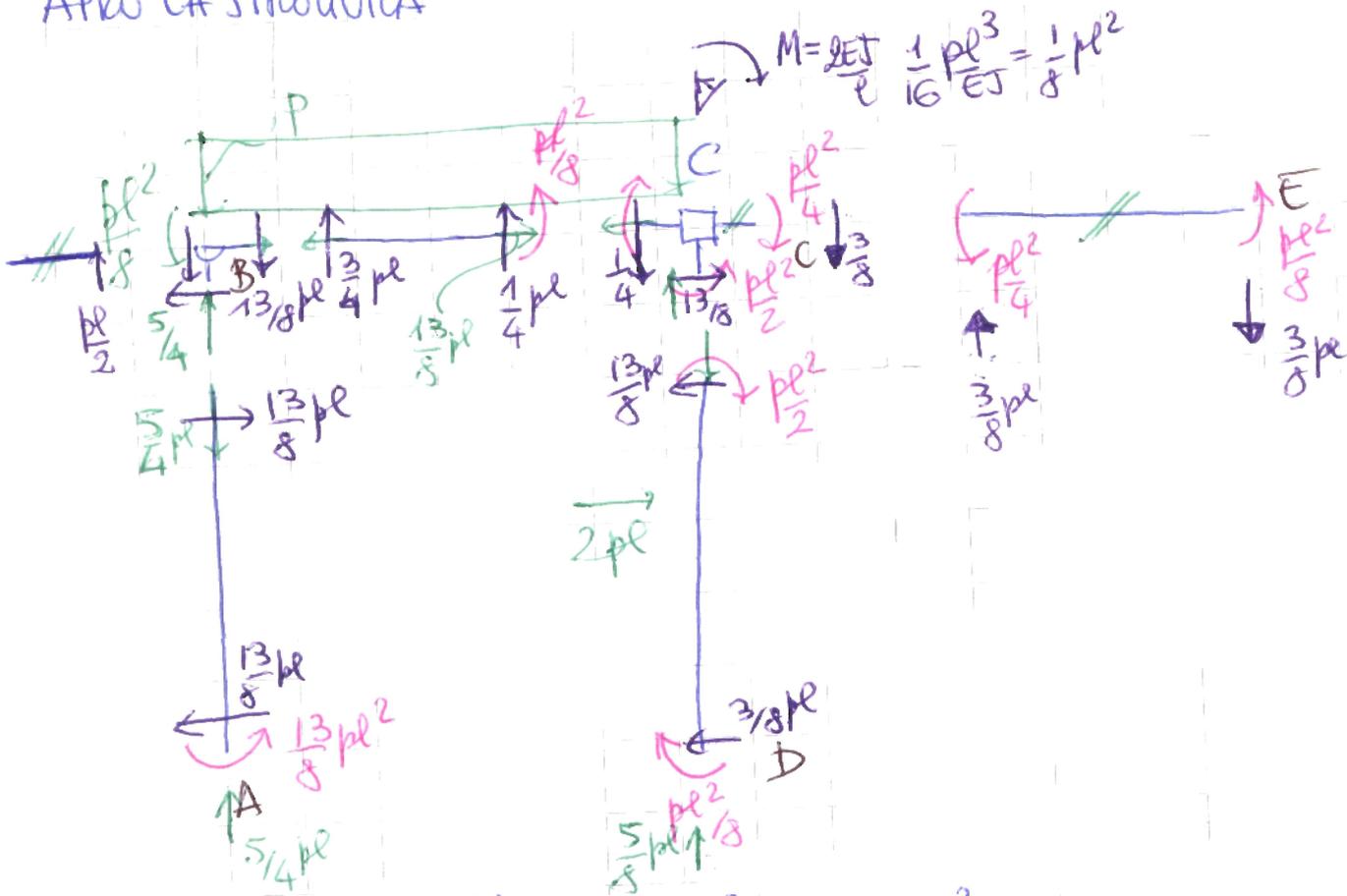


$$\left\{ \begin{aligned} \left(\frac{41EJ}{l} + K \right) \varphi_c - \frac{6EJ}{l^2} \eta + \frac{5}{16} pl^2 &= 0 \\ \frac{6EJ}{l^2} \varphi_c - \frac{15EJ}{l^3} \eta + \left(pl - \frac{3EJ}{l} \frac{\Delta T}{w} \right) &= 0 \end{aligned} \right.$$

$$\left\{ \begin{aligned} \frac{13EJ}{l} \varphi_c - \frac{6EJ}{l^2} \eta + \frac{5}{16} pl^2 &= 0 \\ \frac{6EJ}{l^2} \varphi_c - \frac{15EJ}{l^3} \eta + \frac{7}{8} pl &= 0 \end{aligned} \right.$$

$$\boxed{\begin{aligned} \varphi_c &= -\frac{1}{16} \frac{pl^3}{EJ} \\ \eta &= -\frac{1}{12} \frac{pl^4}{EJ} \end{aligned}}$$

APRO LA STRUCTURA

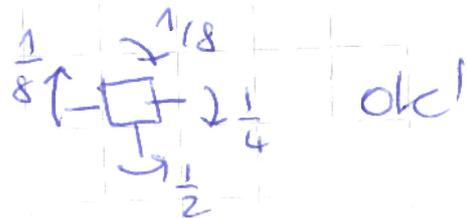


$$\downarrow M_c^{sx} = \frac{2EJ}{e} \left(-\frac{1}{16} \frac{pl^3}{EJ} \right) + 0 + \frac{pl^2}{16} = -\frac{1}{8} pl^2$$

$$\curvearrowright M_c^{dx} = \frac{4EJ}{e} \left(-\frac{1}{16} \frac{pl^3}{EJ} \right) = -\frac{pl^2}{4}$$

$$\curvearrowright M_c^{dodo} = \frac{4EJ}{e} \left(-\frac{1}{16} \frac{pl^3}{EJ} \right) - \frac{6EJ}{e^2} \left(-\frac{1}{12} \frac{pl^4}{EJ} \right) + \frac{pl^2}{4} = \left(-\frac{1}{4} + \frac{1}{2} + \frac{1}{4} \right) pl^2 = \frac{1}{2} pl^2$$

Ep. NODO C



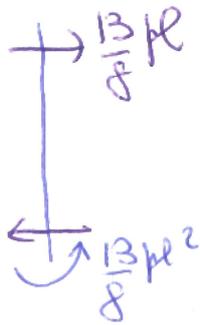
$$\uparrow M_A = \frac{2EJ}{e^2} \left(-\frac{1}{12} \frac{pl^4}{EJ} \right) + \frac{15}{8} pl^2 = \frac{13}{8} pl^2$$

$$\curvearrowright M_D = \frac{2EJ}{e} \left(-\frac{1}{16} \frac{pl^3}{EJ} \right) - \frac{6EJ}{e^2} \left(-\frac{1}{12} \frac{pl^4}{EJ} \right) = \frac{pl^2}{4} = \left(-\frac{1}{8} + \frac{1}{2} - \frac{1}{4} \right) pl^2 = \frac{1}{8} pl^2$$

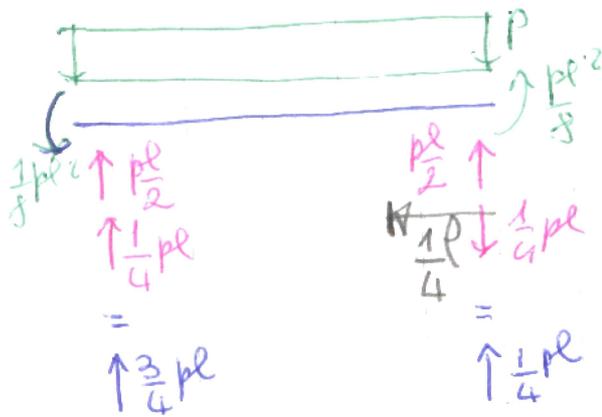
$$\downarrow M_E = \frac{2EJ}{e} \left(-\frac{1}{16} \frac{pl^3}{EJ} \right) = -\frac{1}{8} pl^2$$

Equilibrio ASTA x ASTA

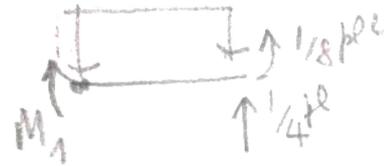
AB



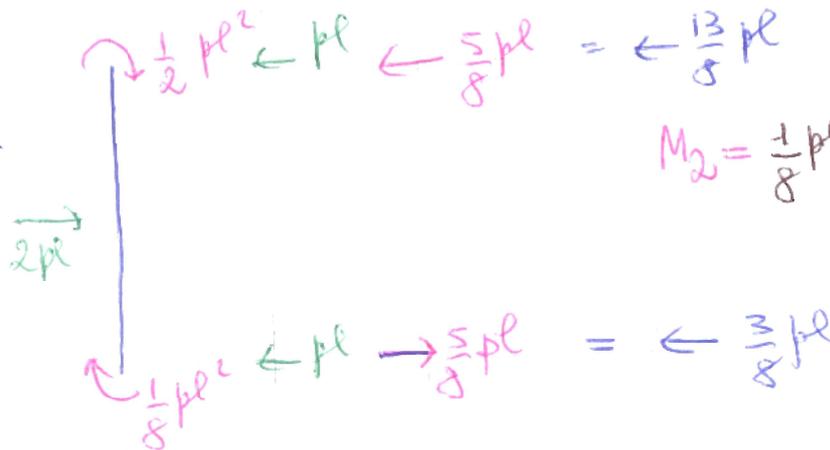
BC



$$M_1 = \frac{1}{8}pl^2 + \frac{1}{16}pl^2 - \frac{pl}{4} \frac{l}{8} = \frac{5}{32}pl^2$$

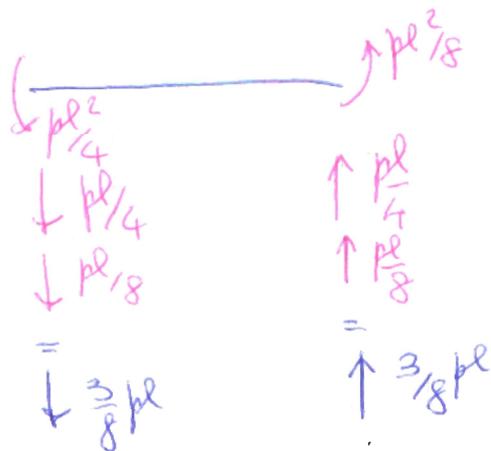


CD



$$M_2 = \frac{1}{8}pl^2 + \frac{3}{8}pl \frac{l}{2} = \frac{5}{16}pl^2$$

CE



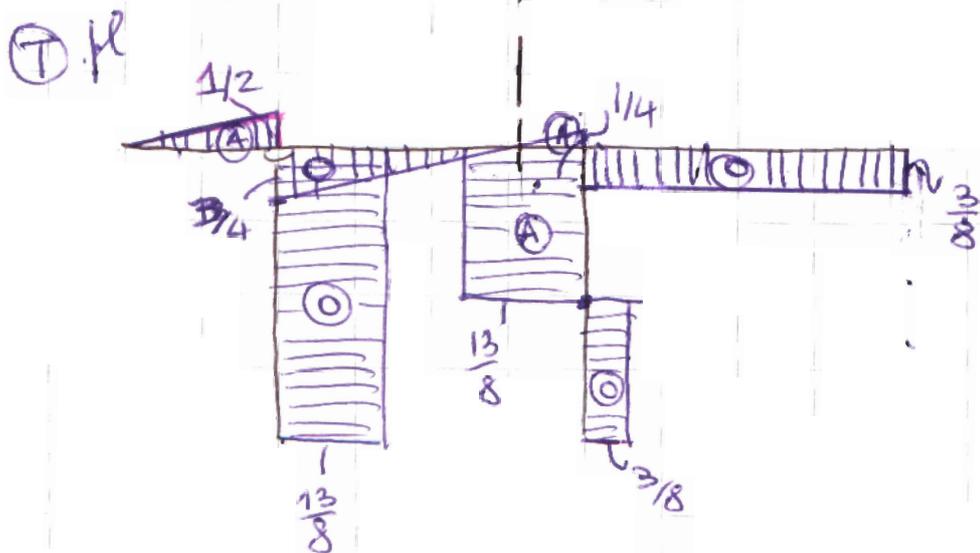
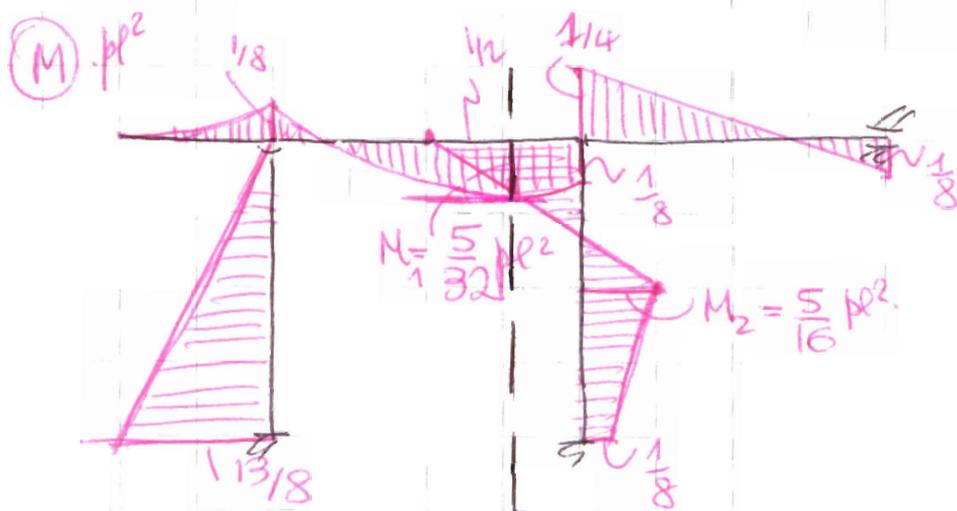
Ep. trasl. orizz.

$$\rightarrow -\frac{13}{8}pl - \frac{3}{8}pl + 2pl \stackrel{?}{=} 0 \quad \underline{ok!}$$

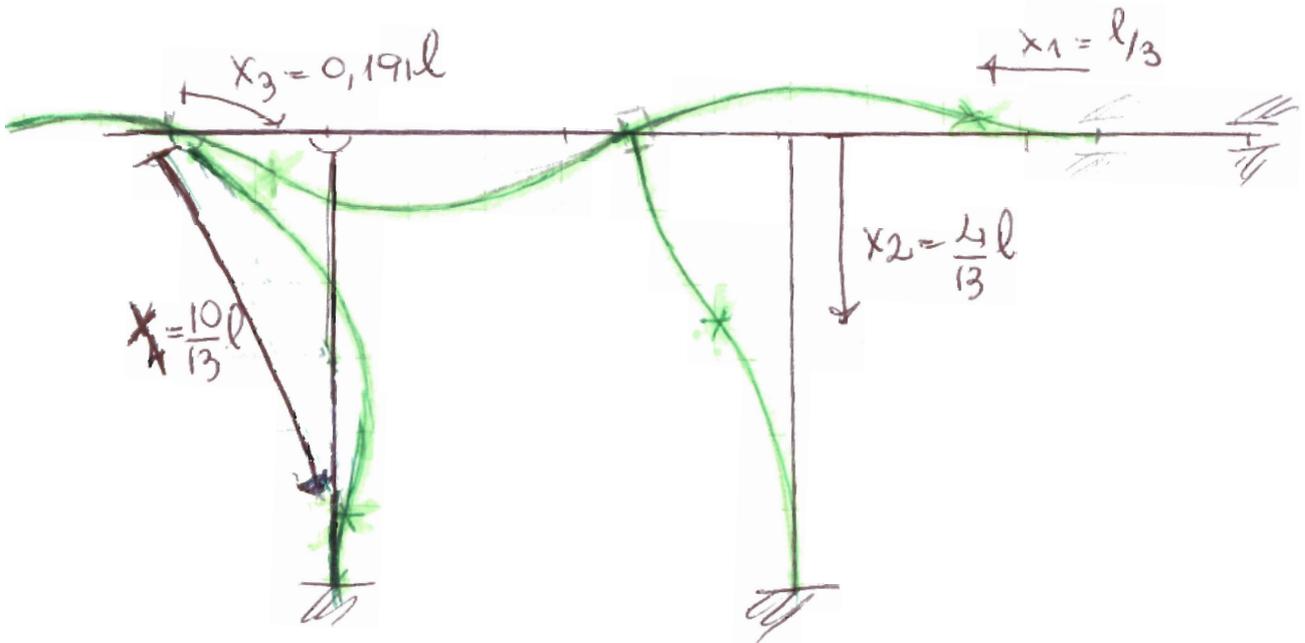
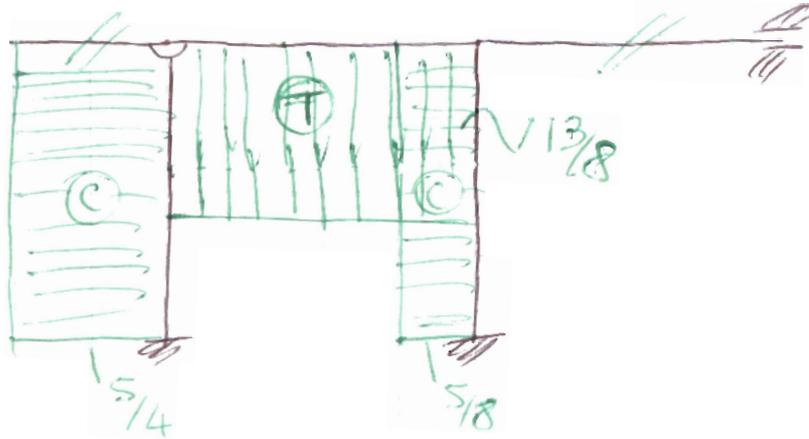
$$\uparrow + \frac{5}{4}pl + \frac{5}{8}pl - \frac{3}{8}pl - \frac{3}{2}pl \stackrel{?}{=} 0 \quad \underline{ok!}$$

$$\curvearrowright + \frac{1}{8}pl^2 + \frac{13}{8}pl^2 + \frac{pl^2}{8} - \frac{pl^2}{8} + \frac{pl}{2} \left(\frac{5l}{4} \right) + \frac{pl}{2} - \frac{5}{4}pl^2 - \frac{3}{8}pl^2 - 2pl \frac{l}{2} \stackrel{?}{=} 0 \quad \underline{ok!}$$

DIAGRAMMI



(N) .pl



Determinazione dei fletti

$$(X_1) \quad x_1 \cdot \frac{1}{8} p l^2 = l \cdot \frac{3}{8} p l^2 \quad \rightarrow \quad x_1 \cdot \frac{3}{8} p l^2 = \frac{1}{8} p l^2 l \quad x_1 = \frac{l}{3}$$

$$(X_2) \quad x_2 \cdot \frac{1}{2} p l^2 = \frac{l}{2} \cdot \frac{13}{16} p l^2 \quad x_2 \cdot \frac{13}{16} p l^2 = \frac{1}{2} p l^2 \cdot \frac{l}{2} \quad x_2 = \frac{4l}{13}$$

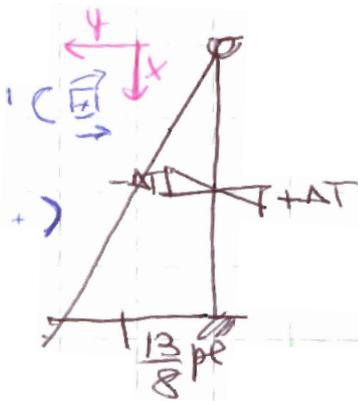
$$(X_3) \quad \begin{array}{l} \text{Diagramma di un elemento di lunghezza } l \text{ con forze:} \\ \text{Forza distribuita } p \text{ verso l'alto} \\ \text{Reazione di vincolo } M=0, T=0 \text{ a destra} \\ \text{Forze esterne: } \frac{p l^2}{8} \text{ verso l'alto, } \frac{3}{4} p l \text{ verso l'alto} \end{array} \quad \begin{array}{l} \int + \frac{p l^2}{48} - 3 p X_3 + \frac{p X_3^2}{2} = 0 \\ 4 X_3^2 - 6 l X_3 + l^2 = 0 \end{array} \quad \begin{array}{l} \rightarrow \\ \text{v dietro} \end{array}$$

$$4x_3^2 - 6lx_3 + l^2 = 0$$

$$x_{12} = \frac{+6 \pm \sqrt{6^2 - 16}}{8} = \frac{+6 \pm \sqrt{36 - 16}}{8} = \begin{cases} \frac{6 - 4,47}{8} = 0,1913l \\ \frac{6 + 4,47}{8} = 1,3l \end{cases}$$

NA

Però determinare i potèncs flexs en AB

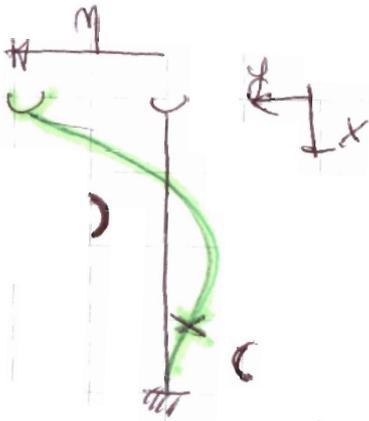


$$y''(x) = -\frac{M(x)}{EJ} + \frac{2\alpha\Delta T}{h}$$

$$y''(x) = -\frac{1}{EJ} \left(\frac{13}{8} px \right) + \frac{2 \cdot 5}{8} \frac{p l^2}{EJ} \leq 0$$

$$-13px + 10pl^2 \leq 0$$

$$x \geq \frac{10}{13} l$$



$$x_4 = \frac{10}{13} l$$

$$x > \frac{10}{13} l \quad y'' < 0$$

$$x < \frac{10}{13} l \quad y'' > 0$$

FONDAMENTI DI PROGETTAZIONE STRUTTURALE

ESAME 17/02/2010, 2h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:

Superato ELEMENTI STRUTTURALI A? Sì NO

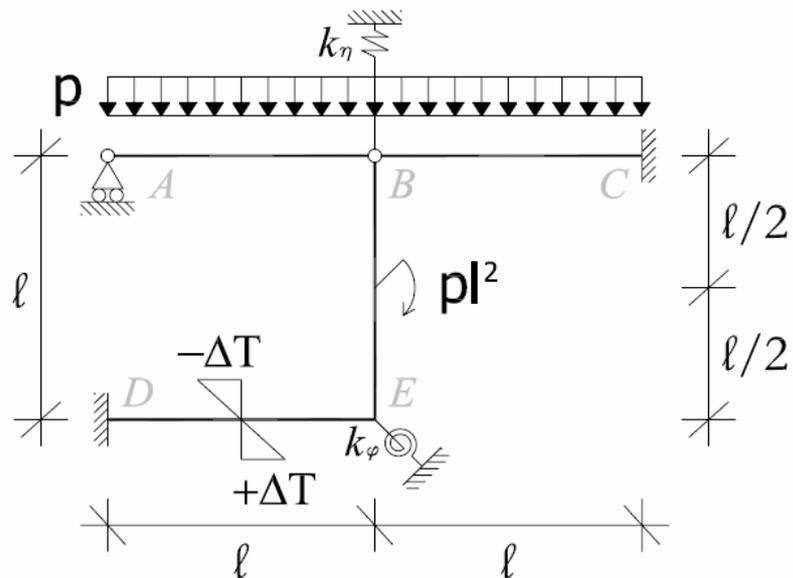
ESERCIZIO 1 (20 PUNTI)

$$k_{\varphi} = 1 \frac{EJ}{\ell};$$

$$k_{\eta} = 17 \frac{EJ}{\ell^3};$$

$$\frac{\alpha \Delta T}{h} = \frac{1}{16} \frac{Pl^2}{EJ}$$

$$EA \rightarrow \infty$$

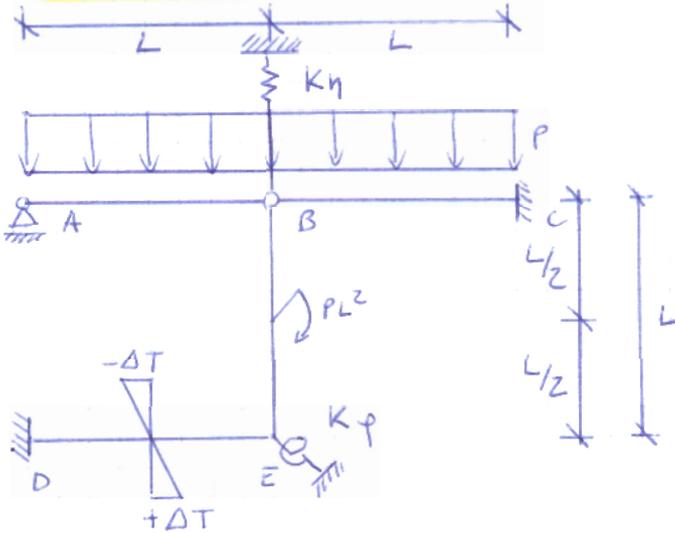


Dato il telaio in figura

Si richiedono:

- 1- Momento flettente (con il valore e la posizione dei massimi)
- 2- Taglio
- 3- Azione assiale
- 4- Deformata qualitativa con posizione dei flessi

TEMA D'ESAME



$$K_f = 1 \times \frac{EJ}{L}$$

$$K_\eta = 17 \times \frac{EJ}{L^3}$$

$$\frac{\alpha \Delta T}{t} = \frac{1}{10} \frac{PL^2}{EJ}$$

$$EJ = \text{cost}$$

$$EA \rightarrow \infty$$

ANALISI CINEMATICA

GDL $3 + 3 + 3 = 9$

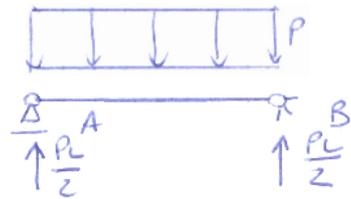
GDU $1 + 4 + 1 + 3 + 3 + 1 = 13$

I $13 - 9 = 4$

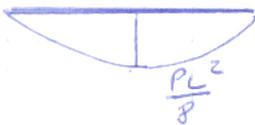
STRUTTURA A NODI SPOSTABILI

APPENDICE ISOSTATICA

ASTA AB



$V(x)$

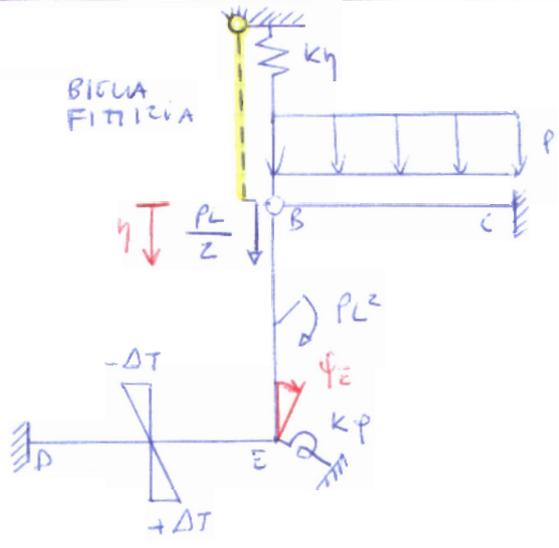


$M(x)$



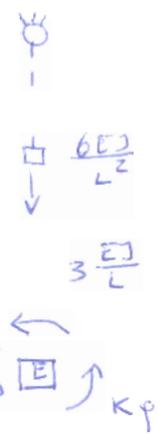
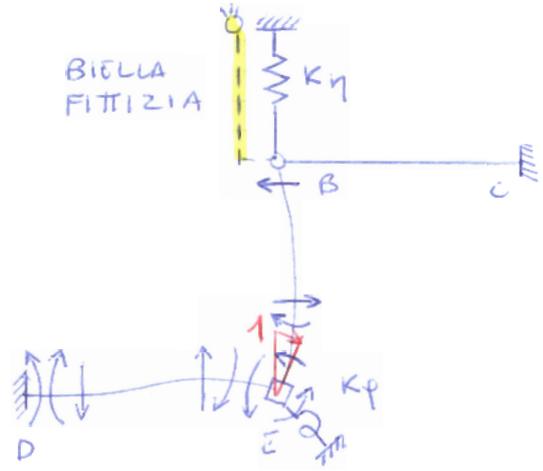
DEFORMATA QUANTITATIVA

SISTEMA RISOLVENTE / CONVENZIONI DI SEGNO



$$\begin{cases} \sum m_E = \phi \\ \sum F_{biella} = \phi \end{cases} \begin{cases} m_{EE} \times \phi_E + m_{E\eta} \times \eta + m_{E\phi} = \phi \\ h_{BE} \times \phi_E + h_{B\eta} \times \eta + h_{B\phi} = \phi \end{cases}$$

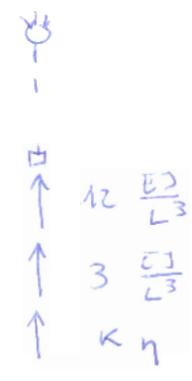
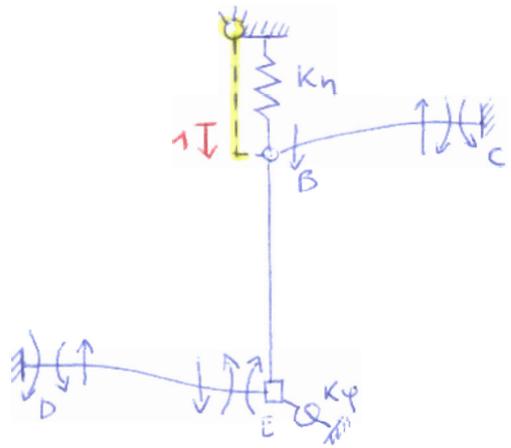
$\phi_E = 1 \quad \eta = \phi \quad P = \phi \quad \Delta T = \phi$



$$m_{EE} = -\frac{7EJ}{L} - k_p$$

$$h_{BE} = -\frac{6EJ}{L^2}$$

$$\phi_s = \phi \quad \eta = 1 \quad P = \phi \quad \Delta T = \phi$$

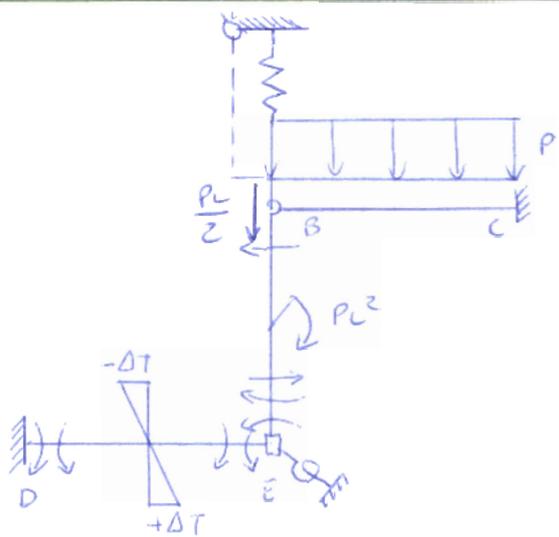


$$\frac{6EJ}{L^2} \left[\square \right]$$

$$m_{BE} = \frac{6EJ}{L^2}$$

$$h_{B\eta} = +15 \frac{EJ}{L^3} + K\eta$$

$$\phi_s = \phi \quad \eta = \phi \quad P \neq \phi \quad \Delta T \neq \phi$$



$$m_{E\phi} = -\frac{2EJ \alpha \Delta T}{t} - \frac{PL^2}{8}$$

$$h_{B\phi} = -\frac{PL}{2} - \frac{3}{8} PL = -\frac{7}{8} PL$$

SISTEMA RISOLVENTE

$$\begin{cases} -\left(\frac{7EJ}{L} + K_{\varphi}\right) \cdot \varphi_E + \left(\frac{6EJ}{L^2}\right) \cdot \eta - \left(\frac{2EJ\alpha\Delta T}{t} + \frac{PL^2}{8}\right) = \phi \\ -\left(\frac{6EJ}{L^2}\right) \cdot \varphi_E + \left(15\frac{EJ}{L^3} + K_{\eta}\right) \cdot \eta - \frac{7}{8} PL = \phi \end{cases}$$

$$\begin{bmatrix} -\left(\frac{7EJ}{L} + K_{\varphi}\right) & + \frac{6EJ}{L^2} \\ -\frac{6EJ}{L^2} & + \left(15\frac{EJ}{L^3} + K_{\eta}\right) \end{bmatrix} \cdot \begin{bmatrix} \varphi_E \\ \eta \end{bmatrix} = \begin{bmatrix} +\frac{2EJ\Delta T}{L} + \frac{PL^2}{8} \\ \frac{7}{8} PL \end{bmatrix}$$

DETERMINAZIONE DELLE COSTANTI K_{φ} , K_{η} , $\frac{\alpha\Delta T}{t}$

$$\begin{bmatrix} -8 & +6 \\ -6 & +21 \end{bmatrix} ; \begin{bmatrix} 1/4 \\ 7/8 \end{bmatrix} \rightarrow \begin{bmatrix} 0 \\ 1/24 \end{bmatrix}$$

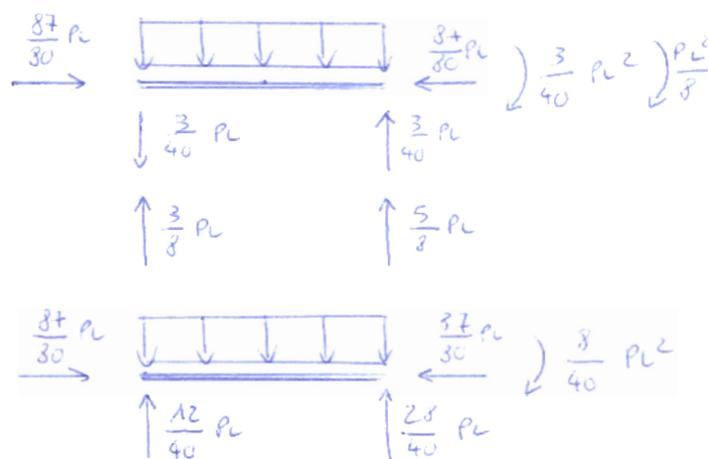
$$\begin{bmatrix} -8 & +6 \\ -6 & +32 \end{bmatrix} ; \begin{bmatrix} 1/4 \\ 7/8 \end{bmatrix} \rightarrow \begin{bmatrix} -1/80 \\ 1/40 \end{bmatrix}$$

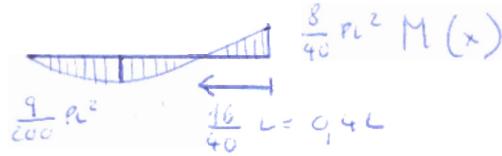
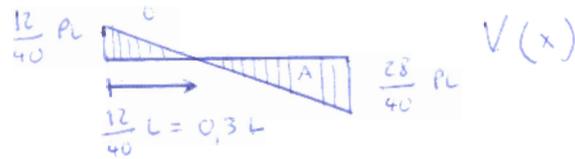
$\varphi_E = -\frac{1}{80} \frac{PL^3}{EJ}$; $\eta = \frac{1}{40} \frac{PL^4}{EJ}$

$K_{\varphi} = 1 \times \frac{EJ}{L}$ $K_{\eta} = 17 \frac{EJ}{L^3}$ $\frac{\alpha\Delta T}{t} = \frac{1}{16} \frac{PL^2}{EJ}$

DETERMINAZIONE DELLE AZIONI INTERNE

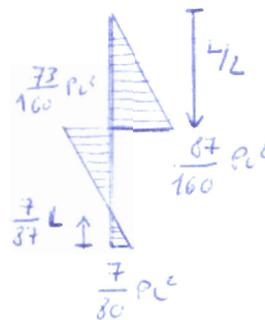
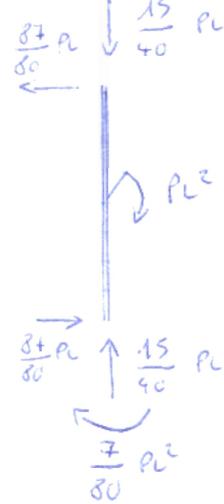
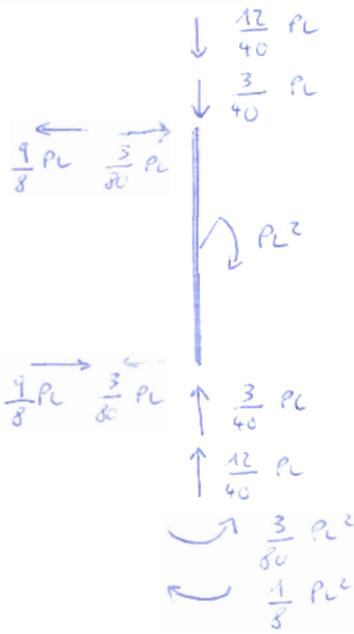
ASTA BC





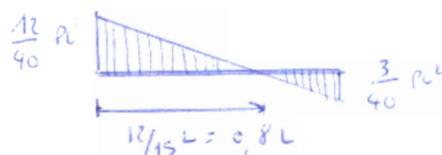
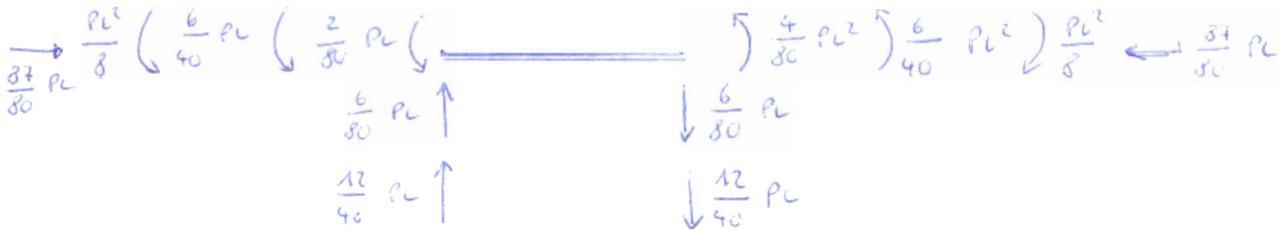
DEFORMATA QUANTITATIVA

ASTA BE



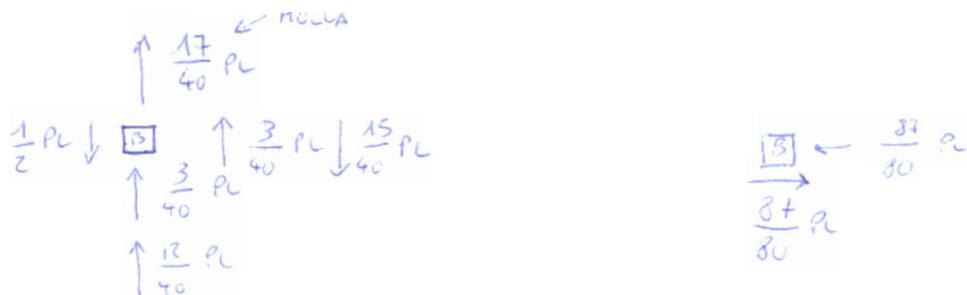
DEFORMATA QUANTITATIVA

ASTA DE



EQUILIBRIO DEI NODI

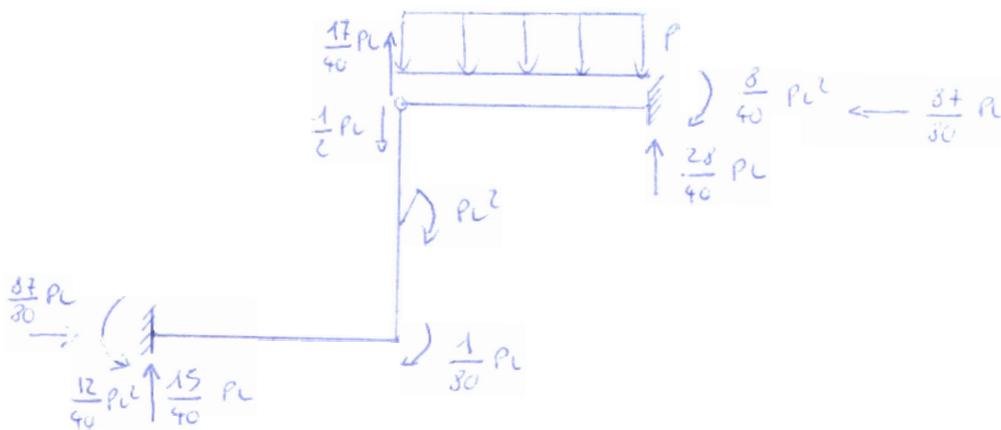
NODO B



NODO I



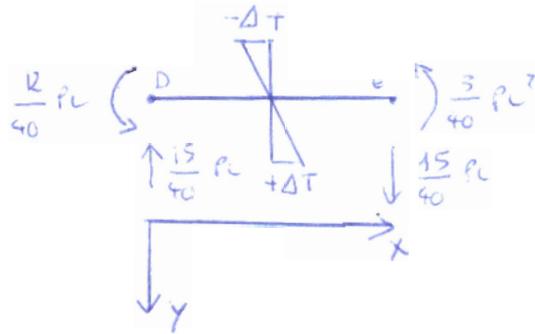
EQUILIBRIO GLOBALE



$$\begin{cases} \sum H = \phi \\ \sum V = \phi \\ \sum M_0 = \phi \end{cases}$$

$$\begin{cases} + \frac{87}{80} PL - \frac{87}{80} PL = \phi \\ + \frac{15}{40} PL + \frac{1}{2} PL + \frac{17}{40} PL + \frac{28}{40} PL - 2 PL = \phi \\ - \frac{17}{40} PL^2 + \frac{1}{2} PL^2 + PL \times \frac{3}{2} L + \frac{8}{40} PL^2 - \frac{87}{80} PL^2 + \\ - \frac{28}{40} PL \times 2L + PL^2 + \frac{1}{30} PL^2 - \frac{12}{40} PL^2 = \phi \end{cases}$$

DETERMINAZIONE DELLA DEFURATA DELL'ASTA DC



$$M(x) = \frac{15}{40} PLx - \frac{12}{40} P \cdot L^2$$

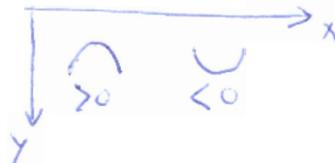
$$\frac{\alpha \Delta T}{t} = \frac{1}{16} \frac{PL^2}{EJ}$$

$$y''(x) = -\frac{M(x)}{EJ} - \frac{2\alpha \Delta T}{t}$$

$$y''(x) = \frac{15}{40} \frac{PLx}{EJ} + \frac{12}{40} \frac{PL^2}{EJ} - \frac{1}{8} \frac{PL^2}{EJ}$$

$$y''(x) = -\frac{15}{40} \frac{PLx}{EJ} + \frac{7}{40} \frac{PL^2}{EJ}$$

$$y''(x) > 0 \quad \rightarrow \quad x < \frac{7}{15} L$$



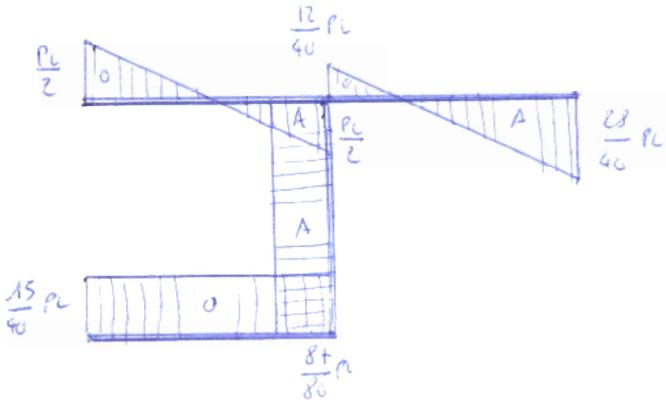
$$\text{SE } x < \frac{7}{15} L$$

CONCAVITA' VERSO IL BASSO

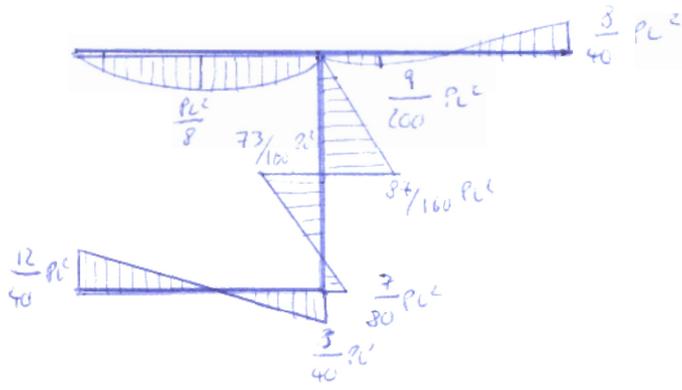
$$\text{SE } x > \frac{7}{15} L$$

CONCAVITA' VERSO L'ALTO

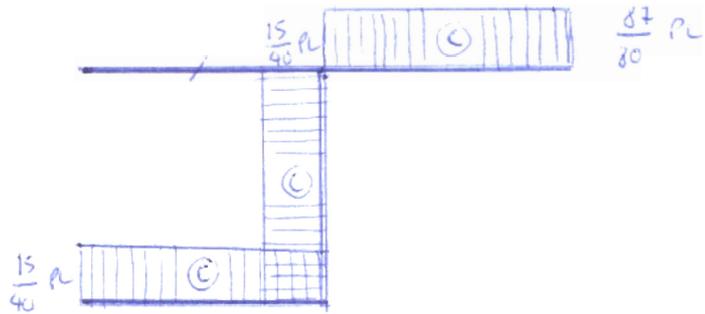
TRACCIAMENTO AZIONI INTERNE



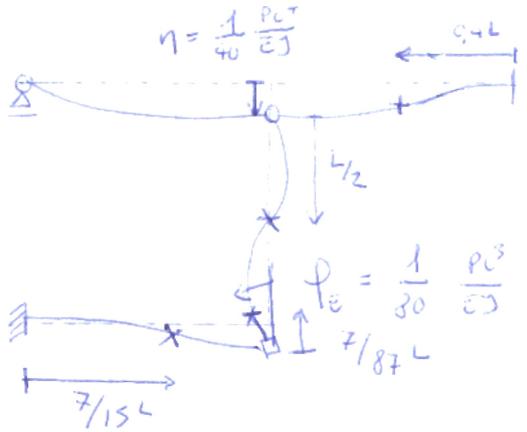
AZIONE DI TAGLIO V



MOMENTO FLETTENTE M



AZIONE ASSIALE N



DEFORMAZIONE QUANTITATIVA

FONDAMENTI DI PROGETTAZIONE STRUTTURALE

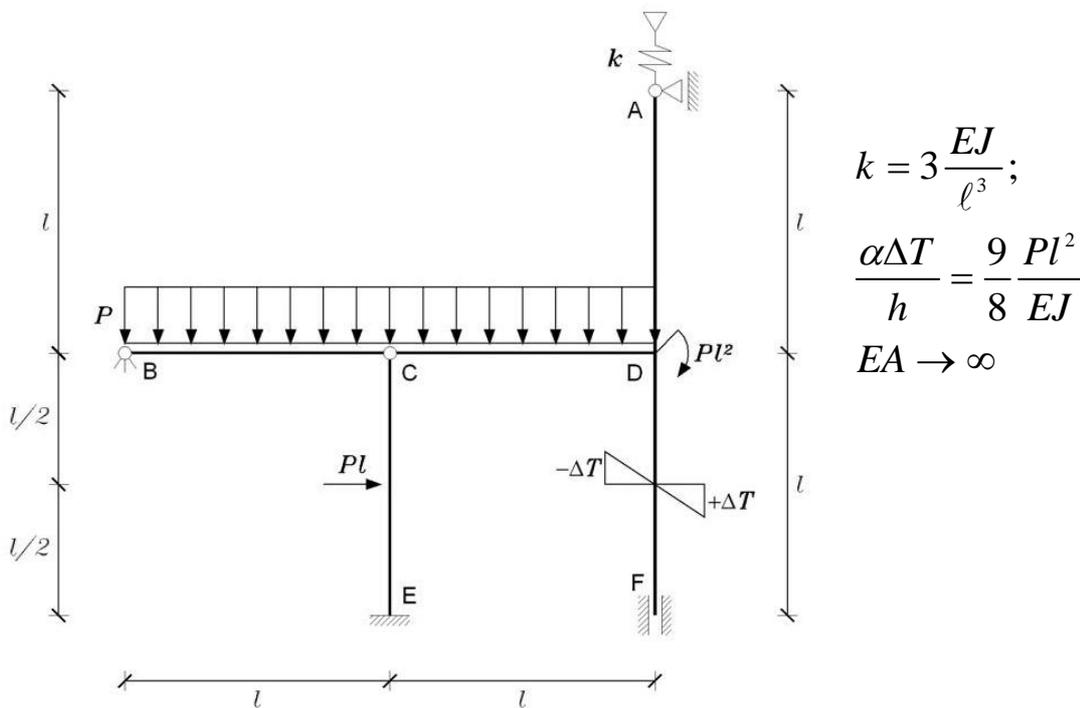
ESAME 23/03/2010, 3h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:

Superato ELEMENTI STRUTTURALI A? Sì NO

ESERCIZIO 1 (20 punti)



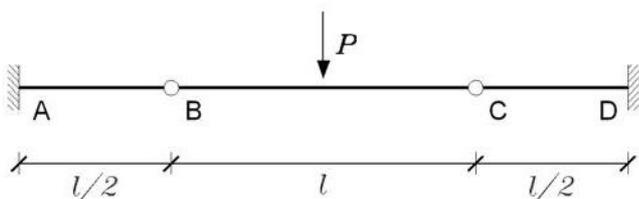
Dato il telaio in figura

Si richiedono:

- 1- Momento flettente (con il valore e la posizione dei massimi)
- 2- Taglio
- 3- Azione assiale
- 4- Deformata qualitativa con posizione dei flessi

ESERCIZIO 2 (6 punti)

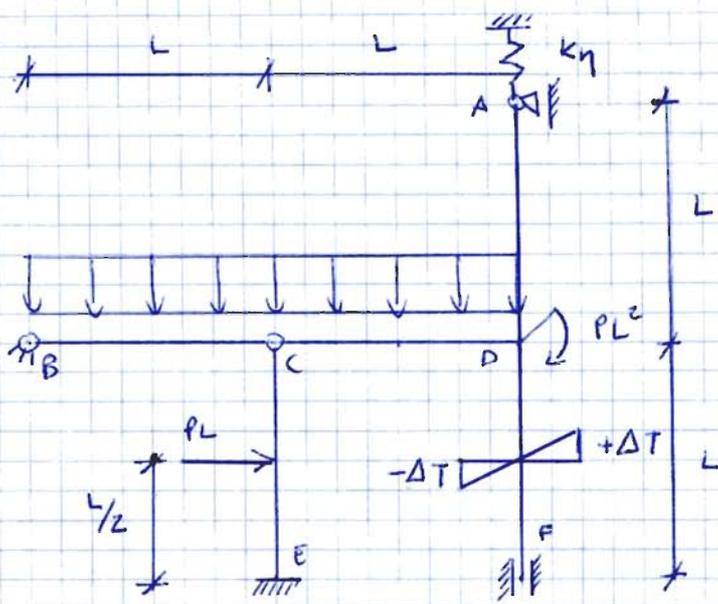
Dato lo schema statico in figura:



$$P = 5500 \text{ kg}$$

$$l = 5 \text{ m}$$

TEMA D'ESAME



$$k_h = 3 \frac{EJ}{L^3}$$

$$\frac{\alpha \Delta T}{t} = \frac{9}{8} \frac{PL^2}{EJ}$$

$$EA \rightarrow \infty$$

$$EJ = \text{cost}$$

ANALISI CINEMATICA

GDL $3 + 3 + 3 = 9$

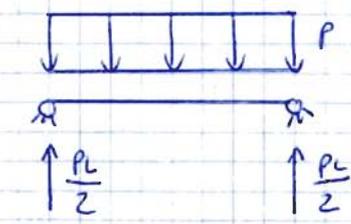
GDV $2 + 2 + 4 + 3 + 2 = 13$

I $13 - 9 = 4$

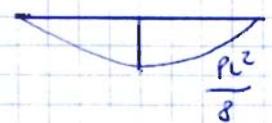
STRUTTURA A NODI SPOSTABILI

APPENDICE ISOSTATICA

ASTA BC



$V(x)$

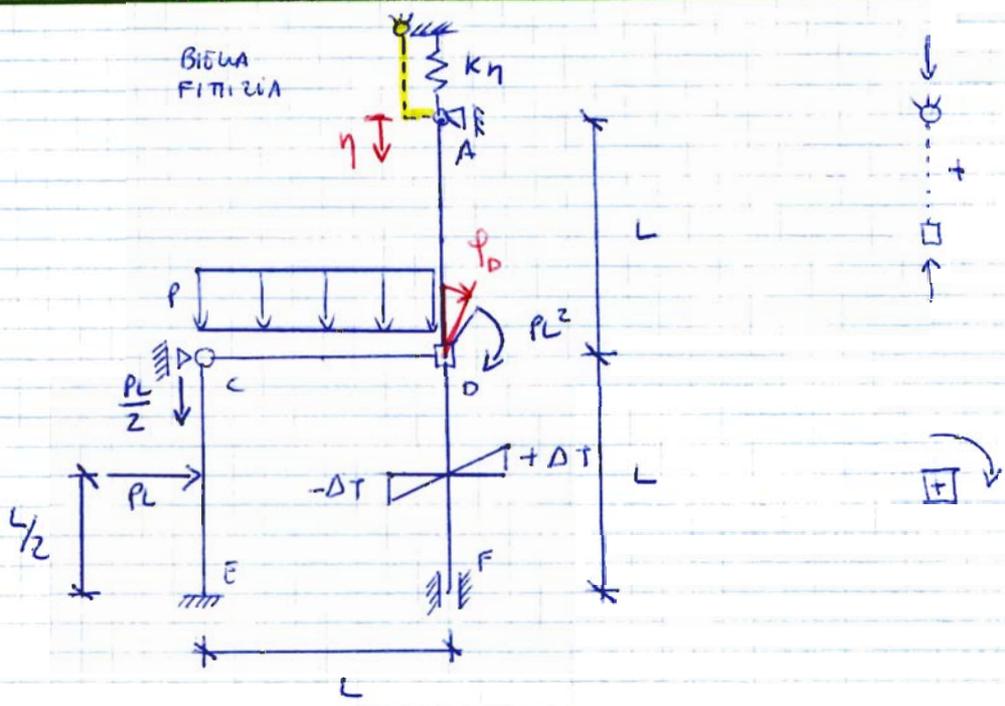


$M(x)$



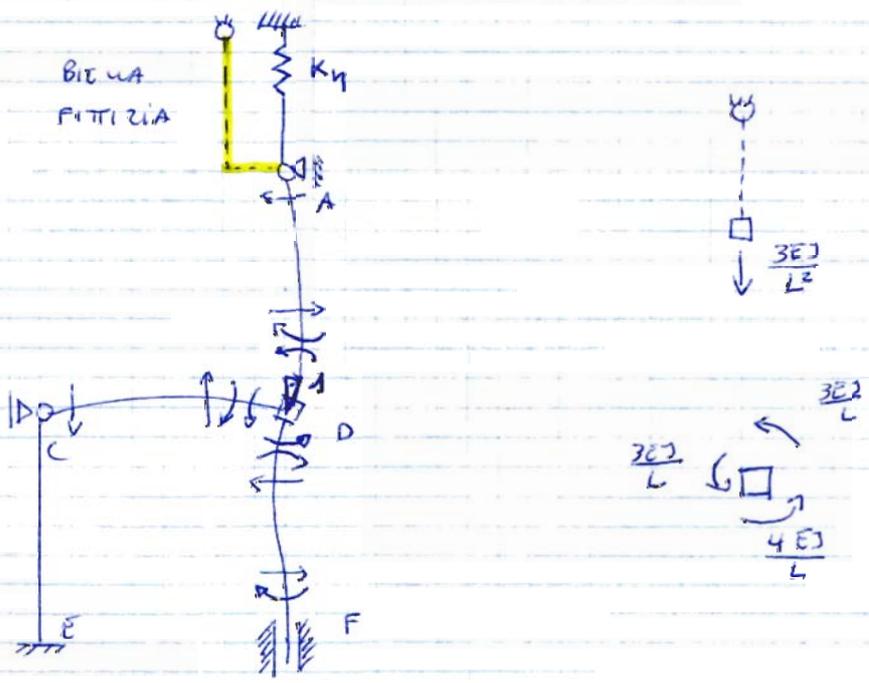
DEFORMATA QUANTITATIVA

SISTEMA RISOLVENTE / CONVENIENZE DI SEGNO



$$\begin{cases} \sum m_D = \phi \\ \sum F_{biella} = \phi \end{cases} \begin{cases} m_{DD} * \phi_D + m_{D\eta} * \eta + m_{D\phi} = \phi \\ h_{AD} * \phi_D + h_{A\eta} * \eta + h_{A\phi} = \phi \end{cases}$$

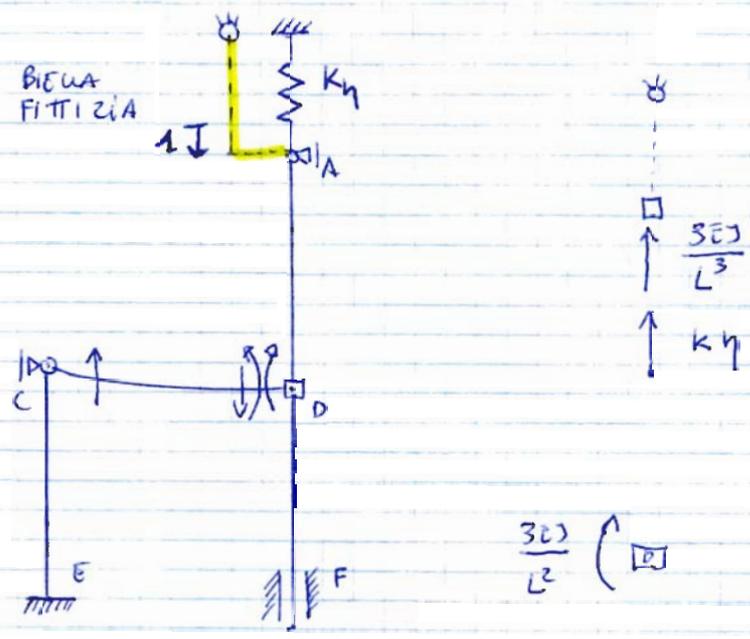
$\phi_D = 1 \quad \eta = \phi \quad P = \phi \quad \Delta T = \phi$



$$m_{DD} = - \frac{10 EJ}{L}$$

$$h_{AD} = - \frac{3 EJ}{L^2}$$

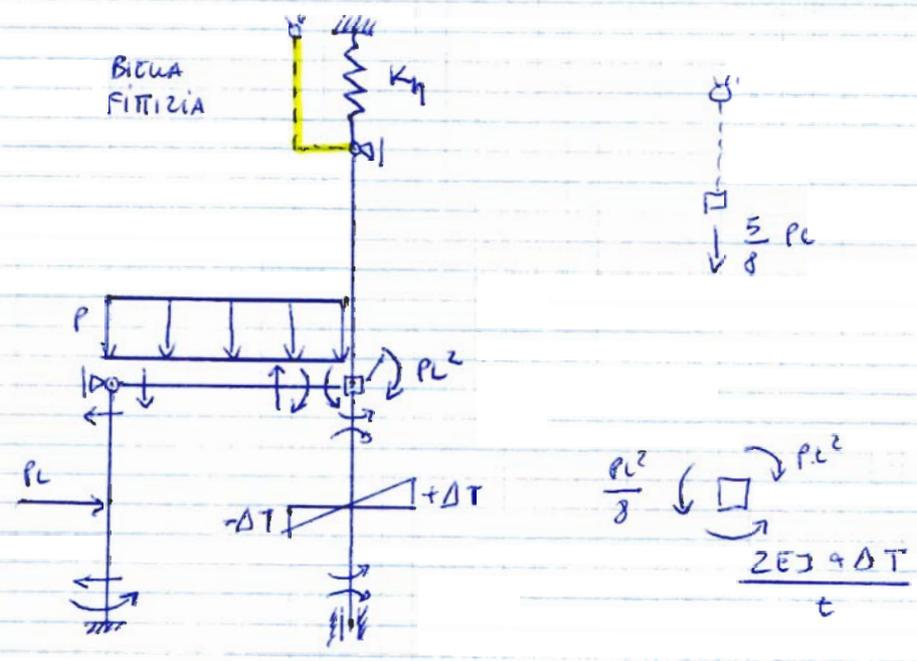
$\phi_D = \phi$ $\eta = 1$ $P = \phi$ $\Delta T = \phi$



$m_{D\eta} = + \frac{3EJ}{L^2}$

$h_{A\eta} = + \frac{3EJ}{L^3} + K\eta$

$\phi_D = \phi$ $\eta = \phi$ $P \neq \phi$ $\Delta T \neq \phi$



$m_{D\phi} = PL^2 - \frac{PL^2}{8} - \frac{2EJ + \Delta T}{t}$

$h_{A\phi} = -\frac{5}{8} PL$

SISTEMA RISOLVENTE

$$\begin{cases} \left(-10 \frac{EJ}{L}\right) \times \rho_0 + \left(\frac{3EJ}{L^2}\right) \times \eta = \left(\frac{2EJ \alpha \Delta T}{t} - \frac{7}{8} PL^2\right) \\ \left(-3 \frac{EJ}{L^2}\right) \times \rho_0 + \left(\frac{3EJ}{L^3} + K_\eta\right) \times \eta = \left(\frac{5}{8} PL\right) \end{cases}$$

$$\begin{bmatrix} -10 \frac{EJ}{L} & + 3 \frac{EJ}{L^2} \\ -3 \frac{EJ}{L^2} & + \frac{3EJ}{L^3} + K_\eta \end{bmatrix} \times \begin{bmatrix} \rho_0 \\ \eta \end{bmatrix} = \begin{bmatrix} \frac{2EJ \alpha \Delta T}{t} - \frac{7}{8} PL^2 \\ \frac{5}{8} PL \end{bmatrix}$$

DETERMINAZIONE DELLE COSTANTI $K_\eta = \frac{\alpha \Delta T}{t}$

$$\begin{bmatrix} -10 & +3 \\ -3 & +15 \end{bmatrix} \cdot \begin{bmatrix} 1/8 \\ 5/8 \end{bmatrix} \rightarrow \begin{bmatrix} 0 \\ 1/24 \end{bmatrix}$$

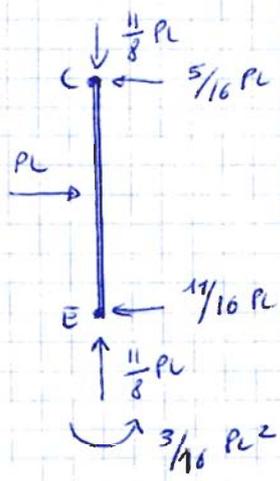
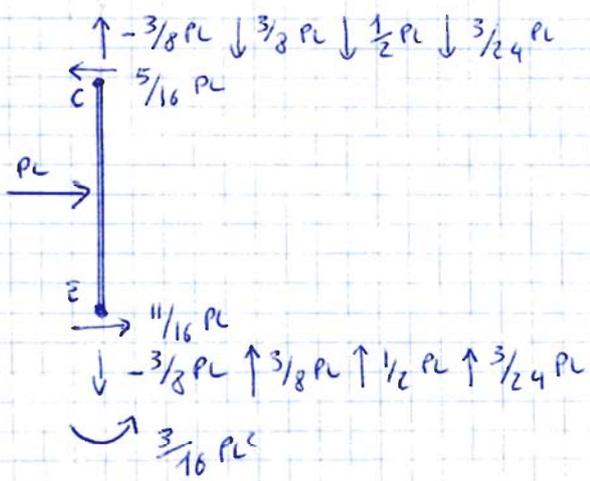
$$\begin{bmatrix} -10 & +3 \\ -3 & +6 \end{bmatrix} \cdot \begin{bmatrix} 11/8 \\ 5/8 \end{bmatrix} \rightarrow \begin{bmatrix} -1/8 \\ 1/24 \end{bmatrix}$$

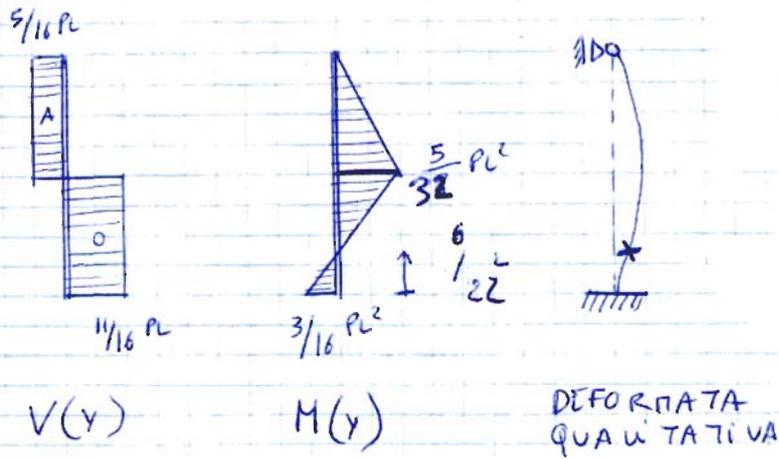
$$\rho_0 = -\frac{1}{8} \frac{PL^3}{EJ} \quad \eta = \frac{1}{24} \frac{PL^4}{EJ}$$

$$K_\eta = 3 \frac{EJ}{L^3} \quad \frac{2EJ \alpha \Delta T}{t} = \frac{18}{8} PL^2 \rightarrow \frac{\alpha \Delta T}{t} = \frac{9}{8} \frac{PL^2}{EJ}$$

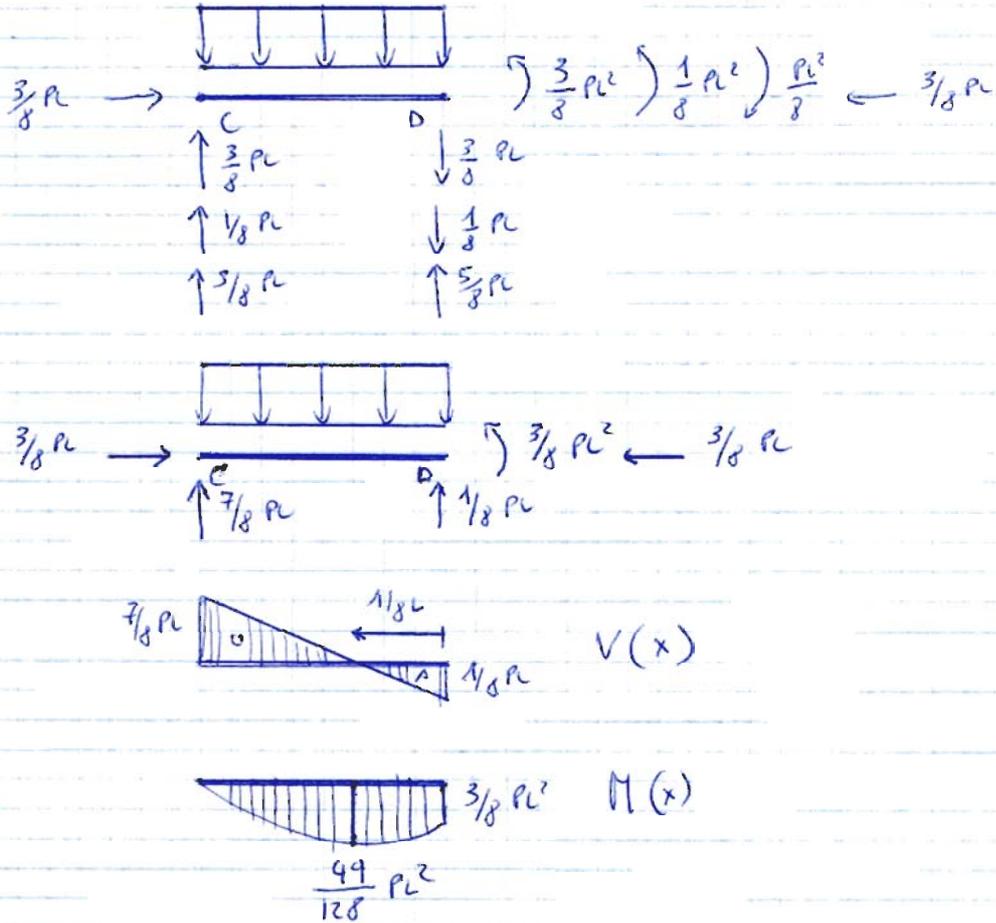
DETERMINAZIONE DEI MOMENTI AZIONI INTERNE

ASTA CE

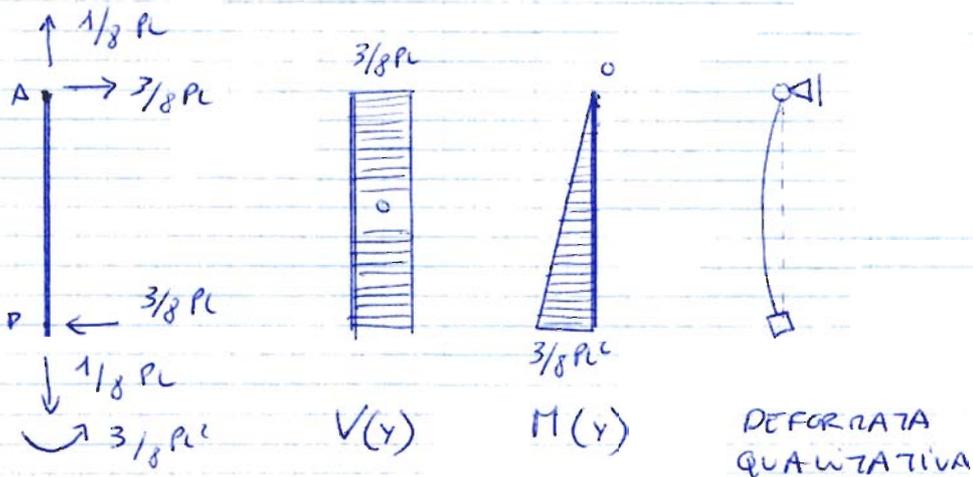




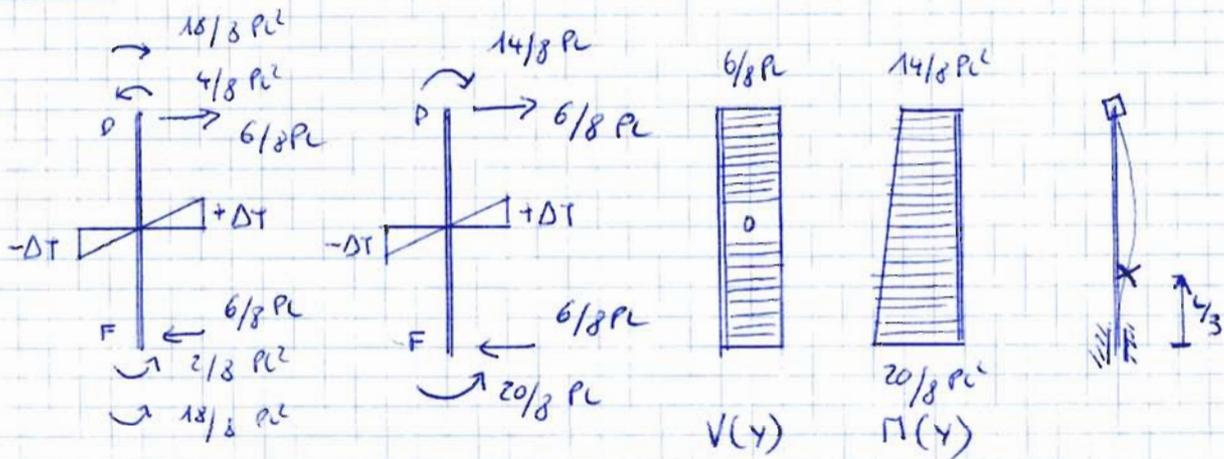
ASTA CD



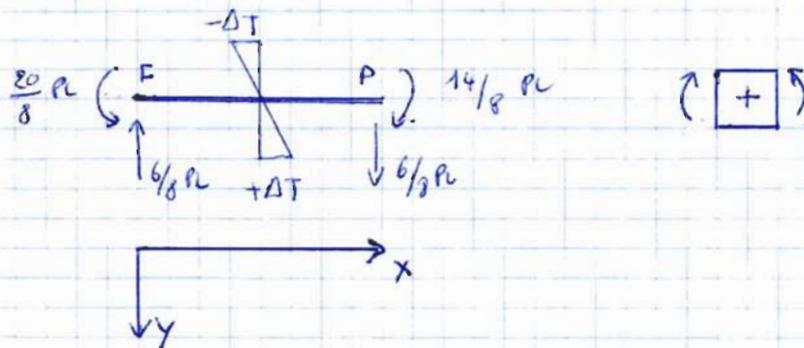
ASTA AD



ASTA DF



ASTA DF : DETERMINAZIONE DELLA DEFORMATA



$$M(x) = \frac{6}{8} PLx - \frac{20}{8} PL^2$$

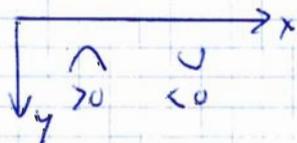
$$\frac{\alpha \Delta T}{t} = \frac{9}{8} \frac{PL^2}{EJ}$$

$$y''(x) = - \frac{M(x)}{EJ} - \frac{2\alpha \Delta T}{t}$$

$$y''(x) = - \frac{6}{8} \frac{PLx}{EJ} + \frac{20}{8} \frac{PL^2}{EJ} - \frac{9}{4} \frac{PL^2}{EJ}$$

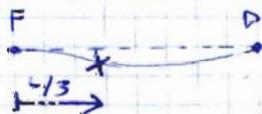
$$y''(x) = - \frac{6}{8} \frac{PLx}{EJ} + \frac{2}{8} \frac{PL^2}{EJ}$$

$$y''(x) > 0 \rightarrow x < \frac{1}{3} L$$



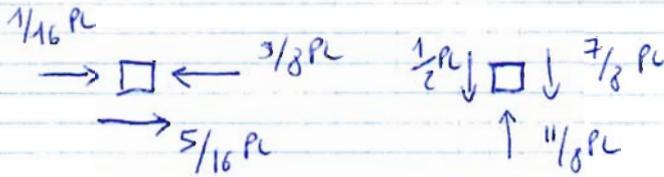
se $x < \frac{1}{3} L$ CONCAVITA' VERSO IL BASSO

se $x > \frac{1}{3} L$ CONCAVITA' VERSO L'ALTO

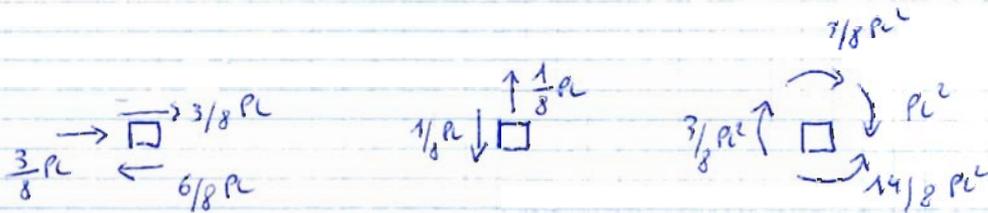


EQUILIBRIO DEI NODI

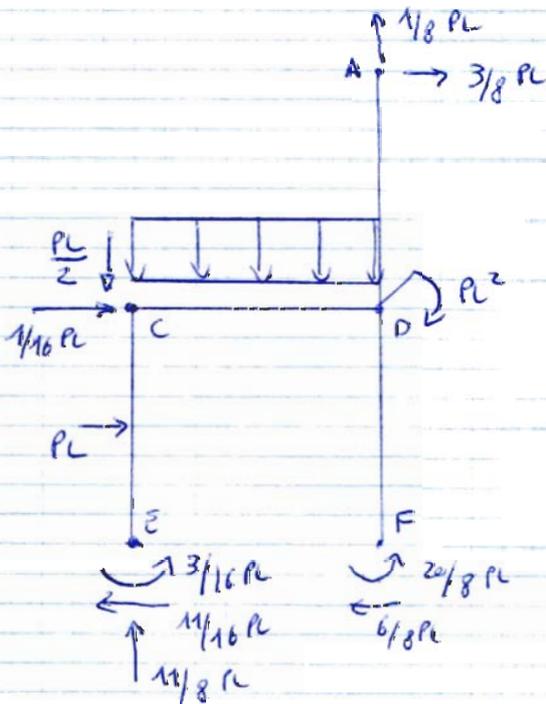
NODO C



NODO D



EQUILIBRIO GLOBALE



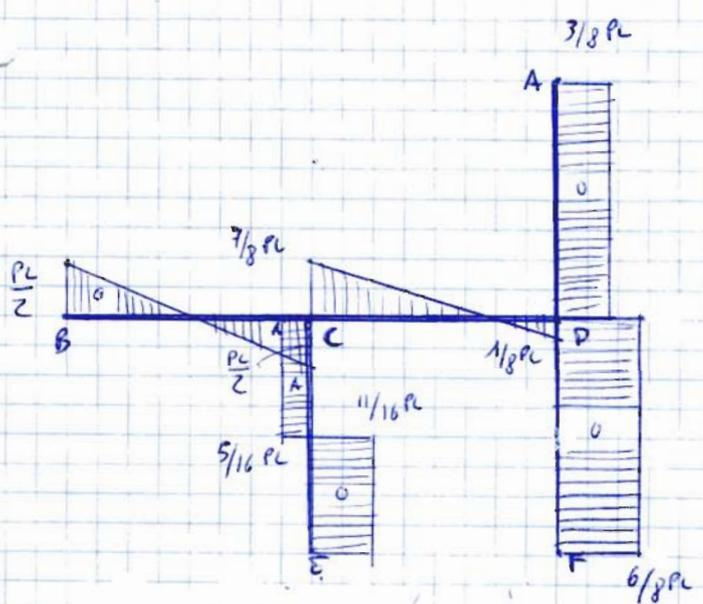
$$\sum V = \phi \quad +\uparrow \quad PL \times \left(-\frac{1}{8} - \frac{8}{8} - \frac{4}{8} + \frac{11}{8} \right) = \phi$$

$$\sum H = \phi \quad +\rightarrow \quad PL \times \left(\frac{3}{8} - \frac{6}{8} + \frac{1}{16} - \frac{11}{16} + 1 \right) =$$

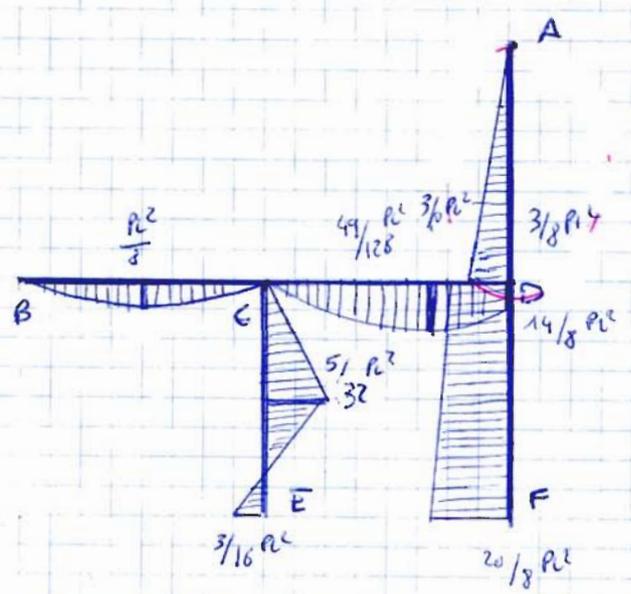
$$= \frac{PL}{16} \times \left(6 - 12 + 1 - 11 + 16 \right) = \phi$$

$$\sum M_D = \phi \quad +\curvearrowright \quad -\frac{1}{8} PL^2 + \frac{3}{4} PL^2 + PL^2 + \frac{PL^2}{2} + \frac{1}{16} PL^2 - \frac{20}{8} PL^2 + \frac{PL^2}{2} - \frac{3}{16} PL^2 = \phi$$

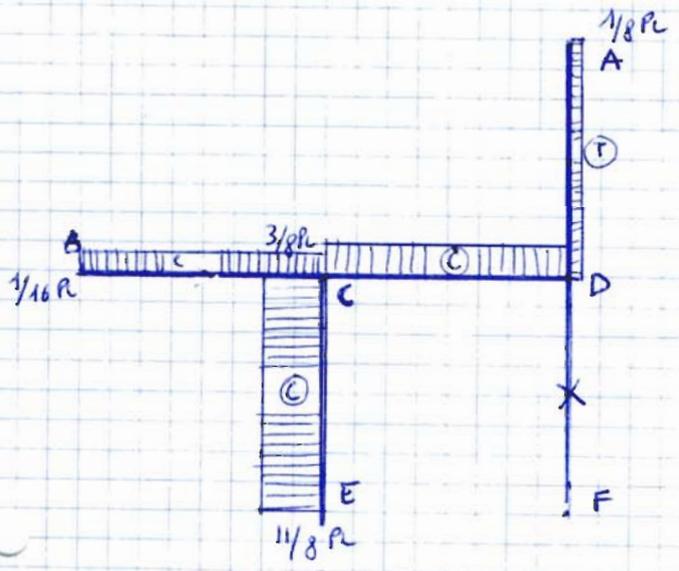
TRACCIAMENTO PER AZIONI INTERNE E DEFORMAZIA



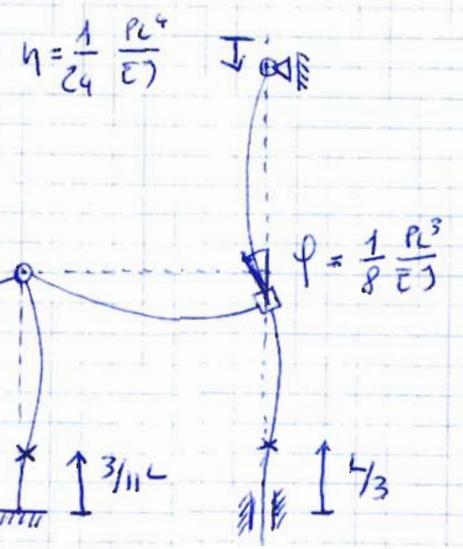
AZIONE DI TAGLIO V



MOMENTO FLESSIONE M



AZIONE ASSIALE N



$$\eta = \frac{1}{24} \frac{PL^4}{EI}$$

$$\phi = \frac{1}{8} \frac{PL^3}{EI}$$

FONDAMENTI DI PROGETTAZIONE STRUTTURALE

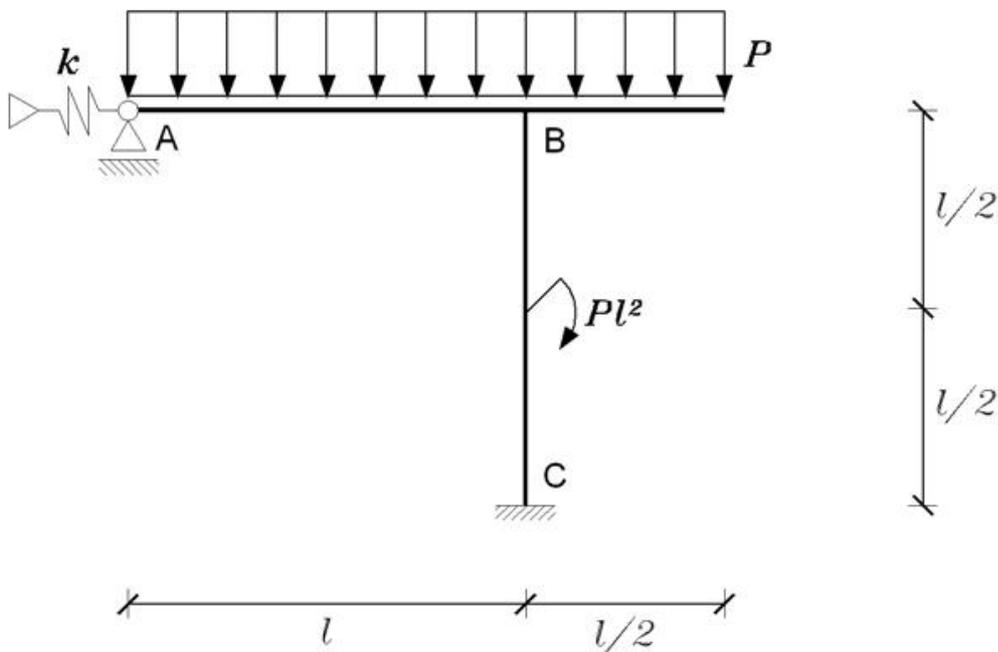
ESAME 12/04/2010, 3h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:.....

Superato ELEMENTI STRUTTURALI A? Sì NO

ESERCIZIO 1 (20 punti)

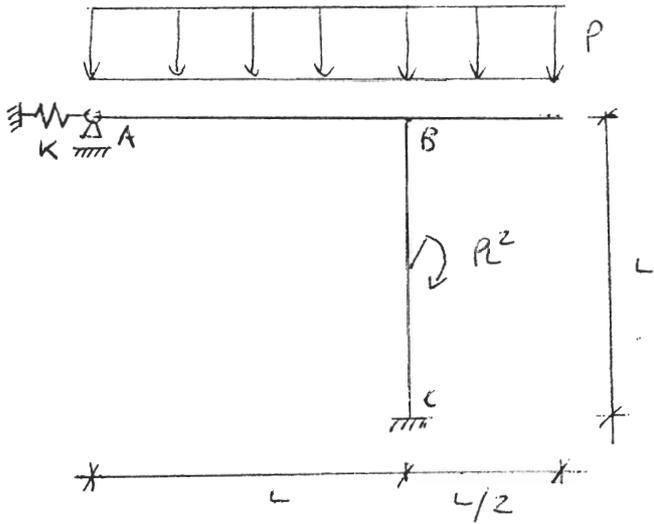


$$k = 6 \frac{EJ}{l^3}; \quad EJ = \cos t \quad EA \rightarrow \infty$$

Dato il telaio in figura

Si richiedono:

- 1- Momento flettente (con il valore e la posizione dei massimi)
- 2- Taglio
- 3- Azione assiale
- 4- Deformata qualitativa con posizione dei flessi



DATI:

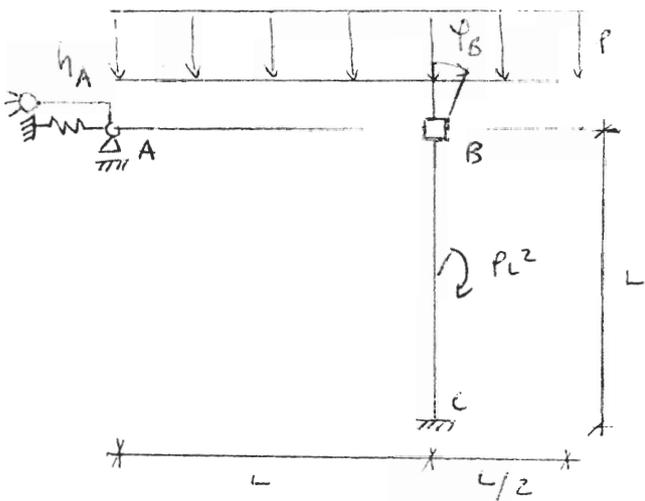
- * $EJ = \text{cost}$
- * $EA = \infty$
- * $K = 6 \frac{EJ}{L^3}$

□ ANALISI CINEMATICA

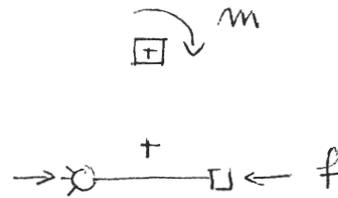
- * GRADI DI LIBERTA' $L = 3$
- * GRADI DI VINCOLO $V = 4$
- * GRADO DI IPERSTATICITA' $I = V - L = 1$

STRUTTURA 1 VOLTA IPERSTATICA A NODI SPOSTABILI

□ METODO DEGLI SPOSTAMENTI



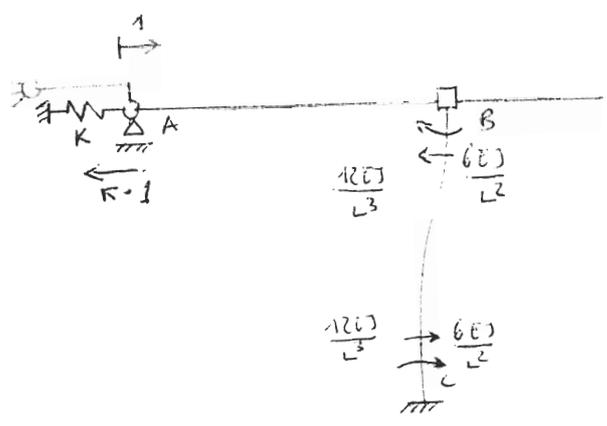
* CONVENZIONI DI SEGNO



* INCOGNITE

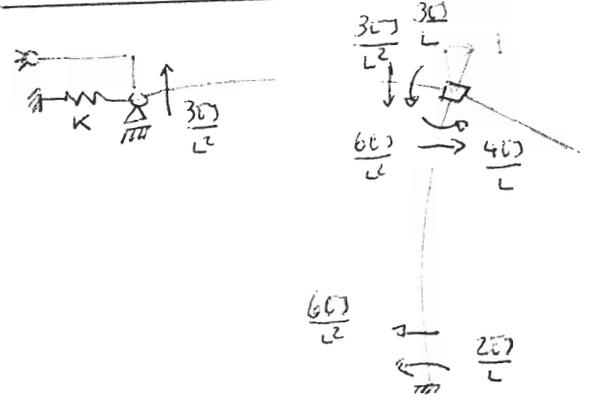
$h_A ; \phi_B$

$\square \eta_A = 1 \quad \psi_B = \phi \quad P = \phi$



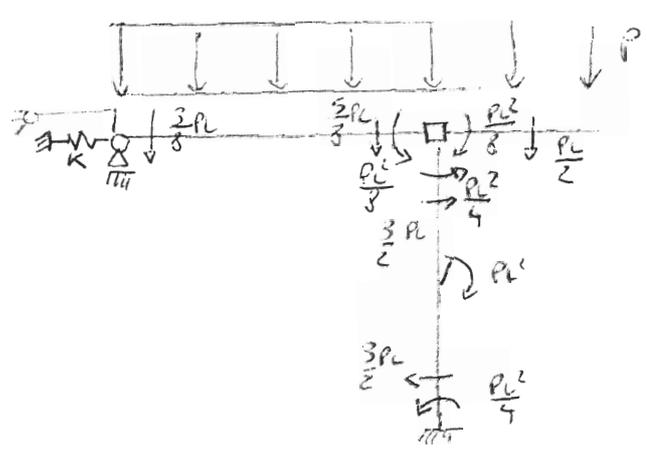
$$\begin{cases} f_{AA} = \frac{12EJ}{L^3} + K \\ m_{BA} = \frac{6EJ}{L^2} \end{cases}$$

$\square \eta_A = \phi \quad \psi_B = 1 \quad P = \phi$



$$\begin{cases} f_{AB} = -\frac{6EJ}{L^2} \\ m_{BB} = -\frac{7EJ}{L} \end{cases}$$

$\square \eta_A = \phi \quad \psi_B = \phi \quad P \neq \phi$



$$\begin{cases} f_{A\phi} = -\frac{3}{2} PL \\ m_{B\phi} = -\frac{PL^2}{4} \end{cases}$$

$$\begin{cases} \left(12 \frac{EJ}{L^3} + k\right) \eta_A + \left(-6 \frac{EJ}{L^2}\right) \psi_B - \frac{3}{2} PL = \phi \\ \left(6 \frac{EJ}{L^2}\right) \eta_A + \left(-7 \frac{EJ}{L}\right) \psi_B - \frac{PL^2}{4} = \phi \end{cases}$$

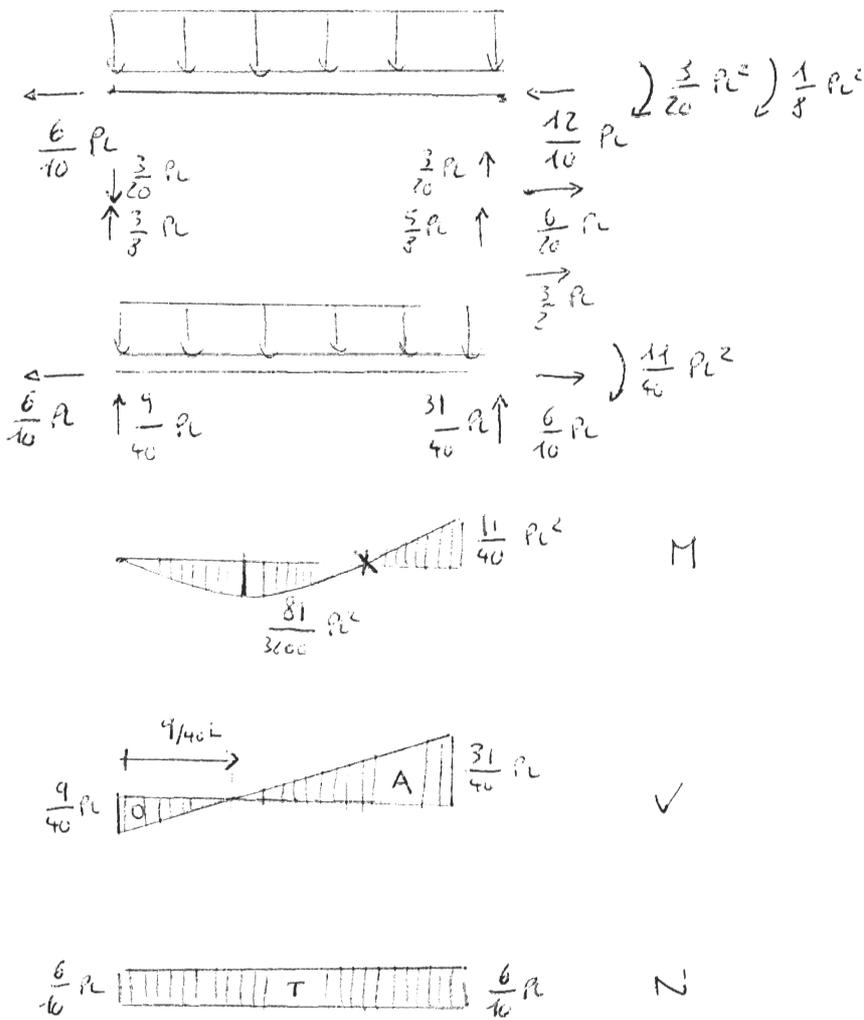
con

$$k = 6 \frac{EJ}{L^3}$$

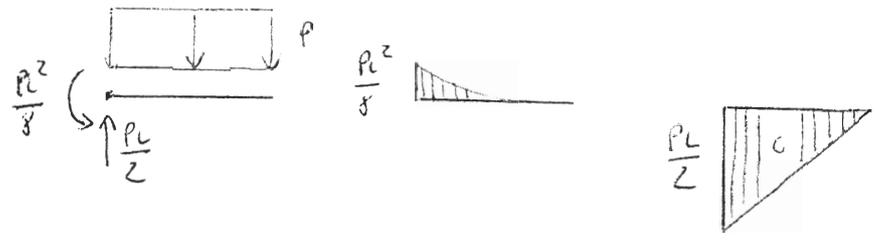
si HANNO

$$\begin{cases} \eta_A = \frac{1}{10} \frac{PL^4}{EJ} \\ \psi_B = \frac{1}{20} \frac{PL^3}{EJ} \end{cases}$$

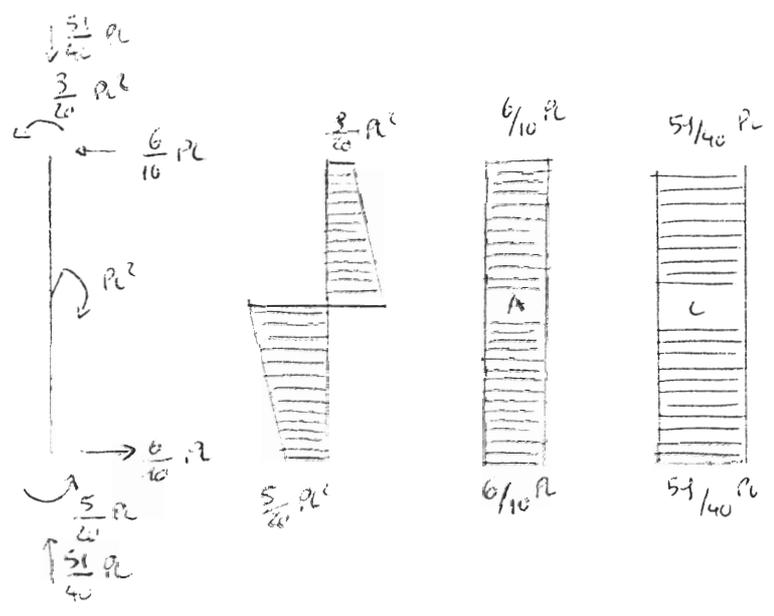
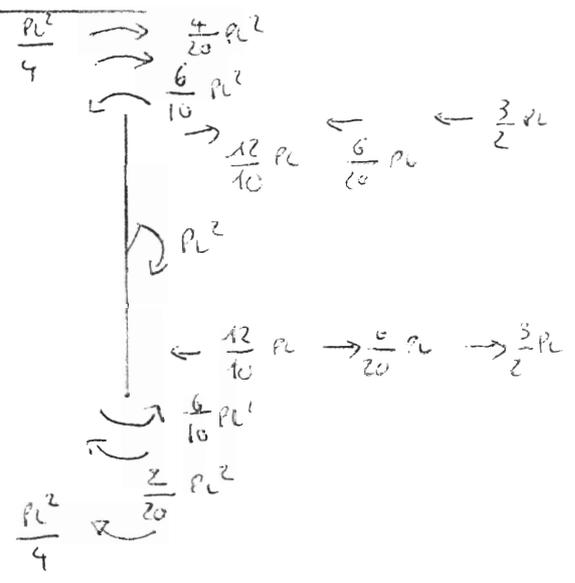
□ ASTA AB



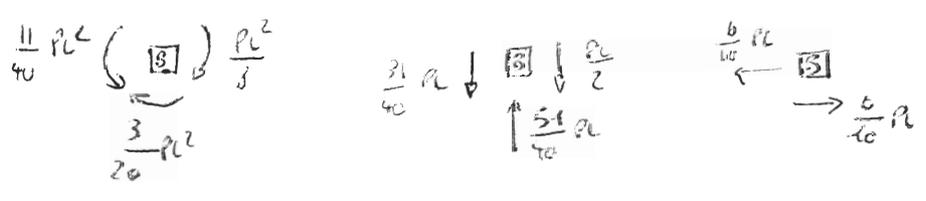
□ MENSOLO IN B



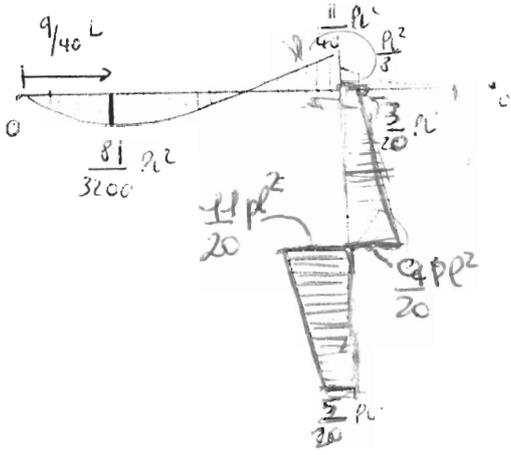
□ ASTA BC



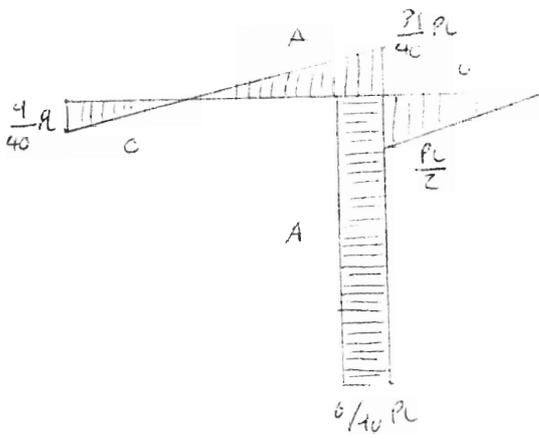
□ EQUILIBRIO NEL NODO B



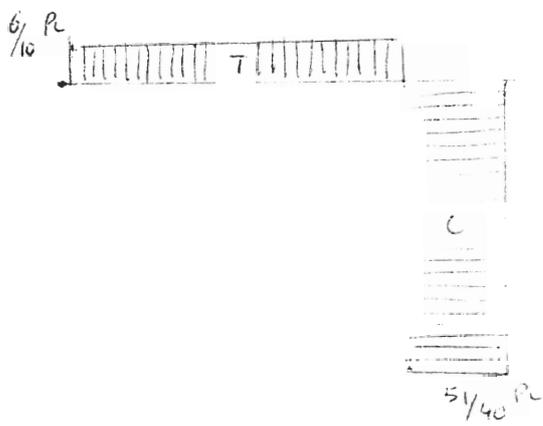
DIAGRAMMI DELLE AZIONI IN TORRE



MOMENTO FLESSIONE M



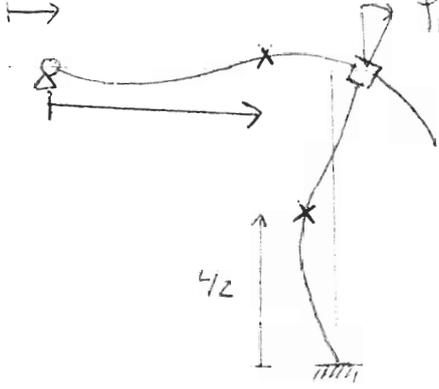
AZIONE DI TAGLIO V



AZIONE ASSIALE N

$$n_A = \frac{1}{10} \frac{PL^4}{EI}$$

$$f_B = \frac{1}{20} \frac{PL^3}{EI}$$



FONDAMENTI DI PROGETTAZIONE STRUTTURALE

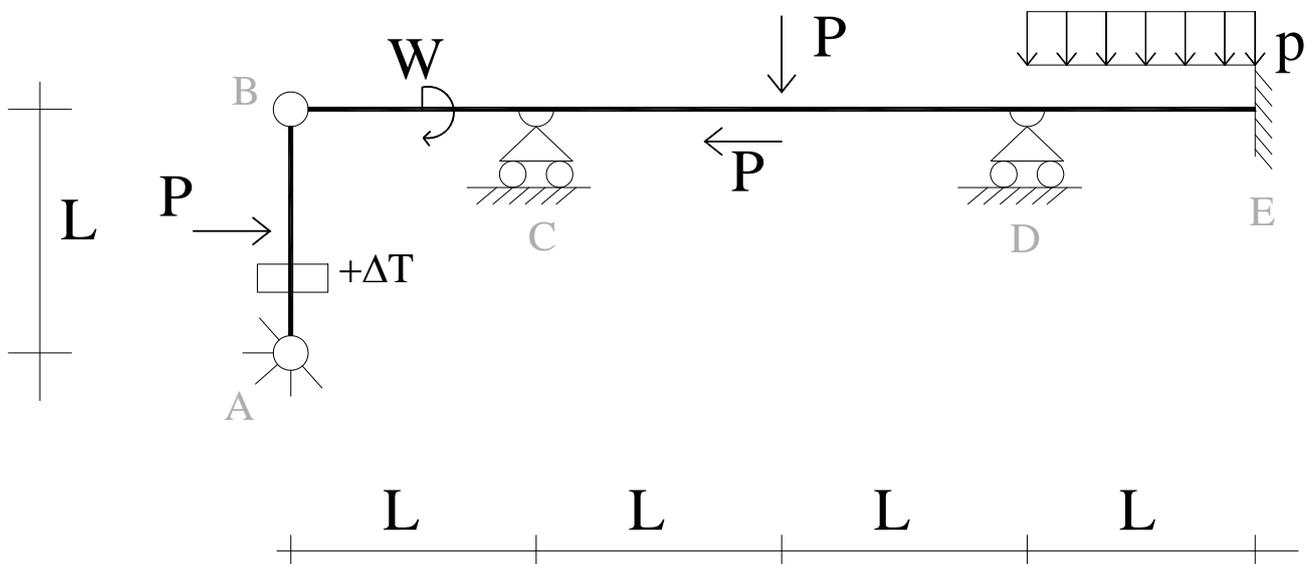
ESAME 05/07/2010, 3h

ing. F. Minelli; ing. F. Germano

Nome:..... Cognome:..... n.matr.:

Superato ELEMENTI STRUTTURALI A? Sì NO

ESERCIZIO 1 (20 punti)



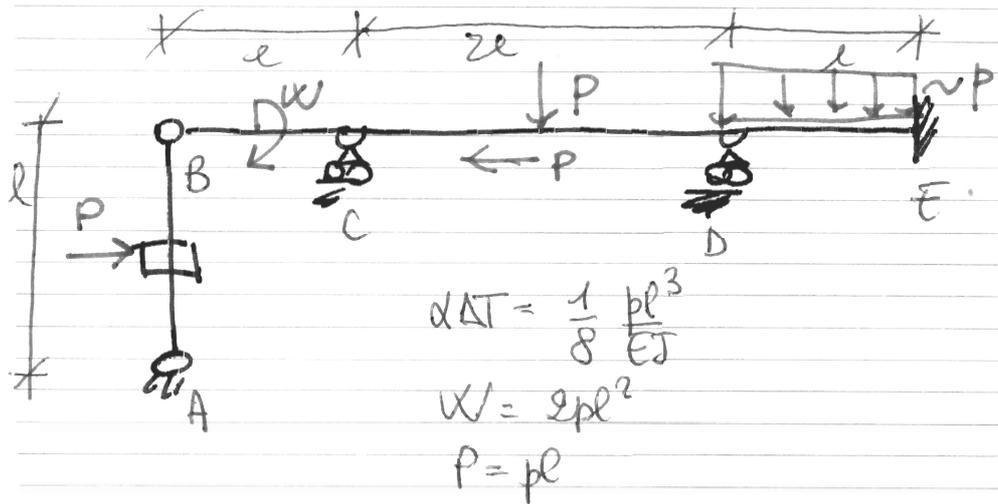
$$P = pL; \quad \alpha\Delta T = \frac{1}{8} \frac{pL^3}{EJ}; \quad W = 2pL^2; \quad EA \rightarrow \infty$$

Dato il telaio in figura

Si richiedono:

- 1- Momento flettente (con il valore e la posizione dei massimi)
- 2- Taglio
- 3- Azione assiale
- 4- Deformata qualitativa con posizione dei flessi

Tema Esame 05/07/2010

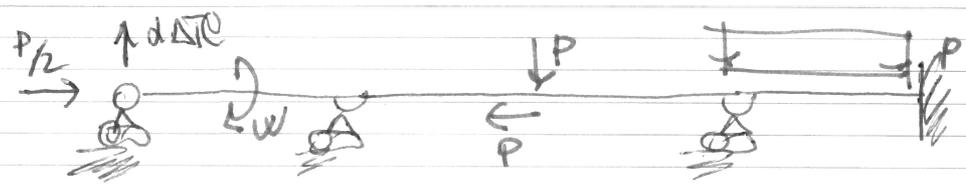


$$\alpha \Delta T = \frac{1}{8} \frac{p l^3}{EJ}$$

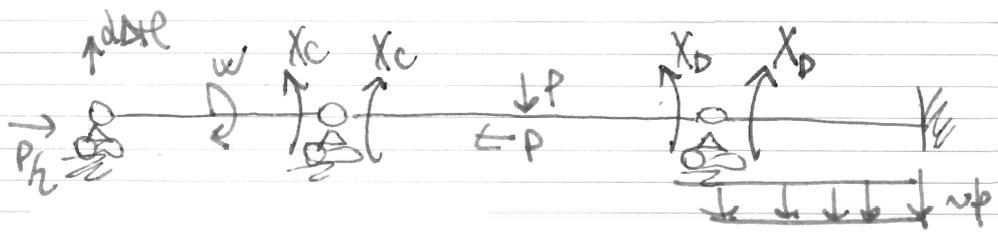
$$W = 2 p l^2$$

$$P = p l$$

Si osserva la presenza della buca AB.
 Si tratta di un'appendice isostatica.
 La possiamo pensare avendo, però, l'accortezza di considerare sulla trave continua BCDE il carico P e il "cedimento" improvviso in B dalle deformazioni termiche.

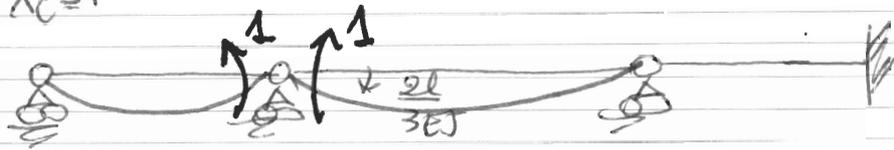


Si tratta di una trave continua. La risolviamo con il METODO delle FORZE



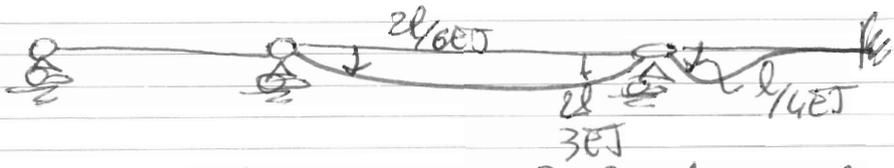
$$\begin{cases} \varphi_{cc} X_c + \varphi_{cd} X_d + \varphi_{c0} = 0 \\ \varphi_{dc} X_c + \varphi_{dd} X_d + \varphi_{d0} = 0 \end{cases}$$

$X_C = 1$



$$Y_{CC} = \frac{2l}{3EJ} - \left(-\frac{l}{3EJ}\right) = \frac{l}{EJ} \quad Y_{DC} = 0 - \left(-\frac{2l}{6EJ}\right) = \frac{l}{3EJ}$$

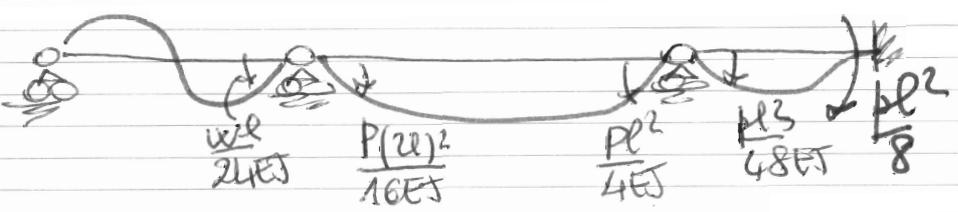
$X_D = 1$



$$Y_{DD} = \frac{l}{4EJ} - \left(-\frac{2l}{3EJ}\right) = \frac{3+8}{12} \frac{l}{EJ} = \frac{11l}{12EJ}$$

$$Y_{CD} = \frac{2l}{6EJ} - 0 = \frac{l}{3EJ}$$

$P \neq 0$



$$Y_{CO}^P = \frac{Pl^2}{4EJ} - \left(-\frac{wl}{24EJ}\right) = \frac{Pl^2}{4EJ} + \frac{wl}{4EJ}$$

$$Y_{DO}^P = \frac{Pl^3}{48EJ} - \left(-\frac{wl}{24EJ}\right) = \frac{Pl^3}{48EJ} + \frac{Pl^2}{4EJ}$$

$\Delta \neq 0$



$$Y_{CO}^{\Delta} = \frac{\Delta \cdot 2l \cdot l}{EJ} = -\Delta \cdot 2l$$

$$Y_{DO}^{\Delta} = 0$$

$$\varphi_{CO}^{TOT} = \varphi_{CO}^P + \varphi_{CO}^{\Delta T} = \frac{Pl^2}{4EJ} + \frac{Wl}{24EJ} - \Delta T$$

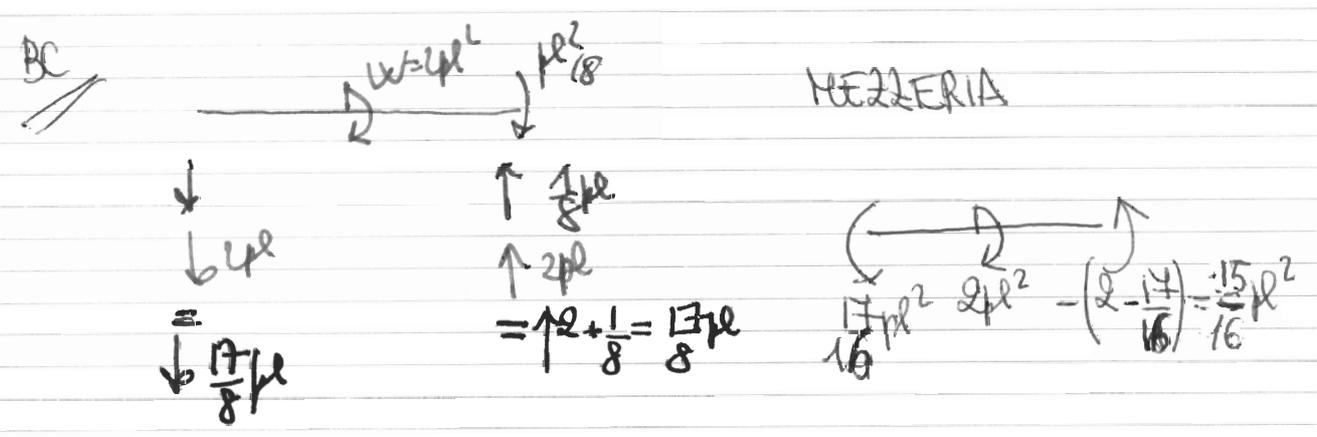
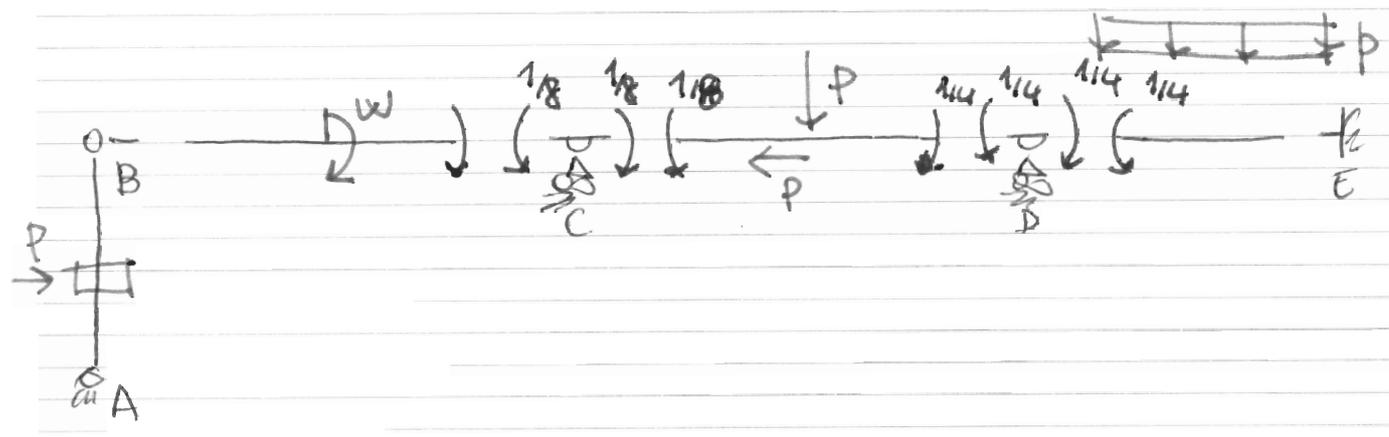
$$\varphi_{DO}^{TOT} = \frac{Pl^3}{48EJ} + \frac{Pl^2}{4EJ}$$

con $\Delta T = \frac{1}{8} \frac{Pl^3}{EJ}$; $W = 2Pl^2$

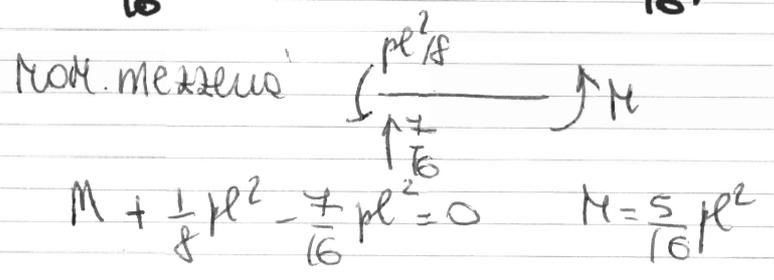
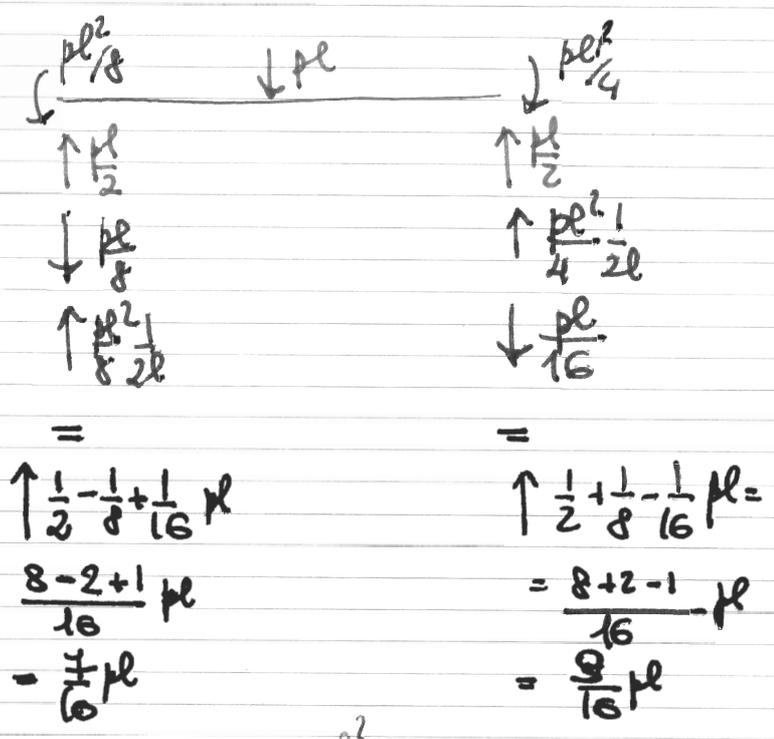
$$\varphi_{CO}^{TOT} = \frac{Pl^3}{4EJ} + \frac{2Pl^2}{24EJ} - \frac{1}{8} \frac{Pl^3}{EJ} = \frac{6+2-3}{24} \frac{Pl^3}{EJ} = \frac{5}{24} \frac{Pl^3}{EJ}$$

$$\varphi_{DO}^{TOT} = \frac{Pl^3}{48EJ} + \frac{Pl^3}{4EJ} = \frac{1+12}{48} \frac{Pl^3}{EJ} = \frac{13}{48} \frac{Pl^3}{EJ}$$

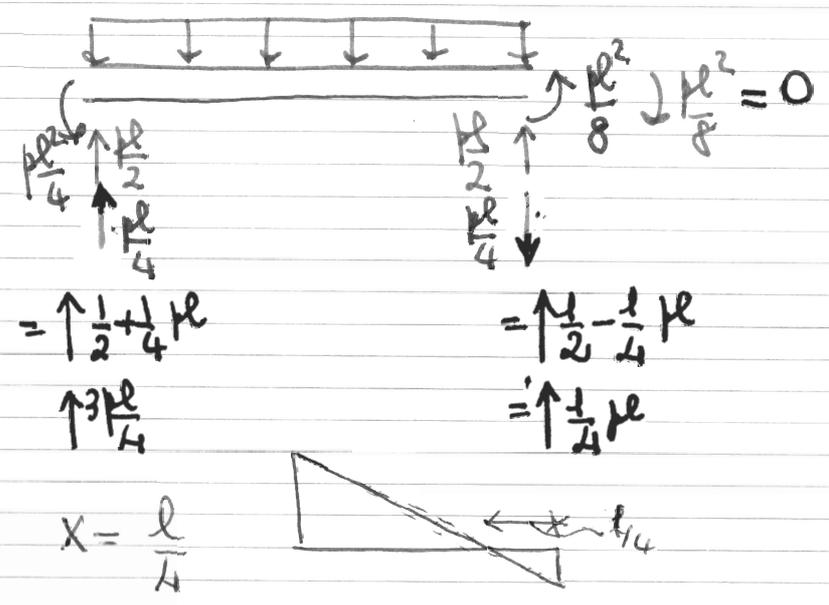
$$\begin{cases} \frac{l}{EJ} X_C + \frac{l}{3EJ} X_D + \frac{5Pl^3}{24EJ} = 0 \\ \frac{l}{3EJ} X_C + \frac{11l}{12EJ} X_D + \frac{13Pl^3}{48EJ} = 0 \end{cases} \rightarrow \begin{cases} X_C = -\frac{1}{8} Pl^2 \\ X_D = -\frac{1}{4} Pl^2 \end{cases}$$



CD



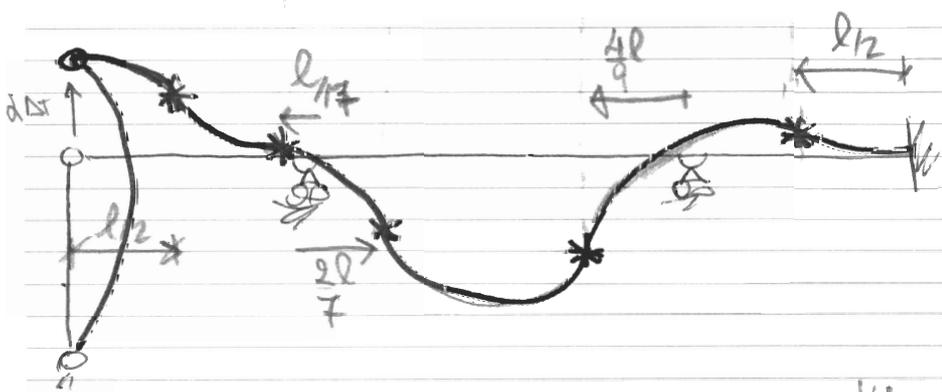
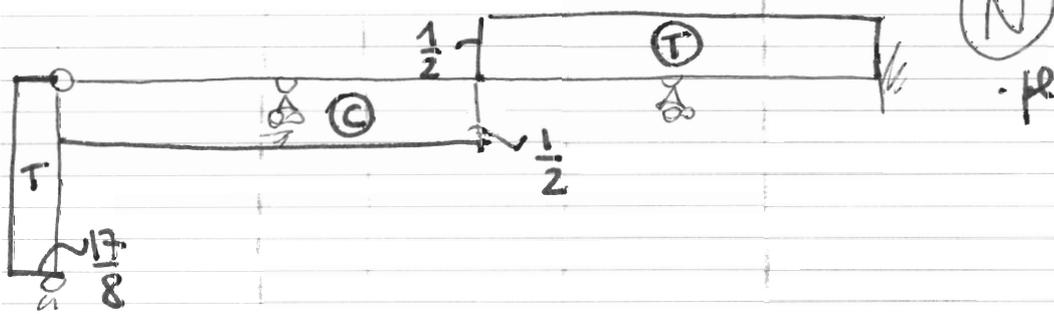
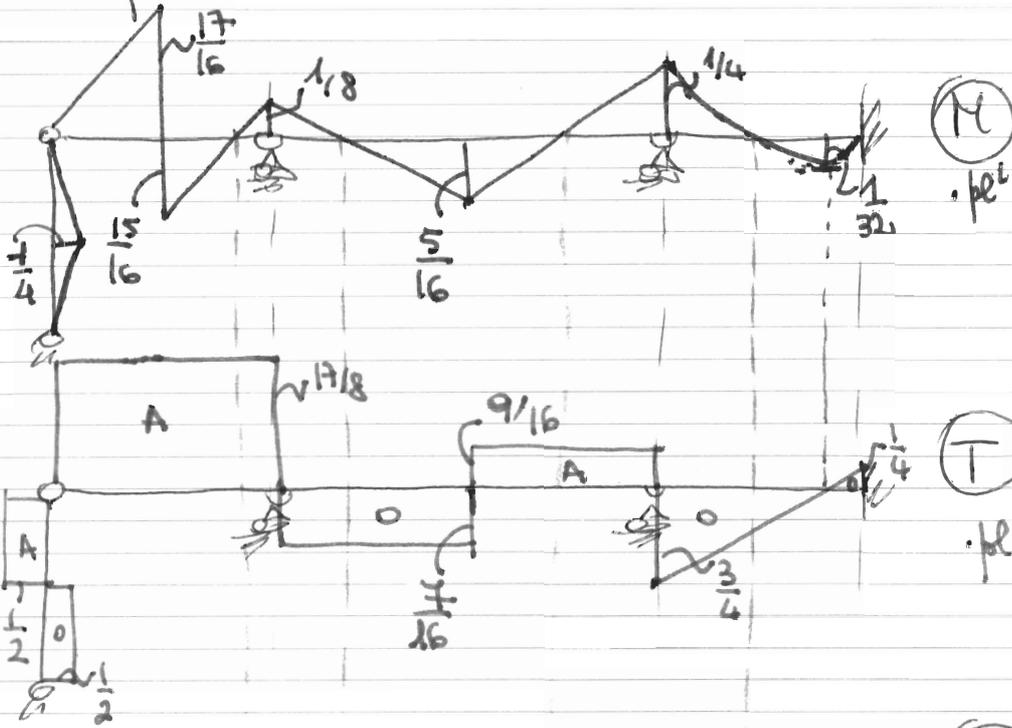
DE



$$M_{max}(\frac{l}{4}) = \frac{p l}{4} \cdot \frac{l}{4} - \frac{p l}{4} \cdot \frac{1}{2} \cdot \frac{l}{4}$$

$$= \frac{p l^2}{32}$$

Diagramma delle AZIONI INTERNE



$$\frac{1}{8} \cdot x = \frac{15}{16} \left(\frac{l}{2} - x \right)$$

$$\frac{15}{16} x = \frac{1}{8} \left(\frac{l}{2} - x \right)$$

$$\left(\frac{15}{2} + 1 \right) x = \frac{l}{8} \Rightarrow x = \frac{l}{17}$$

$$\frac{1}{8} \cdot x = \frac{5}{16} (l - x)$$

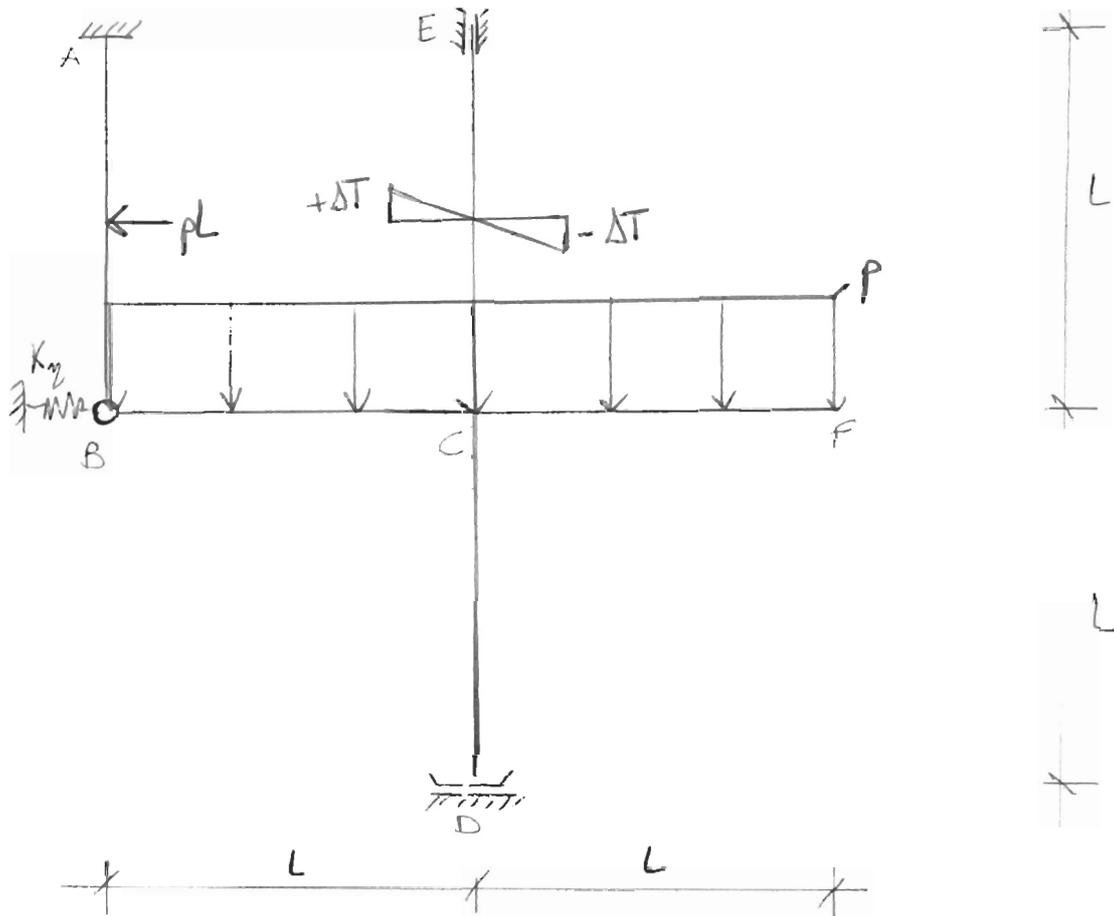
$$\frac{5}{16} x = \frac{1}{8} (l - x)$$

$$\left(\frac{5}{2} + 1 \right) x = l \Rightarrow x = \frac{2l}{7}$$

$$\frac{1}{4} \cdot x = \frac{5}{16} (l - x)$$

$$\frac{5}{16} x = \frac{1}{4} (l - x)$$

$$\left(\frac{5}{4} + 1 \right) x = \dots$$



Analisi cinematica

G.L.: 6

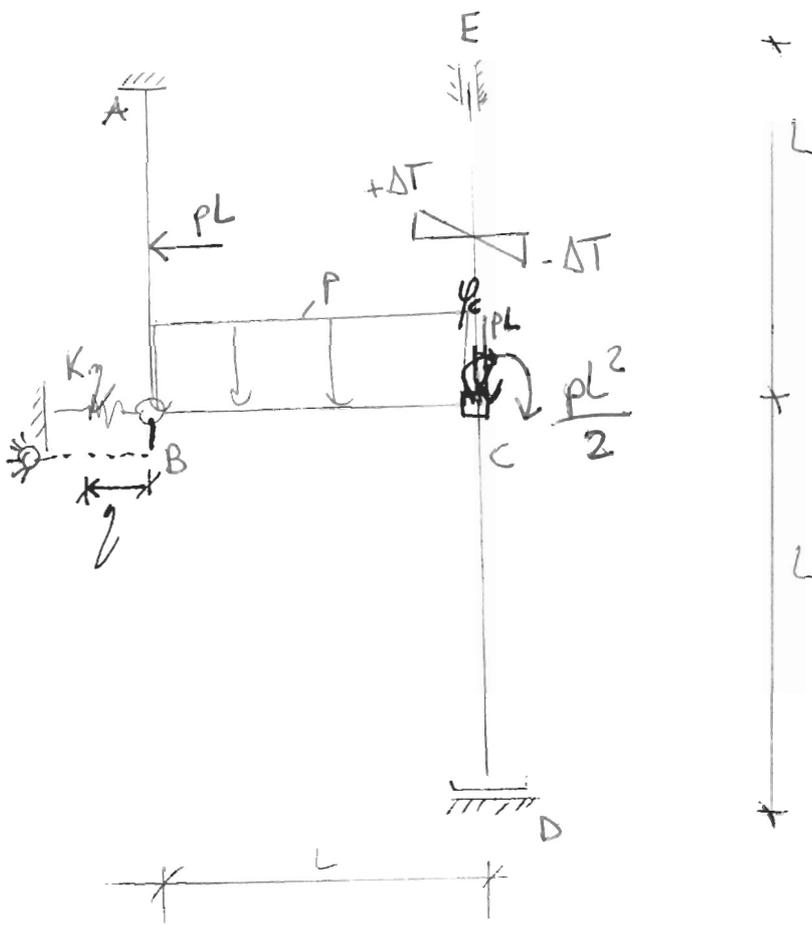
G.V.: 9

c'è un'appendice isostatica → (CF)

si può quindi studiare la seguente struttura

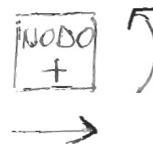
$$k_g = \frac{9ES}{L^3}$$

$$\frac{2\Delta T}{t} = \frac{1}{6} \frac{pL^2}{ES}$$

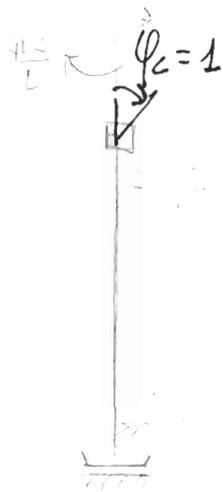
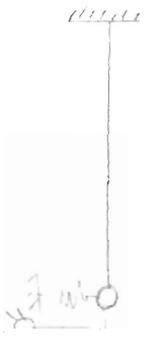


$$\begin{cases} \sum m_c = 0 \\ \sum F_{b, d} = 0 \end{cases}$$

$$\begin{cases} m_{cc} \cdot \varphi_c + m_{c\eta} \cdot \eta + m_{c0} = 0 \\ h_{\eta c} \cdot \varphi_c + h_{\eta\eta} \cdot \eta + h_{\eta c} = 0 \end{cases}$$



$\psi_c = 1 \quad \text{resto} = 0$

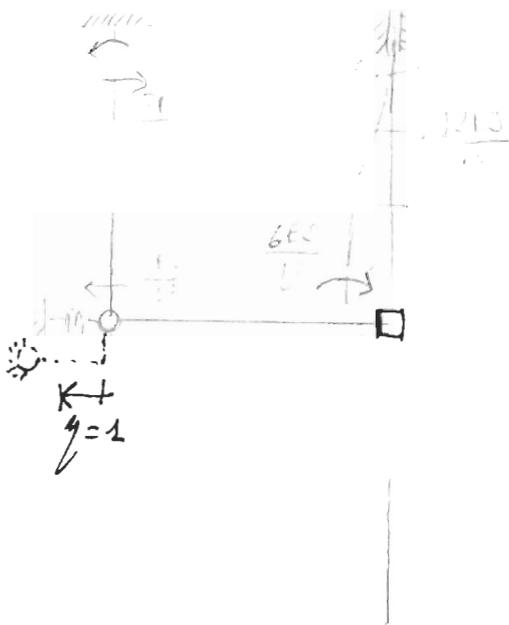


$\frac{4ES}{L}$
 $\frac{3ES}{L}$
 ES

$M_{cc} = -\frac{2ES}{L}$

$f_{nc} = \frac{6ES}{L^2}$

$\eta = 1 \quad \text{resto} = 0$

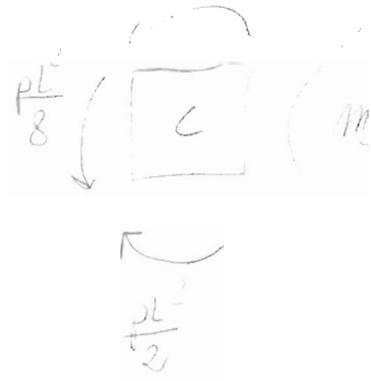
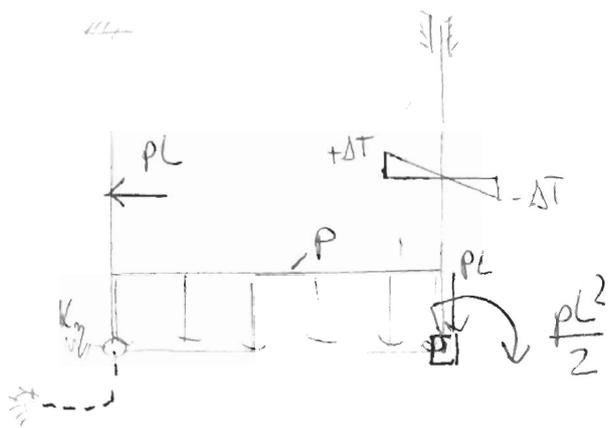


$\frac{6ES}{L^2}$

$M_c = \frac{6ES}{L}$

$= \frac{L}{13} + k_{\eta} \frac{L}{13}$

$$\underline{\varphi=0 \quad \eta=0 \quad \text{resto} \neq 0}$$



$$\frac{2ES\Delta T}{+} - \frac{3}{8} pL$$

$$\eta_0 = \frac{5}{16} pL$$

$$\begin{cases} \frac{8ES}{L} \varphi_c - \frac{6ES}{L^2} \eta + \frac{2ES\Delta T}{+} - \frac{3}{8} pL^2 = 0 \\ -\frac{6ES}{L^2} \varphi_c + \left(\frac{15ES}{L^2} + K_y \right) \eta - \frac{5pL}{16} = 0 \end{cases}$$

$$\begin{cases} \frac{8ES}{L} \varphi_c - \frac{6ES}{L^2} \eta = \frac{3}{8} pL^2 - \frac{2ES\Delta T}{+} \\ -\frac{6ES}{L^2} \varphi_c + \left(\frac{15ES}{L^2} + K_y \right) \eta = \frac{5pL}{16} \end{cases}$$

$$\begin{bmatrix} \frac{8ES}{L} & -\frac{6ES}{L^2} \\ -\frac{6ES}{L^2} & \left(\frac{15ES}{L^2} + K_y \right) \end{bmatrix} \begin{bmatrix} \varphi_c \\ \eta \end{bmatrix} = \begin{bmatrix} \frac{3}{8} pL^2 - \frac{2ES\Delta T}{+} \\ \frac{5pL}{16} \end{bmatrix}$$

considerando

$$k_{ng} = \frac{9EJ}{L^3} \quad \text{ed} \quad \frac{2\Delta T}{t} = \frac{1}{6} \frac{pL^2}{EJ}$$

s. ha $\varphi_c = \frac{4}{217} \frac{pL^3}{EJ}$

$$\gamma = \frac{11}{624} \frac{pL^4}{EJ}$$

Diagrammi delle azioni interne

EC

$$\uparrow \frac{2EJ}{L} \cdot \frac{4}{217} \frac{pL^3}{EJ} - \frac{6EJ}{L^2} \cdot \frac{11}{624} \frac{pL^4}{EJ} - \frac{1}{3} pL^2 = -0,402 pL^2$$

$$\leftarrow \frac{4EJ}{L} \cdot \frac{4}{217} \frac{pL^3}{EJ} - \frac{6EJ}{L^2} \cdot \frac{11}{624} \frac{pL^4}{EJ} + \frac{1}{3} pL^2 = 0,301 pL^2$$

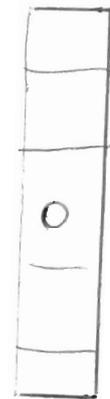
$$\leftarrow 0,402 pL^2$$

$$\rightarrow 0,10 pL$$

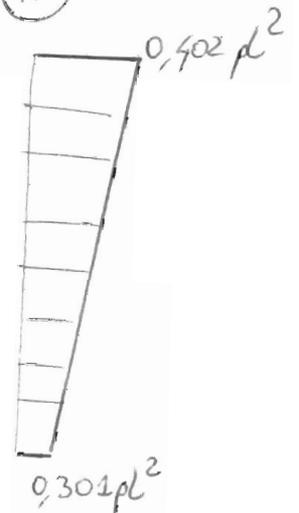
$$\leftarrow 0,10 pL$$

$$\leftarrow 0,301 pL^2$$

(V) $0,10 pL$



(M)



(V)



(M)



$$\frac{4}{217} pL^2$$

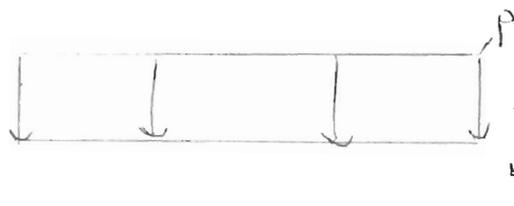
(5)

CD

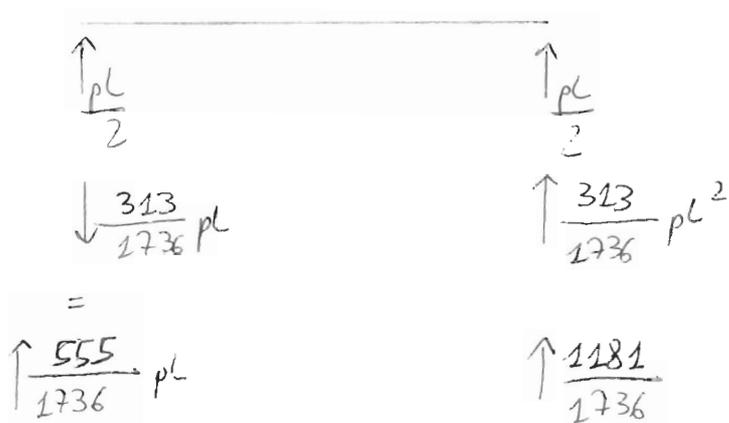
$$\uparrow \frac{EJ}{L} \cdot \frac{4}{217} \frac{pL^3}{EJ}$$

$$\leftarrow \frac{EJ}{L} \cdot \frac{4}{217} \frac{pL^3}{EJ}$$

BC

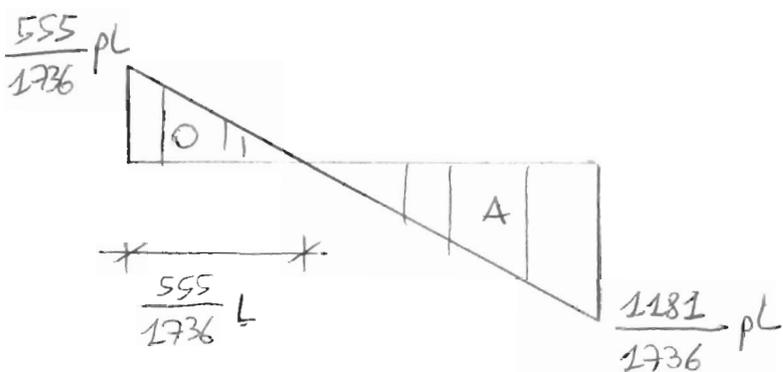


$$\frac{3EI}{L} \cdot \frac{4}{217} \frac{pL^3}{EI} + \frac{1}{8} pL^2 = \frac{313}{1736} pL^2 = 0,18 pL^2$$

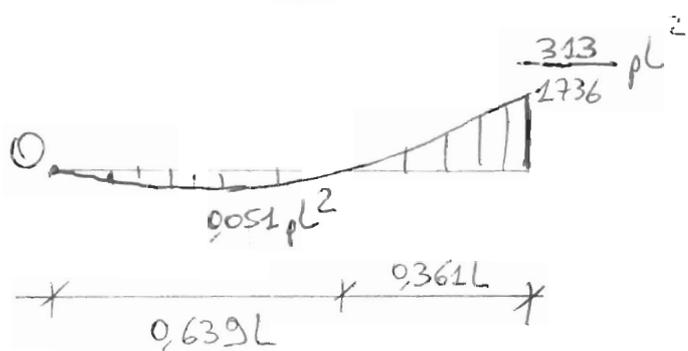


$$= \uparrow \frac{555}{1736} pL \quad \uparrow \frac{1181}{1736} pL^2$$

(V)



(II)



$$M = \frac{555}{1736} pL \cdot \frac{555}{1736} L - p \frac{\left(\frac{555}{1736}\right)^2 L^2}{2} = 0,051 pL^2$$

$$\rightarrow \frac{555}{1736} pL \cdot x - p \cdot x \cdot \frac{x}{2} = 0$$

$$-\frac{1}{2} p x^2 + \frac{555}{1736} pL x = 0 \quad \begin{cases} x=0 \\ x=0,639 \end{cases}$$

(6)

AB

$$\frac{3EJ}{L^2} \cdot \frac{11}{624} \frac{PL^4}{EJ} + \frac{3}{16} PL^2 = \frac{25}{104} PL^2$$



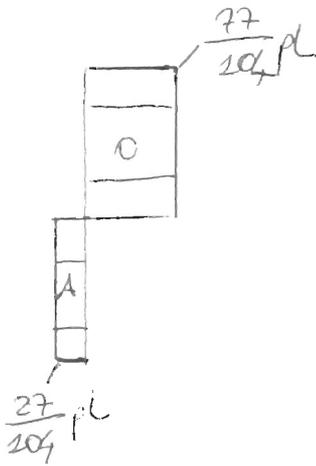
$$\frac{25}{104} PL^2$$

$$\frac{25}{104} PL \rightarrow \frac{PL}{2}$$

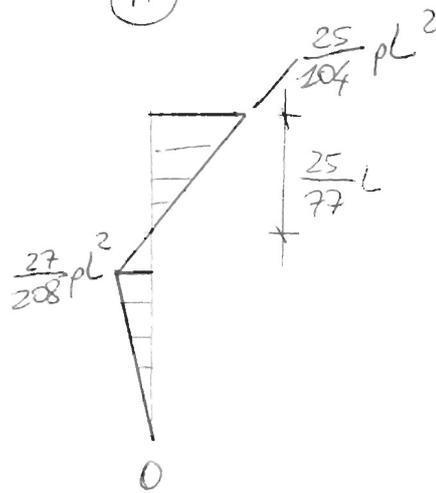
$$\frac{25}{104} PL \leftarrow \frac{PL}{2}$$

$$\frac{27}{104} PL \rightarrow$$

(V)



(M)

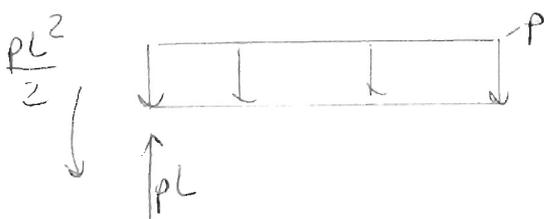


$$\frac{27}{104} PL \cdot \frac{L}{2} = \frac{27}{208} PL^2$$

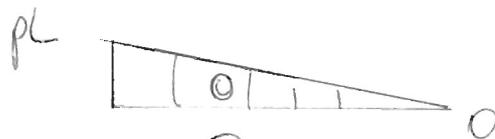
$$\frac{25}{104} PL^2 - \frac{27}{104} PL \cdot x = 0$$

$$x = \frac{25}{77} L$$

CF

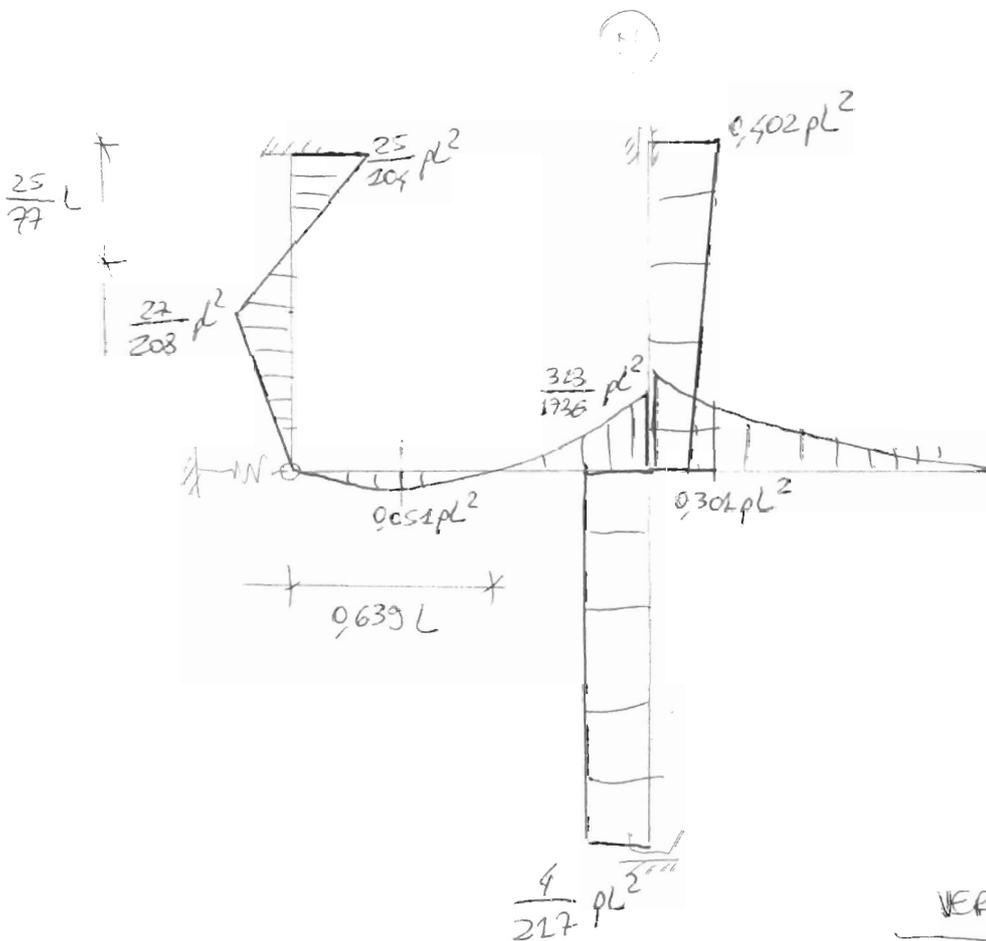
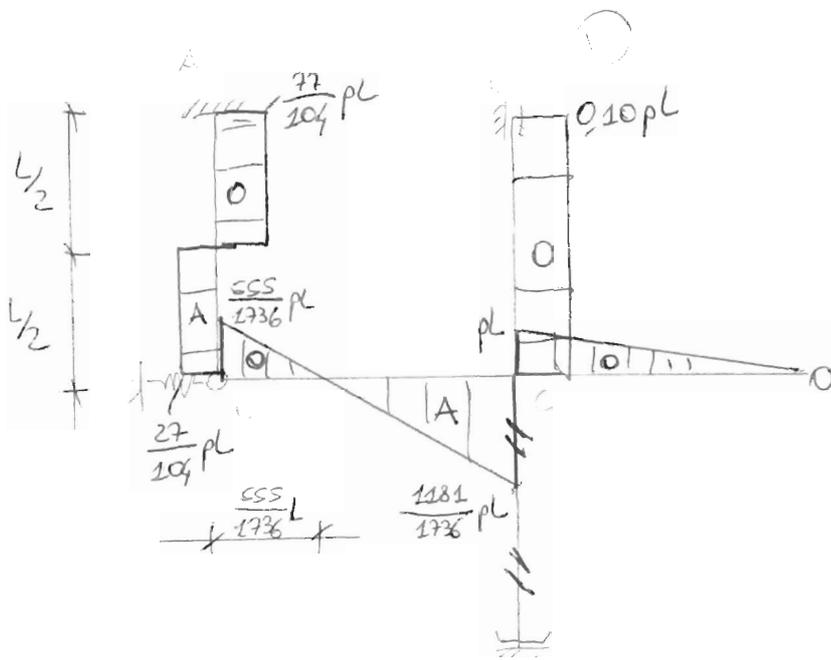


(V)

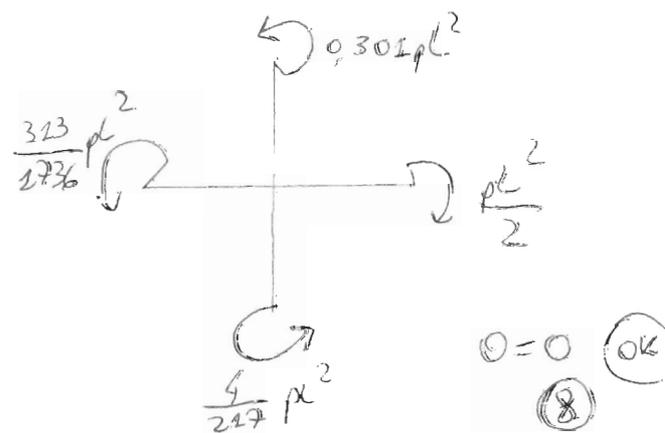


(M)

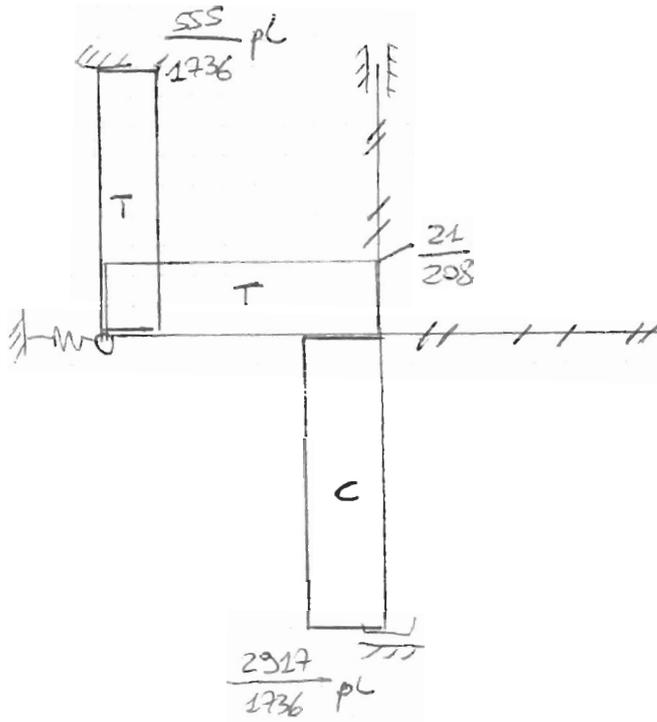




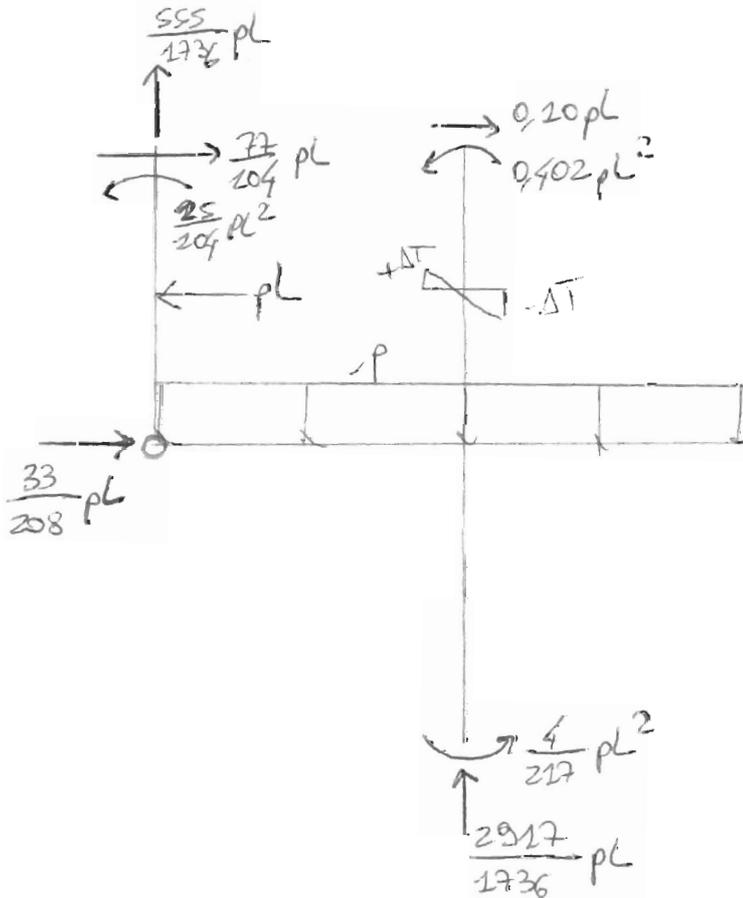
VERIFICA EQUILIBRIO NODI



(N)



VERIFICA EQUILIBRIO GLOBALE

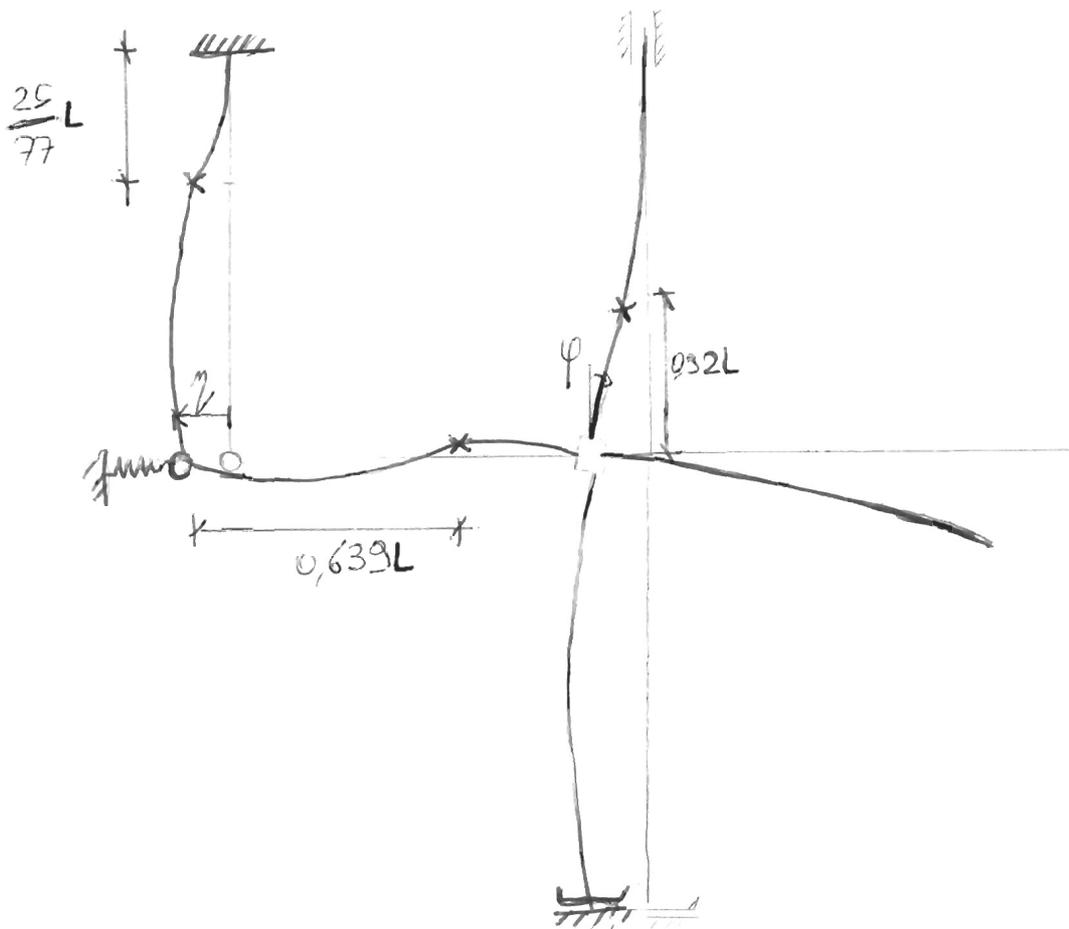


$$\rightarrow \frac{33}{208} pl + \frac{77}{204} pl + 0,10 pl - pl = 0 \quad 0=0 \quad \checkmark$$

$$\uparrow \frac{555}{1736} pl - 2pl + \frac{2917}{1736} pl = 0 \quad 0=0 \quad \checkmark$$

$$\curvearrowright \frac{4}{217} pl^2 - pl \frac{L}{2} + pl \frac{L}{2} - \frac{33}{208} pl \cdot L + 0,402 pl^2 - 0,10 pl \cdot 2L + pl \cdot \frac{3}{2} L - \frac{25}{204} pl^2 - \frac{77}{204} pl \cdot 2L - \frac{555}{1736} pl \cdot L = 0$$

DEFORMATA QUANTITATIVA



$$v''(x) = -\frac{M(x)}{EJ} + \frac{22\Delta T}{h}$$

$$M(x) = 0.301pL^2 + 0.10pL \cdot x$$

$$\frac{22\Delta T}{h} = 2 \cdot \frac{1}{6} \frac{pL^2}{EJ} = \frac{1}{3} \frac{pL^2}{EJ}$$

$$v''(x) = -\frac{0.301pL^2}{EJ} - \frac{0.10pLx}{EJ} + \frac{1}{3} \frac{pL^2}{EJ} = -\frac{0.10pLx}{EJ} + 0.032 \frac{pL^2}{EJ}$$

$$v(x) = -0.10 \frac{pL}{EJ} x + 0.032 \frac{pL^2}{EJ}$$

$$v'' > 0 \quad -0.10 \frac{pL}{EJ} x > -0.032 \frac{pL^2}{EJ} \rightarrow 0.10x < 0.032L \quad x < 0.32L$$

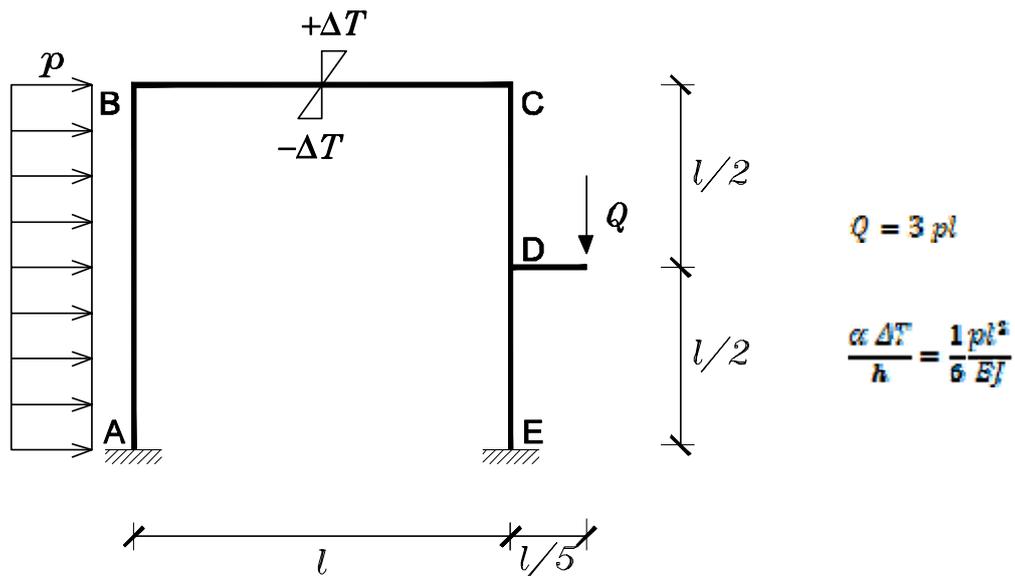
$$v'' < 0 \quad x > 0.32L \rightarrow$$

TECNICA DELLE COSTRUZIONI

ESAME DEL 24 GENNAIO 2011

DOCENTE: ING. FAUSTO MINELLI

ESERCIZIO

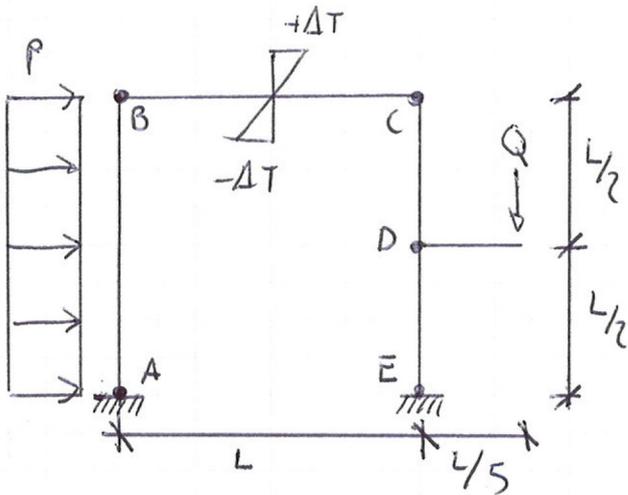


Dato il telaio in figura si richiede il tracciamento:

1. Diagramma del momento flettente (con il valore e la posizione dei massimi);
2. Diagramma del taglio;
3. Diagramma dell'azione assiale;
4. Deformata qualitativa con posizione dei flessi.

PORTALE IPERSTATICO

①



$$\begin{cases} \frac{\alpha \Delta T}{t} = \frac{1}{6} \frac{PL^2}{EI} \\ Q = 3PL \end{cases}$$

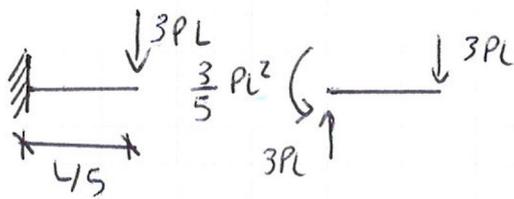
① ANALISI CINEMATICA

$G_{di L} = 3$

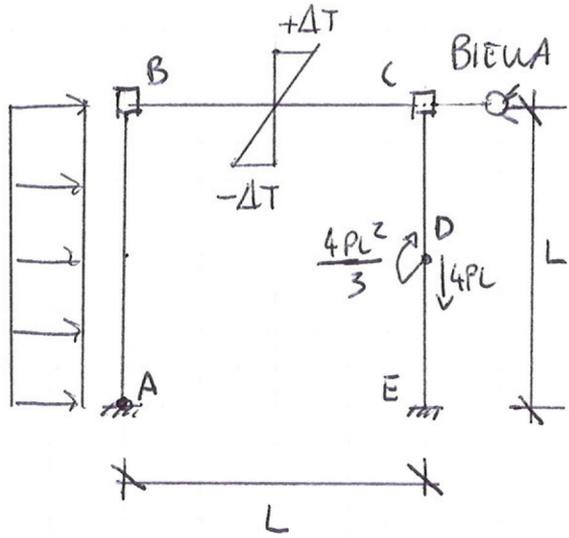
$G_{di V} = 6$

→ STRUTTURA 3 VOLTE IPERSTATICA A
NODI SPOSTABILI

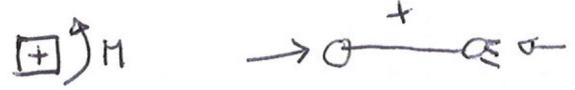
APPENDICE ISOSTATICA: MENSOLA IN D



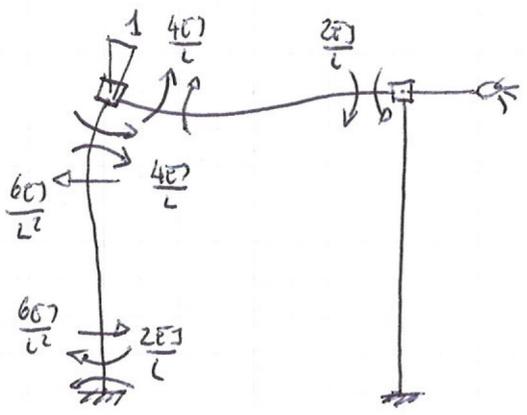
2



$$\begin{cases} m_{BB} \varphi_B + m_{BC} \varphi_C + m_{B\eta} \eta + m_{B0} = 0 \\ m_{CB} \varphi_B + m_{CC} \varphi_C + m_{C\eta} \eta + m_{C0} = 0 \\ h_{\eta B} \varphi_B + h_{\eta C} \varphi_C + h_{\eta\eta} \eta + h_{\eta 0} = 0 \end{cases}$$



* $\varphi_B = 1$

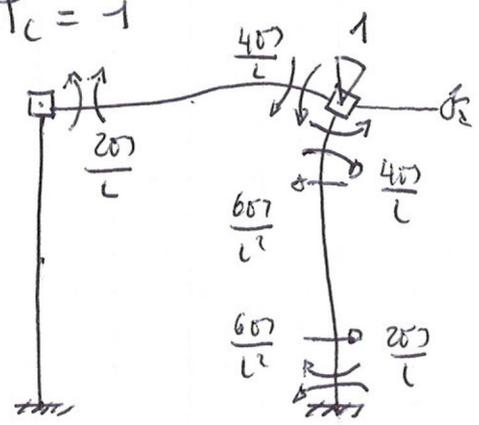


$$m_{BB} = + \frac{4EJ}{L} + \frac{4EJ}{L} = \frac{8EJ}{L}$$

$$m_{CB} = + \frac{2EJ}{L}$$

$$h_{\eta B} = + \frac{6EJ}{L^2}$$

* $P_c = 1$

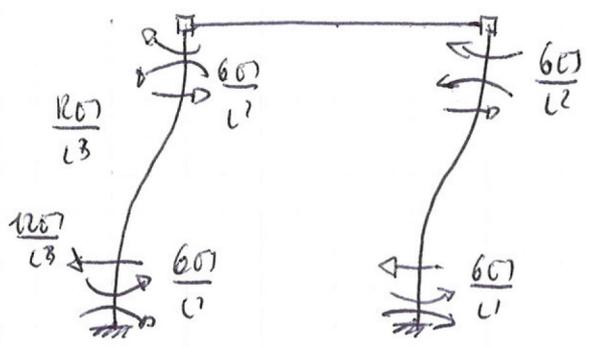


$$m_{BC} = + \frac{2EJ}{L}$$

$$m_{CC} = + \frac{4EJ}{L} + \frac{4EJ}{L} = \frac{8EJ}{L}$$

$$h_{\eta C} = + \frac{6EJ}{L^2}$$

* $\eta = 1$



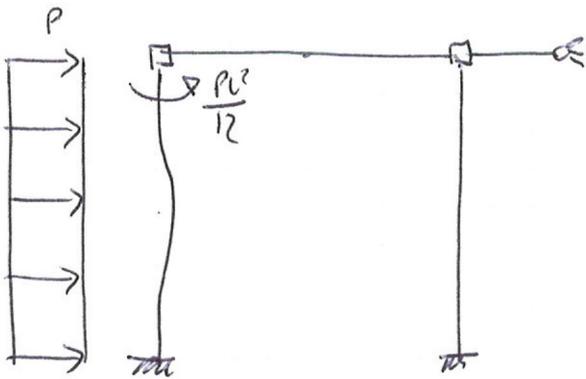
$$m_{BH} = - \frac{6EJ}{L^2}$$

$$m_{CH} = - \frac{600}{L^2}$$

$$h_{\eta\eta} = - \frac{24EJ}{L^3}$$

* $P \neq 0$

4

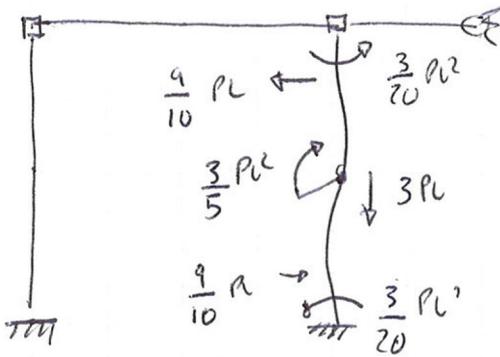


$$M_{BC, I} = \frac{PL^2}{12}$$

$$m_{CO, I} = 0$$

$$h_{CO, I} = \frac{PL}{2}$$

* $Q \neq 0$



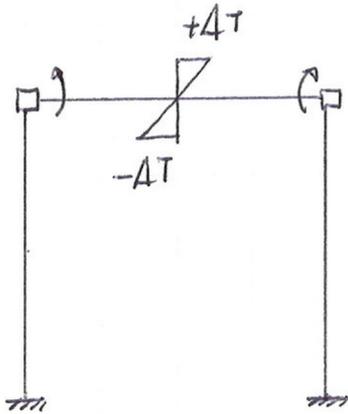
$$M_{BC, II} = 0$$

$$m_{CO, II} = \frac{3}{20} PL^2$$

$$h_{CO, II} = \frac{9}{10} PL$$

* $\Delta T \neq 0$

5



$$m_{B_0, III} = \frac{2EJ \alpha \Delta T}{t} = \frac{1}{3} PL^2$$

$$m_{C_0, III} = \frac{2EJ \alpha \Delta T}{t} = -\frac{1}{3} PL^2$$

$$h_{q_0, III} = \phi$$

* SISTEMA RISOLVENTE

$$1) \left(8 \frac{EJ}{L} \right) \varphi_B + \left(2 \frac{EJ}{L} \right) \varphi_C + \left(-\frac{6EJ}{L^2} \right) \eta = -\frac{5}{12} PL^2$$

$$2) \left(2 \frac{EJ}{L} \right) \varphi_B + \left(8 \frac{EJ}{L} \right) \varphi_C + \left(-\frac{6EJ}{L^2} \right) \eta = +\frac{11}{60} PL^2$$

$$3) \left(6 \frac{EJ}{L^2} \right) \varphi_B + \left(6 \frac{EJ}{L^2} \right) \varphi_C + \left(-\frac{24EJ}{L^3} \right) \eta = -\frac{7}{5} PL$$

$$\boxed{\varphi_B = -\frac{1}{60} \frac{PL^3}{EJ} \quad \varphi_C = +\frac{1}{12} \frac{PL^3}{EJ} \quad \eta = +\frac{3}{40} \frac{PL^4}{EJ}}$$

$$-1 \begin{cases} 8 \frac{EJ}{L} \varphi_B + 2 \frac{EJ}{L} \varphi_C - \frac{6EJ}{L^2} \eta = -\frac{5}{12} PL^2 \\ 2 \frac{EJ}{L} \varphi_B + 8 \frac{EJ}{L} \varphi_C - \frac{6EJ}{L^2} \eta = +\frac{11}{60} PL^2 \\ 6 \frac{EJ}{L^2} \varphi_B + \frac{6EJ}{L^2} \varphi_C - 24 \frac{EJ}{L^3} \eta = -\frac{7}{5} PL^2 \end{cases} \quad (6)$$

$$-4 \begin{cases} 8 \varphi_B + 2 \varphi_C - 6 \frac{\eta}{L} = -\frac{5}{12} \frac{PL^3}{EJ} \\ 2 \varphi_B + 8 \varphi_C - 6 \frac{\eta}{L} = +\frac{11}{60} \frac{PL^3}{EJ} \\ 6 \varphi_B + 6 \varphi_C - 24 \frac{\eta}{L} = -\frac{7}{5} \frac{PL^3}{EJ} \end{cases}$$

$$3 \begin{cases} 6 \varphi_B - 6 \varphi_C = \left(-\frac{5}{12} + \frac{11}{60}\right) \frac{PL^3}{EJ} = \left(-\frac{25+11}{60}\right) \frac{PL^3}{EJ} = -\frac{3}{5} \frac{PL^3}{EJ} \\ -2 \varphi_B - 10 \varphi_C = \left(-\frac{11}{15} - \frac{7}{5}\right) \frac{PL^3}{EJ} = -\left(\frac{11+21}{15}\right) \frac{PL^3}{EJ} = -\frac{32}{15} \frac{PL^3}{EJ} \\ -84 \varphi_C = \left(-\frac{3}{5} - \frac{32}{5}\right) \frac{PL^3}{EJ} \end{cases}$$

$$84 \varphi_C = \frac{37}{5} \frac{PL^3}{EJ}$$

$$\boxed{\varphi_C = \frac{1}{12} \frac{PL^3}{EJ}}$$

$$6 \varphi_B - 6 \left(\frac{1}{12} \frac{PL^3}{EI} \right) = -\frac{3}{5} \frac{PL^3}{EI} \quad (7)$$

$$6 \varphi_B = \left(\frac{1}{2} - \frac{3}{5} \right) \frac{PL^3}{EI} \quad 6 \varphi_B = \frac{5-6}{10} \frac{PL^3}{EI}$$

$$\boxed{\varphi_B = -\frac{1}{60} \frac{PL^3}{EI}}$$

$$6 \cdot \left(-\frac{1}{60} \frac{PL^3}{EI} \right) + 6 \left(\frac{1}{12} \frac{PL^3}{EI} \right) - 24 \left(\frac{\eta}{L} \right) = -\frac{7}{5} \frac{PL^3}{EI}$$

$$-\frac{1}{10} \frac{PL^4}{EI} + \frac{1}{2} \frac{PL^4}{EI} - 24 \eta = -\frac{7}{5} \frac{PL^4}{EI}$$

$$+ 24 \eta = -\left(\frac{1}{10} - \frac{1}{2} - \frac{7}{5} \right) \frac{PL^4}{EI}$$

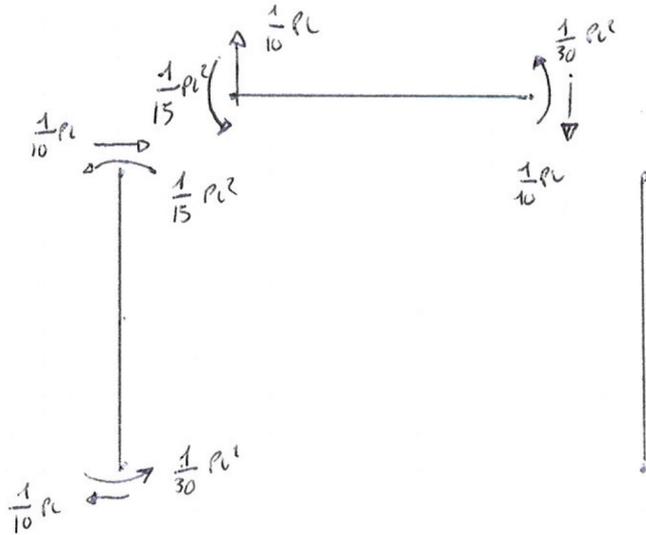
$$\eta = \frac{1}{24} \cdot \left[-\left(\frac{1-5-14}{10} \right) \frac{PL^4}{EI} \right] = \frac{1}{24} \cdot \frac{18}{10} \frac{PL^4}{EI}$$

$$\boxed{\eta = \frac{3}{40} \frac{PL^4}{EI}}$$

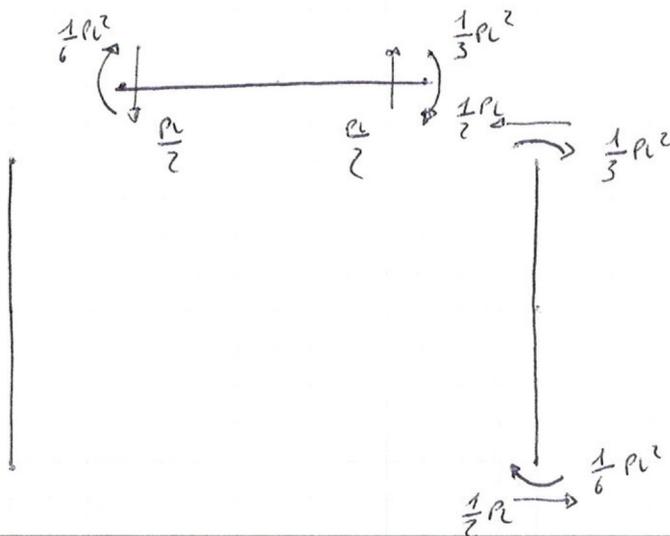
APPLICANDO IL PRINCIPIO DI SOVR. EFFETTI

$$\textcircled{*} f_B = -\frac{1}{60} \frac{PL^3}{EI}$$

⑧

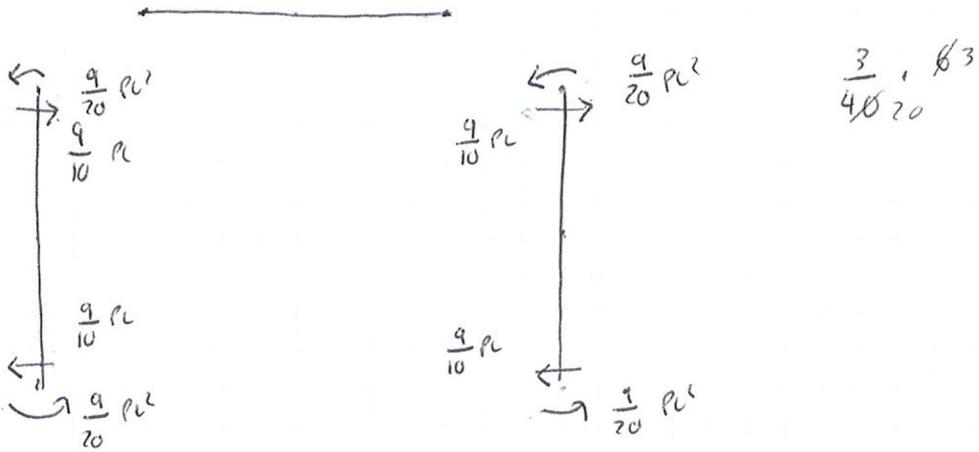


$$\textcircled{*} f_C = \frac{1}{12} \frac{PL^3}{EI}$$

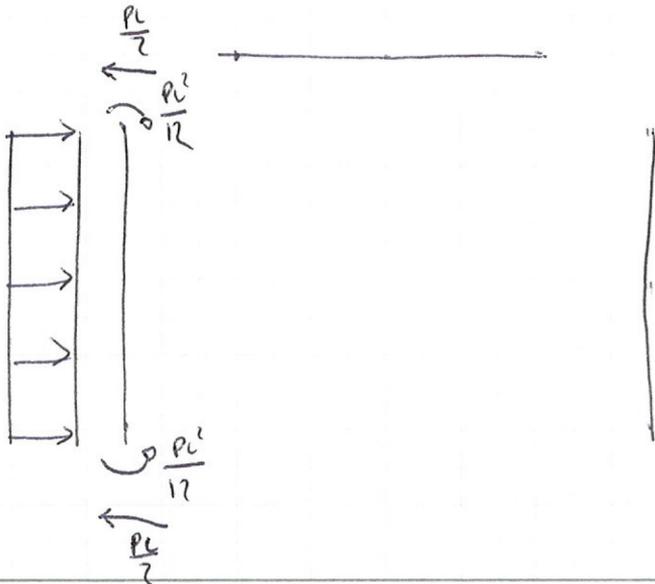


* $\eta = + \frac{3}{40} \frac{PL^4}{EI}$

(9)

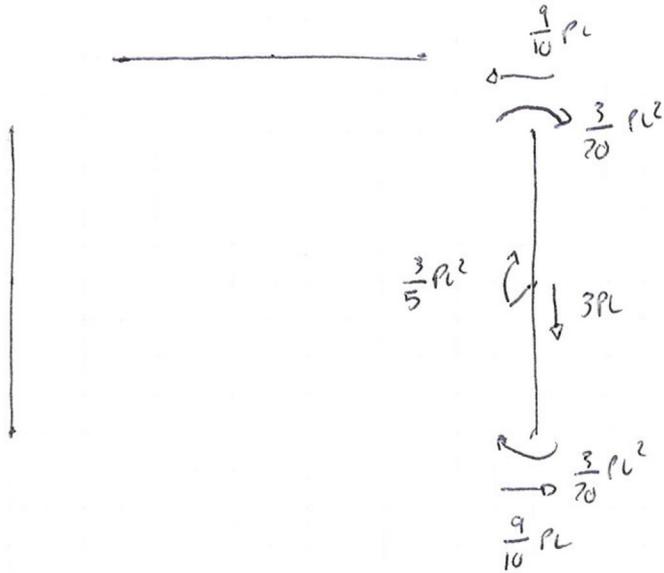


* $P \neq 0$

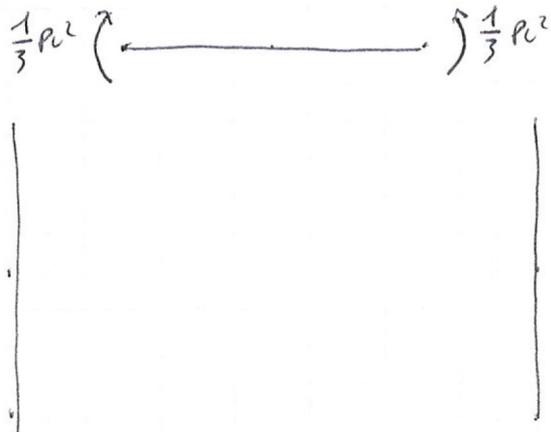


(*) $Q \neq 0$

(10)

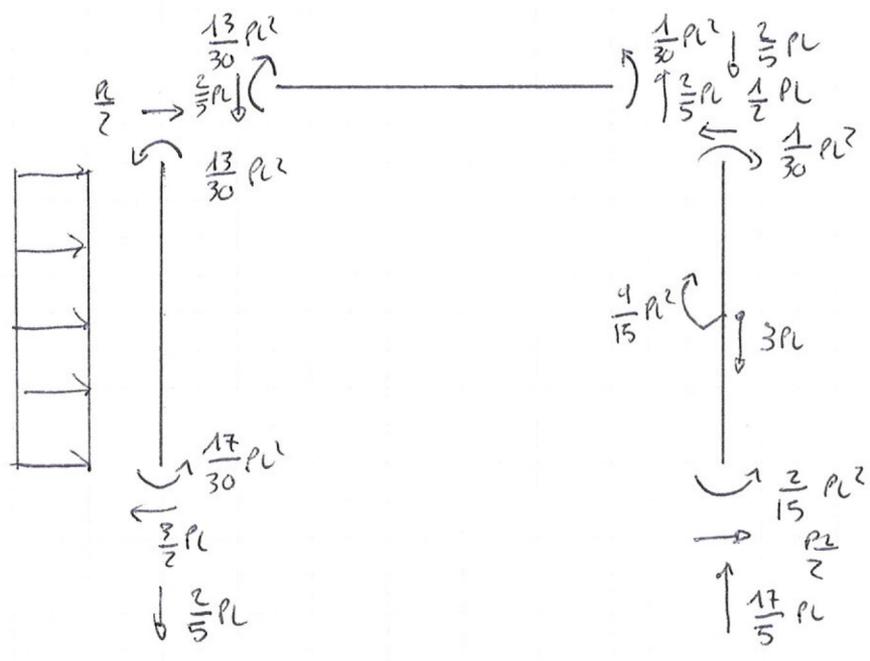


(*) $\Delta T \neq 0$



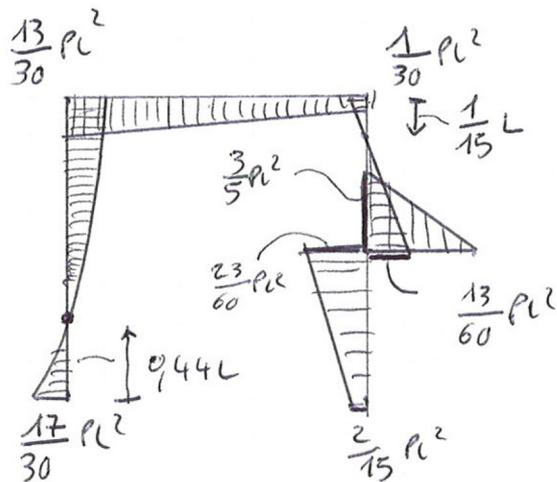
TO TAKE

11



(M)

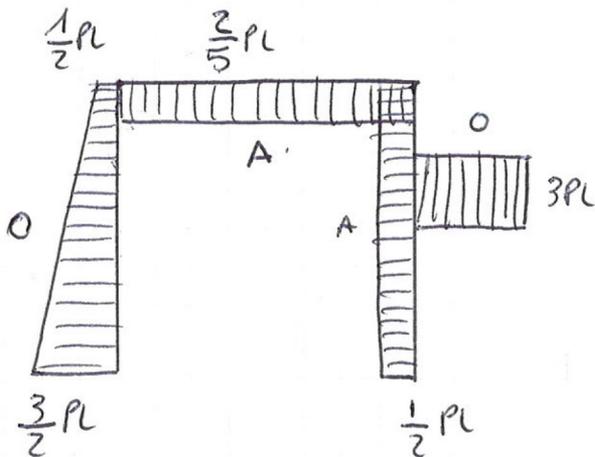
(12)



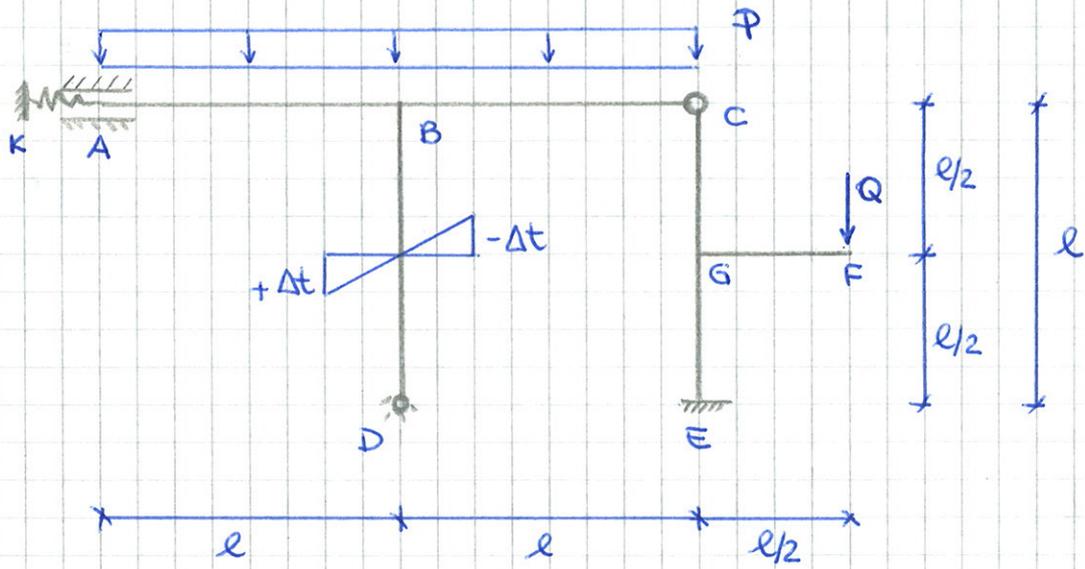
$$M_{AB}(x) = \frac{17}{30} PL^2 - \frac{3}{2} PLx - \frac{Px^2}{2} = 0 \rightarrow x^2 - 3xL + \frac{17}{15} L^2 = 0$$

$$x_{1,2} = \frac{+3L \pm \sqrt{9L^2 - \frac{68}{15} L^2}}{2}$$

(V)



$$= \begin{cases} 3.17L \\ 9.44L \end{cases}$$



$$K = \frac{3EJ}{l^3}$$

$$EJ = \text{cost.}$$

$$Q = 10pe$$

$$EA \rightarrow \infty$$

$$\frac{\alpha \Delta t}{t} = \frac{13}{24} \frac{pe^2}{EJ}$$

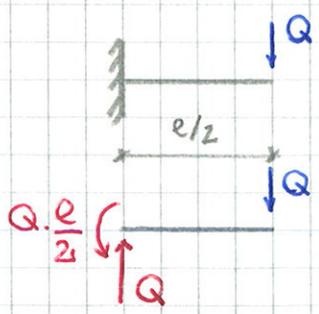
. Analisi cinematica

$$\text{GDL } 3+3 = 6$$

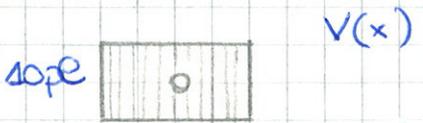
$$\text{GDV } 2+4+2+2+2 = 12$$

Struttura a nodi spostabili.

. Appendice isostatica - asta FG

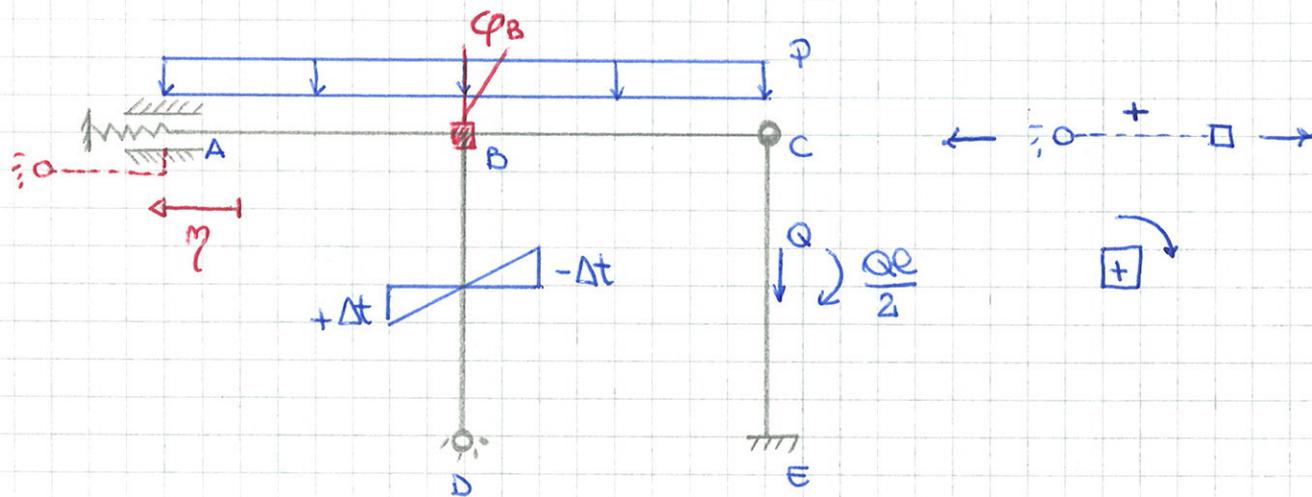


$$Q = 10pe$$



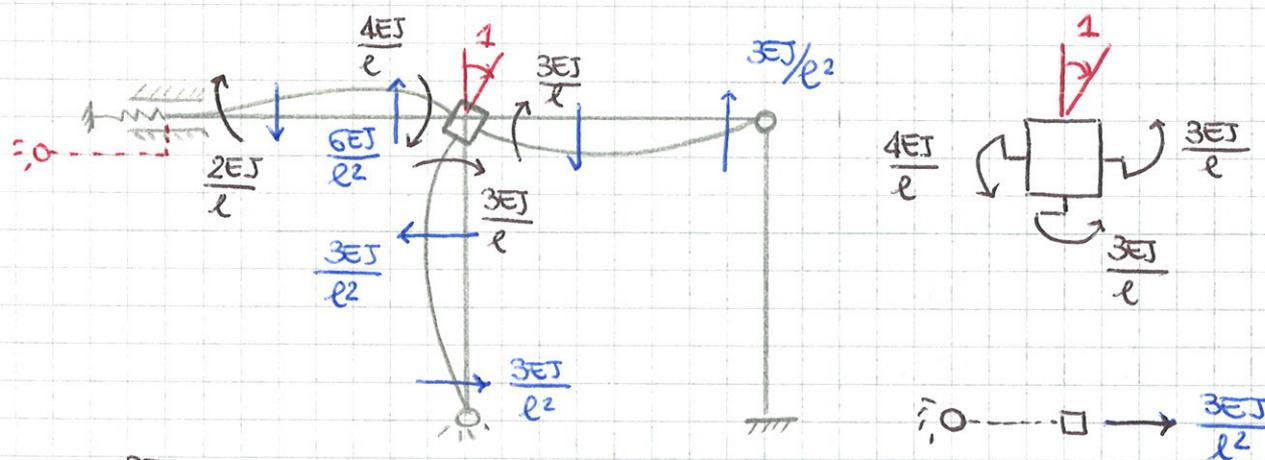
• Sistema risolvibile / convenzione di segno

②



$$\begin{cases} \sum M_B = \phi \\ \sum F_{\text{betta}} = \phi \end{cases} \quad \begin{cases} m_{BB} \cdot \phi_B + m_{B\eta} \cdot \eta + m_{B0} = \phi \\ r_{AB} \cdot \phi_B + r_{A\eta} \cdot \eta + r_{A0} = \phi \end{cases}$$

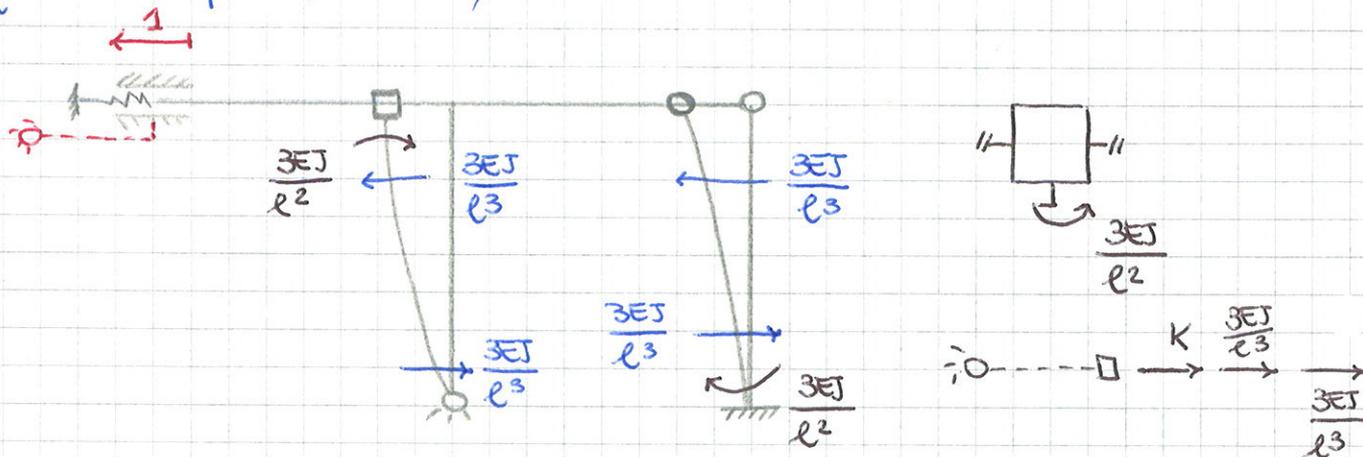
① $\phi_B = 1 \quad \eta = p = \Delta t = \phi$



$$M_{BB} = -\frac{3EJ}{l} - \frac{3EJ}{l} - \frac{4EJ}{l} = -\frac{10EJ}{l}$$

$$r_{AB} = \frac{3EJ}{l^2}$$

② $\eta = 1 \quad \phi_B = p = \Delta t = \phi$



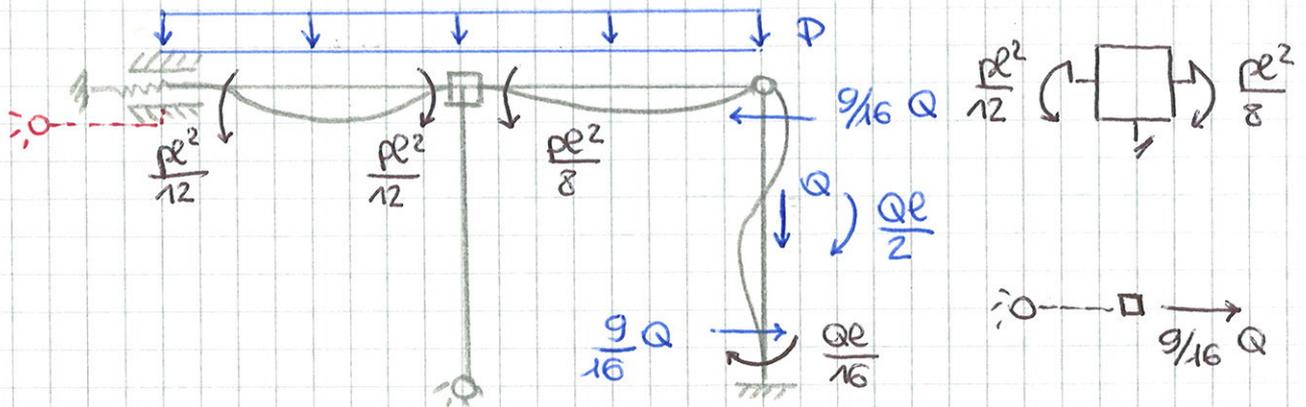
$$M_{B\eta} = -\frac{3EJ}{l^2}$$

③

$$R_{A\eta} = K + \frac{3EJ}{l^3} + \frac{3EJ}{l^3} = \frac{9EJ}{l^3}$$

③ $\varphi \neq 0$

$$\varphi_B = \eta = \Delta t = 0$$

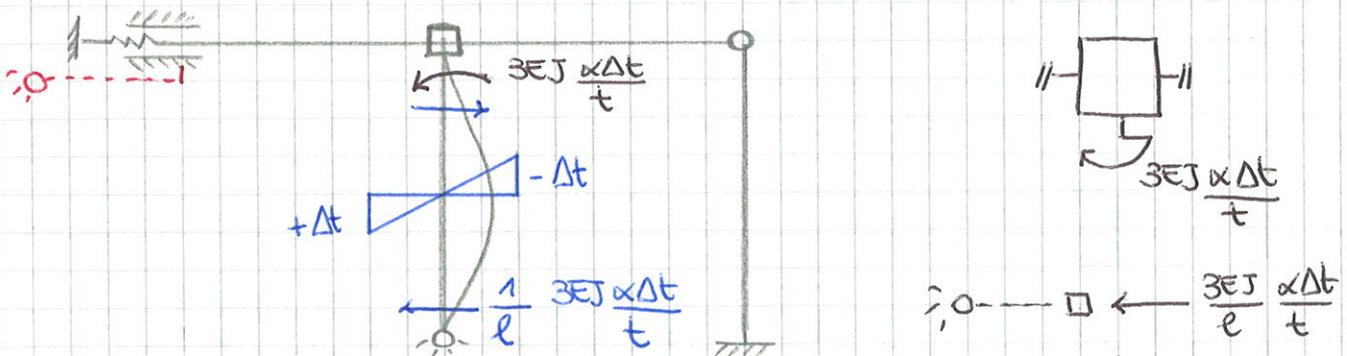


$$M_{B0} = \frac{pe^2}{8} - \frac{pe^2}{12} = \frac{pe^2}{24}$$

$$R_{A0} = \frac{9}{16} Q = \frac{9}{16} (10pe) = \frac{45}{8} pe$$

④ $\Delta t \neq 0$

$$\varphi_B = \eta = p = 0$$



$$M_{B0} = 3EJ \frac{\alpha \Delta t}{t} = 3EJ \cdot \frac{13}{24} \frac{pe^2}{EJ} = \frac{13}{8} pe^2$$

$$R_{A0} = -\frac{3EJ}{l} \frac{\alpha \Delta t}{t} = -\frac{3EJ}{l} \cdot \frac{13}{24} \frac{pe^2}{EJ} = -\frac{13}{8} pe$$

Sistema risolvibile

④

$$\begin{cases} \varphi_B \left(-\frac{10EJ}{l} \right) - \frac{3EJ}{l^2} \eta + \frac{1}{24} \rho l^2 + \frac{13}{8} \rho l^2 = 0 & \left(\cdot \frac{3}{l} \right) \\ \varphi_B \cdot \frac{3EJ}{l^2} + \frac{9EJ}{l^3} \eta + \frac{45}{8} \rho l - \frac{13}{8} \rho l = 0 \end{cases}$$

$$\begin{cases} -30 \frac{EJ}{l^2} \varphi_B - \frac{9EJ}{l^3} \eta + \frac{1}{8} \rho l + \frac{39}{8} \rho l = 0 \\ \frac{3EJ}{l^2} \varphi_B + \frac{9EJ}{l^3} \eta + \frac{45}{8} \rho l - \frac{13}{8} \rho l = 0 \end{cases}$$

$$-27 \frac{EJ}{l^2} \varphi_B + \frac{72}{8} \rho l = 0$$

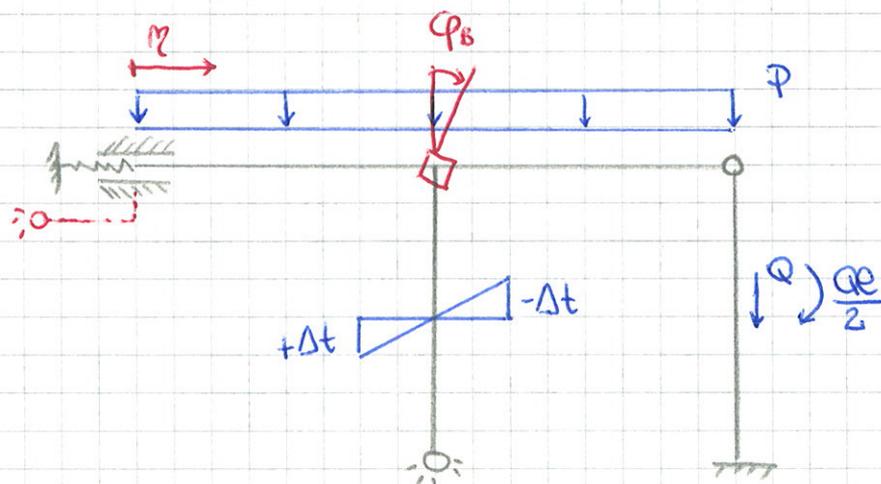
$$\varphi_B = \frac{72}{8} \frac{\rho l^3}{EJ} \cdot \frac{1}{27} = \frac{1}{3} \frac{\rho l^3}{EJ}$$

$$-10 \frac{EJ}{l} \left(\frac{1}{3} \frac{\rho l^3}{EJ} \right) - \frac{3EJ}{l^2} \eta + \frac{40}{24} \rho l^2 = 0$$

$$\frac{3EJ}{l^2} \eta = -\frac{40}{24} \rho l^2$$

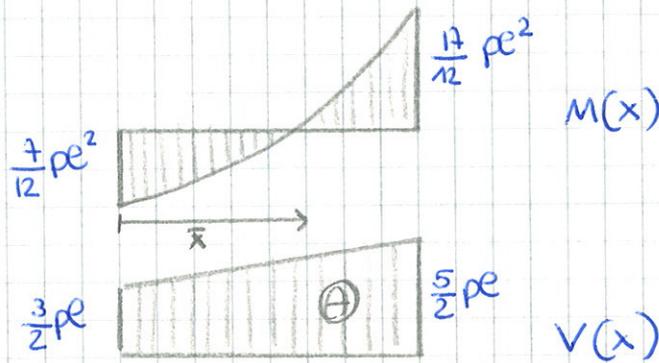
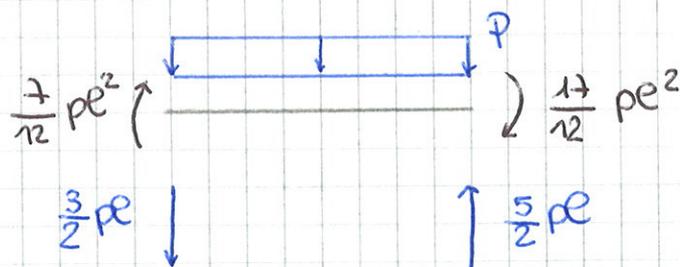
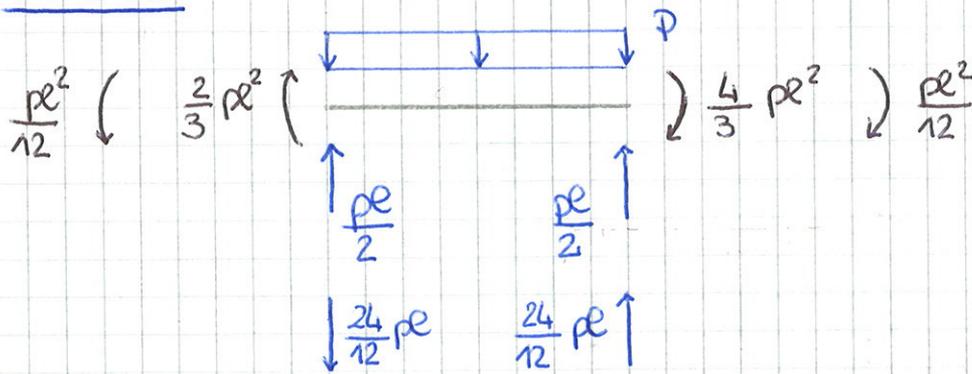
$$\eta = -\frac{5}{3} \rho l^2 \cdot \frac{1}{3} \frac{l^2}{EJ} = -\frac{5}{9} \frac{\rho l^4}{EJ}$$

$$\begin{cases} \varphi_B = \frac{1}{3} \frac{\rho l^3}{EJ} \\ \eta = -\frac{5}{9} \frac{\rho l^4}{EJ} \end{cases}$$

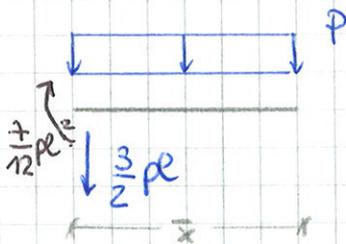


Determinazione delle azioni interne

Asta AB



$\bar{x} = ?$



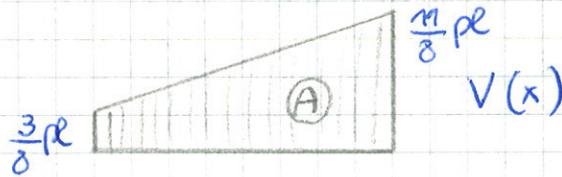
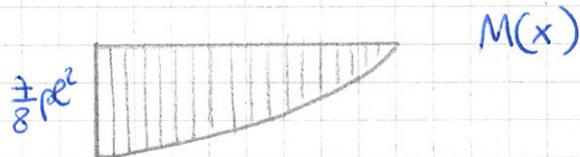
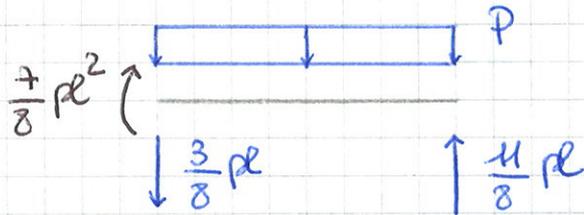
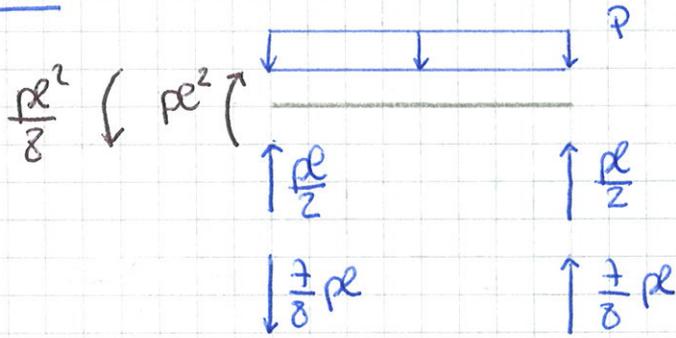
$$\frac{7}{12} pe^2 - \frac{3}{2} pe \bar{x} - \frac{p \bar{x}^2}{2} = 0$$

$$x_{1,2} = \frac{3/2 \pm \sqrt{9/4 + 7/6}}{(-1)}$$

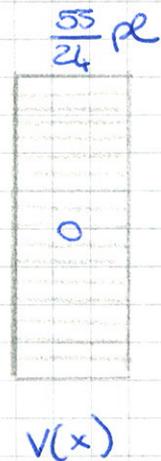
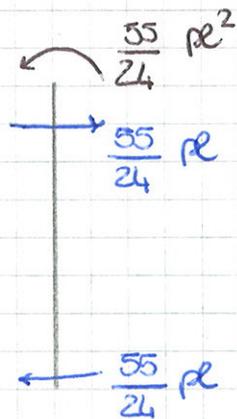
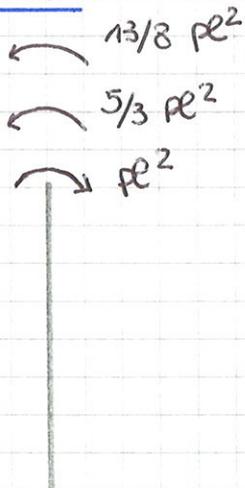
$$x_{1,2} = \begin{cases} -3,34 & \text{non accettabile} \\ 0,348 \end{cases}$$

$$\bar{x} = 0,348 \text{ l}$$

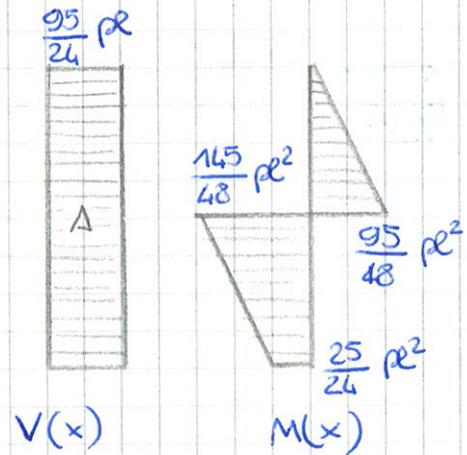
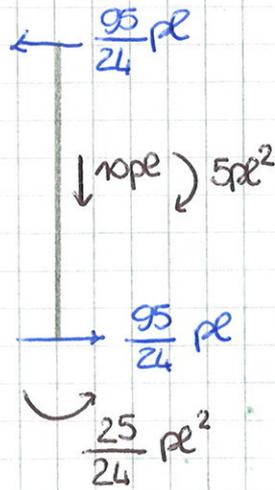
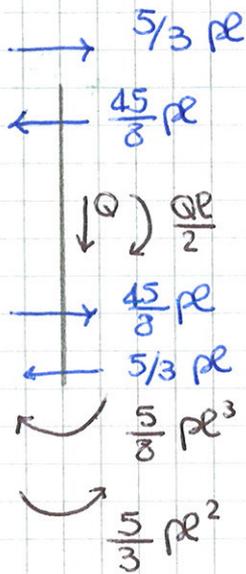
ASTA BC



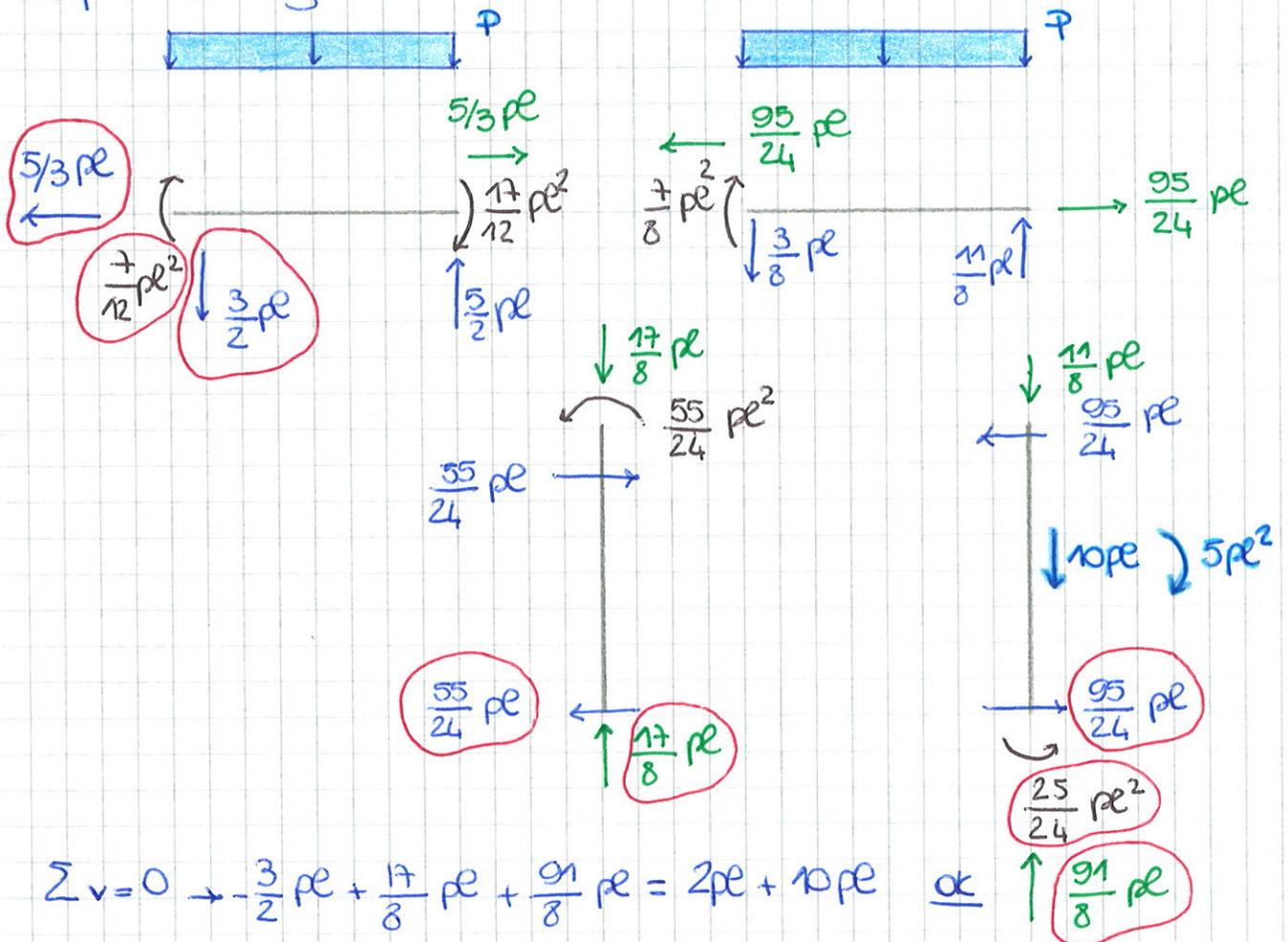
ASTA BD



ASTA CE



• Equilibrio globale

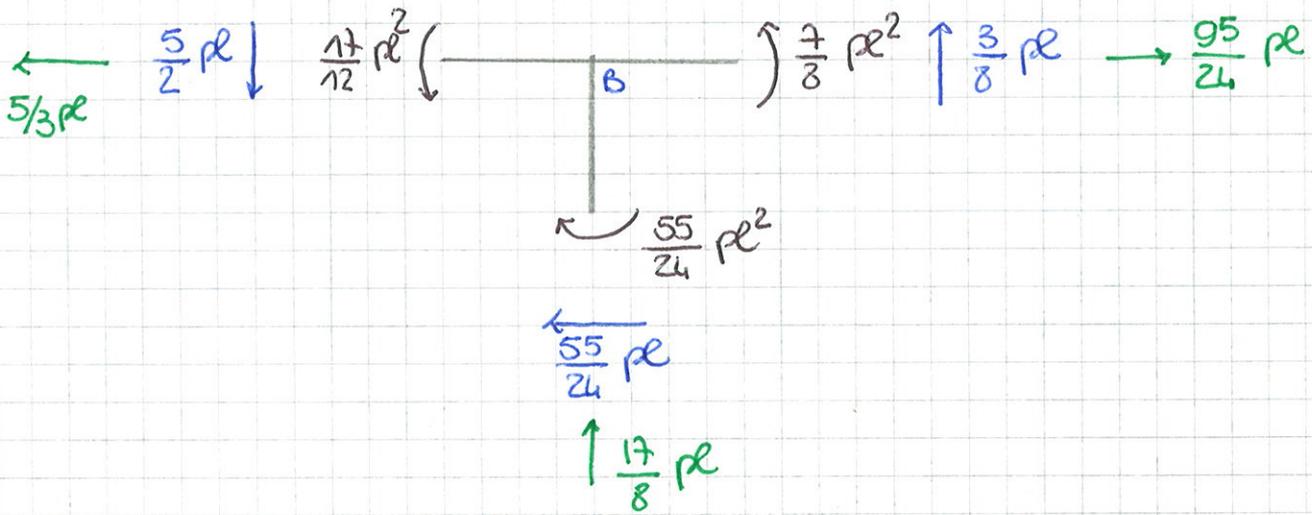


$$\sum v = 0 \rightarrow -\frac{3}{2}pl + \frac{17}{8}pl + \frac{91}{8}pl = 2pl + 10pe \quad \underline{\alpha}$$

$$\sum H = 0 \rightarrow -\frac{5}{3}pl - \frac{55}{24}pl + \frac{95}{24}pl = 0 \quad \underline{\alpha}$$

$$\sum M(b) = 0 \rightarrow 10pl \cdot e + 5pl^2 - \frac{25}{24}pl^2 - \frac{91}{8}pl^2 - \frac{5}{3}pl^2 + \frac{7}{12}pl^2 - \frac{3}{2}pl^2 = 0 \quad \underline{\alpha}$$

Equilibrio al nodo B

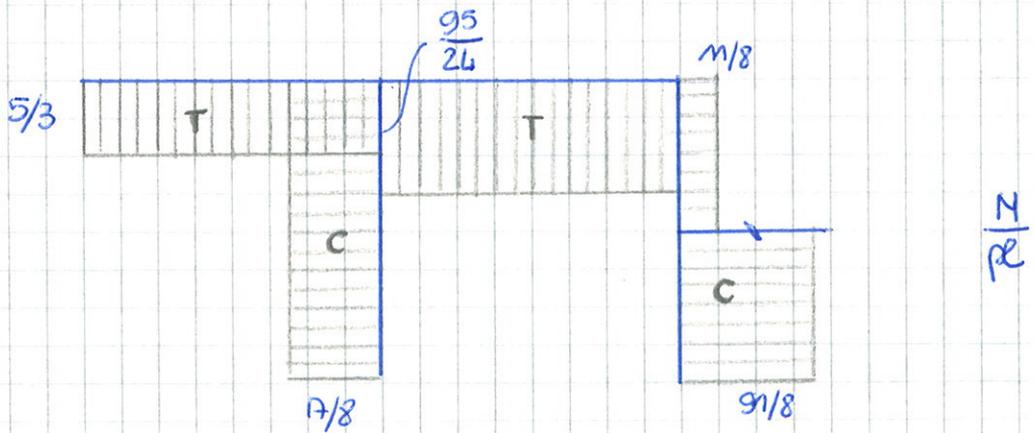
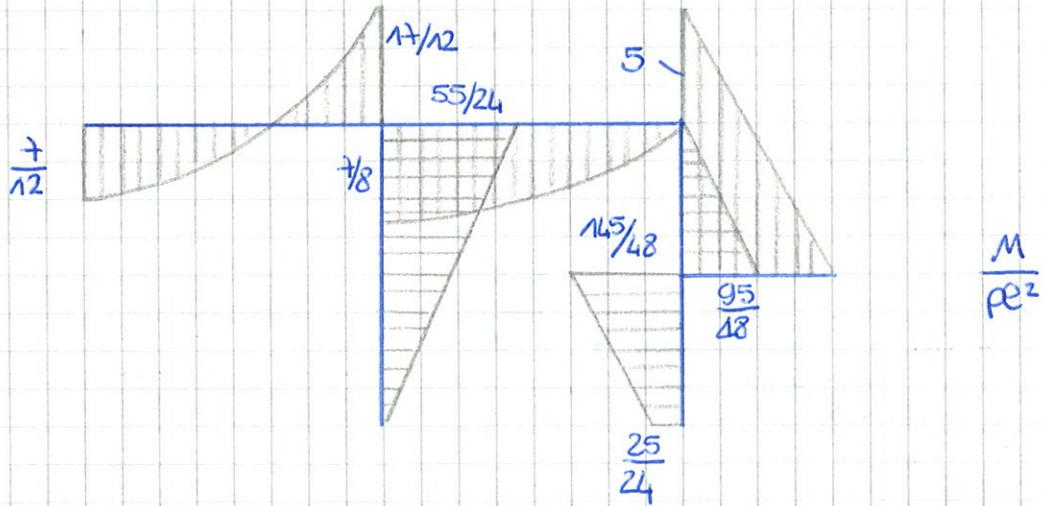
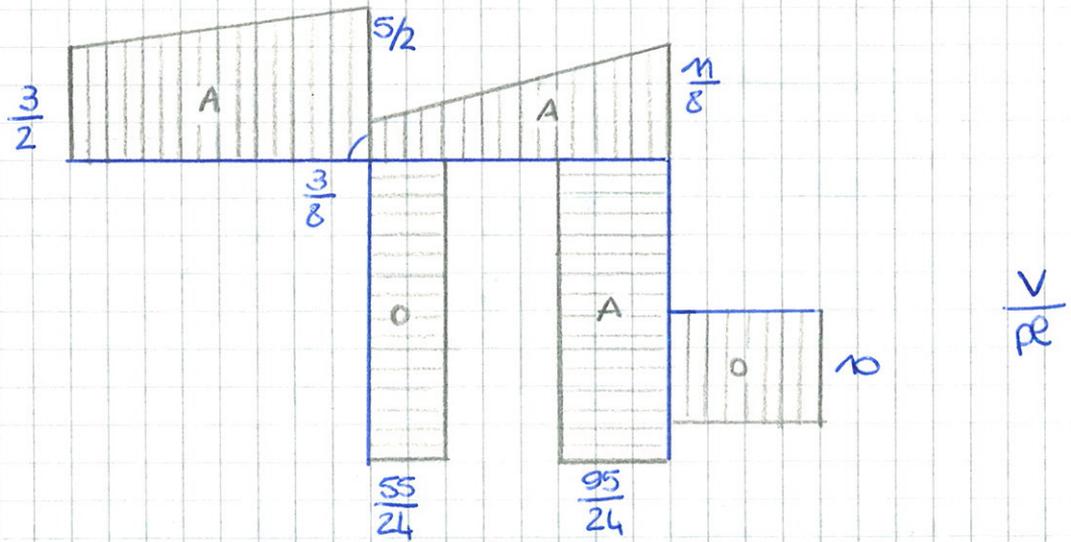


$$\left\{ \begin{array}{l} \sum H = 0 \rightarrow -\frac{5}{3} \rho + \frac{95}{24} \rho - \frac{55}{24} \rho = \frac{-40 + 95 - 55}{24} \rho = \phi \end{array} \right.$$

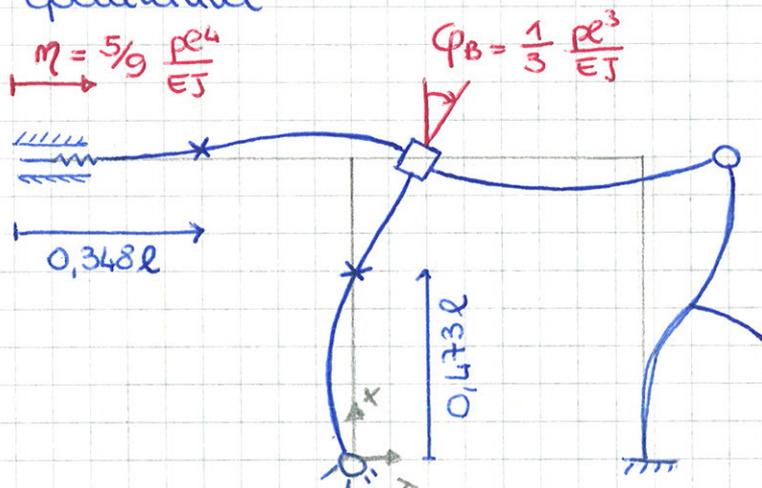
$$\left\{ \begin{array}{l} \sum V = 0 \rightarrow -\frac{5}{2} \rho + \frac{17}{8} \rho + \frac{3}{8} \rho = \frac{-20 + 17 + 3}{8} \rho = \phi \end{array} \right.$$

$$\left\{ \begin{array}{l} \sum M(B) = 0 \rightarrow \frac{17}{12} \rho^2 - \frac{55}{24} \rho^2 + \frac{7}{8} \rho^2 = \frac{34 - 55 + 21}{24} \rho^2 = \phi \end{array} \right.$$

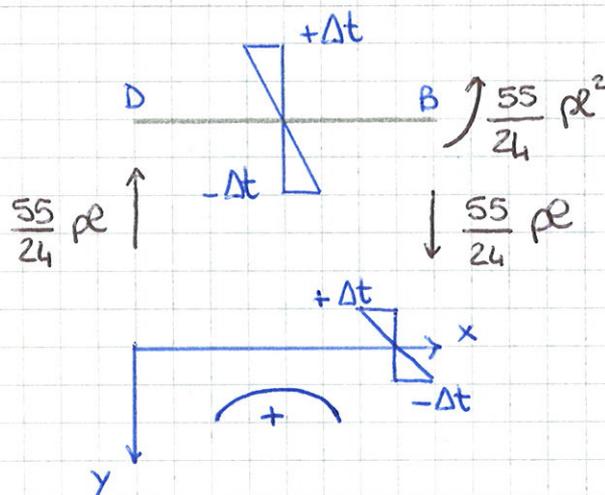
Tacciamiento delle azioni interne



• Deformata qualitativa



- Determinazione della deformati sulla asta BD:



$$\frac{\alpha \Delta t}{t} = \frac{13}{24} \frac{\rho l^2}{EJ}$$

$$M(x) = \frac{55}{24} \rho l \cdot x$$

$$y''(x) = -\frac{M(x)}{EJ} + \frac{2\alpha \Delta t}{t}$$

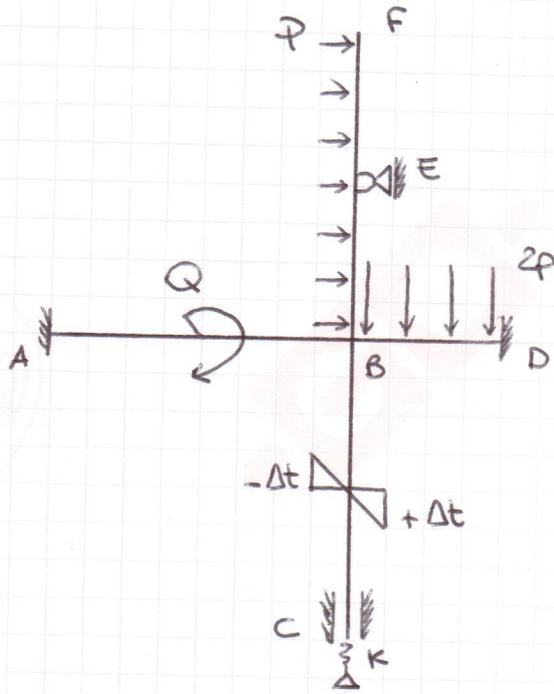
$$y''(x) = -\frac{55}{24} \rho l \frac{x}{EJ} + \frac{13}{12} \frac{\rho l^2}{EJ}$$

$$y''(x) > 0 \rightarrow x < \frac{26}{55} l = 0,473 l$$

se $x < 0,473 l$ concavità verso il basso

se $x > 0,473 l$ concavità verso l'alto

T.E. 4/07/2011

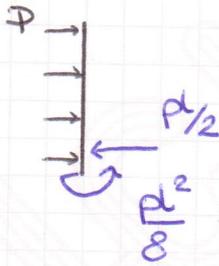
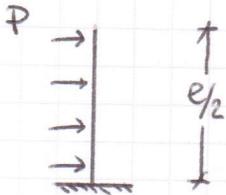


$$\frac{\alpha \Delta t}{h} = \frac{35}{64} \frac{p l^2}{EJ}$$

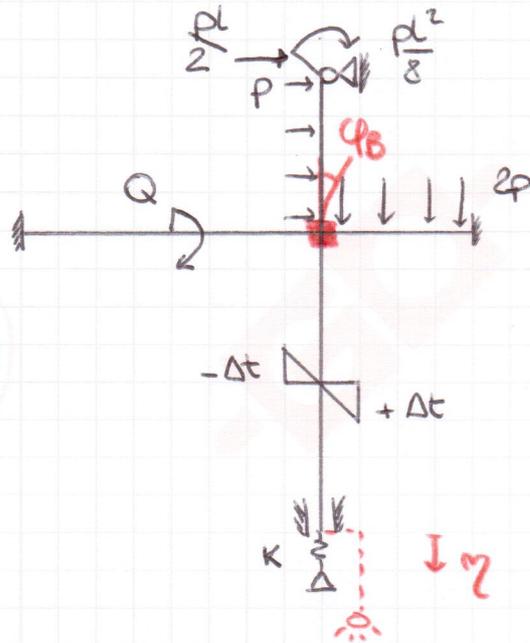
$$Q = p l^2$$

$$K = 18 \frac{EJ}{l^3}$$

- Appendice isostatica EF:



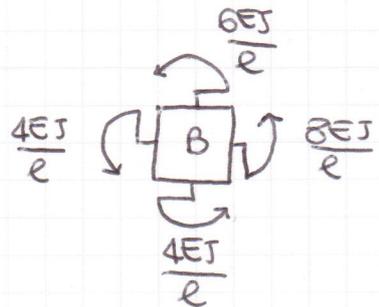
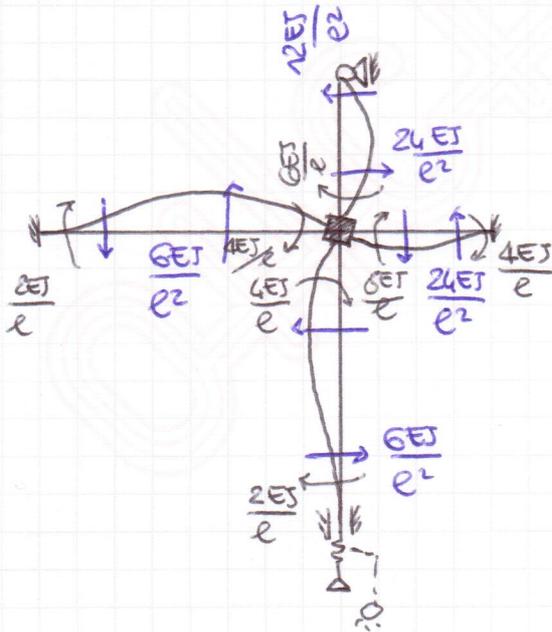
la struttura diventa:



$$\begin{cases} \varphi_B \cdot M_{BB} + \eta \cdot M_{B\eta} + M_{B0} = 0 \\ r_{CB} \cdot \varphi_B + r_{C\eta} \cdot \eta + r_{C0} = 0 \end{cases}$$



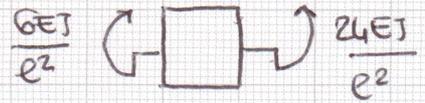
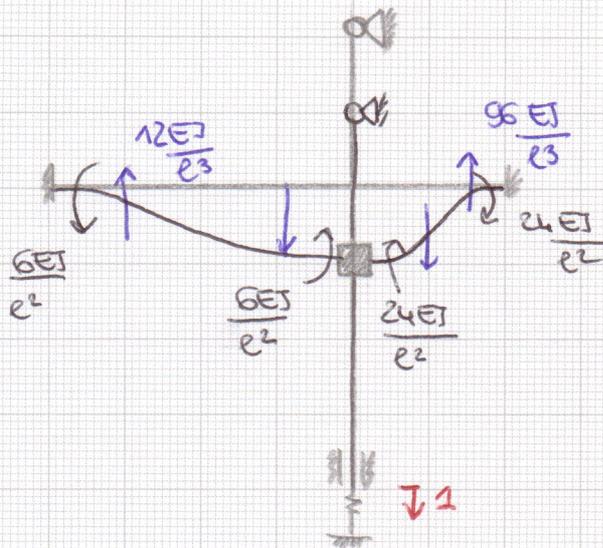
①



$$M_{BB} = -22 \frac{EJ}{e}$$

$$r_{CB} = 18 \frac{EJ}{e^2}$$

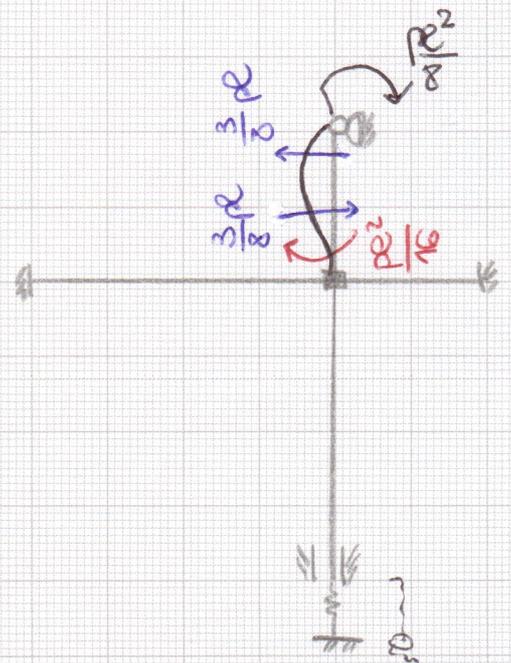
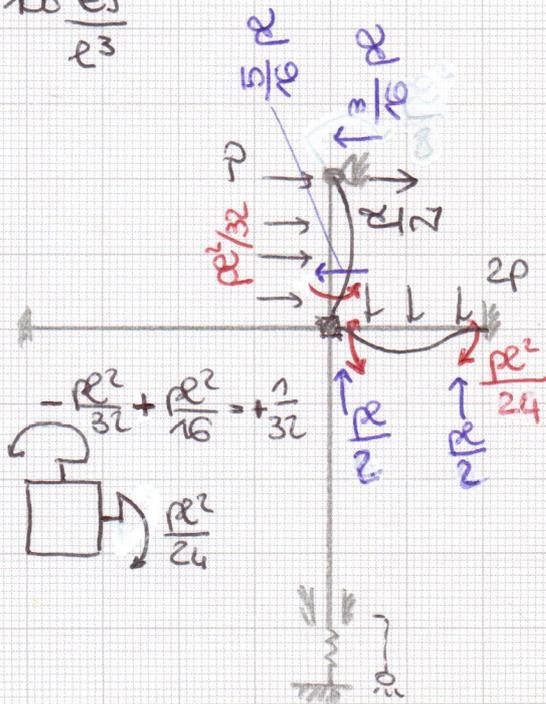
② $\eta \neq 0$



$$M_{BM} = -\frac{18EJ}{e^2}$$

$$r_{cc} = K + \frac{108EJ}{e^3}$$

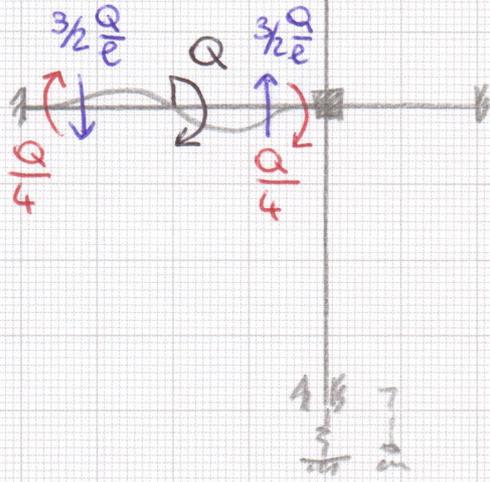
③ $\eta \neq 0$



$$M_{B0} = +\frac{1}{96} ql^2$$

$$r_{cc} = -ql$$

④ $Q \neq 0$



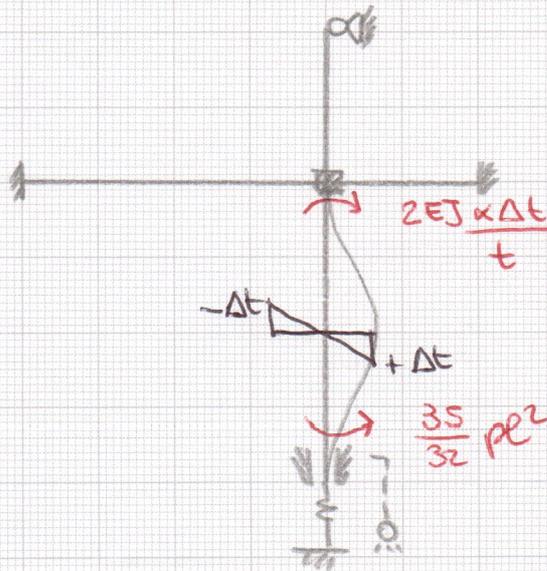
$$\frac{Q}{4} = \frac{pe^2}{4}$$

$$\frac{3}{2} \frac{Q}{e} = \frac{3}{2} pe$$

$$M_{B0} = -\frac{pe^2}{4}$$

$$H_{C0} = -\frac{3}{2} pe$$

⑤ $\Delta t \neq 0$



$$\frac{35}{32} pe^2$$

$$M_{B0} = -\frac{35}{32} pe^2$$

$$H_{C0} = \emptyset$$

Risoluzione del sistema:

$$\begin{cases} -22 \frac{EJ}{l} \varphi_B - 18 \frac{EJ}{l^2} \eta + \frac{1}{96} \rho l^2 - \frac{\rho l^2}{4} - \frac{35}{32} \rho l^2 = 0 \\ 18 \frac{EJ}{l^2} \varphi_B + \left(K + 108 \frac{EJ}{l^3} \right) \eta - \frac{\rho l}{2} - \frac{3}{2} \rho l = 0 \end{cases}$$

$$\begin{cases} -22 \frac{EJ}{l} \varphi_B - 18 \frac{EJ}{l^2} \eta - \frac{4}{3} \rho l^2 = 0 \\ 18 \frac{EJ}{l^2} \varphi_B + 126 \frac{EJ}{l^3} \eta - 2\rho l = 0 \end{cases}$$

$$126 \frac{EJ}{l^3} \eta = \left(2\rho l - 18 \frac{EJ}{l^2} \varphi_B \right)$$

$$\eta = \frac{1}{63} \frac{\rho l^4}{EJ} - \frac{9}{63} l \varphi_B$$

$$-22 \frac{EJ}{l} \varphi_B - 18 \frac{EJ}{l^2} \left(\frac{1}{63} \frac{\rho l^4}{EJ} - \frac{9}{63} l \varphi_B \right) - \frac{4}{3} \rho l^2 = 0$$

$$-22 \frac{EJ}{l} \varphi_B - \frac{18}{63} \rho l^2 + \frac{162}{63} \varphi_B \frac{EJ}{l} - \frac{4}{3} \rho l^2 = 0$$

$$-22 \frac{EJ}{l} \varphi_B + \frac{162}{63} \varphi_B \frac{EJ}{l} = \frac{4}{3} \rho l^2 + \frac{18}{63} \rho l^2$$

$$\frac{-1386 + 162}{63} \varphi_B \frac{EJ}{l} = \frac{84 + 18}{63} \rho l^2$$

$$\varphi_B = -\frac{102}{1224} \frac{\rho l^3}{EJ}$$

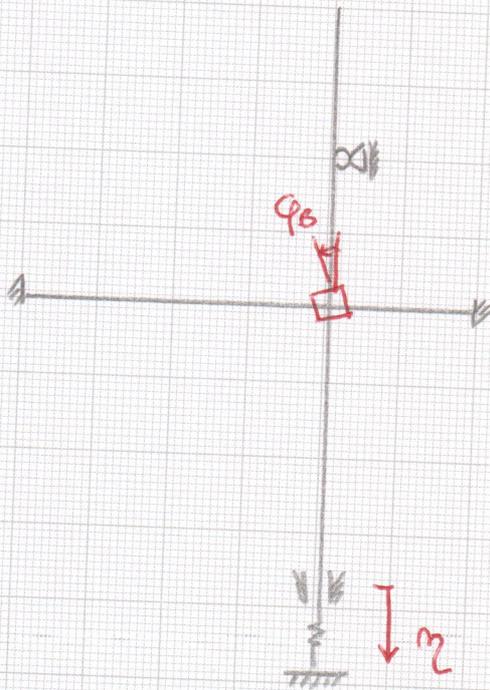
$$\boxed{\varphi_B = -\frac{1}{12} \frac{\rho l^3}{EJ}}$$

$$18 \frac{EJ}{l^2} \left(-\frac{1}{12} \frac{pl^3}{EJ} \right) + 126 \frac{EJ}{l^3} \eta - 2pl = 0$$

$$-\frac{3}{2} pl + 126 \frac{EJ}{l^3} \eta - 2pl = 0$$

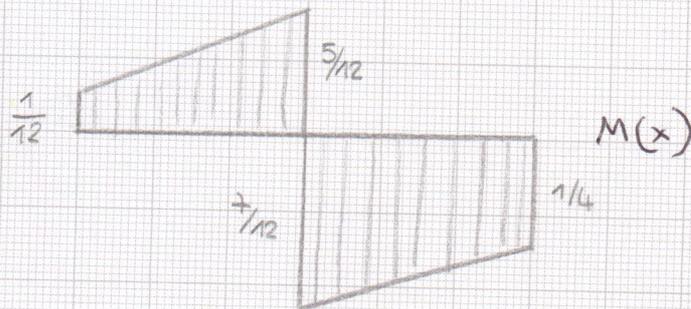
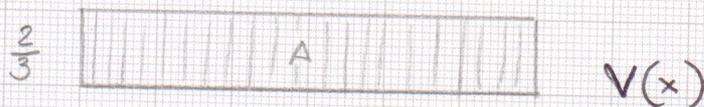
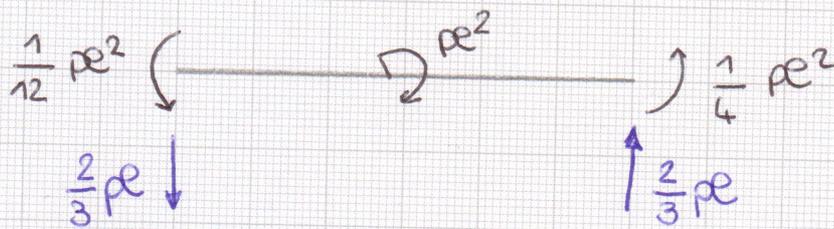
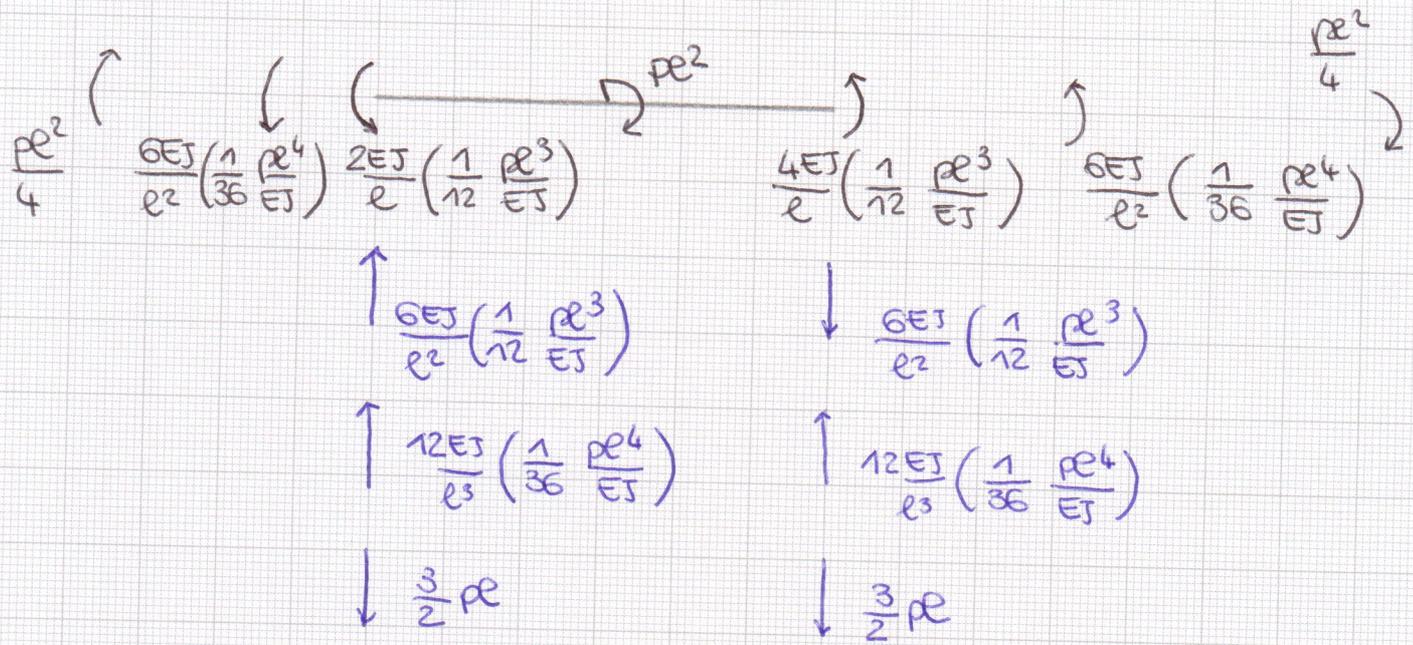
$$126 \frac{EJ}{l^3} \eta = \frac{7}{2} pl$$

$$\eta = \frac{1}{36} \frac{pl^4}{EJ}$$

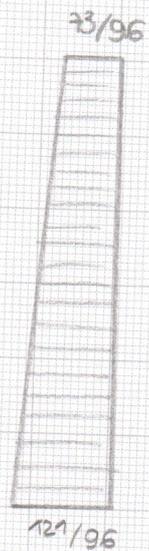
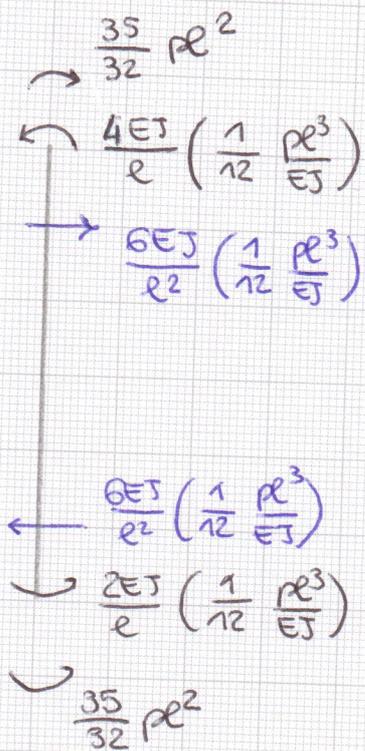


$$\begin{cases} \varphi_B = \frac{1}{12} \frac{pl^3}{EJ} \\ \eta = \frac{1}{36} \frac{pl^4}{EJ} \end{cases}$$

AB AB



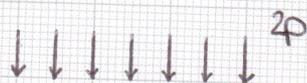
Asta BC



V

M

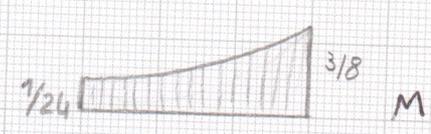
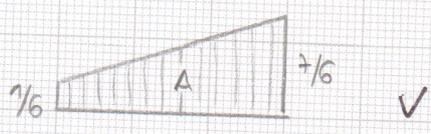
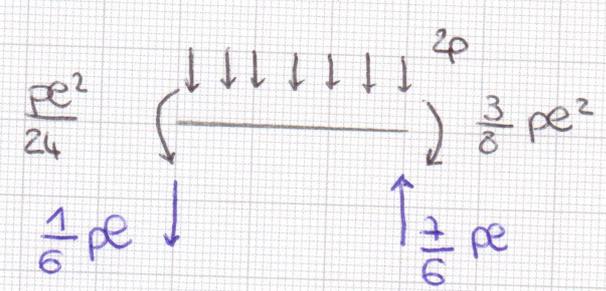
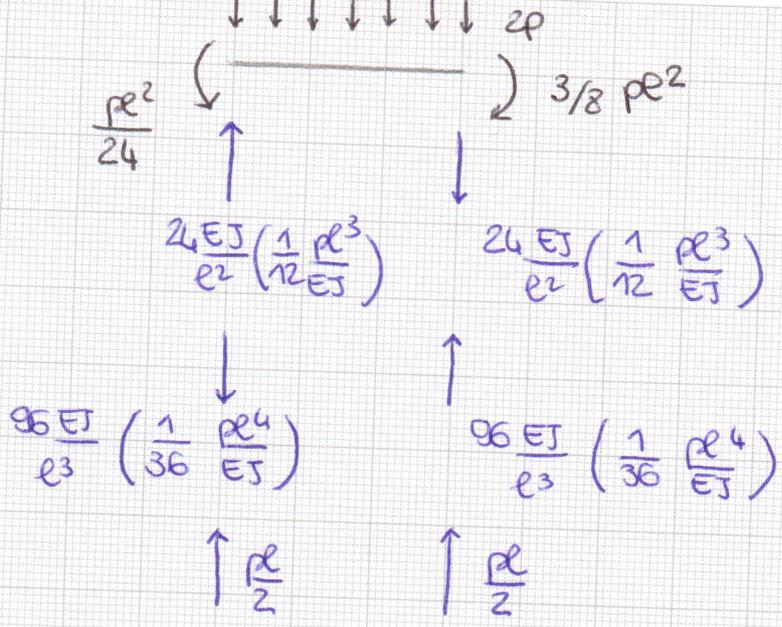
Asta BD



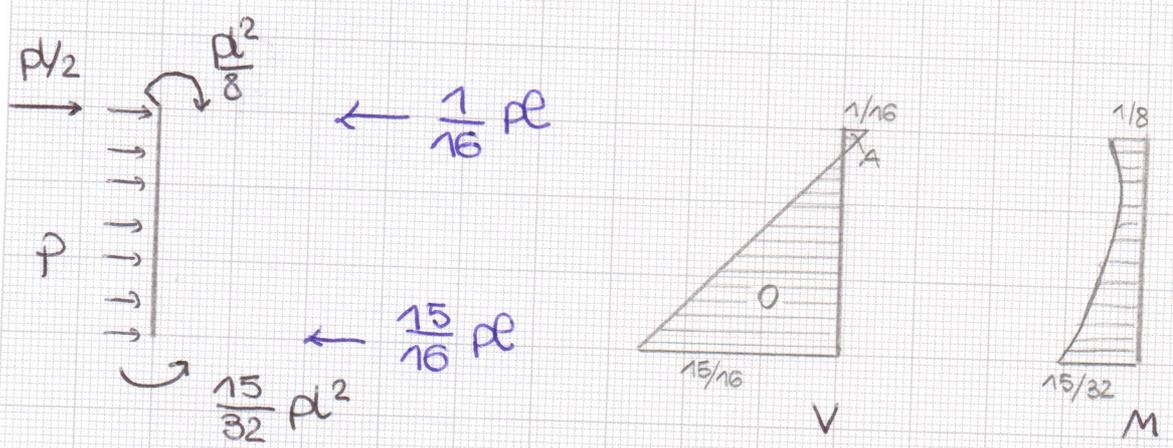
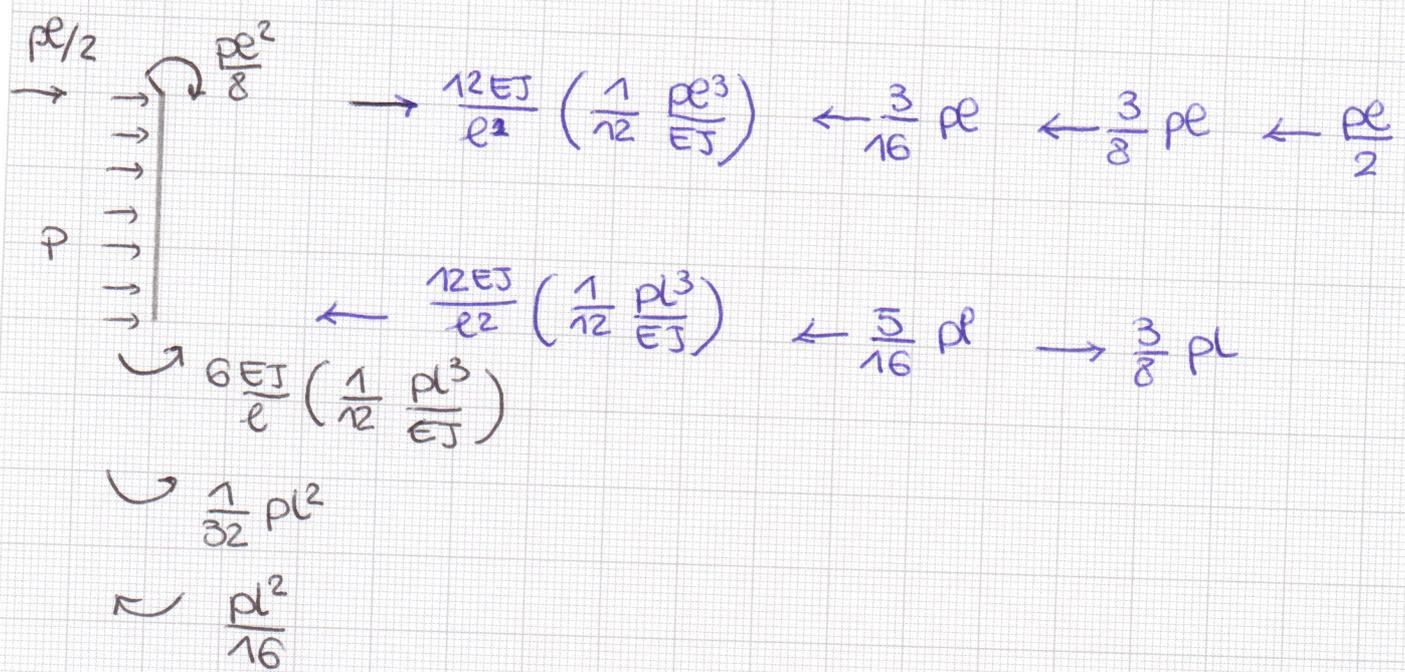
$$\left(\frac{8EJ}{l} \left(\frac{1}{12} \frac{l^3}{EJ} \right) \right) \rightarrow \frac{4EJ}{l} \left(\frac{1}{12} \frac{l^3}{EJ} \right) \rightarrow \frac{24EJ}{l^2} \left(\frac{1}{36} \frac{l^4}{EJ} \right) \rightarrow \frac{l^2}{24}$$

$$\left(\frac{24EJ}{l^2} \left(\frac{1}{36} \frac{l^4}{EJ} \right) \right)$$

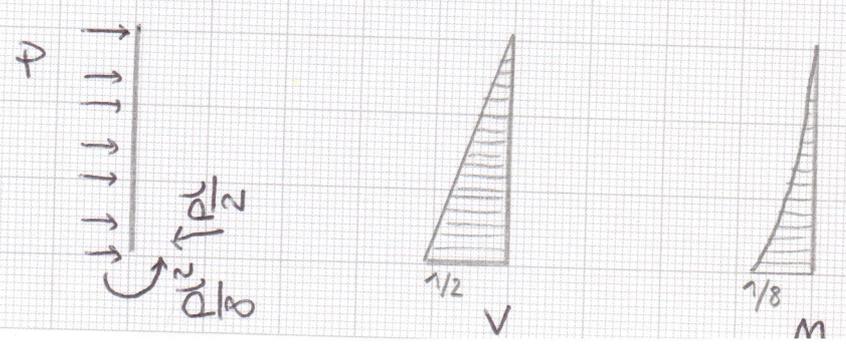
$$\left(\frac{l^2}{24} \right)$$



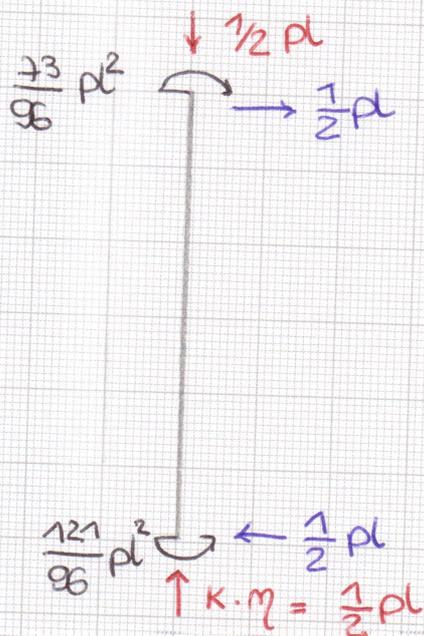
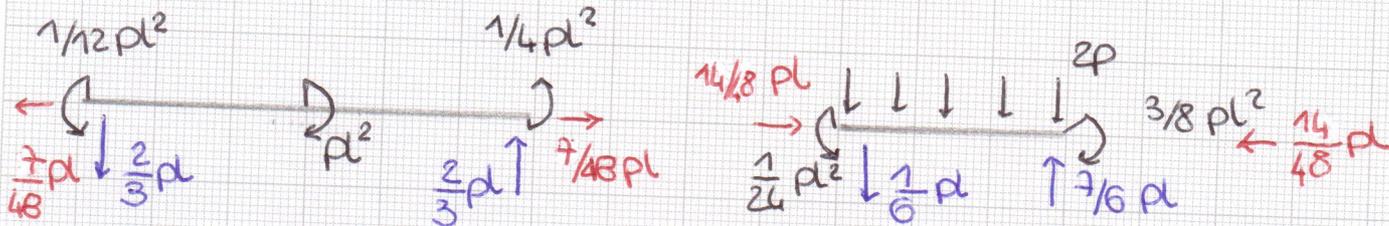
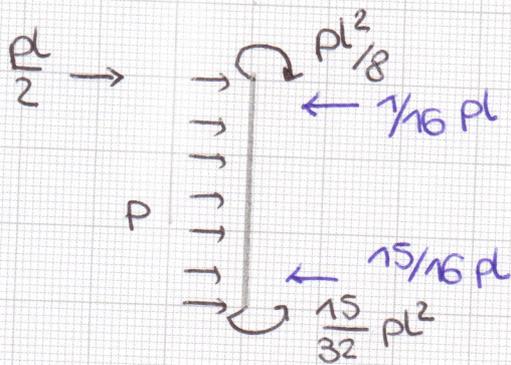
Asta BE



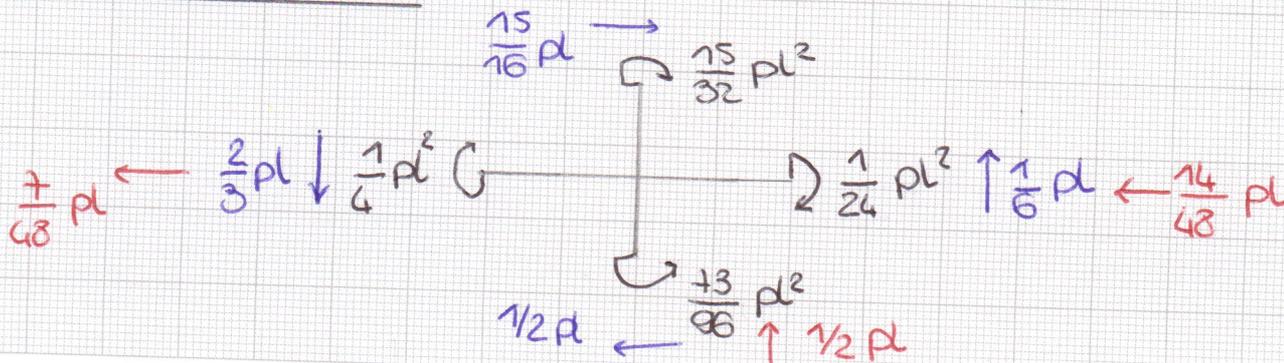
Appendice ISOSTATICA



Equilibrio globale



Equilibrio al nodo



$$\cdot \sum v = 0 \rightarrow \frac{2}{3} p - \frac{1}{2} p - \frac{1}{6} p = 0$$

$$\frac{4 - 3 - 1}{6} p = 0 \quad (\text{OK})$$

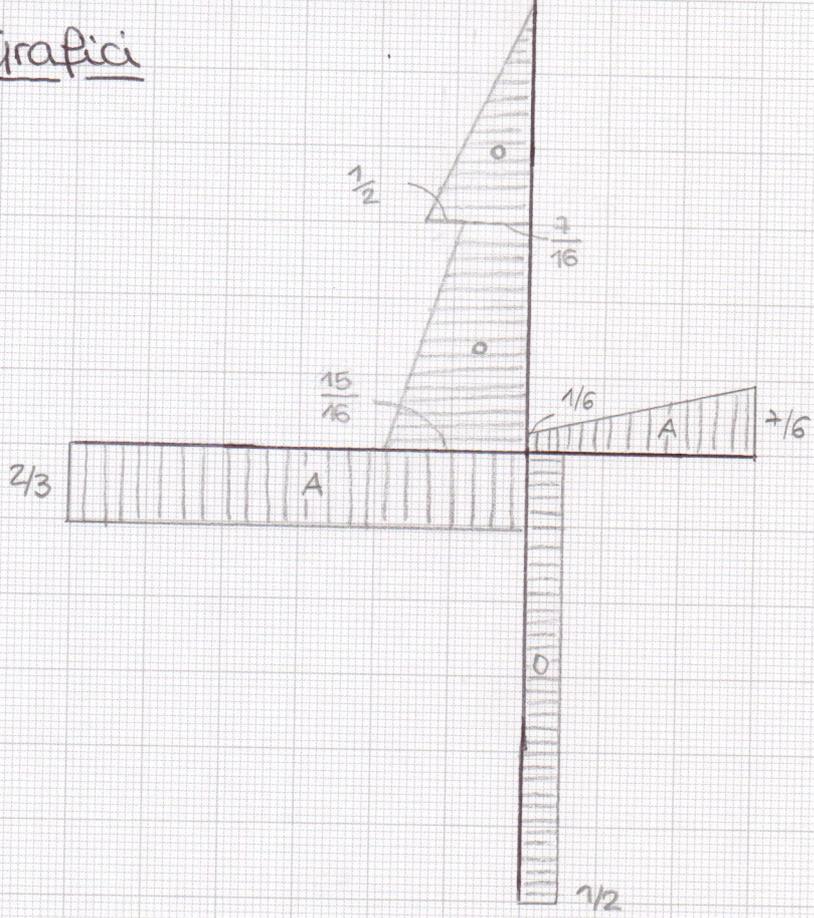
$$\cdot \sum H = 0 \rightarrow \frac{7}{48} p - \frac{15}{16} p + \frac{1}{2} p + \frac{14}{48} p = 0$$

$$\frac{7 - 45 + 24 + 14}{48} p = 0 \quad (\text{OK})$$

$$\cdot \sum M = 0 \rightarrow \frac{1}{4} p^2 + \frac{15}{32} p^2 + \frac{1}{24} p^2 - \frac{73}{96} p^2 = 0$$

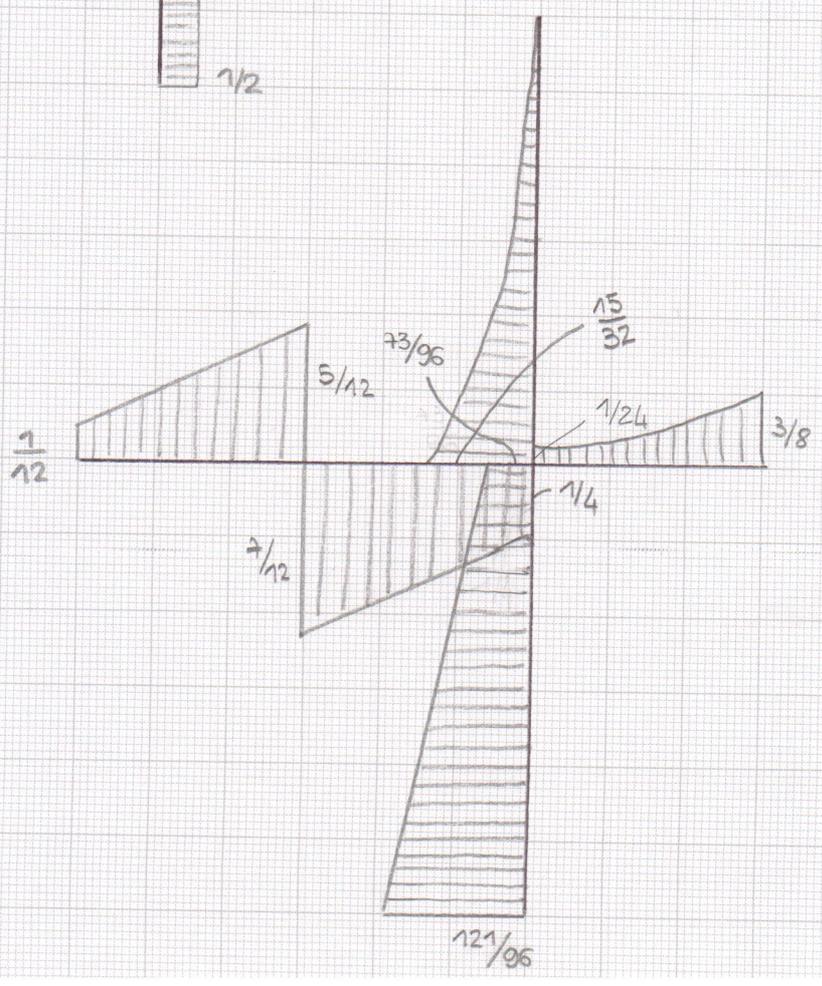
$$\frac{24 + 45 + 4 - 73}{96} p^2 = 0 \quad (\text{OK})$$

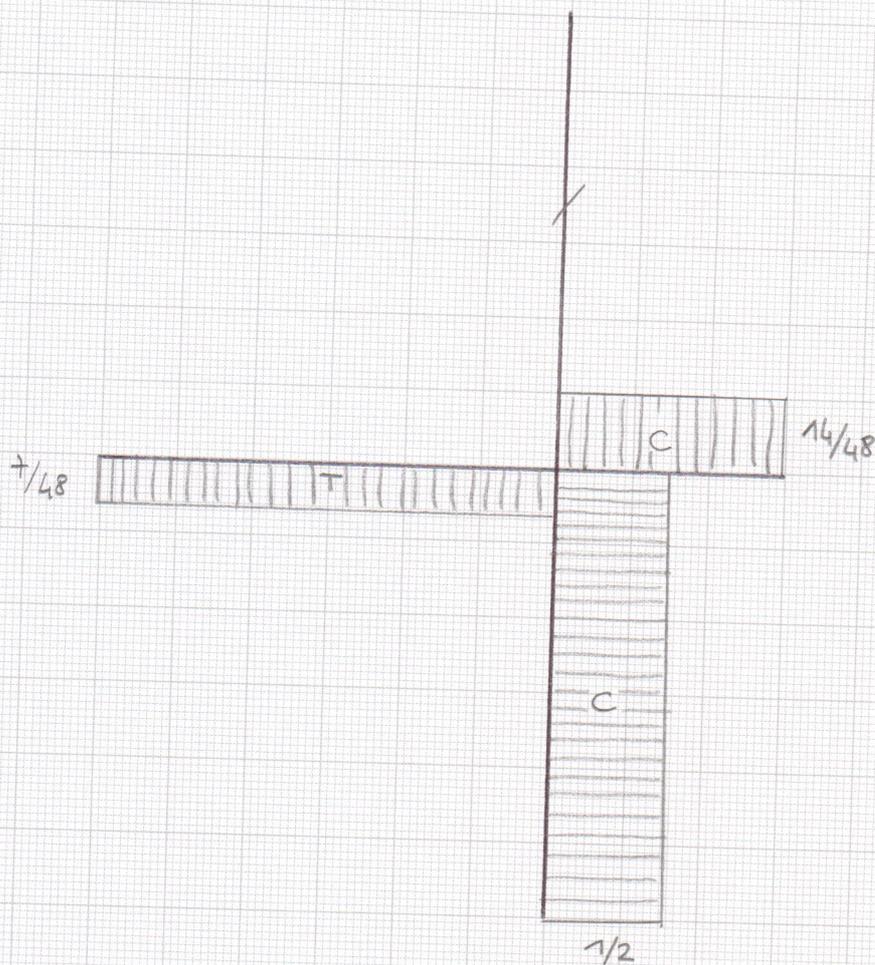
Grafici



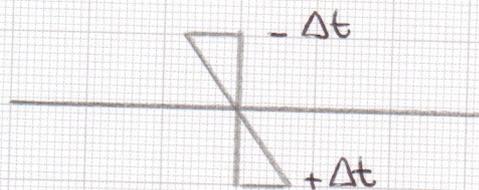
$$\left[\frac{v}{\rho} \right]$$

$$\left[\frac{M}{\rho^2} \right]$$





Calcolo deformata sull'asta BC



$$y''(x) = -\frac{M(x)}{EJ} \pm \frac{2x\Delta t}{t}$$

$$y''(x) = -\frac{1}{EJ} \left(-\frac{121}{96} pl^2 + \frac{1}{2} plx \right) - \frac{35}{32} \frac{pl^2}{EJ}$$

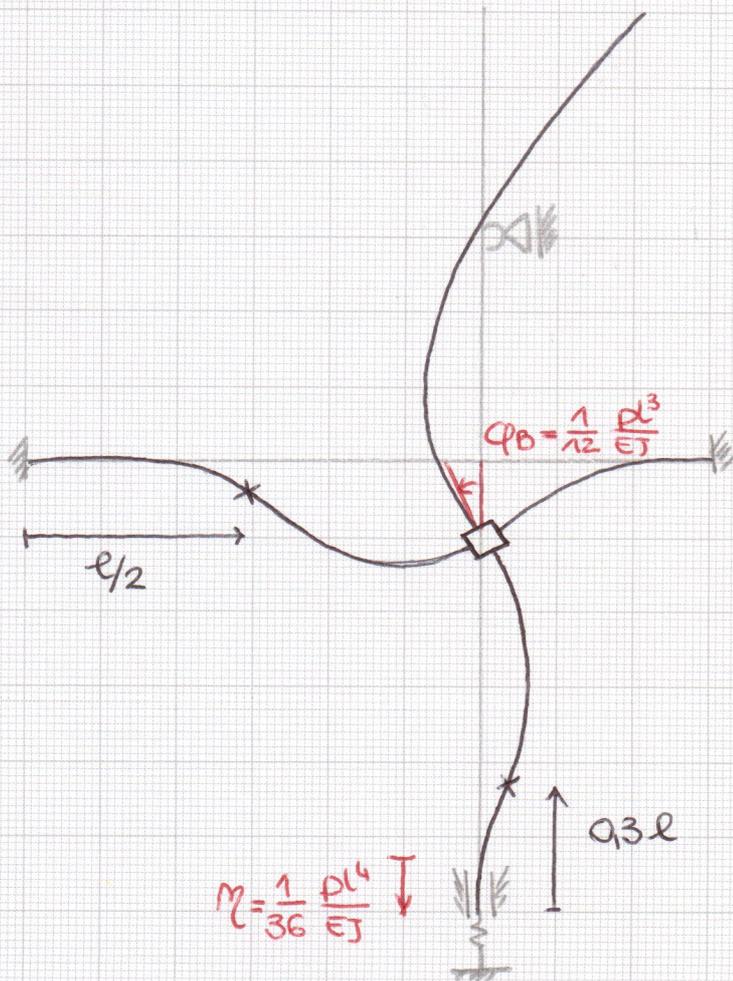
$$\frac{121}{96} \frac{Pl^2}{EJ} - \frac{1}{2} \frac{Plx}{EJ} - \frac{35}{32} \frac{Pl^2}{EJ} > 0$$

$$-\frac{1}{2}x > \frac{105-121}{96} l$$

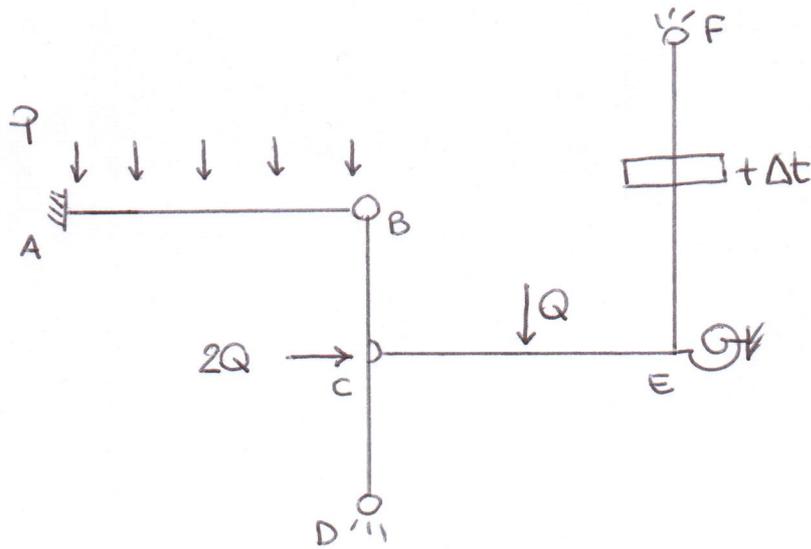
$$-\frac{1}{2}x > -\frac{16}{96} l$$

$$x < \frac{16}{48} l$$

$$x < 0,3 l$$



7. E, 18/07/2011



$$K = \frac{3EJ}{l}$$

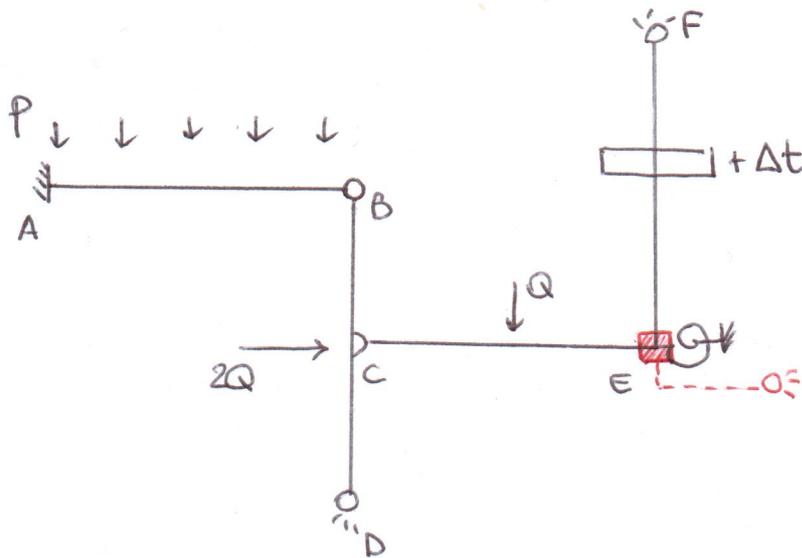
$$\alpha \Delta t = \frac{49}{48} \frac{QL^2}{EJ}$$

$$P = \frac{Q}{l}$$

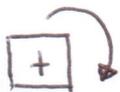
Telaio a nodi spostabili.

Sistema risolvente:

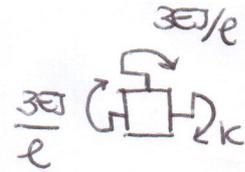
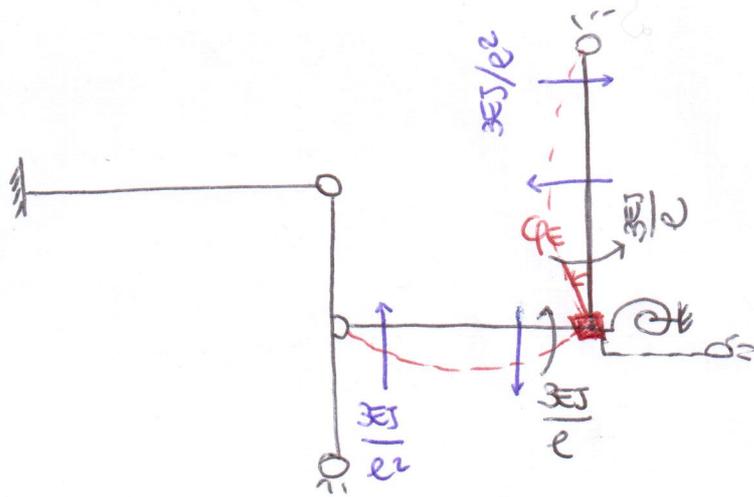
$$\begin{cases} m_{EE} \cdot \varphi_E + m_{E\eta} \cdot \eta + m_{EO} = 0 \\ h_{\eta E} \cdot \varphi_E + h_{\eta\eta} \cdot \eta + h_{\eta O} = 0 \end{cases}$$



Convenzioni di segno:

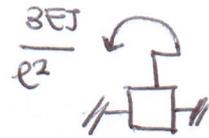
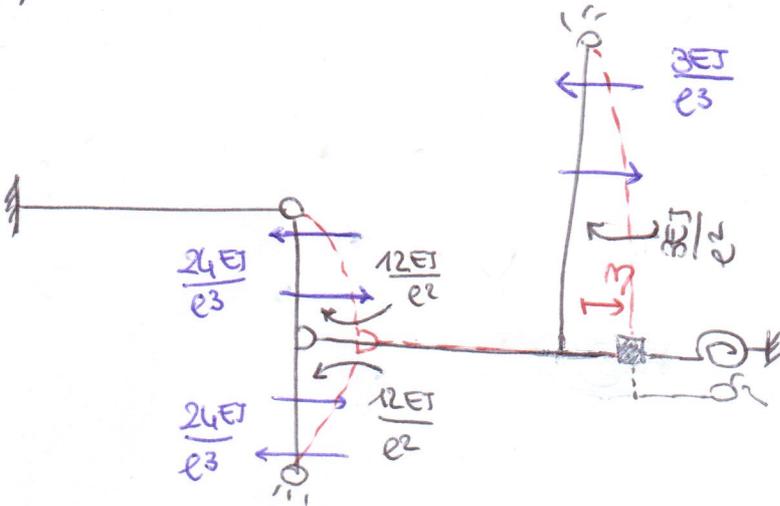


① $\varphi_E \neq 0$



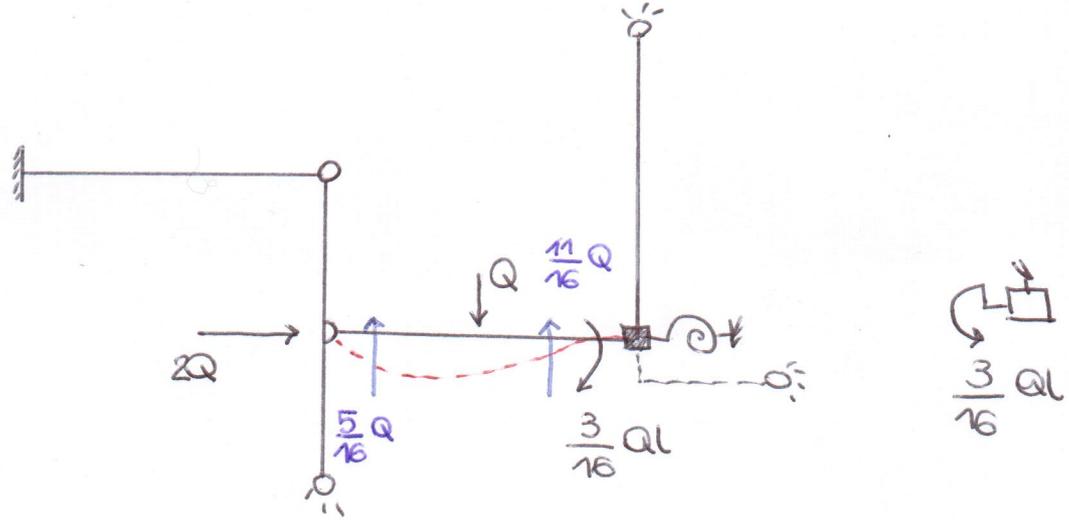
$$\begin{cases} m_E = \frac{3EJ}{l} + \frac{3EJ}{l} + k = \frac{6EJ}{l} + k = \frac{6EJ}{l} + \frac{3EJ}{l} = \frac{9EJ}{l} \\ h_E = \frac{3EJ}{l^2} \end{cases}$$

② $\varphi_E \neq 0$



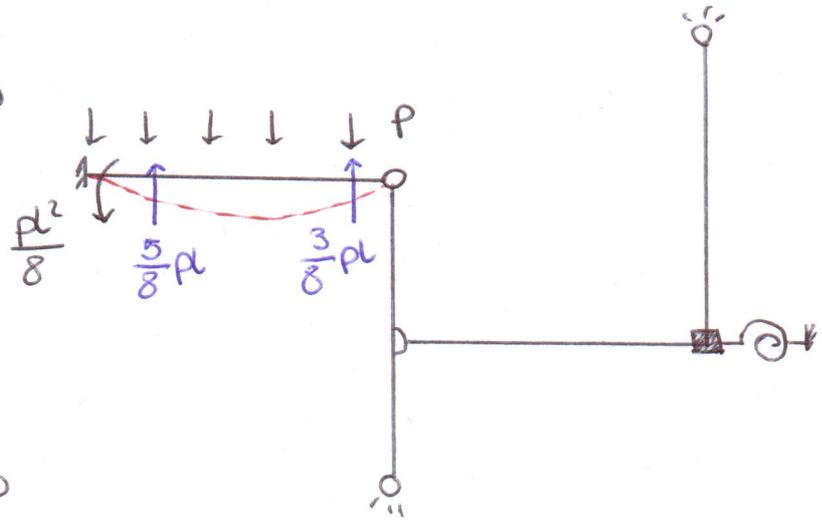
$$\begin{cases} m_E = -\frac{3EJ}{l^2} \\ h_E = -\frac{48EJ}{l^3} - \frac{3EJ}{l^3} = -\frac{51EJ}{l^3} \end{cases}$$

③: $Q \neq 0$



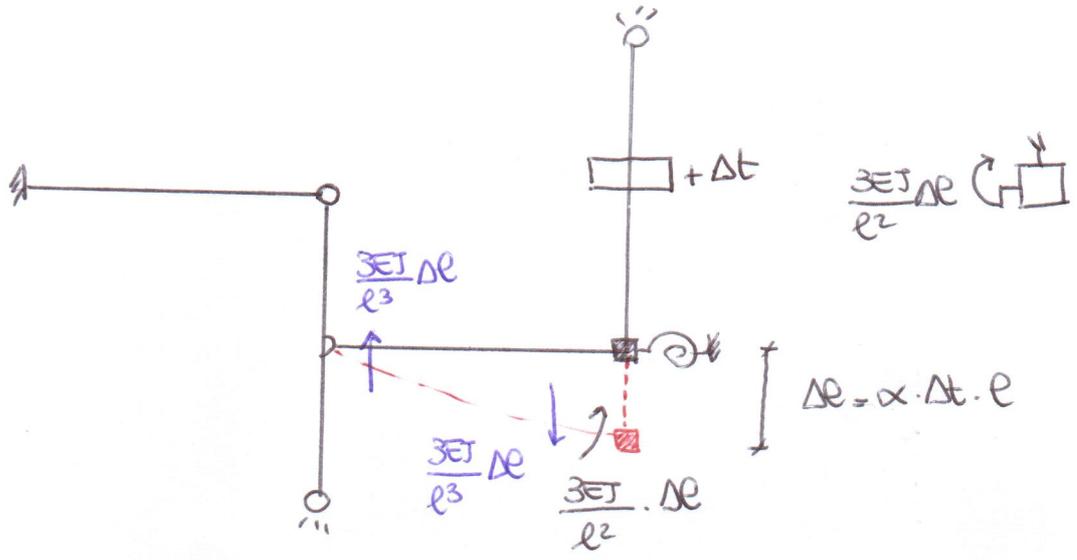
$$\begin{cases} m_{EO} = -\frac{3}{16} Ql \\ r_{EO} = 2Q \end{cases}$$

④: $P \neq 0$



$$\begin{cases} m_{EO} = 0 \\ r_{EO} = 0 \end{cases}$$

⑤: $\Delta t \neq 0$



$$\begin{cases} m_{EO} = \frac{3EJ}{l^2} \Delta e \\ r_{EO} = 0 \end{cases}$$

$$\begin{cases} \left(\frac{6EJ}{e} + K \right) \varphi - \frac{3EJ}{e^2} \cdot \eta - \frac{3}{16} Qe + \frac{3EJ}{e^2} \cdot \alpha \Delta t \cdot e = 0 \\ \frac{3EJ}{e^2} \cdot \varphi - \frac{51EJ}{e^3} \cdot \eta + 2Q = 0 \end{cases}$$

$$-\frac{3}{16} + \frac{49}{16}$$

$$\begin{cases} \left(\frac{6EJ}{e} + \frac{3EJ}{e} \right) \varphi - \frac{3EJ}{e^2} \eta - \frac{3}{16} Qe + \frac{3EJ}{e^2} \frac{49}{48} \frac{Qe^3}{EJ} = 0 \\ \frac{3EJ}{e^2} \cdot \varphi - \frac{51EJ}{e^3} \cdot \eta + 2Q = 0 \end{cases}$$

$$\begin{cases} \frac{9EJ}{e} \varphi - \frac{3EJ}{e^2} \eta + \frac{46}{16} Qe = 0 \\ \frac{3EJ}{e^2} \cdot \varphi - \frac{51EJ}{e^3} \eta + 2Q = 0 \quad (-3e) \end{cases}$$

$$\begin{cases} \frac{9EJ}{e} \varphi - \frac{3EJ}{e^2} \eta + \frac{46}{16} Qe = 0 \\ -\frac{9EJ}{e} \varphi + \frac{153EJ}{e^2} \eta - 6Qe = 0 \end{cases}$$

$$\frac{46}{16} - 6 = \frac{46-96}{16}$$

$$\equiv \frac{150EJ}{e^2} \eta - \frac{50}{16} Qe = 0$$

$$\frac{150EJ}{e^2} \eta = \frac{25}{8} Qe$$

$$\eta = \frac{25}{8} \cdot \frac{1}{150} \frac{e^2}{EJ} Qe$$

$$\boxed{\eta = \frac{1}{48} \frac{Qe^3}{EJ}}$$

$$\frac{288-153}{48} = \frac{135}{48}$$

$$-\frac{9EJ}{e} \varphi + \frac{153EJ}{e^2} \left(\frac{1}{48} \frac{Qe^3}{EJ} \right) - 6Qe = 0$$

$$-\frac{9EJ}{e} \varphi = 6Qe - \frac{153}{48} Qe$$

$$-\frac{9EI}{l} \varphi = \frac{135}{48} Ql$$

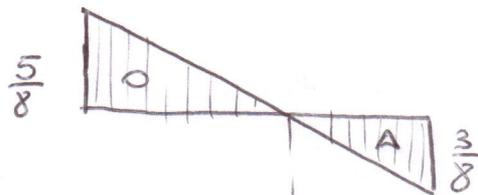
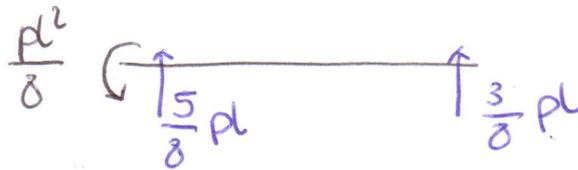
$$\varphi = \frac{135}{48} \cdot \frac{1}{9} \frac{Ql^2}{EI}$$

$$\eta = \frac{1}{48} \frac{Ql^3}{EI}$$

$$\varphi = -\frac{15}{48} \frac{Ql^2}{EI}$$

$$\varphi = -\frac{5}{16} \frac{Ql^2}{EI}$$

Asta AB

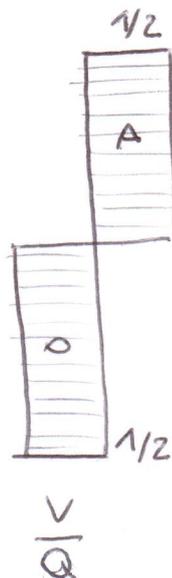
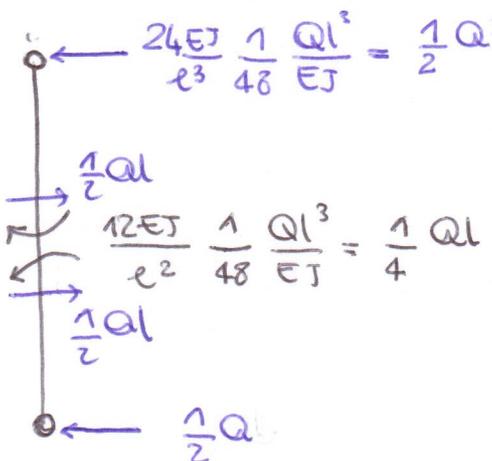


$$\frac{V}{Pl}$$

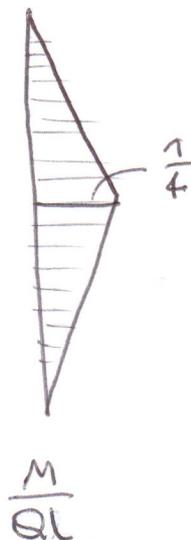


$$\frac{M}{Pl^2}$$

Asta BD

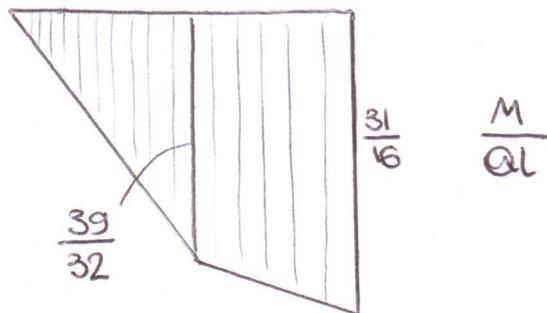
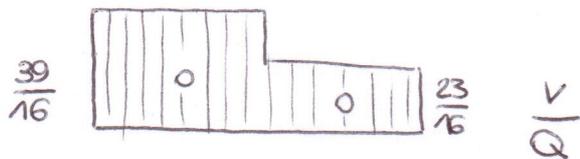
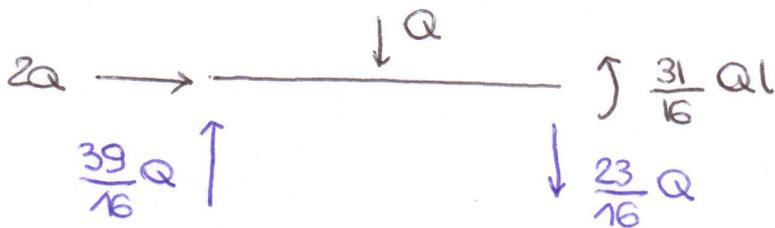
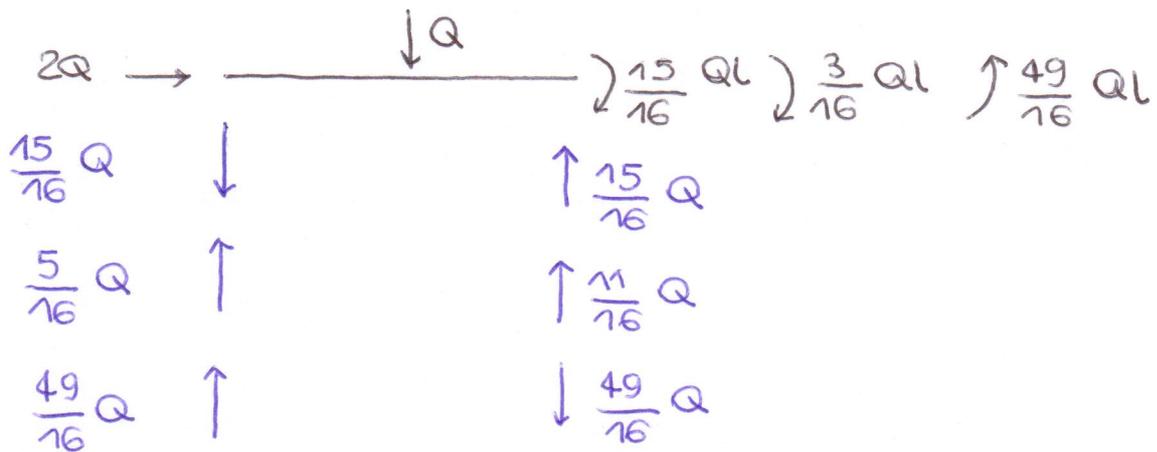


$$\frac{V}{Q}$$

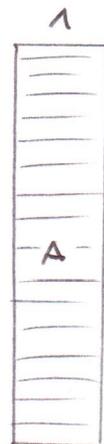
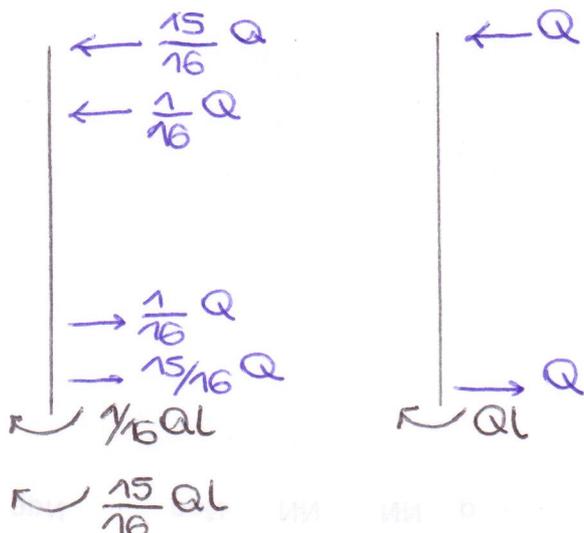


$$\frac{M}{Ql}$$

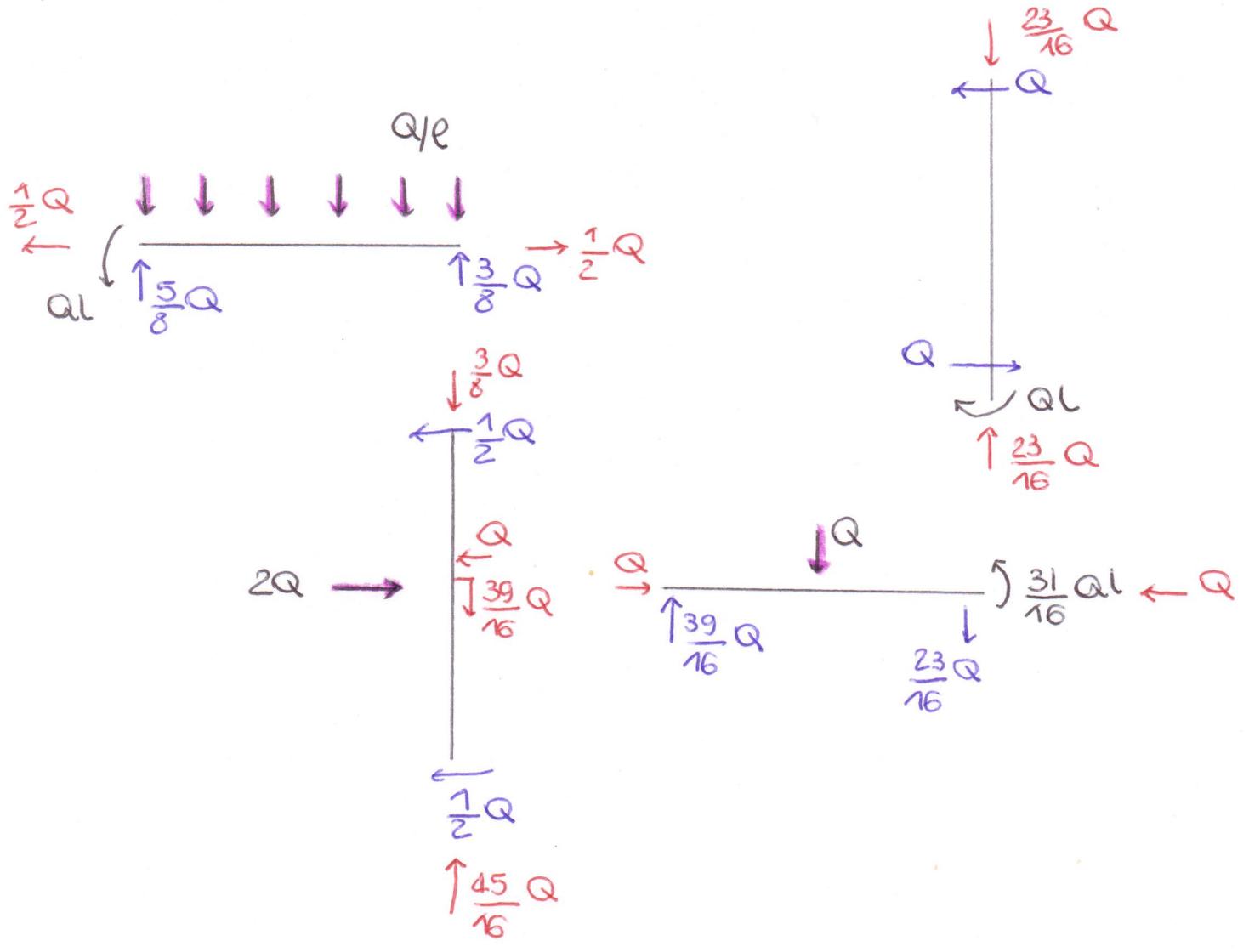
Asta CE



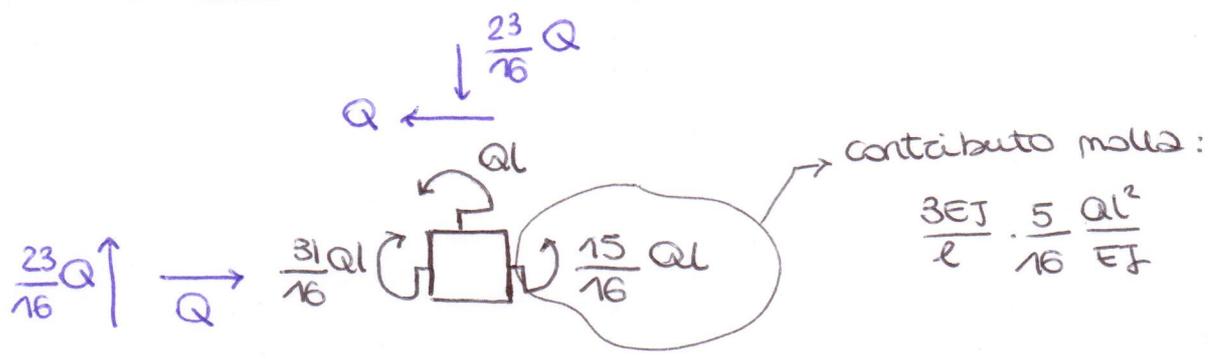
Asta EF



Equilibrio globale



Equilibrio al nodo

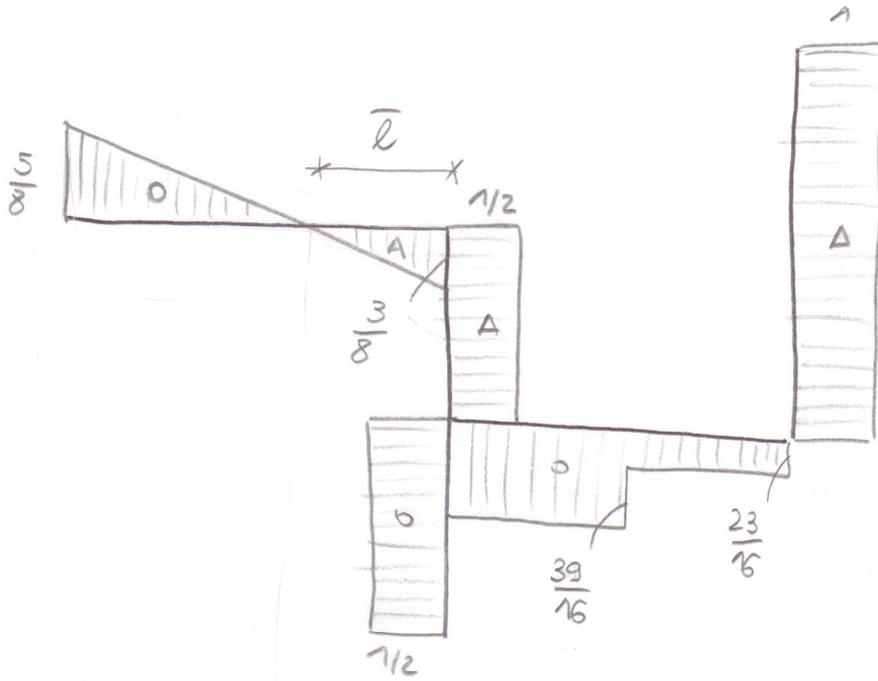


$$\sum M = 0 \rightarrow \frac{31}{16} Ql - Ql - \frac{15}{16} Ql = 0$$

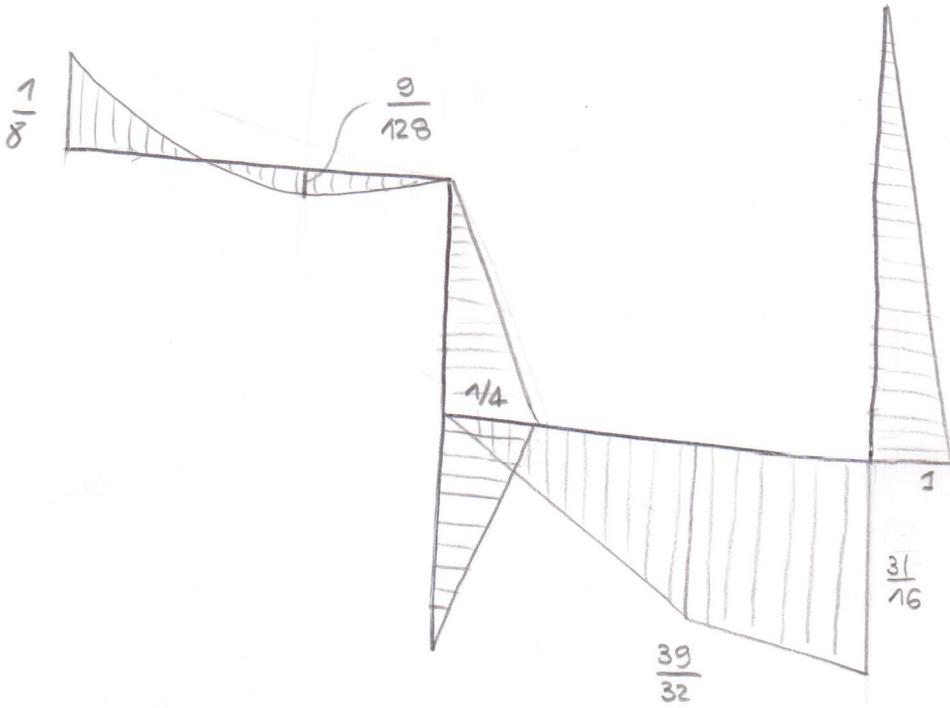
$$\sum V = 0 \rightarrow \frac{23}{16} Q - \frac{23}{16} Q = 0$$

$$\sum H = 0 \rightarrow Q - Q = 0$$

(OK)



$$\frac{Q}{V}$$



$$\frac{M}{ql}$$

$$\bar{l} = \frac{3}{8} l$$

$$\begin{aligned}
 M_{\max}^+ &= \frac{3}{8} pl \cdot \frac{3}{8} l - p \cdot \frac{3}{8} l \cdot \frac{3}{16} l \\
 &= \frac{9}{64} pl^2 - \frac{9}{128} pl^2 \\
 &= \frac{9}{128} pl^2
 \end{aligned}$$

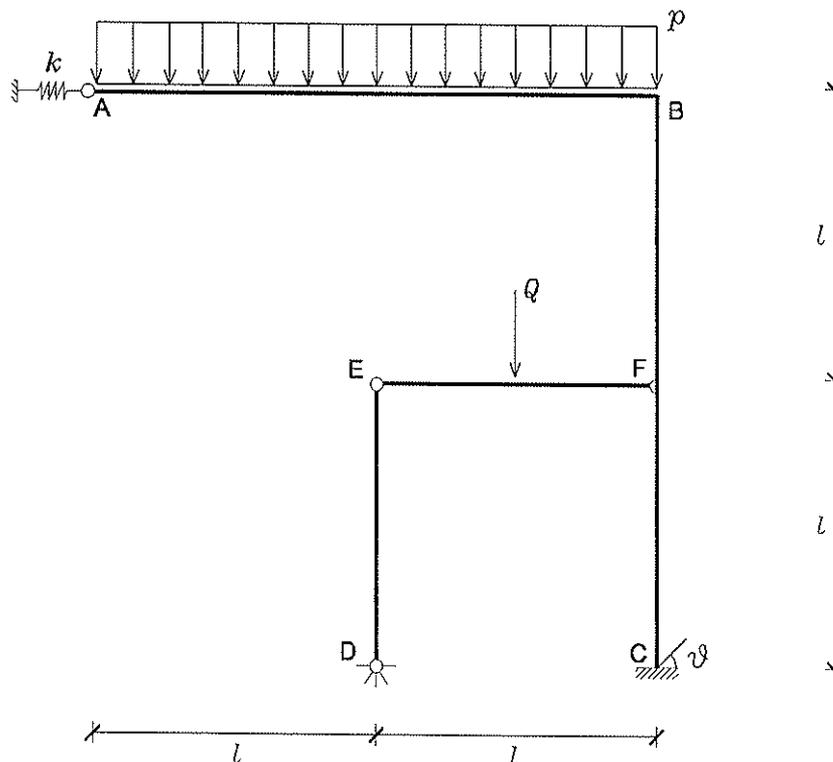
TECNICA DELLE COSTRUZIONI

TEMA ESAME DEL 12 SETTEMBRE 2011

DOCENTE: ING. FAUSTO MINELLI

ESERCITATORE: ING. ADRIANO REGGIA

Esercizio



$$Q = \frac{9}{8}pl$$

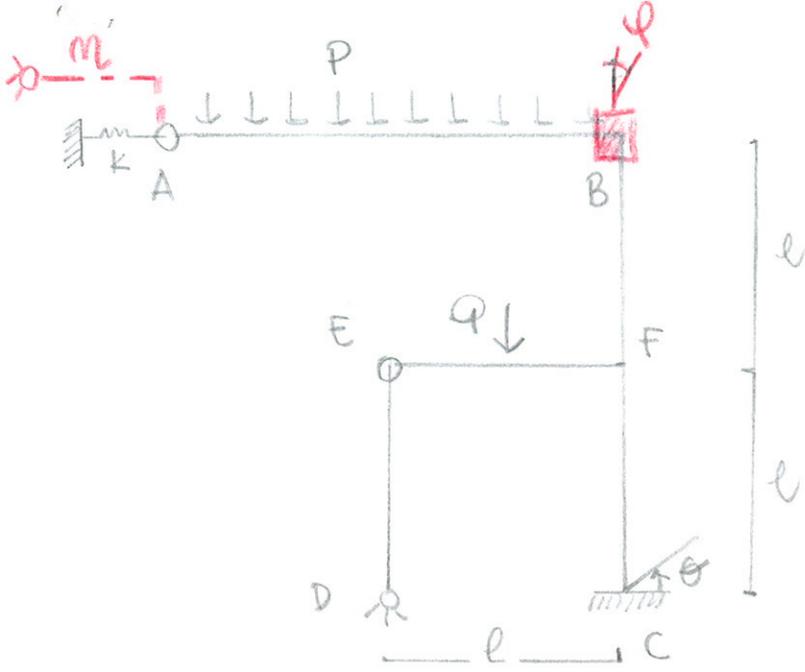
$$k = 2 \frac{EJ}{l^3}$$

$$\vartheta = \frac{5pl^3}{4EJ}$$

Dato il telaio in figura

Si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.

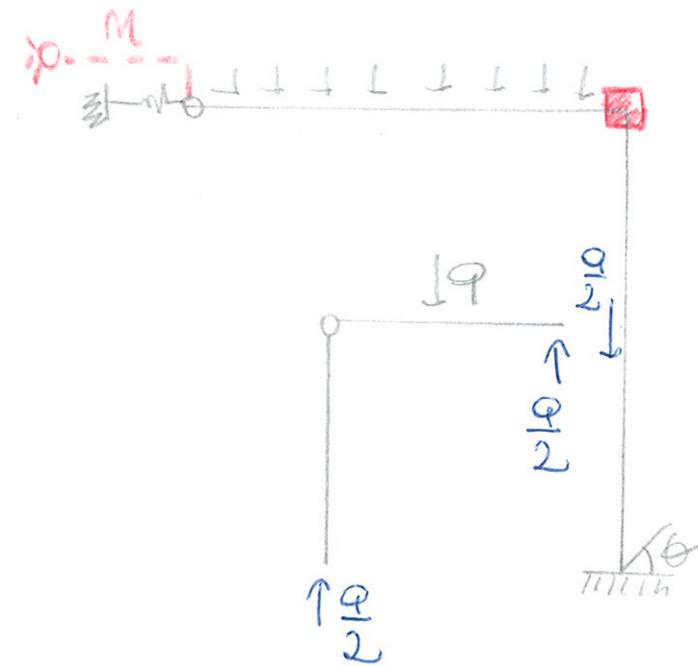


$$Q = \frac{9}{8} p l$$

$$\theta = \frac{5}{4} \frac{p l^3}{EJ}$$

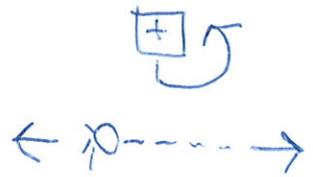
$$k = 2 \frac{EJ}{l^3}$$

* Appendice isostatica DEF:



$$\frac{Q}{2} = \frac{9}{16} p l$$

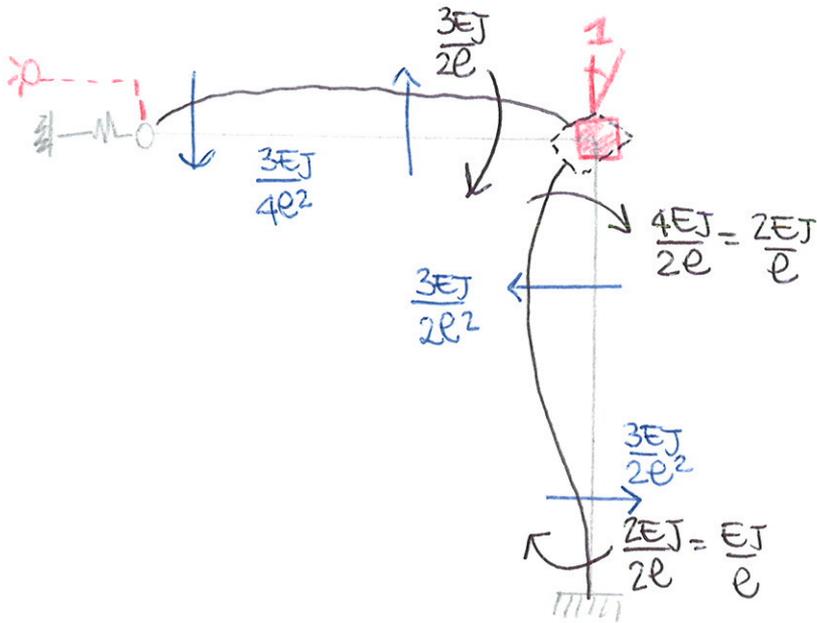
* CONVENZIONI



* Sistema risolvibile:

$$\begin{cases} M_{BB} \varphi_B + M_{B\eta} \eta + M_{B0} = 0 \\ h_{B\eta} \varphi_B + h_{\eta\eta} \eta + h_{\eta 0} = 0 \end{cases}$$

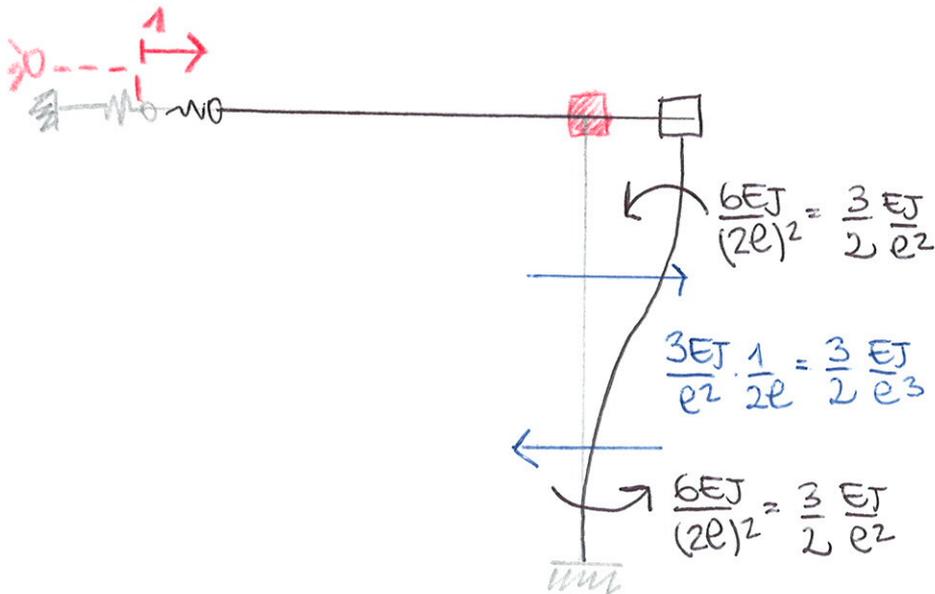
① $p_B=1$ $\eta=p=\theta=0$:



$$M_{BB} = \frac{3}{2} \frac{EJ}{e} + \frac{2EJ}{e} = \frac{(3+4)EJ}{2e} = \frac{7}{2} \frac{EJ}{e}$$

$$h_{MB} = \frac{3}{2} \frac{EJ}{e^2}$$

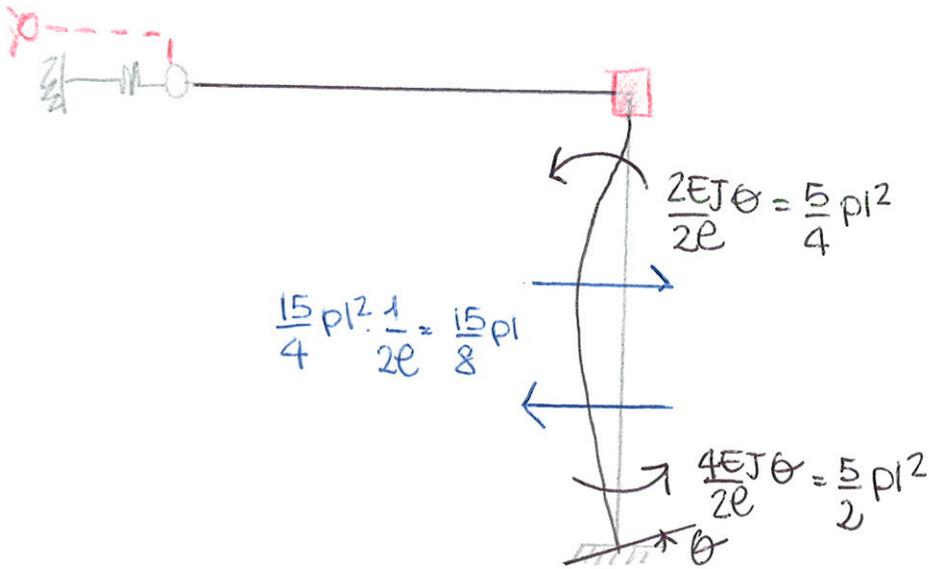
② $\eta=1$ $p_B=p=\theta=0$:



$$M_{B\eta} = -\frac{3}{2} \frac{EJ}{e^2}$$

$$h_{M\eta} = -\frac{3}{2} \frac{EJ}{e^3} - k = -\frac{3}{2} \frac{EJ}{e^3} - \frac{2EJ}{e^3} = \frac{(-3-4)EJ}{2e^3} = -\frac{7}{2} \frac{EJ}{e^3}$$

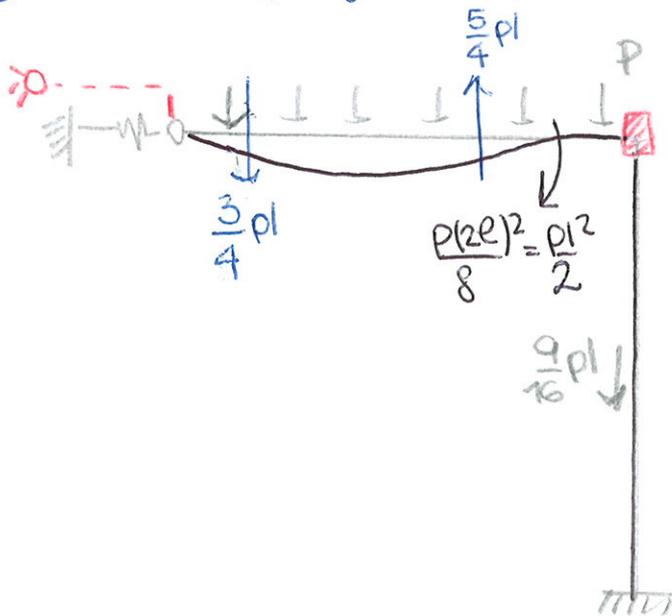
③ $\theta \neq 0$ $\rho_B = \eta = \rho = 0$



$$M_{B0}^{\theta} = -\frac{5}{4} p l^2$$

$$h_{mp}^{\theta} = -\frac{15}{8} p l$$

④ $\rho \neq 0$ $\rho_B = \eta = \theta = 0$:



$$M_{B0}^{\rho} = +\frac{p l^2}{2}$$

$$h_{mp}^{\rho} = 0$$

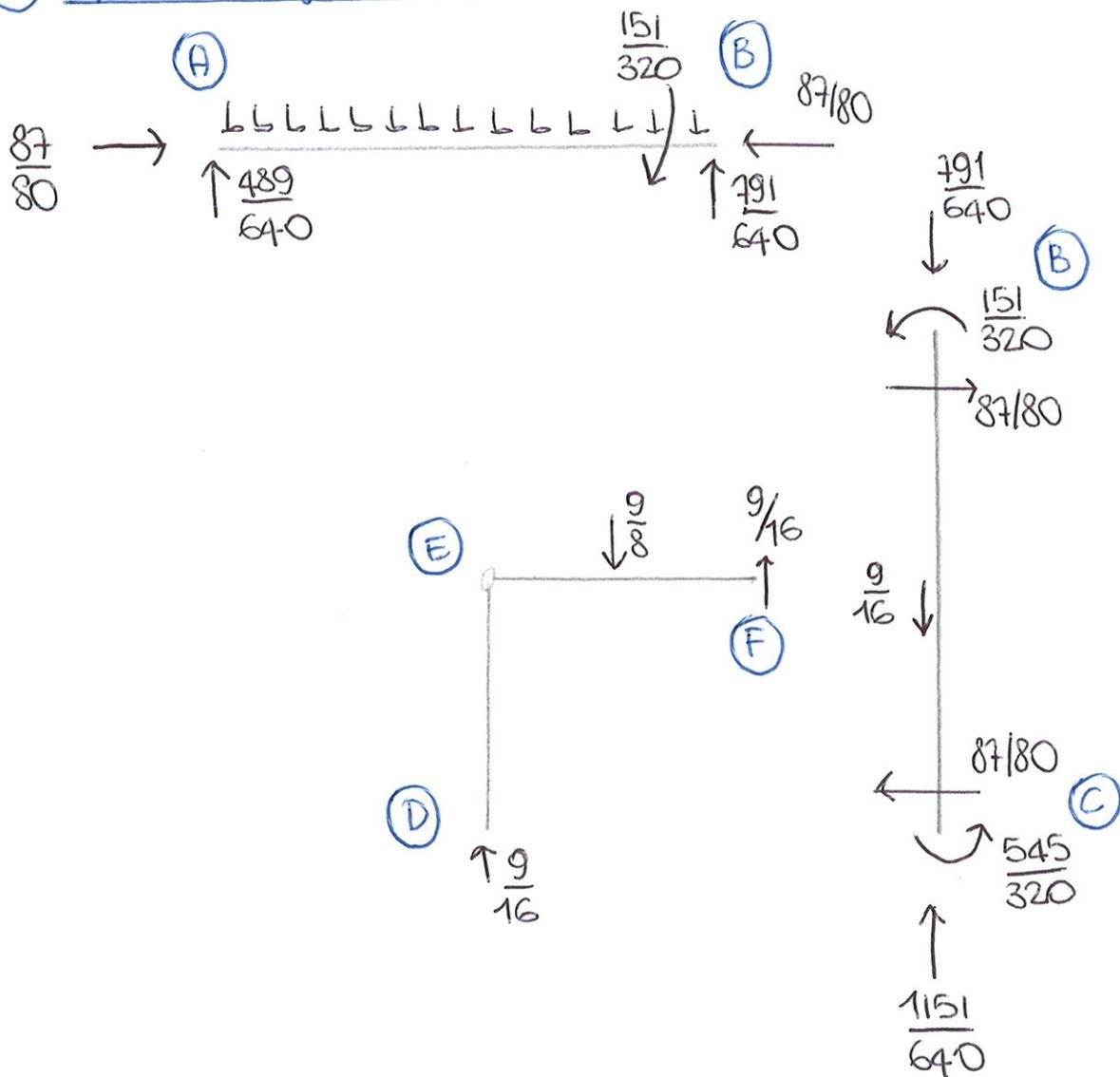
* Sistema isostemico:

$$\begin{cases} \frac{7}{2} \frac{EJ}{l} \varphi_B - \frac{3}{2} \frac{EJ}{l^2} M - \frac{5}{4} pl^2 + \frac{pl^2}{2} = 0 \\ \frac{3}{2} \frac{EJ}{l^2} \varphi_B - \frac{7}{2} \frac{EJ}{l^3} M - \frac{15}{8} pl = 0 \end{cases}$$

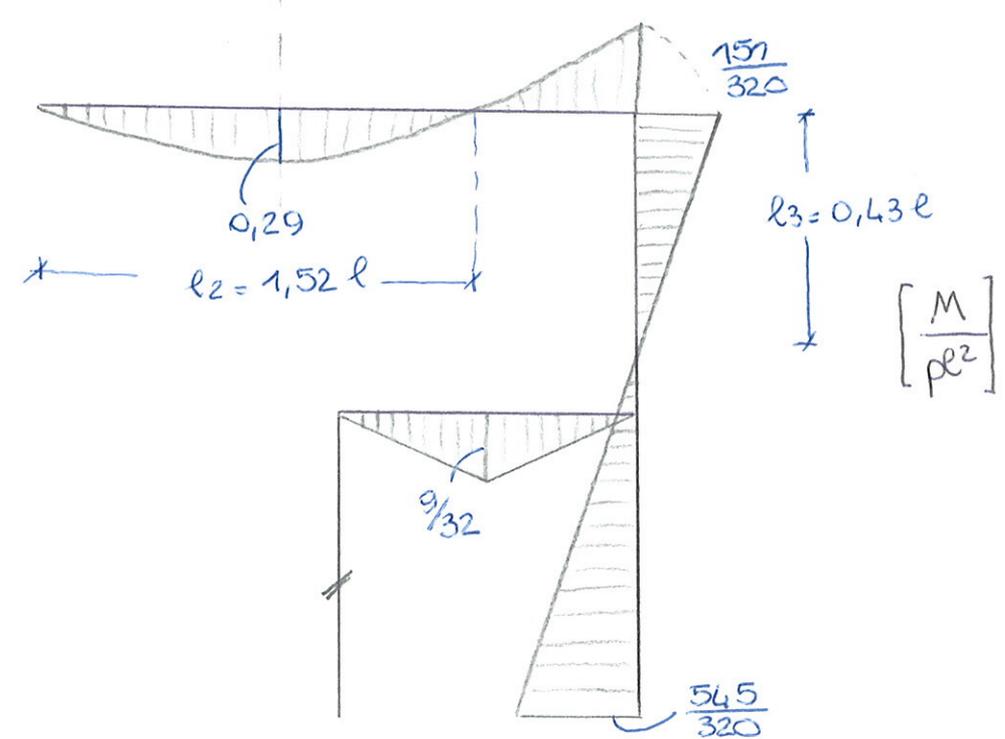
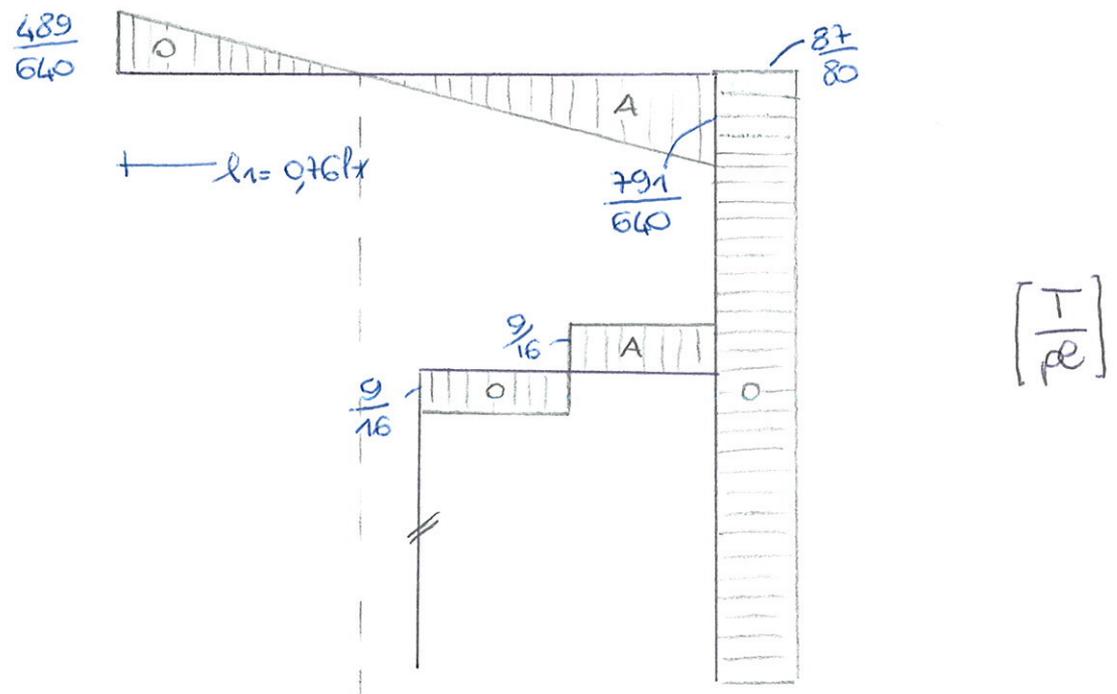
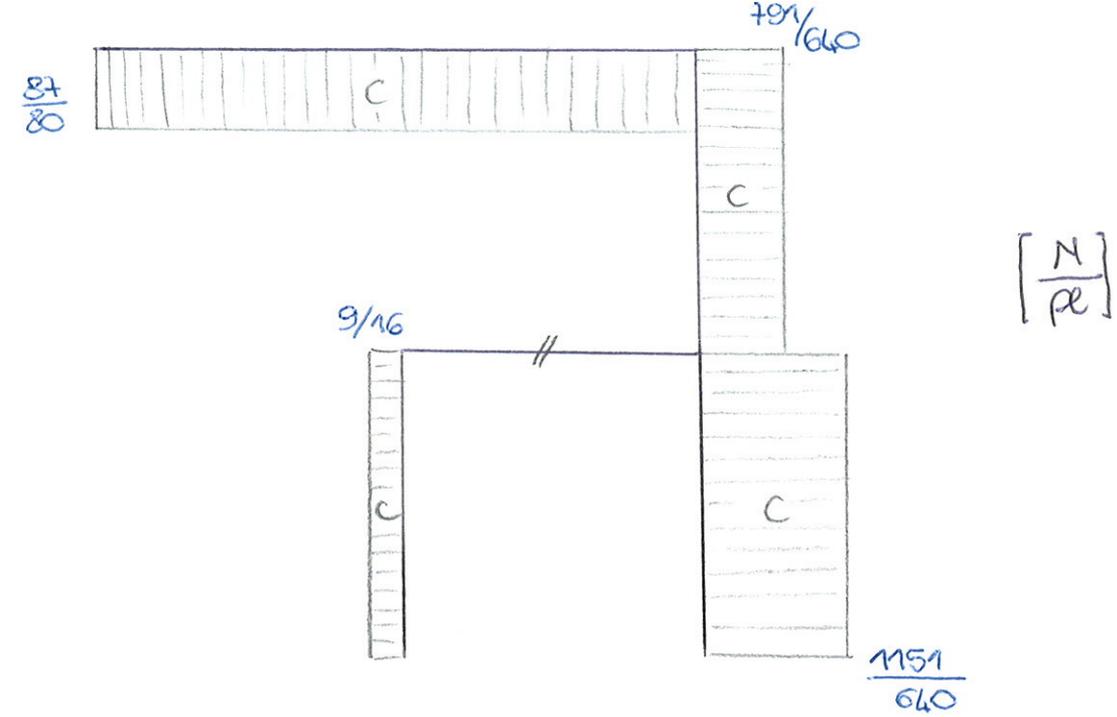
$$M = -\frac{87}{160} \frac{pl^4}{EJ}$$

$$\varphi_B = -\frac{3}{160} \frac{pl^3}{EJ}$$

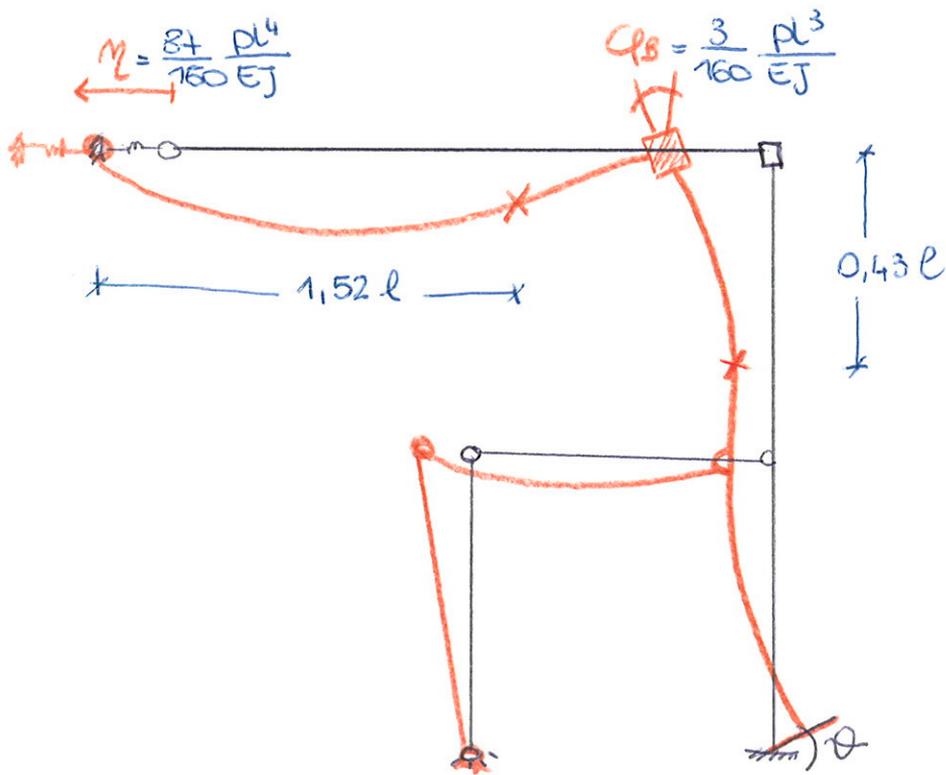
* Equilibrio globale:



* Grafici:



* Deformata:



* Posizione flessi:

. Calcolo $l_1 \rightarrow \frac{489}{640} pl - pl_1 = 0$

$$l_1 = \frac{489}{640} l = 0,76 l$$

. Calcolo $l_2 \rightarrow \frac{489}{640} pl \cdot l_2 - p \cdot \frac{l_2^2}{2} = 0$

$$l_2 = \frac{489}{640} \cdot 2l = \frac{489}{320} l = 1,52 l$$

. Calcolo $l_3 \rightarrow \frac{151}{320} pl^2 - \frac{87}{80} pl \cdot l_3 = 0$

$$l_3 = \frac{151}{320} \cdot \frac{80}{87} l = 0,43 l$$

. Calcolo $M_{\max}(0,76l) \rightarrow \frac{489}{640} pl \frac{489}{640} l - p \left(\frac{489}{640} l \right)^2 \frac{1}{2} =$

$$= \left(\frac{489}{640} \right)^2 pl^2 - \left(\frac{489}{640} \right) \frac{1}{2} pl^2$$

$$= \frac{239121}{819200} pl^2 = 0,29 pl^2$$

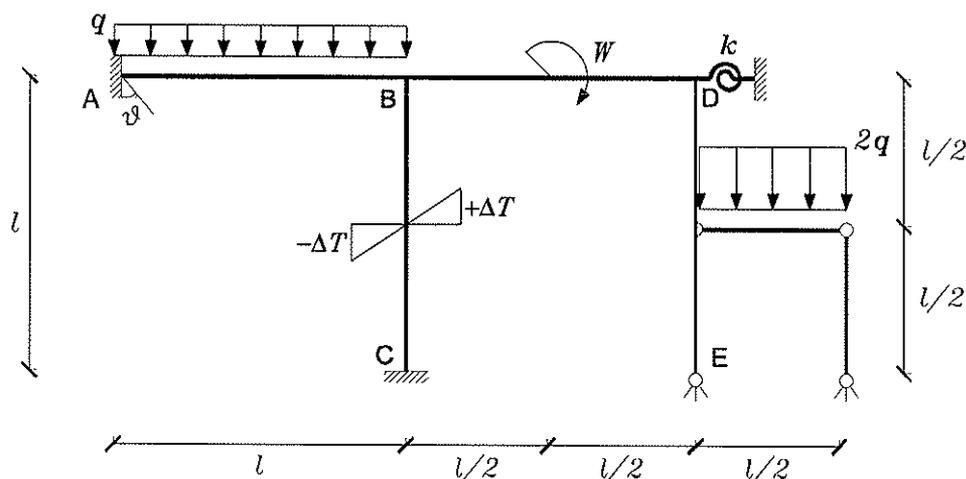
TECNICA DELLE COSTRUZIONI

TEMA ESAME DEL 25 NOVEMBRE 2011

DOCENTE: ING. FAUSTO MINELLI

ESERCITATORE: ING. FRANCESCA FEROLDI

FILA B



$$\frac{\alpha \Delta T}{h} = \frac{1}{6} \frac{ql^2}{EJ}$$

$$k = \frac{EJ}{l}$$

$$\vartheta = \frac{29}{24} \frac{ql^3}{EJ}$$

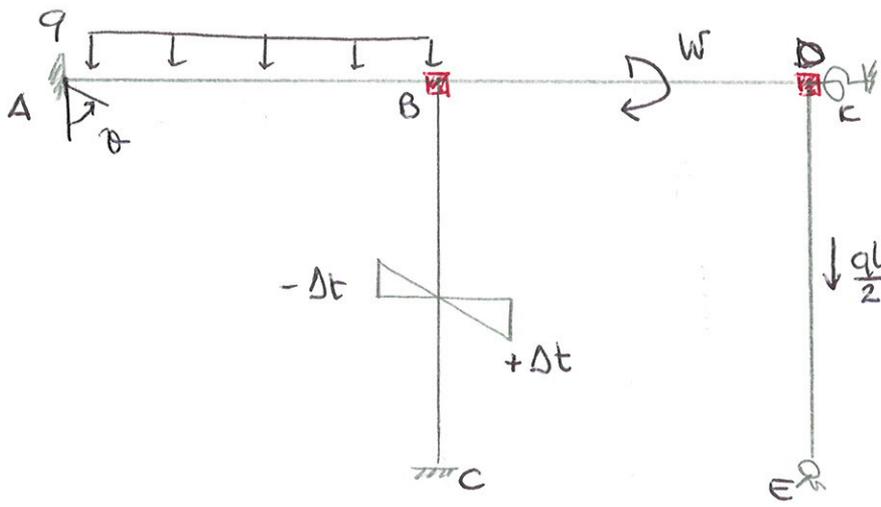
$$W = 2ql^2$$

Dato il telaio in figura

Si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.

(B)

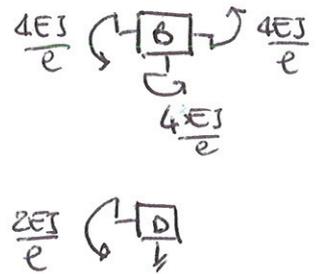
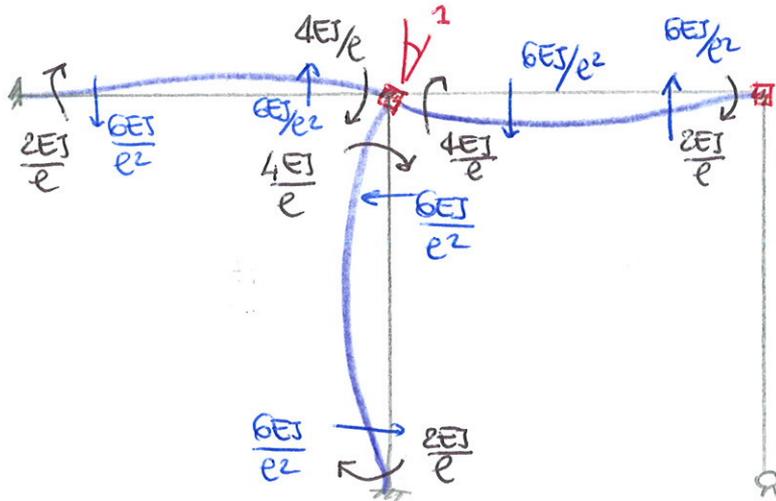


(1)

$$\begin{cases} k = \frac{EJ}{e} \\ W = 2ql^2 \\ \theta = \frac{29}{24} \frac{ql^3}{EJ} \\ \frac{\alpha \Delta t}{h} = \frac{1}{6} \frac{ql^2}{EJ} \end{cases}$$

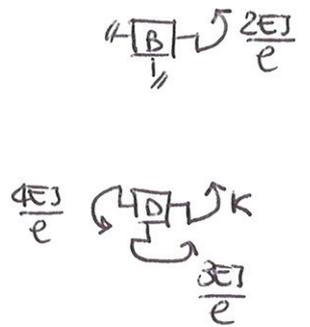
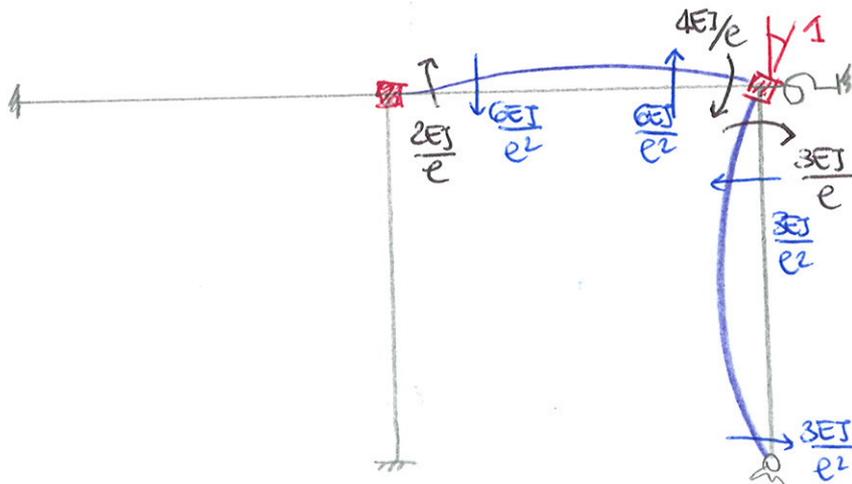
$$\begin{cases} M_{BB} \cdot \varphi_B + M_{BD} \cdot \varphi_D + M_{B0} = 0 \\ M_{DB} \cdot \varphi_B + M_{DD} \cdot \varphi_D + M_{D0} = 0 \end{cases}$$

(1) $\varphi_B = 1$



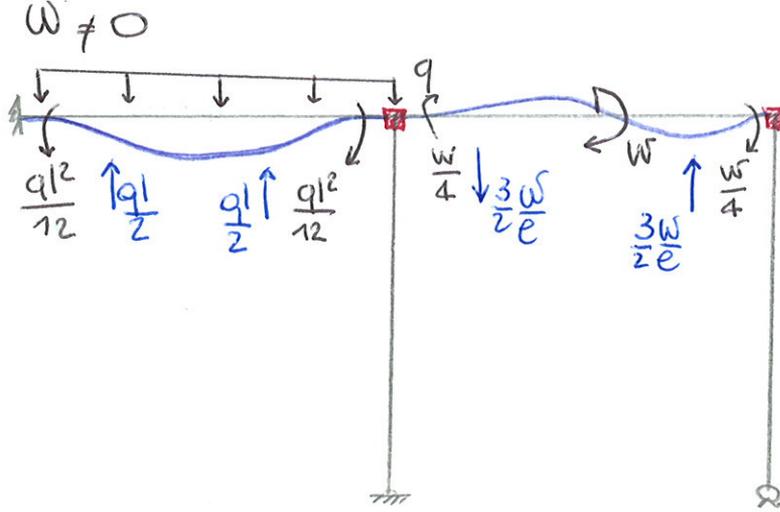
$$\begin{cases} M_{BB} = \frac{12EJ}{e} \\ M_{DB} = \frac{2EJ}{e} \end{cases}$$

(2) $\varphi_D = 1$



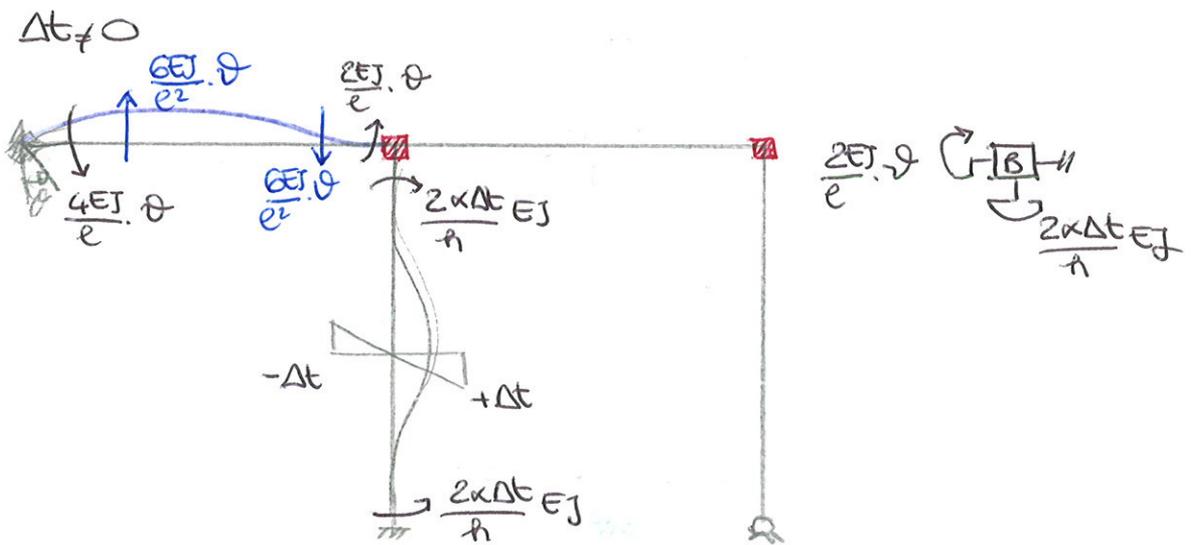
$$\begin{cases} M_{BD} = \frac{2EJ}{e} \\ M_{DD} = \frac{7EJ}{e} + K \end{cases}$$

③ $q \neq 0$



$$\begin{cases} m_{B0} = \frac{ql^2}{12} + \frac{w}{4} \\ m_{C0} = \frac{w}{4} \end{cases}$$

④ $\vartheta \neq 0$



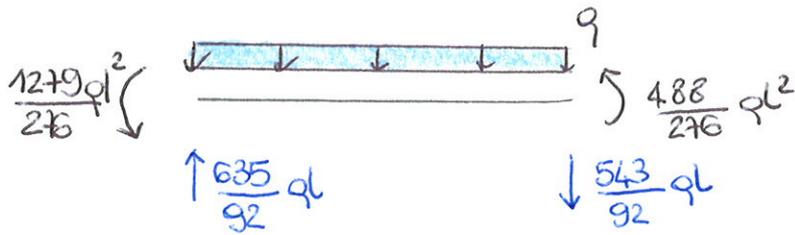
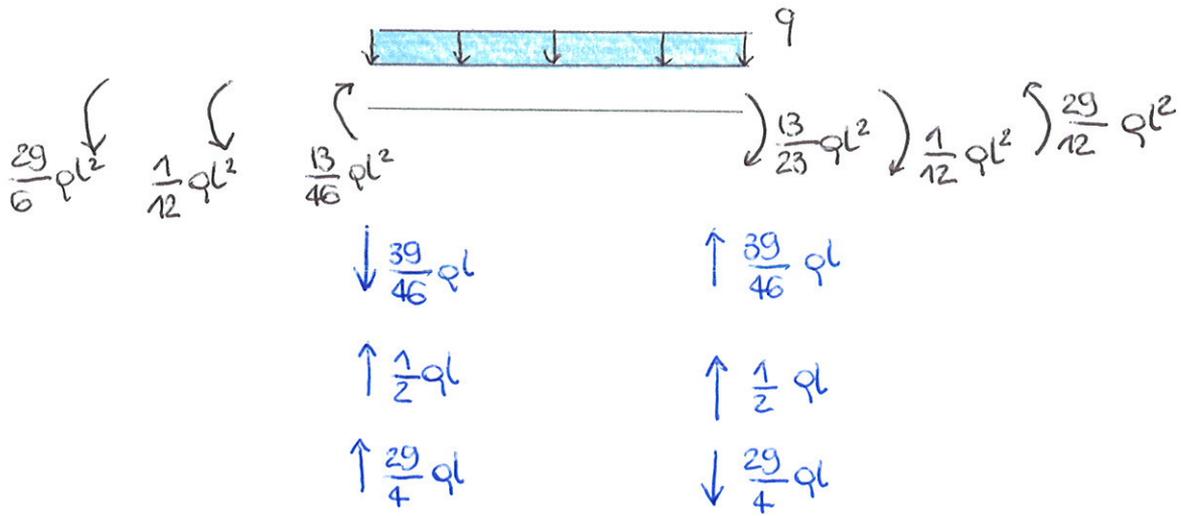
$$\begin{cases} m_{B0} = \frac{2 \times \Delta t}{h} EJ - \frac{2EJ}{e} \cdot \vartheta \\ m_{C0} = 0 \end{cases}$$

Sistema:

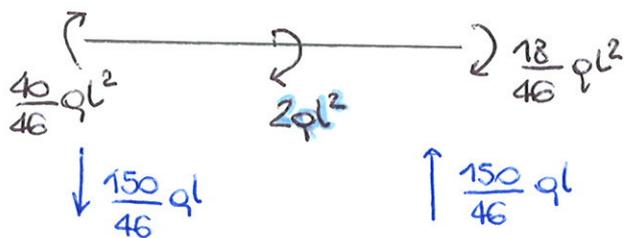
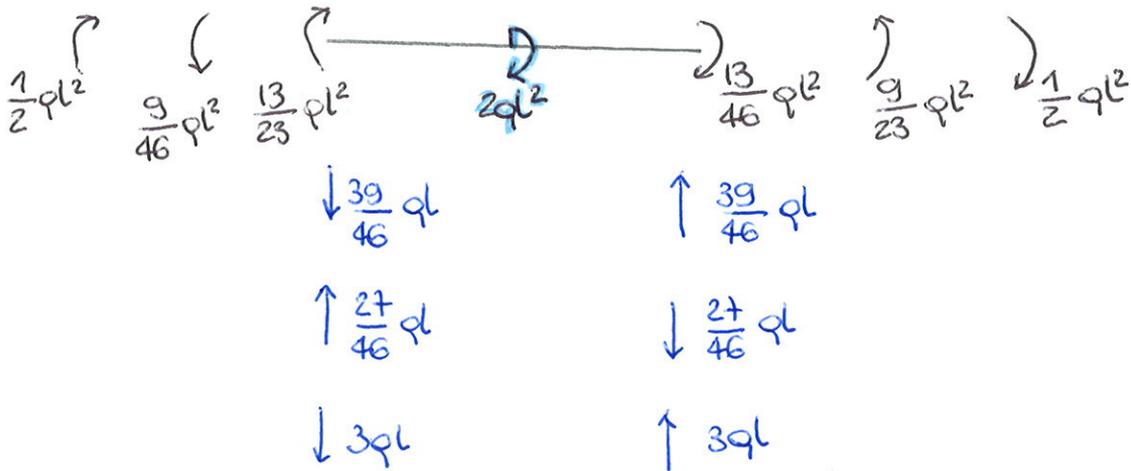
$$\begin{cases} 12 \frac{EJ}{e} \cdot \varphi_B + \frac{2EJ}{e} \cdot \varphi_D + \frac{ql^2}{12} + \frac{w}{4} + \frac{2 \times \Delta t}{h} EJ - \frac{2EJ}{e} \cdot \vartheta = 0 \\ \frac{2EJ}{e} \cdot \varphi_B + (7+k) \frac{EJ}{e} \cdot \varphi_D + \frac{w}{4} = 0 \end{cases}$$

$$\begin{cases} \varphi_B = \frac{13}{92} \frac{ql^3}{EJ} \\ \varphi_D = -\frac{9}{92} \frac{ql^3}{EJ} \end{cases}$$

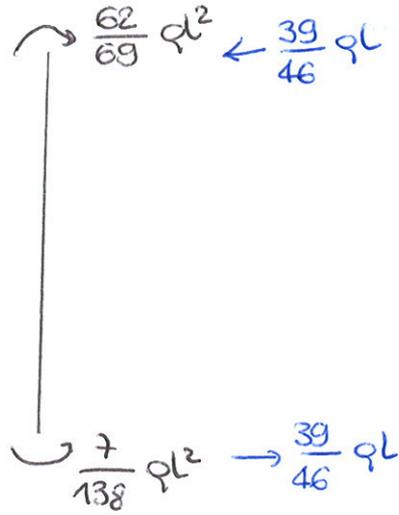
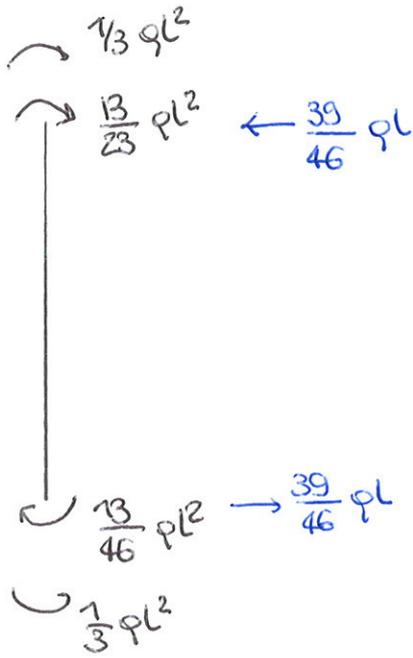
Asta AB



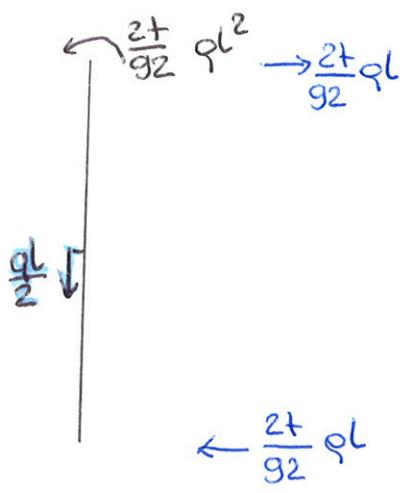
Asta BD



Asta BC

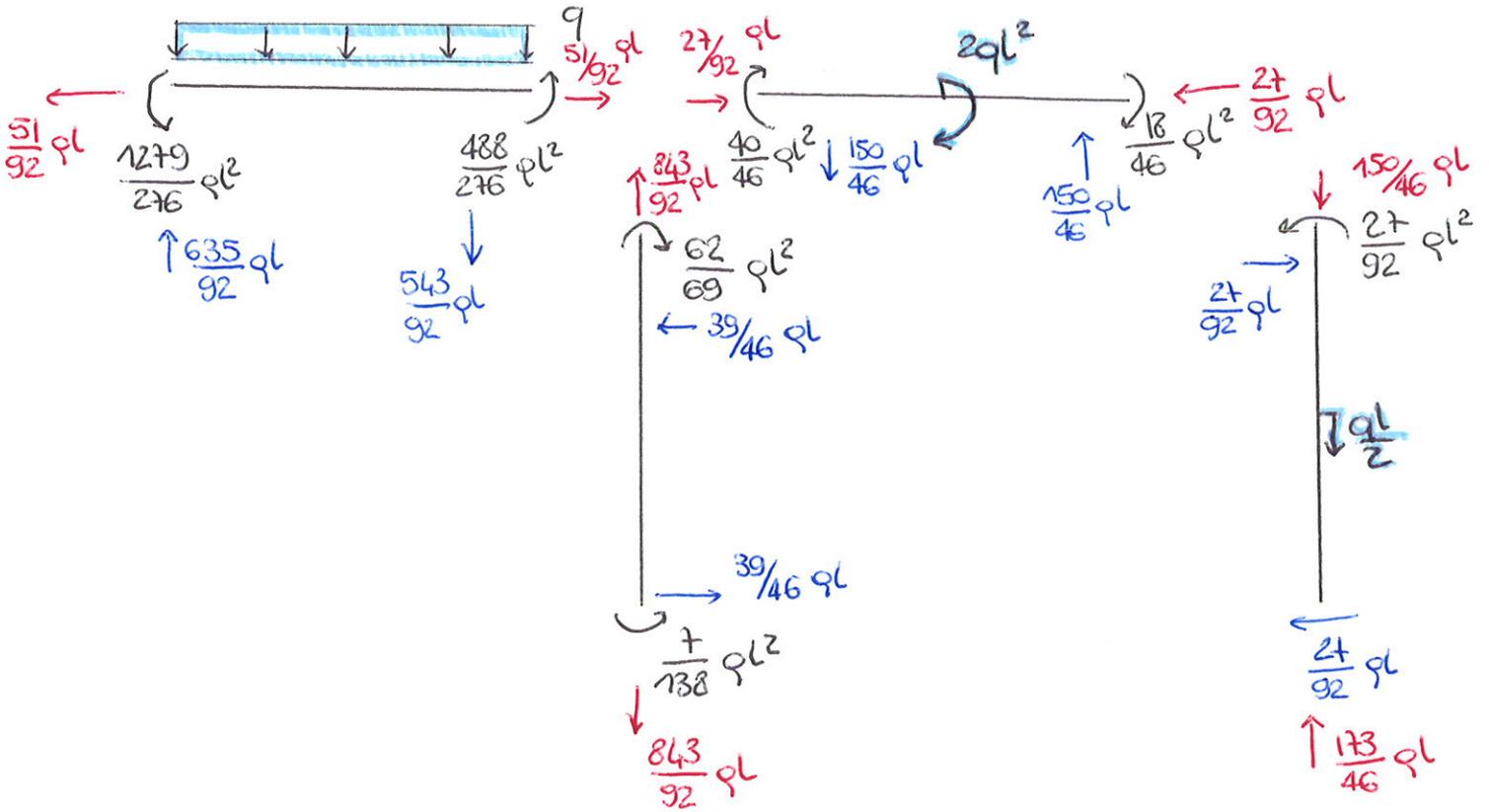


Asta DE

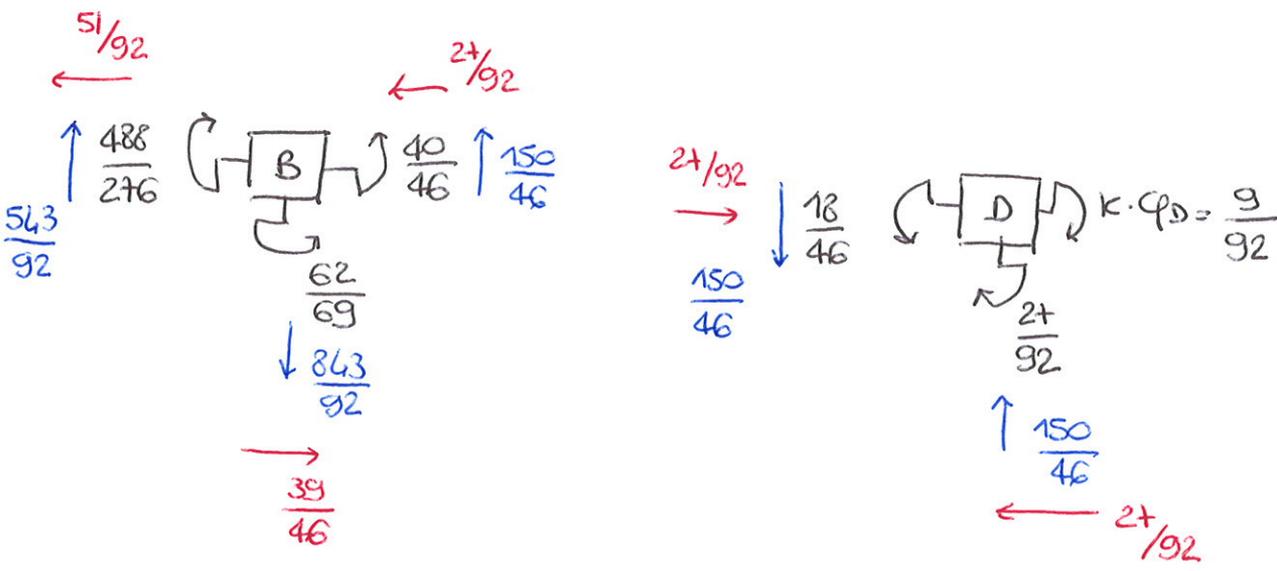


Equilibrio globale

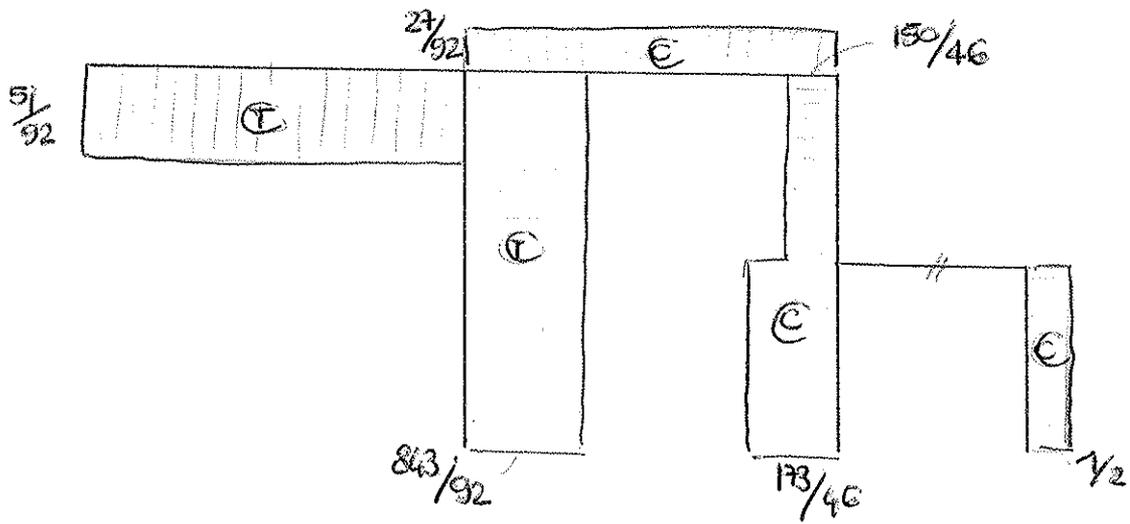
(3)



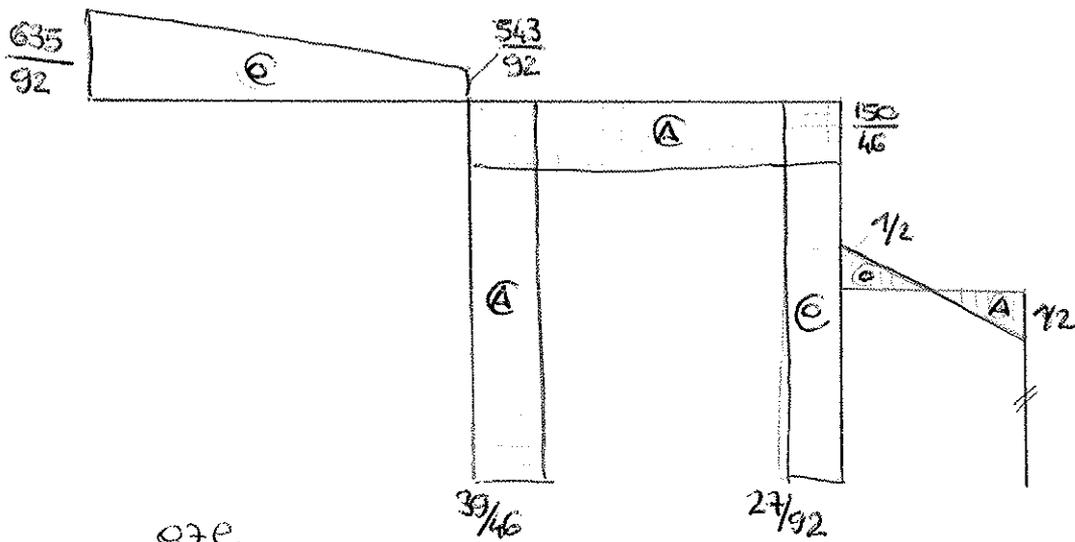
Equilibrio ai nodi:



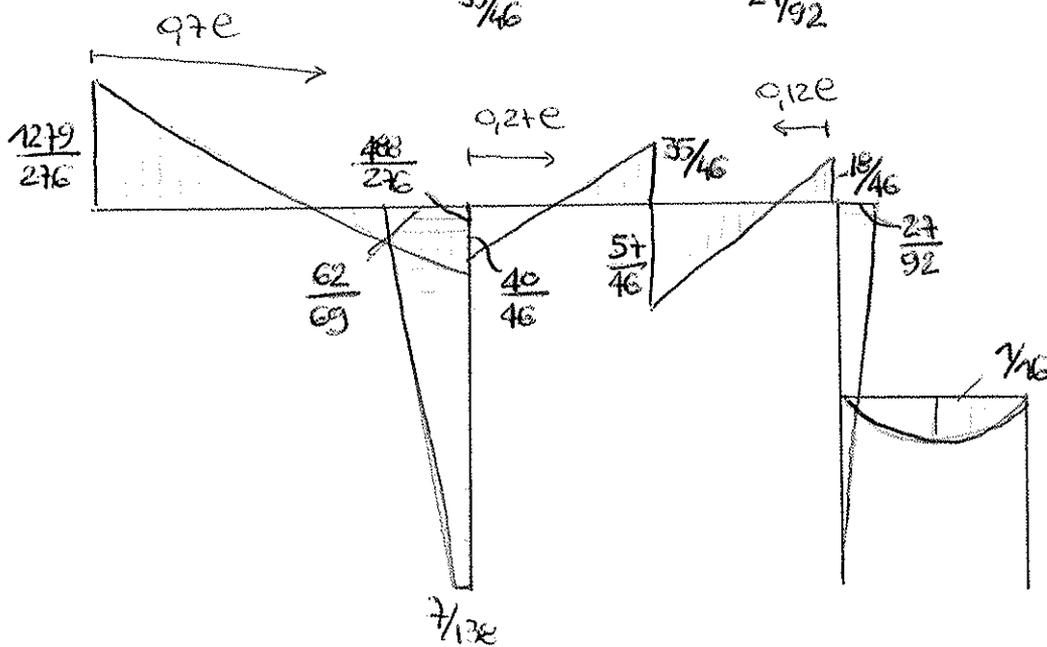
Diagrammi



$\frac{N}{ql}$

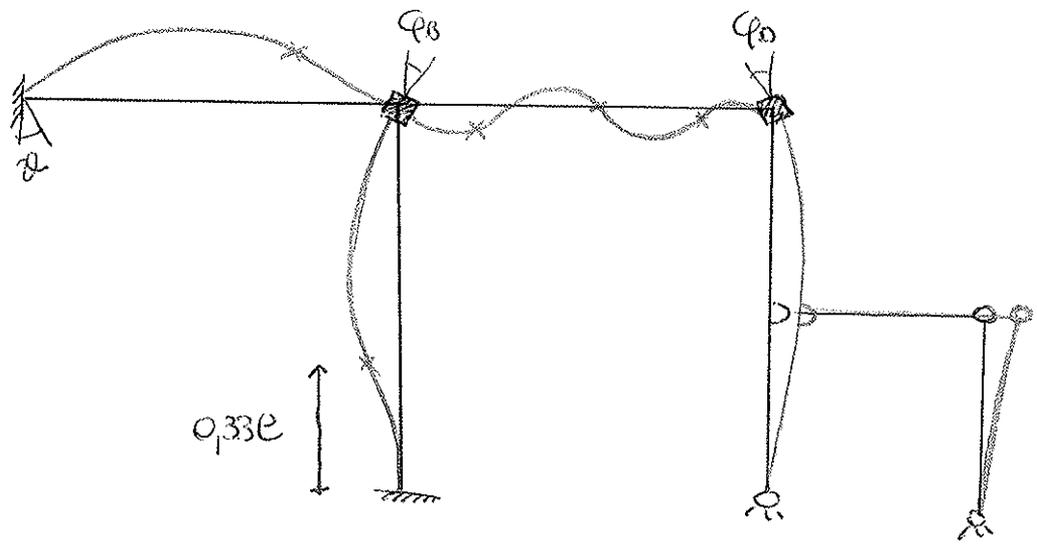


$\frac{V}{ql}$

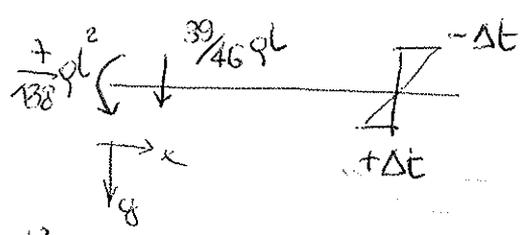


$\frac{M}{ql^2}$

Deformata



$$y''(x) = -\frac{M(x)}{EI} + \frac{2x\Delta t}{h}$$



$$y''(x) = -\frac{1}{EI} \left(-\frac{7}{138} ql^2 - \frac{39}{46} qlx \right) - \frac{1}{3} \frac{ql^2}{EI}$$

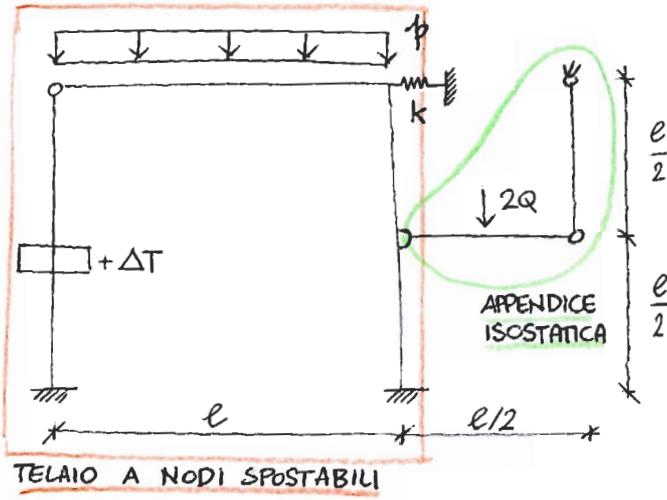
$$y''(x) = \frac{7}{138} ql^2 + \frac{1}{3} ql^2 + \frac{39}{46} qlx$$

$$y''(x) > 0$$

$$-\frac{39}{138} ql^2 + \frac{39}{46} qlx > 0$$

$$x > \frac{13}{46} \cdot \frac{46}{39} e$$

$$x > 0,33e$$

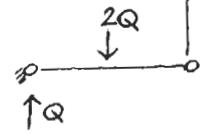


$$\alpha \Delta T = \frac{11}{8} \frac{pl^3}{EJ}$$

$$k = \frac{3EJ}{e^3}$$

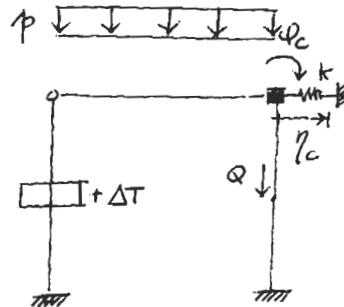
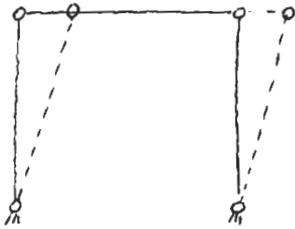
$$Q = \frac{3}{2} pl$$

Appendice isostatica: ↑Q

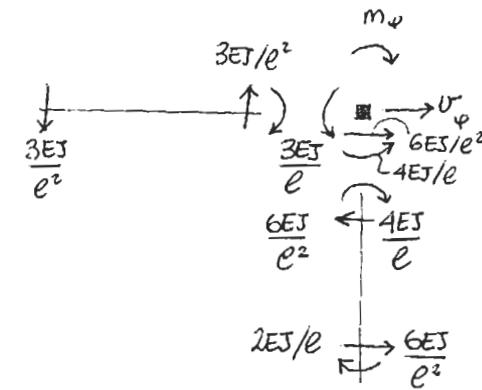
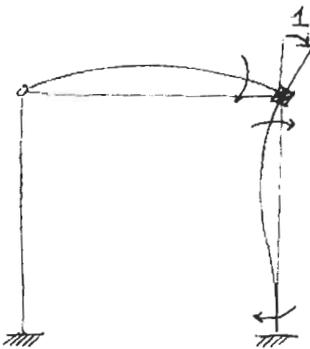


2 INCOGNITE:

$$\varphi_c, \eta_c$$



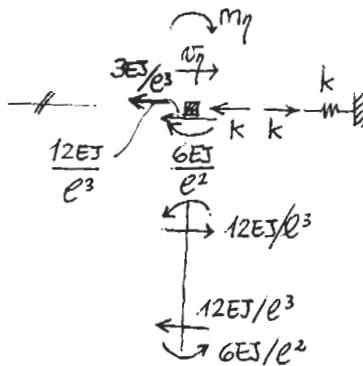
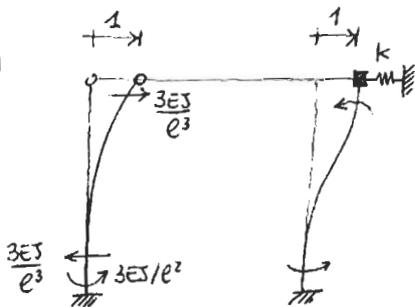
$\varphi_c = 1$)



$$m_\varphi = \frac{7EJ}{e}$$

$$v_\varphi = -\frac{6EJ}{e^2}$$

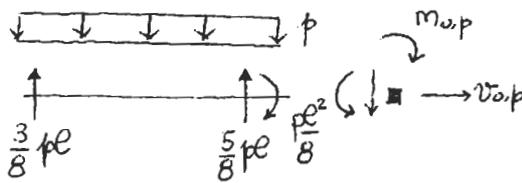
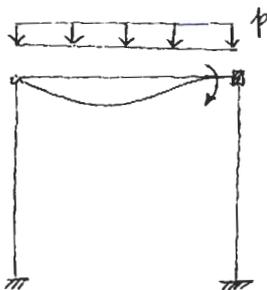
$\eta_c = 1$)



$$m_\eta = -\frac{6EJ}{e^2}$$

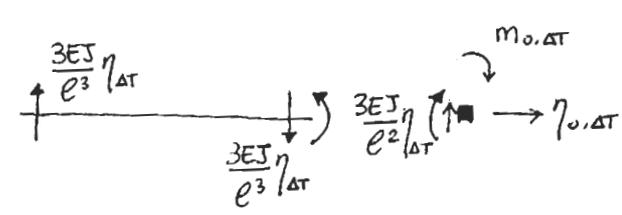
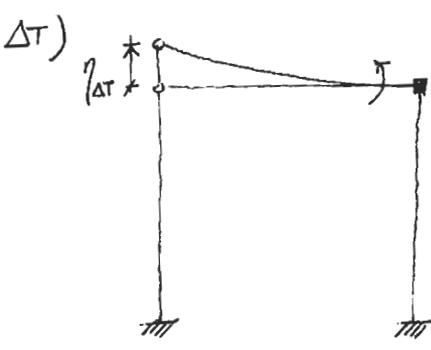
$$v_\eta = \frac{12EJ}{e^3} + k = \frac{18EJ}{e^3}$$

p)



$$m_{o,p} = pl^2/8$$

$$v_{o,p} = \phi$$



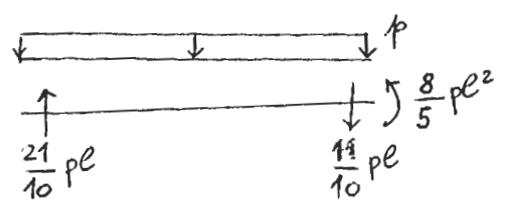
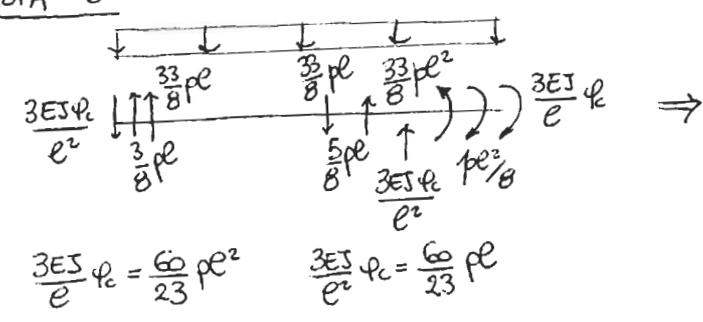
$$m_{0,\Delta T} = -\frac{3EJ}{e^2} \eta_{\Delta T} = \dots \quad \eta_{\Delta T} = \alpha \Delta T \cdot e = \frac{11}{8} \frac{pe^4}{EJ} = \dots = -\frac{33}{8} pe^2$$

SISTEMA RISOLVENTE:

$$\begin{cases} \frac{7EJ}{e} \varphi_c - \frac{6EJ}{e^2} \eta_c + \frac{pe^2}{8} - \frac{33}{8} pe^2 = 0 \Rightarrow \frac{7EJ}{e} \varphi_c - \frac{6EJ}{e^2} \eta_c = 4pe^2 \\ -\frac{6EJ}{e^2} \varphi_c + \frac{18EJ}{e^3} \eta_c = 0 \Rightarrow \eta_c = \frac{1}{3} \varphi_c \cdot e \end{cases}$$

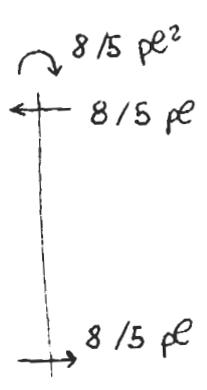
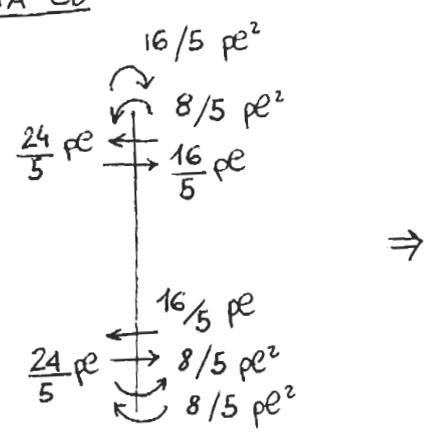
$$\frac{7EJ}{e} \varphi_c - \frac{2EJ}{e} \varphi_c = 4pe^2 \Rightarrow \boxed{\varphi_c = +\frac{4}{5} \frac{pe^3}{EJ}} \quad e \quad \boxed{\eta_c = +\frac{4}{15} \frac{pe^4}{EJ}}$$

ASTA BC

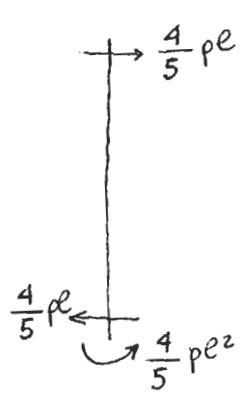


$$\frac{3EJ}{e} \varphi_c = \frac{60}{23} pe^2 \quad \frac{3EJ}{e^2} \varphi_c = \frac{60}{23} pe$$

ASTA CD



ASTA AB



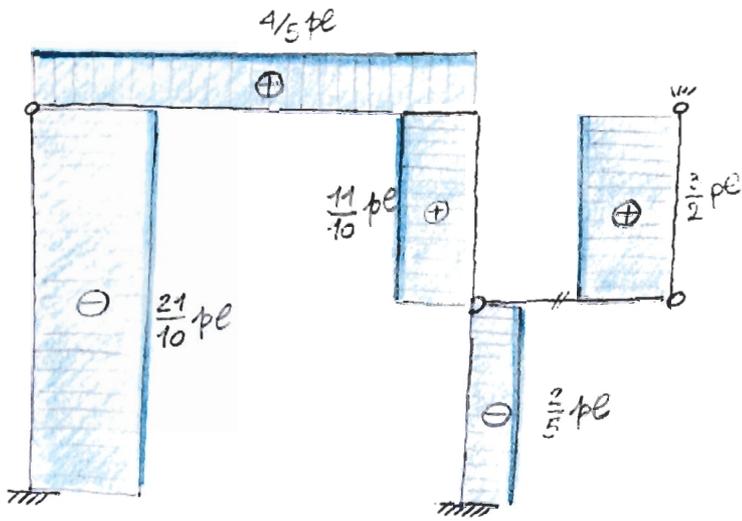
EQUILIBRIO AL NODO C:

$$\begin{array}{c} N_{bc} \quad N_{bc} \\ \leftarrow \quad \rightarrow \end{array} \quad \leftarrow k \eta_c = \frac{3EJ}{e^3} \cdot \frac{4}{15} \cdot \frac{pe^4}{EJ} = \frac{4}{5} pe \Rightarrow N_{bc} = \frac{4}{5} pe; \text{ CONFERMA L'EQUILIBRIO DEL NODO B}$$

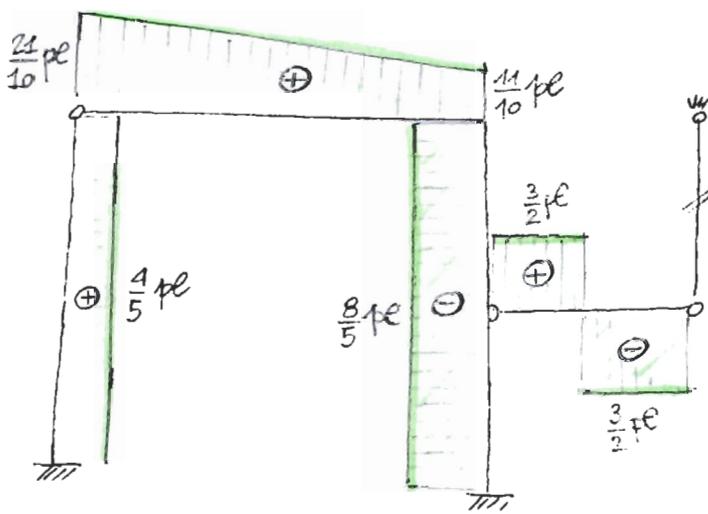
$$\frac{3EJ}{e^3} \eta_c = \frac{4}{5} pe$$

DIAGRAMMI AZIONI INTERNE

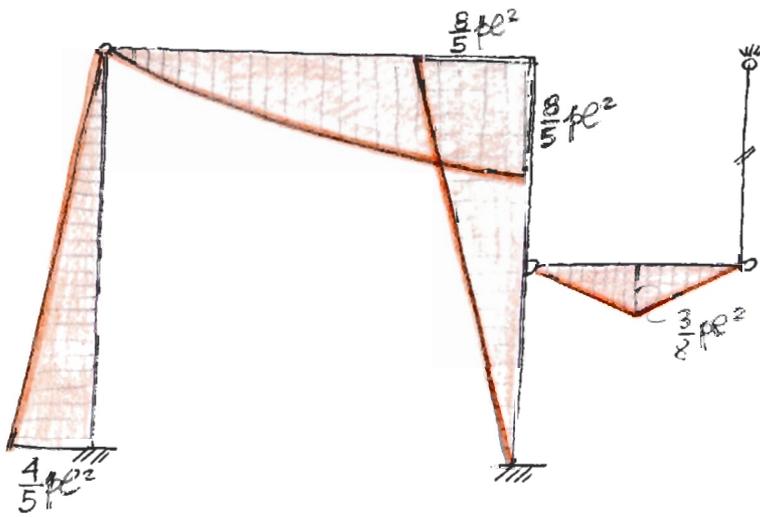
(N) ← ⊠ →



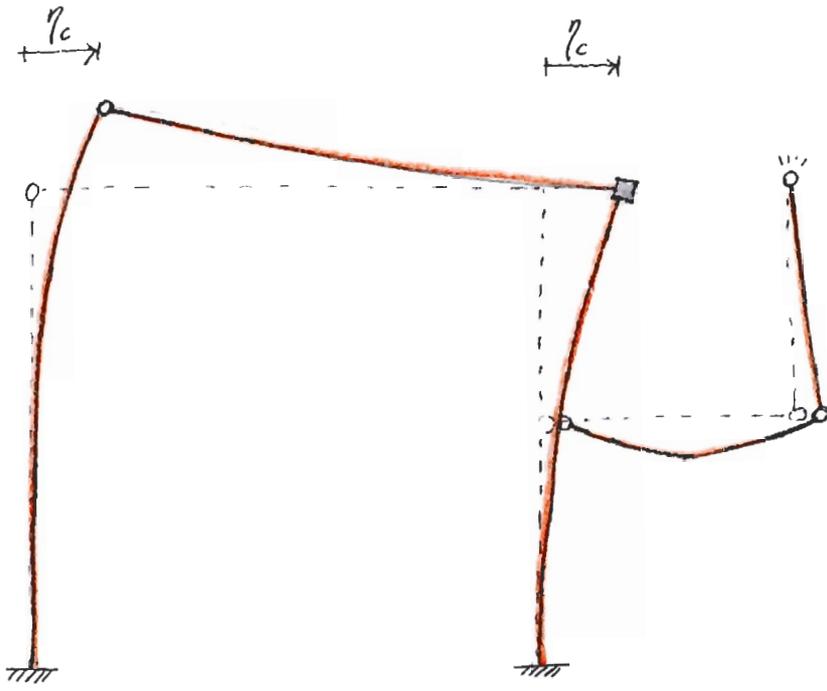
(V) ↑ ⊠ ↓



(M)



DEFORMATA QUALITATIVA



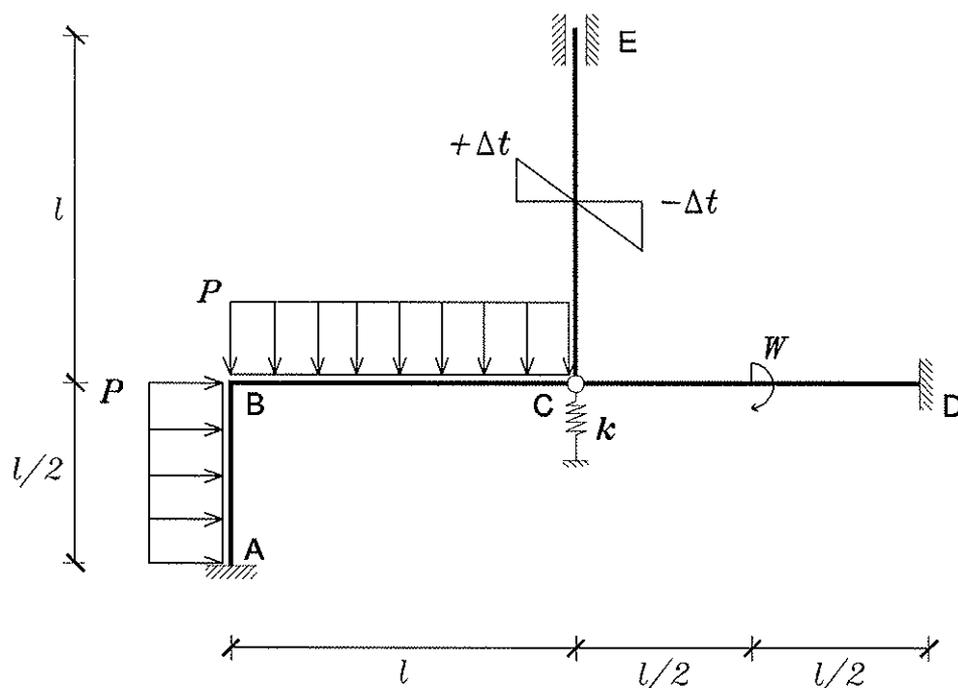
FONDAMENTI DI PROGETTAZIONE STRUTTURALE

ESAME 30/01/2012, 3h

Ing. F. Minelli

Nome: Cognome: n.matr.:

ESERCIZIO 1 (20 punti)



$$W = \frac{1}{18} Pl^2$$

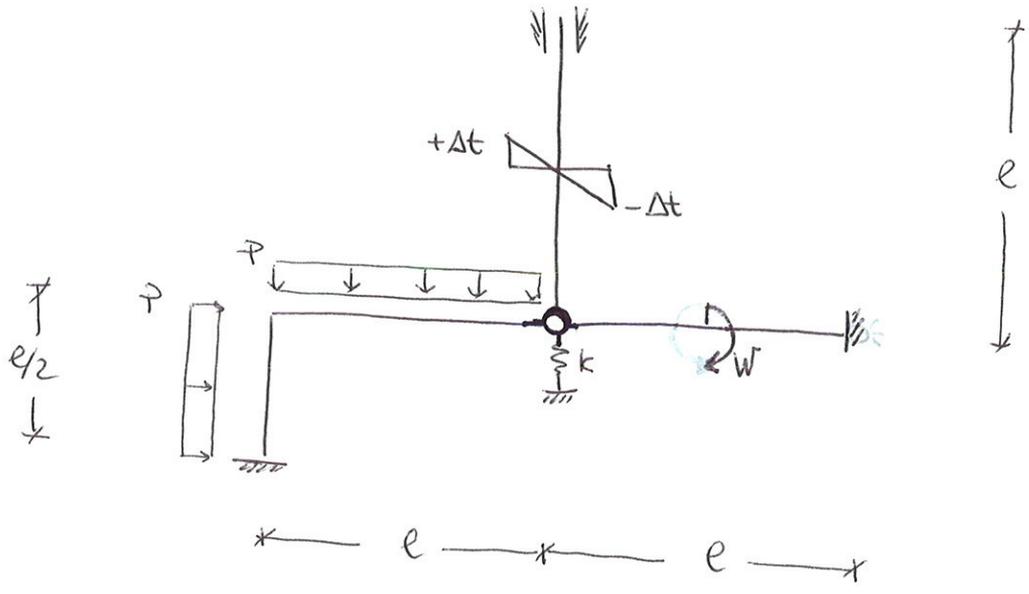
$$k = 3 \frac{EJ}{l^3}$$

$$\frac{\alpha \Delta t}{h} = \frac{5 Pl^2}{12 EJ}$$

Dato il telaio in figura

Si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi)
2. Taglio
3. Azione assiale
4. Deformata qualitativa con posizione dei flessi

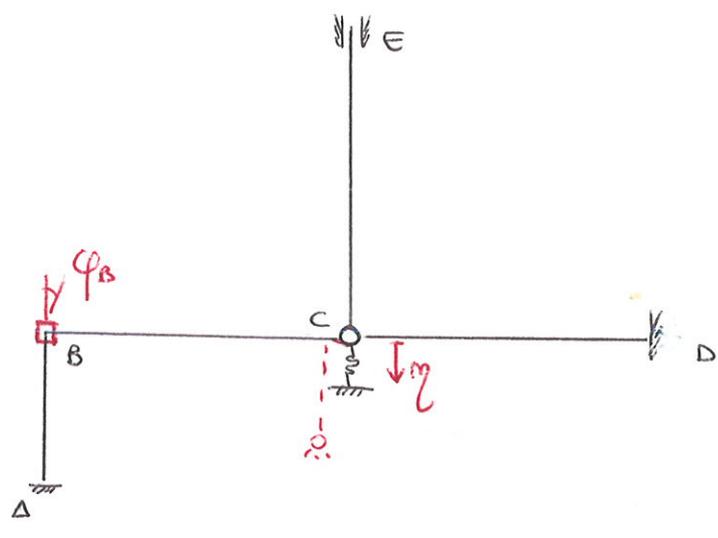


$$\frac{\alpha \Delta t}{R} = \frac{5}{12} \frac{\rho l^2}{EJ}$$

$$K = \frac{3EJ}{l^3}$$

$$W = \frac{1}{18} \rho l^2$$

Struttura a nodi spostabili

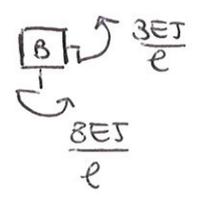
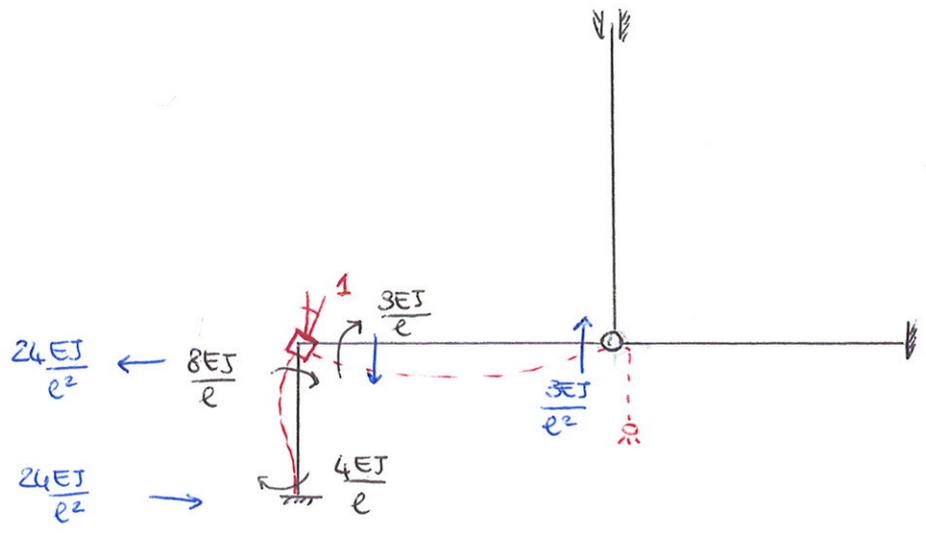


$$\begin{cases} m_{BS} \cdot \phi_B + m_{B\eta} \cdot \eta + m_{B0} = 0 \\ r_{\eta B} \cdot \phi_B + r_{\eta\eta} \cdot \eta + r_{\eta 0} = 0 \end{cases}$$

Convenzioni di segno:

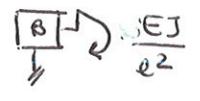
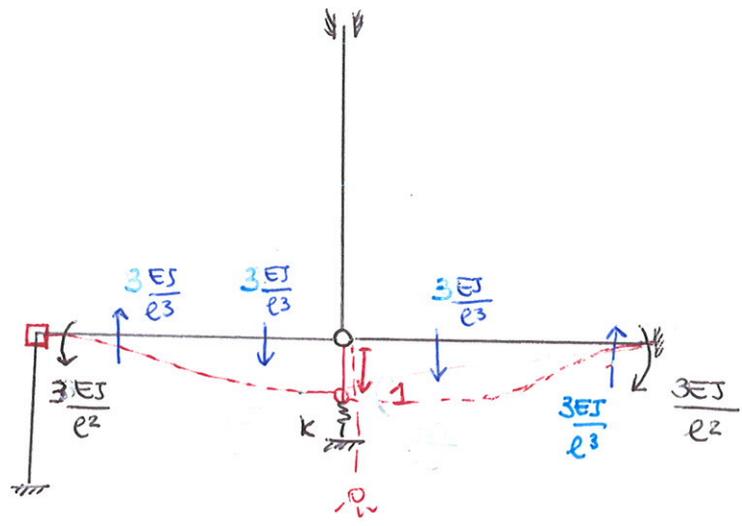


① $\varphi_B = 1$



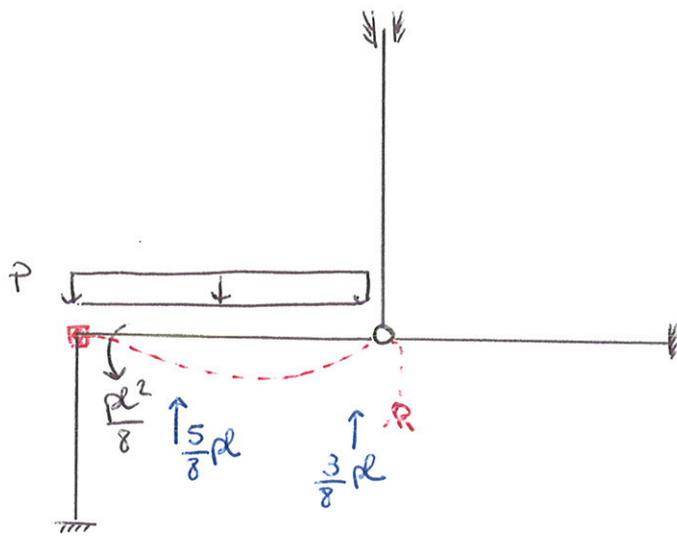
$$\begin{cases} M_{BB} = 11 \frac{EJ}{l} \\ R_{\eta B} = - \frac{3EJ}{l^2} \end{cases}$$

② $\eta = 1$

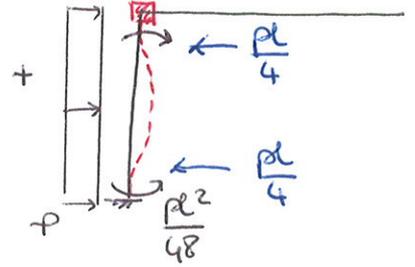


$$\begin{cases} M_{B\eta} = - \frac{3EJ}{l^2} \\ R_{\eta\eta} = \frac{6EJ}{l^3} + K \end{cases}$$

③ $P \neq 0$

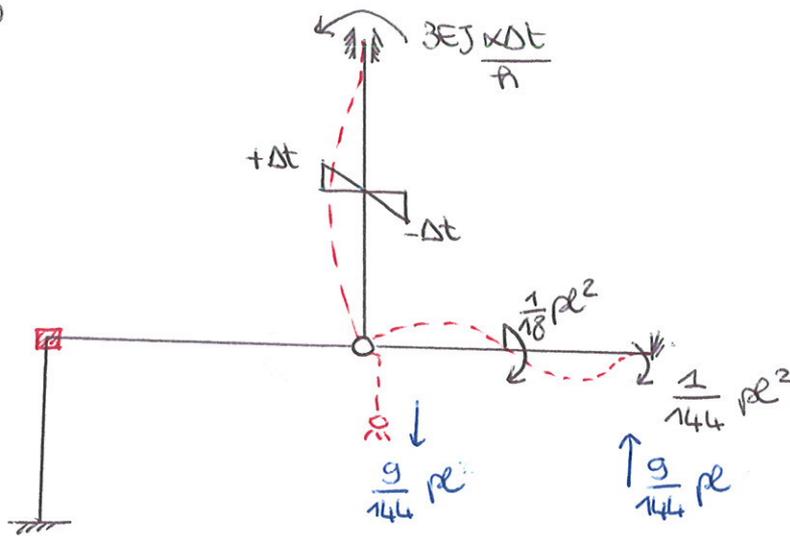


$$\left. \begin{array}{l} \square B \curvearrowright \frac{Pl^2}{8} \\ \curvearrowright \frac{Pl^2}{48} \end{array} \right\}$$



$$\left. \begin{array}{l} M_{Bo} = -\frac{5}{48} Pl^2 \\ R_{mp} = -\frac{3}{8} Pl \end{array} \right\}$$

④ $\Delta t \neq 0$ $W \neq 0$

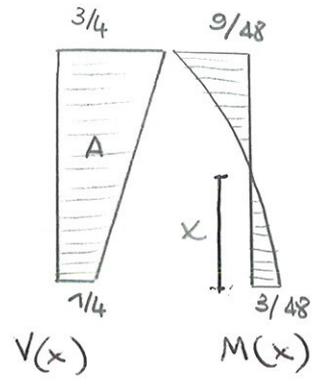
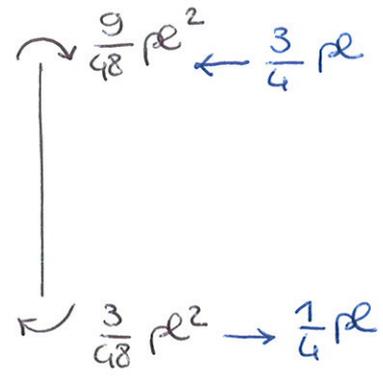
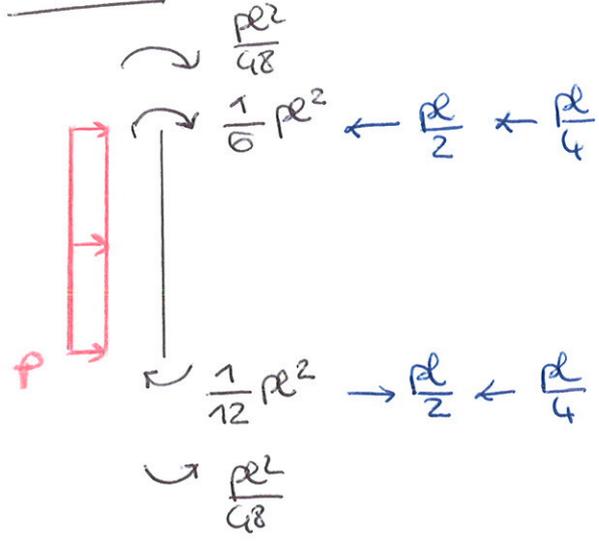


$$\left. \begin{array}{l} M_{Bo} = 0 \\ R_{mp} = \frac{9}{144} Pl \end{array} \right\}$$

Sistema isovalente:

$$\left. \begin{array}{l} \frac{PlEJ}{e} \varphi_B - \frac{3EJ}{e^2} \eta - \frac{5}{48} Pl^2 = 0 \\ -\frac{3EJ}{e^2} \varphi_B + \frac{9EJ}{e^3} \eta - \frac{3}{8} Pl + \frac{9}{144} Pl = 0 \end{array} \right\} \begin{array}{l} \varphi_B = \frac{1}{48} \frac{Pl^3}{EJ} \\ \eta = \frac{1}{24} \frac{Pl^4}{EJ} \end{array}$$

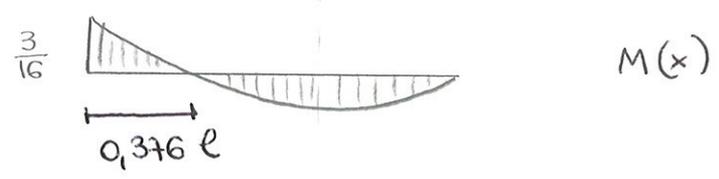
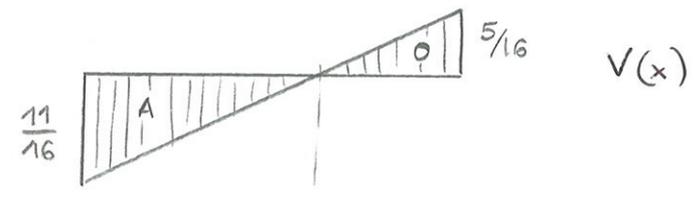
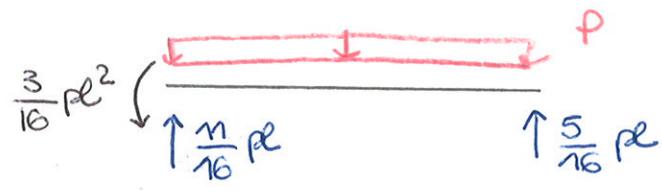
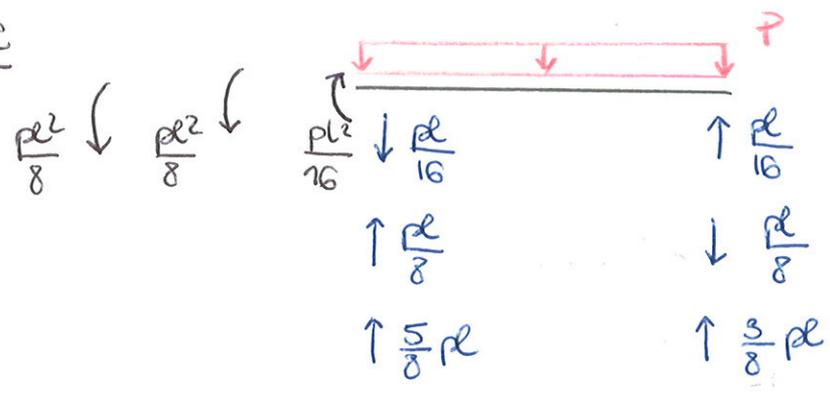
Asta AB



$$M(x) = 0 \rightarrow \frac{3}{48} pl^2 - \frac{1}{4} plx - \frac{px^2}{2} = 0$$

$$x = 0,183 l$$

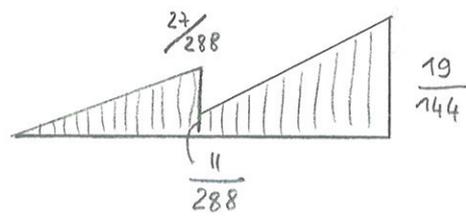
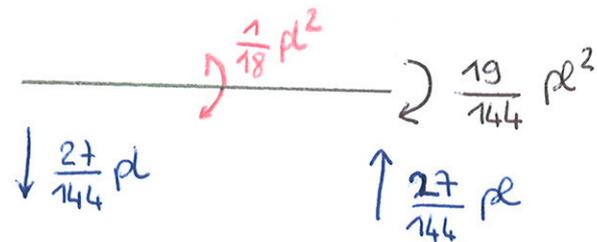
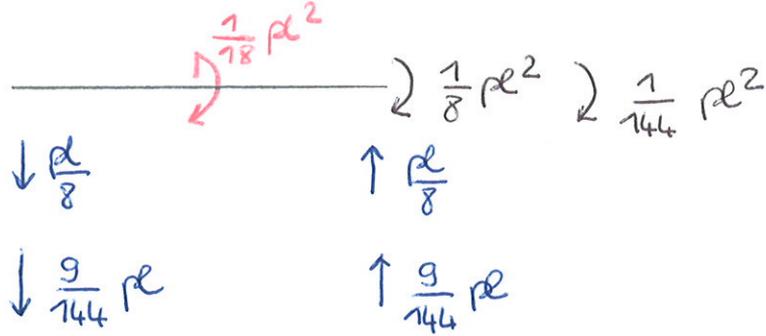
Asta BC



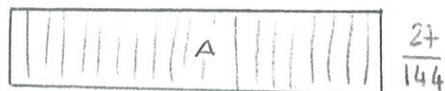
$$M(\bar{x}) = 0 \rightarrow \frac{11}{16} pl\bar{x} - \frac{3}{16} pl^2 - \frac{P\bar{x}^2}{2} = 0$$

$$\bar{x} = \frac{-\frac{11}{16} \pm \sqrt{0,472 - 4 \cdot (-1/2) \cdot (-3/16)}}{(-1)} = \begin{cases} +1 l \\ 0,376 l \end{cases}$$

Asta CD

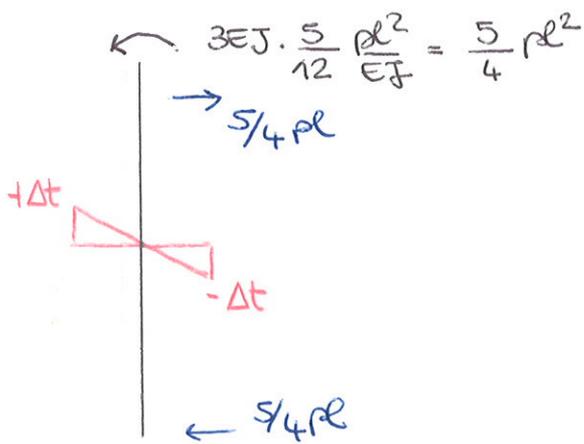


$M(x)$



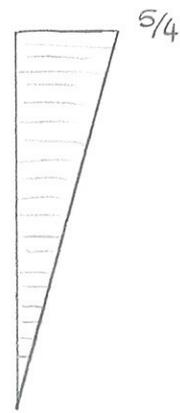
$V(x)$

Asta CE

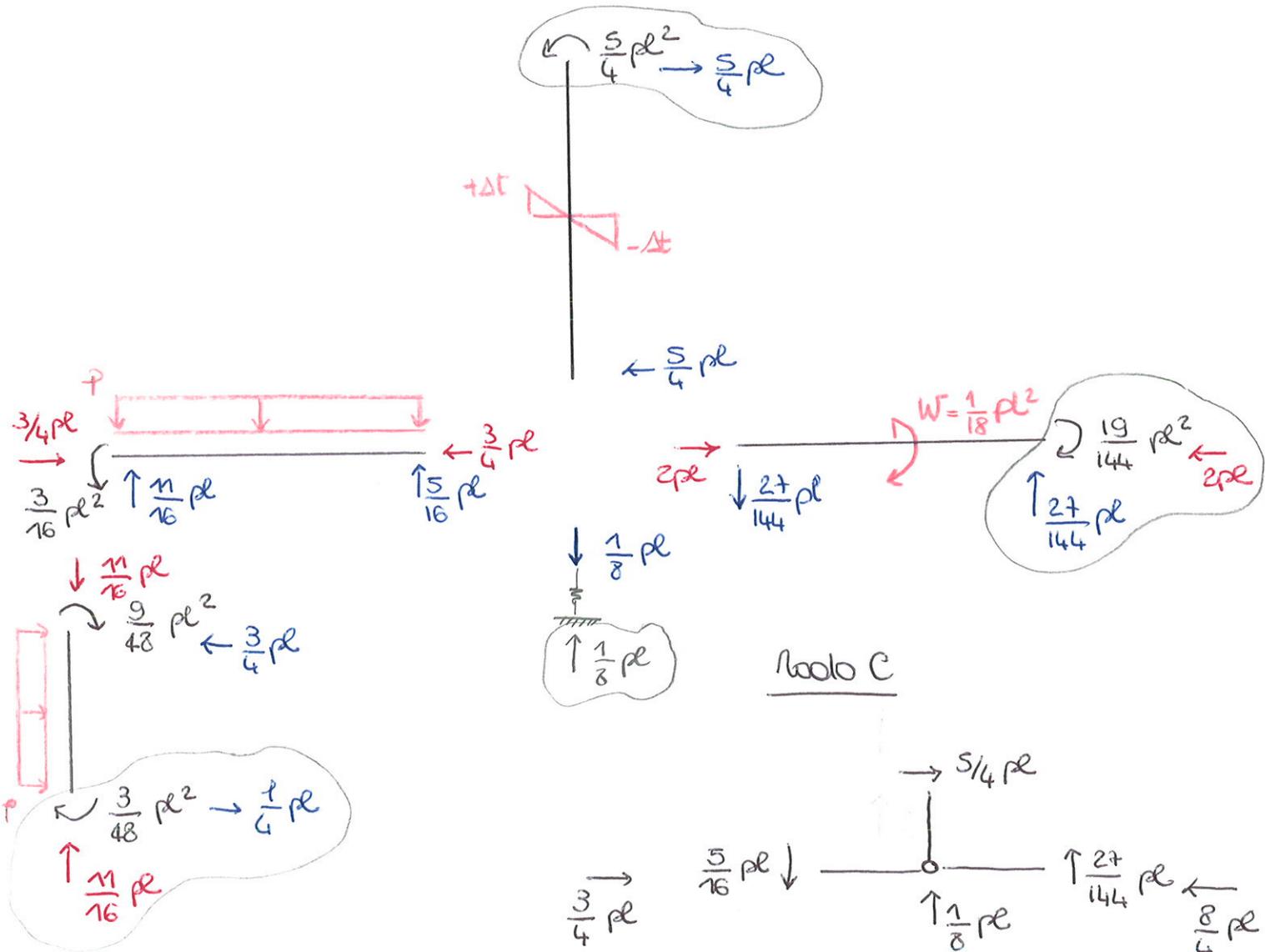


$5/4$

$V(x)$



$M(x)$



Equilibrio globale

$$\sum v = 0 \rightarrow \frac{11}{16}pl - pl + \frac{27}{144}pl + \frac{1}{8}pl = 0$$

$$\frac{99 - 144 + 27 + 18}{144}pl = 0 \quad \text{OK}$$

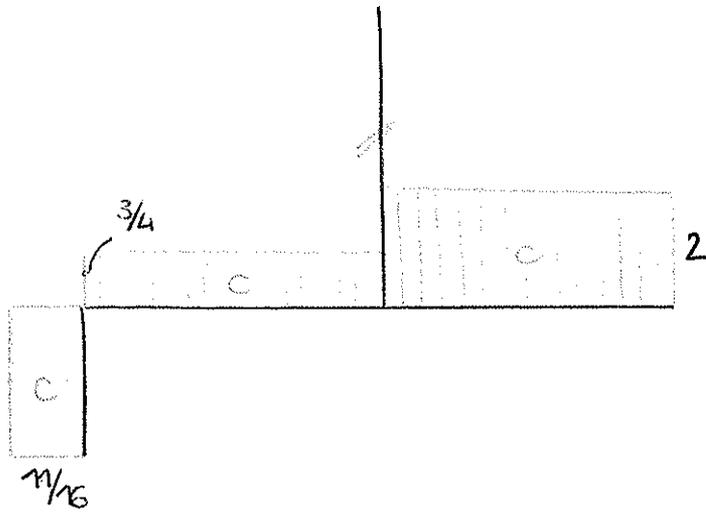
$$\sum H = 0 \rightarrow \frac{pl}{2} + \frac{1}{4}pl + \frac{5}{4}pl - \frac{8}{4}pl = 0$$

$$\frac{2 + 1 + 5 - 8}{4}pl = 0 \quad \text{OK}$$

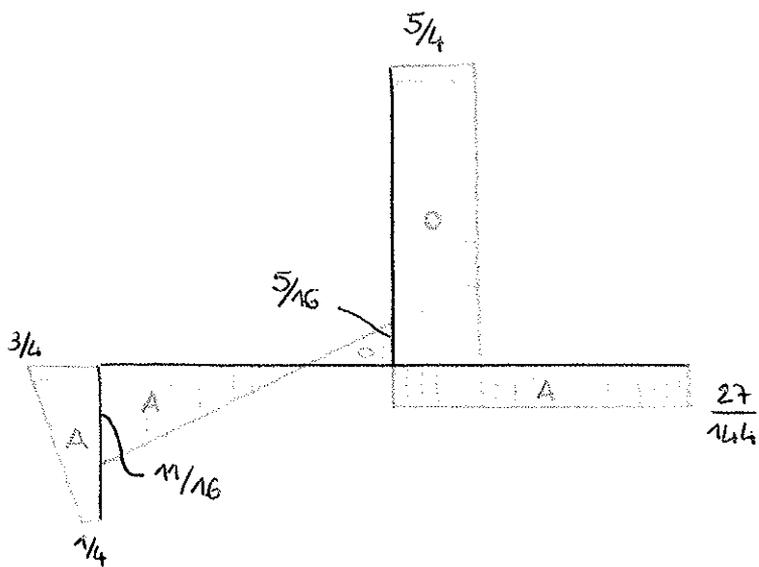
$$\begin{aligned} \sum M(A) = 0 \rightarrow & \frac{3}{48}pl^2 - \frac{1}{8}pl^2 - \frac{1}{8}pl^2 - \frac{54}{144}pl^2 + \frac{19}{144}pl^2 - \frac{5}{4}pl^2 + \frac{5}{4}pl + \frac{1}{18}pl^2 + \\ & + \frac{pl^2}{2} - \frac{pl^2}{8} = 0 \end{aligned}$$

$$\frac{9 - 36 - 54 + 19 - 180 + 180 + 8 + 72 - 18}{144}pl^2 = 0 \quad \text{OK}$$

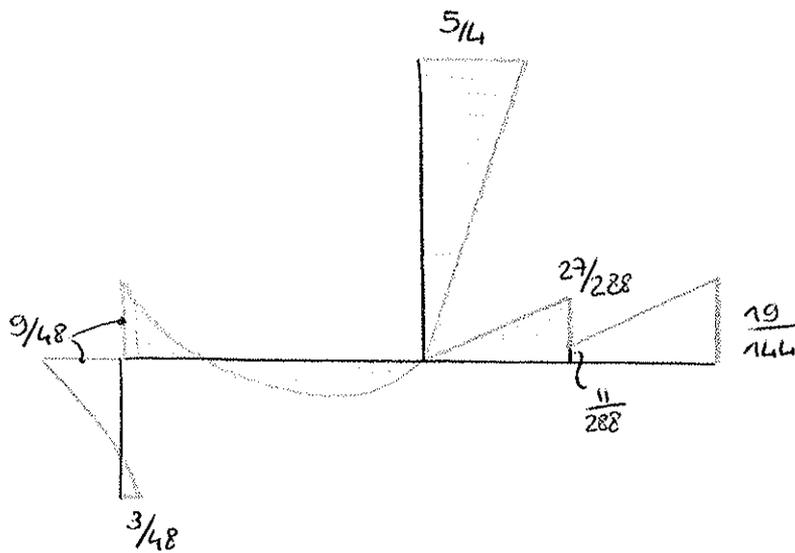
Diagrammi



$$\left[\frac{Z}{I} \right]$$

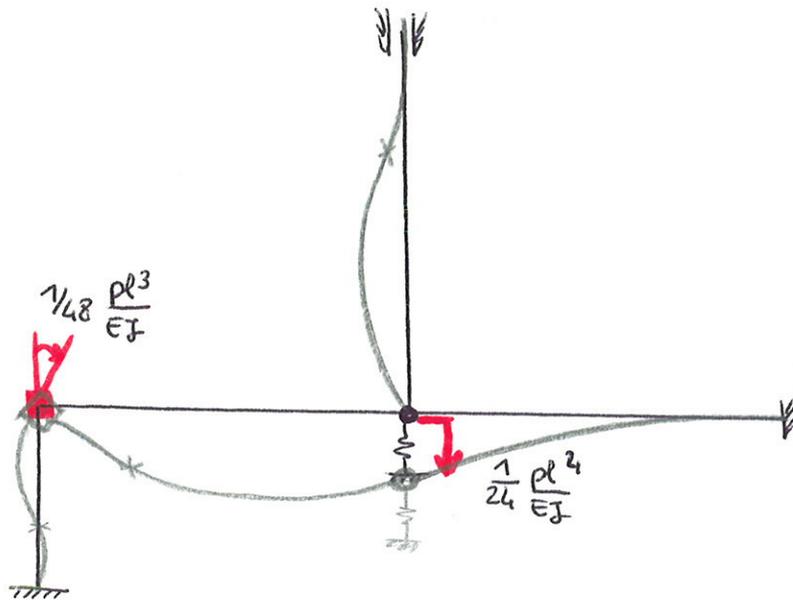


$$\left[\frac{V}{I} \right]$$

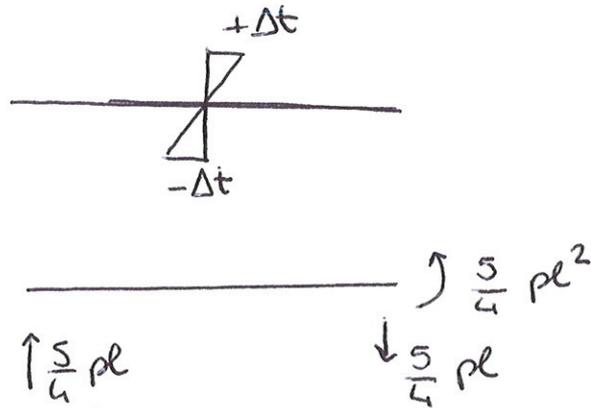


$$\left[\frac{M}{I} \right]$$

Deformato



Calcolo deformato dell'asta CE:



$$y''(x) = -\frac{M(x)}{EI} + \frac{2x\Delta t}{l}$$

$$y''(x) = -\frac{1}{EI} \left(\frac{5}{4} Pl x \right) + \frac{5}{6} \frac{Pl^2}{EI}$$

$$-\frac{5}{4} \frac{Pl x}{EI} + \frac{5}{6} \frac{Pl^2}{EI} > 0$$

$$-\frac{5}{4} x > -\frac{5}{6} l$$

$$x < \frac{2}{3} l$$

$$x < 0,66l$$

curvatura positiva

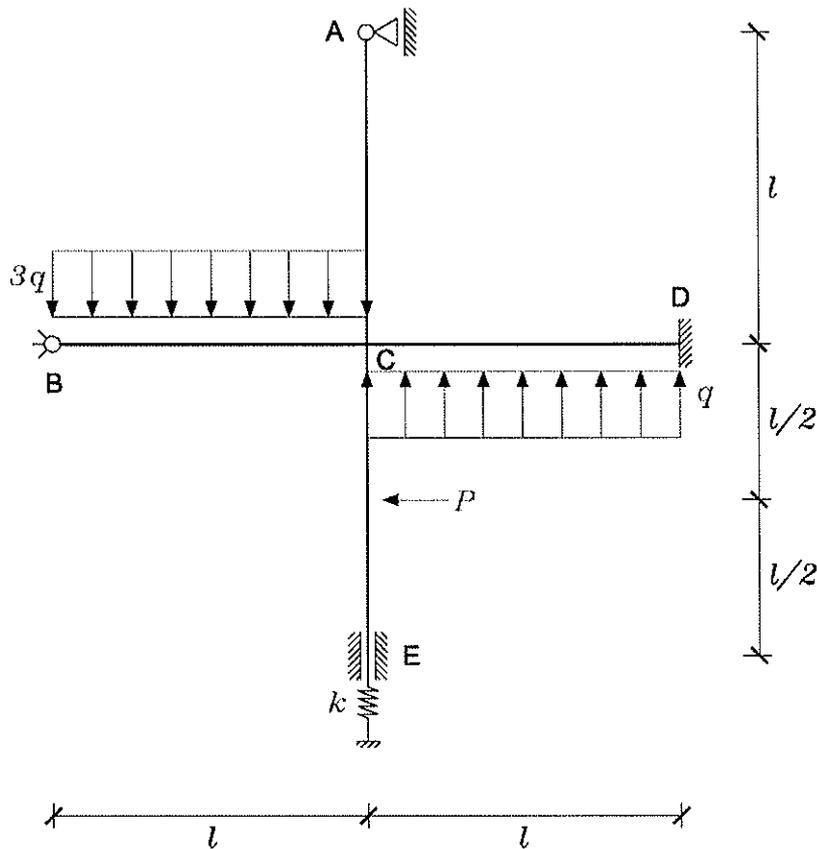
TECNICA DELLE COSTRUZIONI

TEMA ESAME DEL 02 APRILE 2012

DOCENTE: ING. FAUSTO MINELLI

ESERCITATORE: ING. FRANCESCA FEROLDI

Esercizio



$$P = ql$$

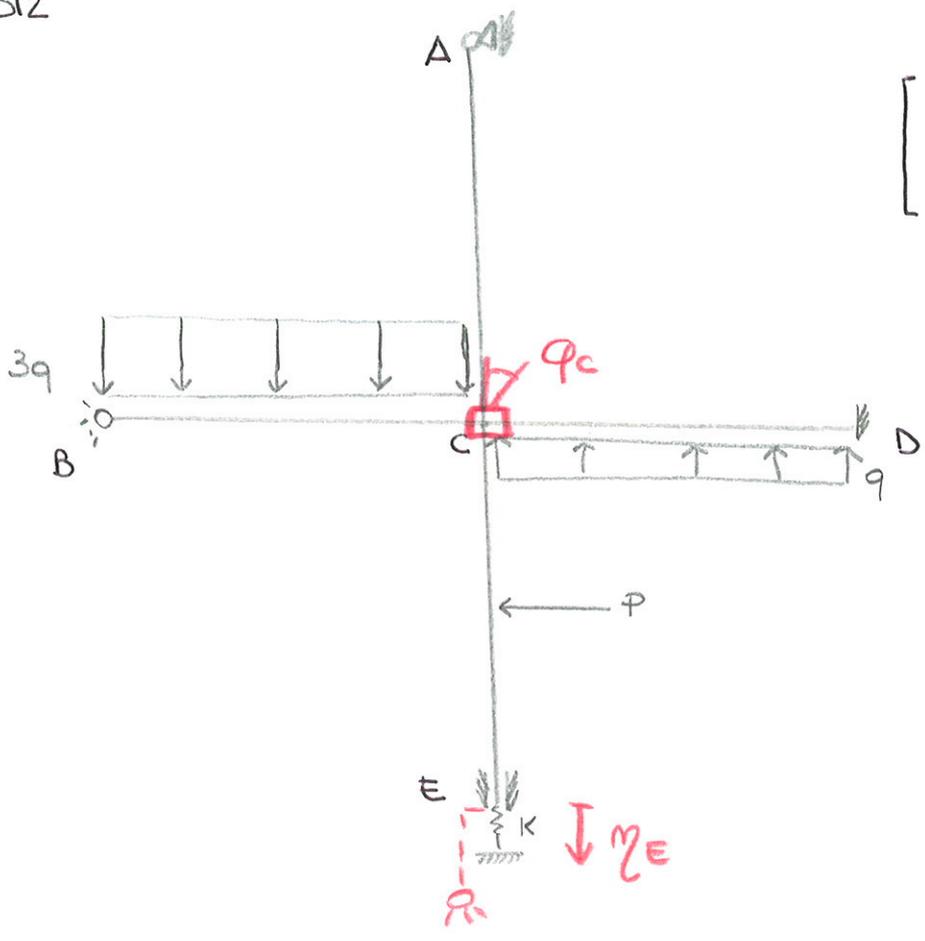
$$k = 3 \frac{EJ}{l^3}$$

Dato il telaio in figura

Si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.

$$\begin{cases} P = q \cdot l \\ k = \frac{3EJ}{e^3} \end{cases}$$

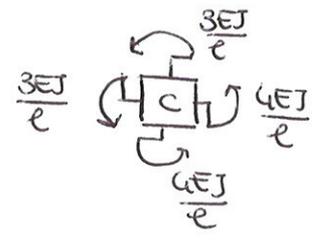
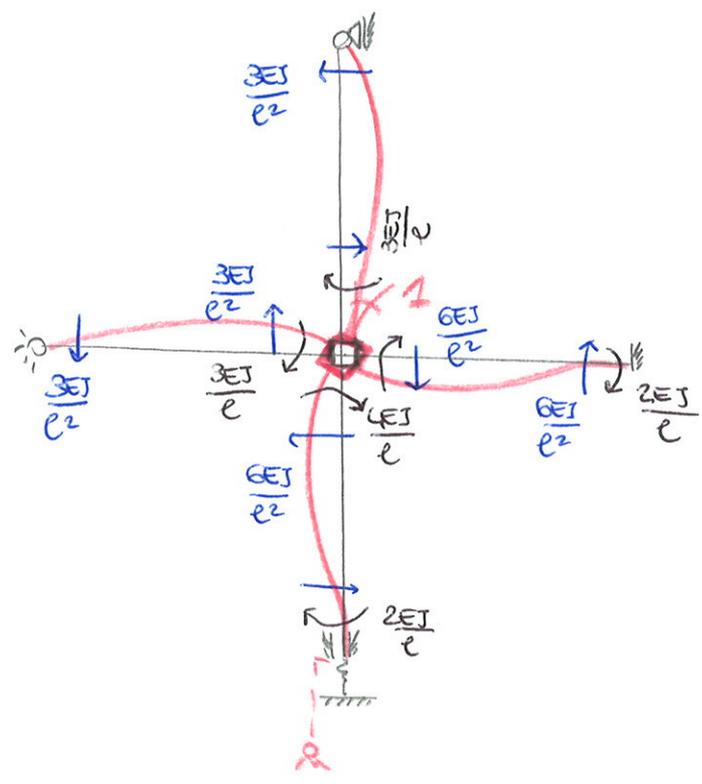


$$\begin{cases} \varphi_C \cdot M_{CC} + M_{CE} \cdot \varphi_E + M_{CD} = 0 \\ \varphi_C \cdot h_{EC} + \varphi_E \cdot h_{EE} + h_{ED} = 0 \end{cases}$$

Convenzioni di segno



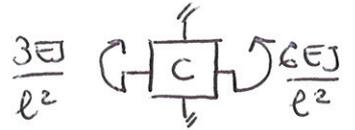
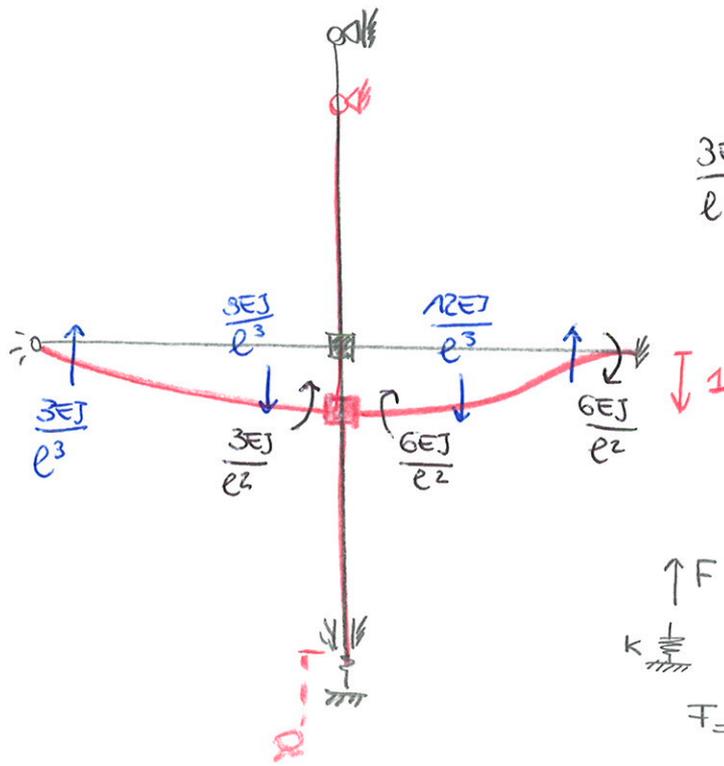
① $\varphi_C = 1$



$$M_{CC} = \frac{14EJ}{e}$$

$$h_{EC} = \frac{3EJ}{e^2}$$

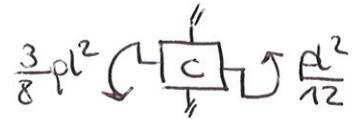
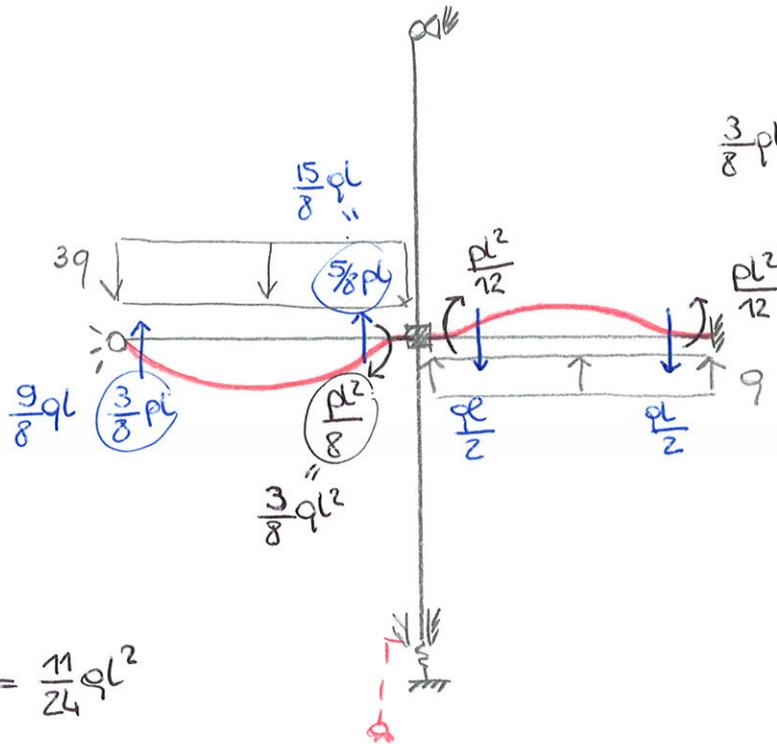
② $M=1$



$\uparrow F$
 $\downarrow 1$
 k
 $F = k \cdot \eta$
 $F = \frac{3EJ}{l^3}$

$M_{C0} = \frac{3EJ}{l^2}$
 $R_{M0} = \frac{15EJ}{l^3} + \frac{3EJ}{l^3} = \frac{18EJ}{l^3}$

③ $q \neq 0$

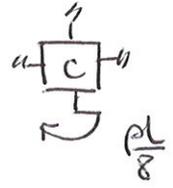
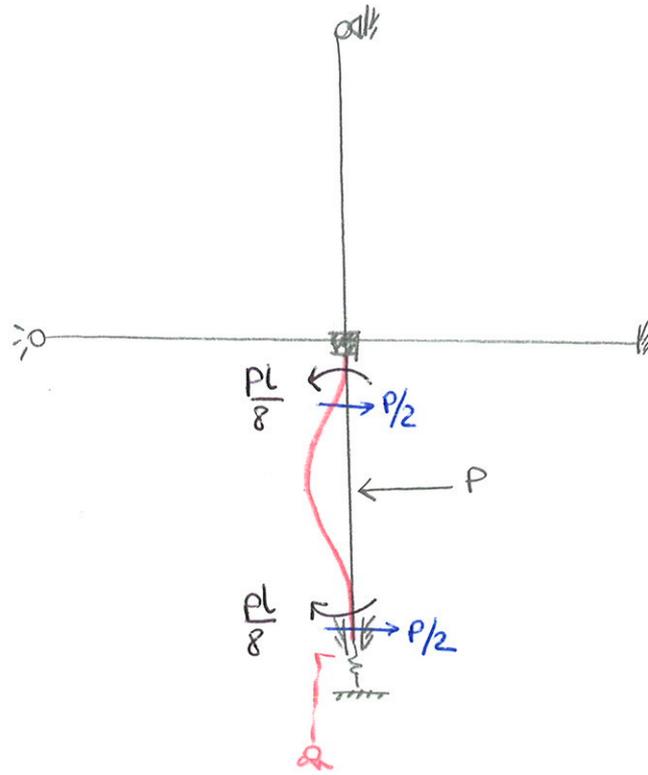


$\uparrow \frac{15}{8} ql$ $\downarrow \frac{1}{2} ql$
 $\frac{15-6}{8} = \frac{11}{8}$

$M_{C0} = \left(\frac{3}{8} + \frac{1}{12}\right) ql^2 = \frac{11}{24} ql^2$
 $R_{M0} = -\frac{11}{8} ql$

④ $P \neq 0$

②



$$\begin{cases} m_{co} = -\frac{Pl}{8} \\ R_{mp} = \varnothing \end{cases}$$

sistema risolvente

$$\begin{cases} \varphi_c \cdot \frac{14EJ}{l} + \eta \cdot \frac{3EJ}{l^2} + \frac{11}{24} ql^2 - \frac{Pl}{8} = 0 \\ \varphi_c \cdot \frac{3EJ}{l^2} + \eta \cdot \frac{18EJ}{l^3} - \frac{11}{8} ql = 0 \end{cases}$$

$$\frac{11}{24} ql^2 - \frac{Pl}{8}$$

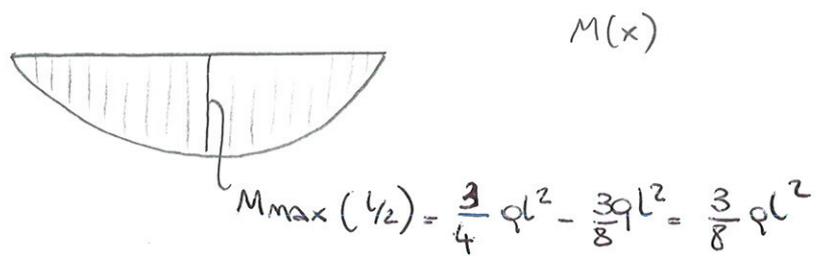
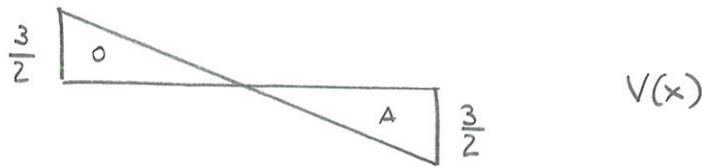
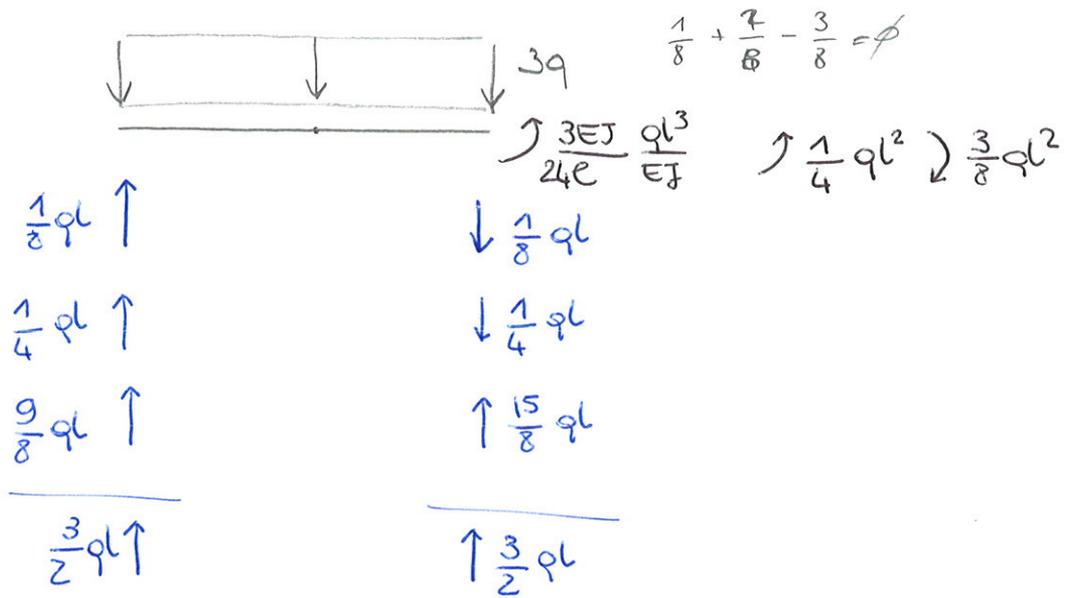
$$\left. \begin{array}{l} \\ \end{array} \right\} P = ql$$

$$\frac{11-3}{24} = \frac{8}{24} = \frac{1}{3}$$

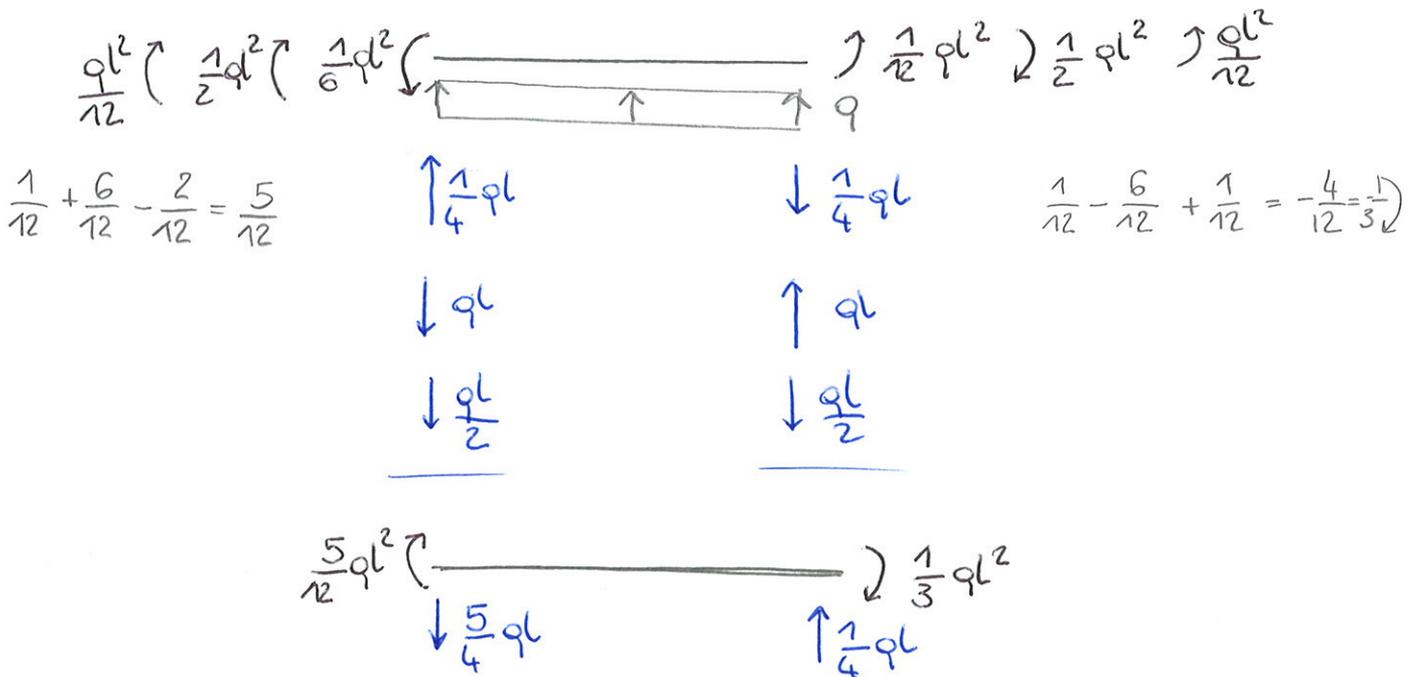
$$\begin{cases} \varphi_c \cdot \frac{14EJ}{l} + \eta \cdot \frac{3EJ}{l^2} + \frac{1}{3} ql^2 = 0 \\ \varphi_c \cdot \frac{3EJ}{l^2} + \eta \cdot \frac{18EJ}{l^3} - \frac{11}{8} ql = 0 \end{cases}$$

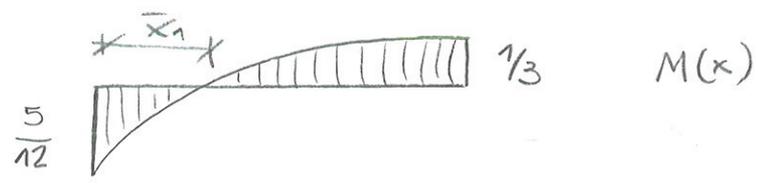
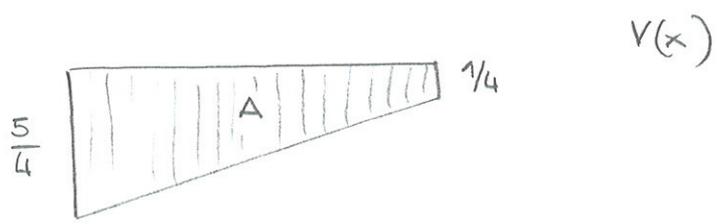
$$\begin{cases} \varphi_c = -\frac{1}{24} \frac{ql^3}{EJ} \\ \eta = \frac{1}{12} \frac{ql^4}{EJ} \end{cases}$$

Asta BC



Asta CD

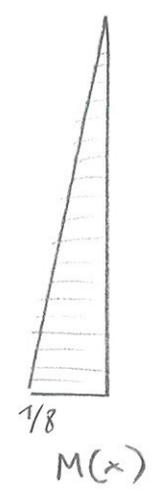
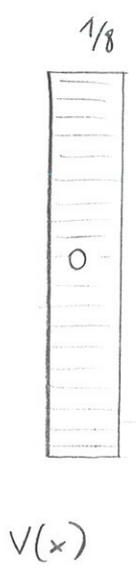
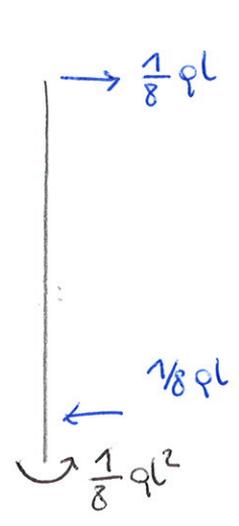




$M(\bar{x}) = 0 \rightarrow \frac{5}{12} ql^2 - \frac{5}{4} ql\bar{x} + q \frac{\bar{x}^2}{2} = 0$
 $x_{1,2} = \frac{5/4 \pm \sqrt{25/16 - 4 \cdot 1/2 \cdot 5/12}}{1} = 0,4l$

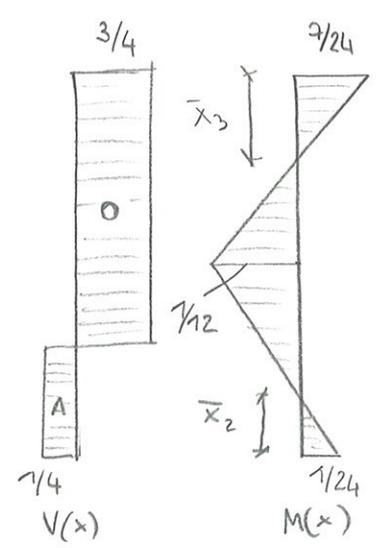
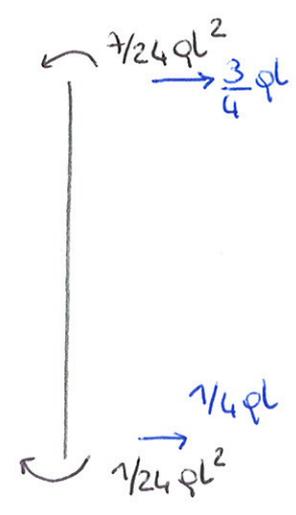
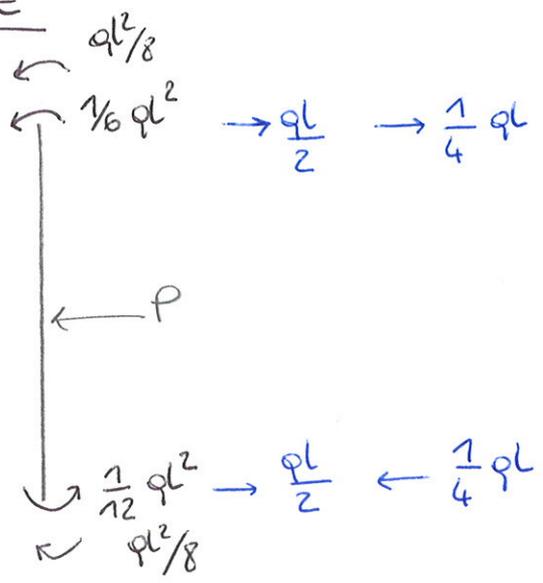
$\bar{x}_3 = ? \rightarrow \frac{7}{24} ql^2 = \frac{3}{4} ql\bar{x}_3$
 $\bar{x}_3 = \frac{7}{24} \cdot \frac{4}{3} l = \frac{7}{18} l \approx 0,38l$

Asta AC

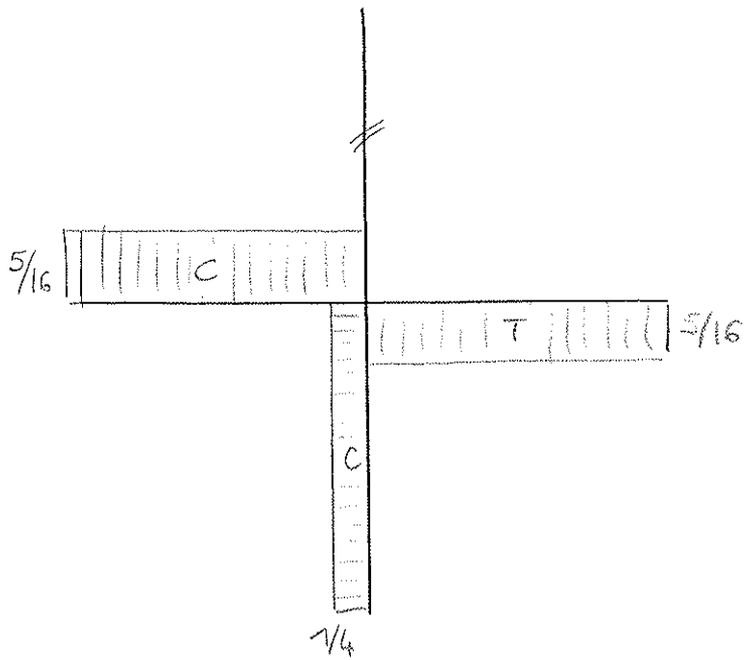


$\bar{x}_2 = ? \rightarrow \frac{1}{24} ql^2 - \frac{1}{4} ql\bar{x} = 0$
 $\frac{1}{4} ql\bar{x} = \frac{1}{24} ql^2$
 $\bar{x} = \frac{4}{24} l = 0,16l$

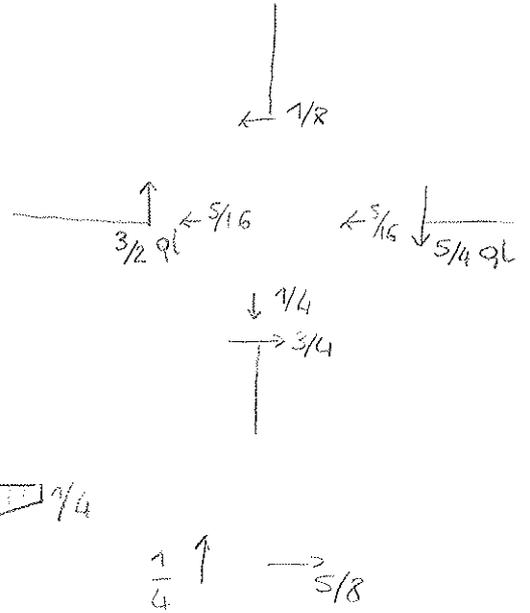
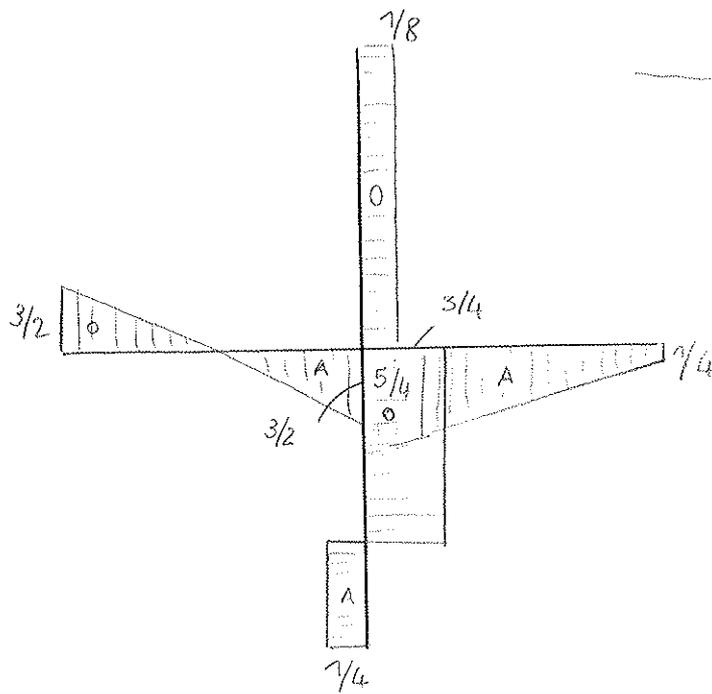
Asta CE



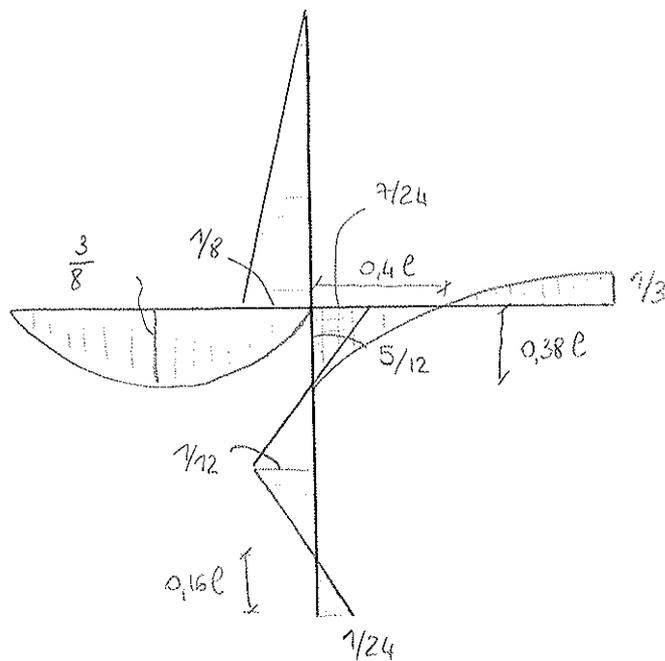
$$\frac{H}{ql}$$



$$\frac{V}{ql}$$

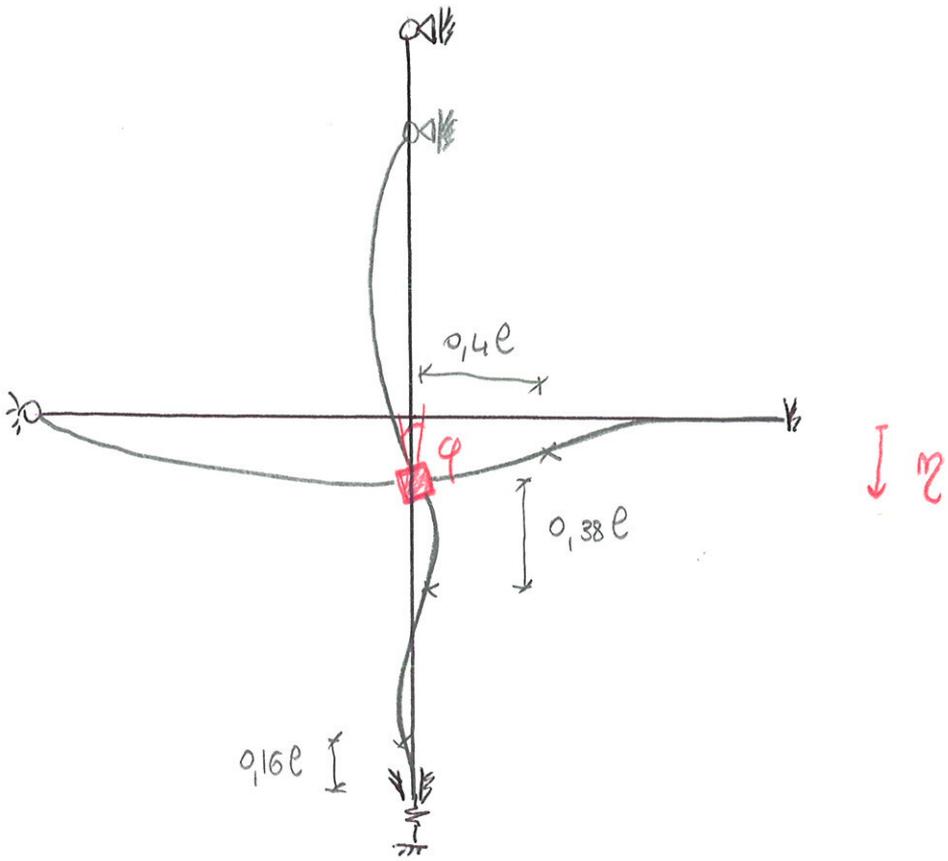


$$\frac{M}{ql^2}$$



Deformata

5



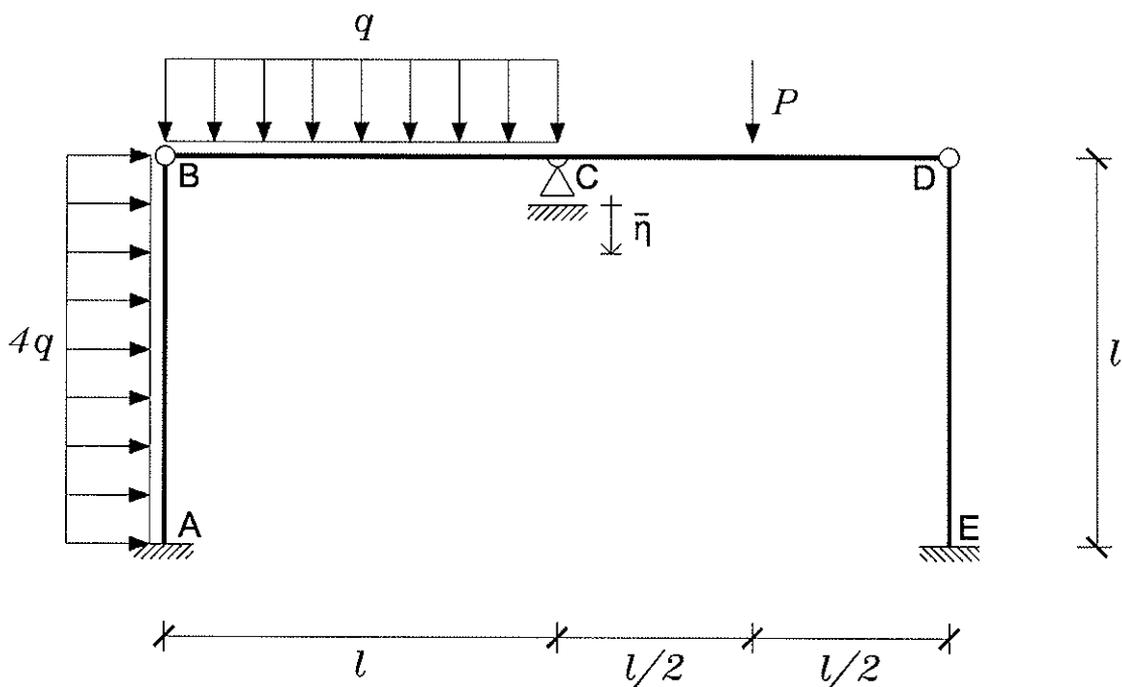
TECNICA DELLE COSTRUZIONI

PROVA IN ITINERE DEL 23 APRILE 2012

DOCENTE: PROF. GIOVANNI PLIZZARI

ESERCITATORE: ING. FAUSTO MINELLI

Esercizio



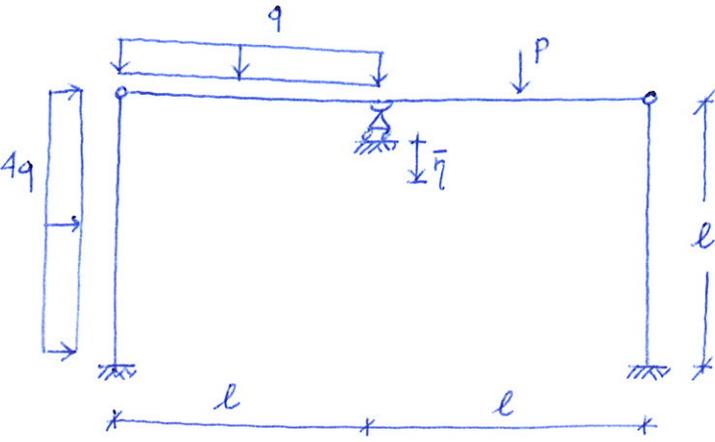
$$P = 2ql$$

$$\bar{\eta} = \frac{qL^4}{12EJ}$$

Dato il telaio in figura

Si richiedono i grafici di:

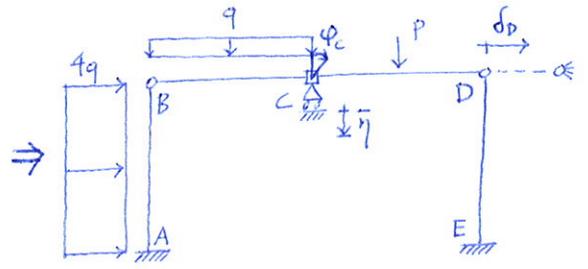
1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.



$$P = 2qL$$

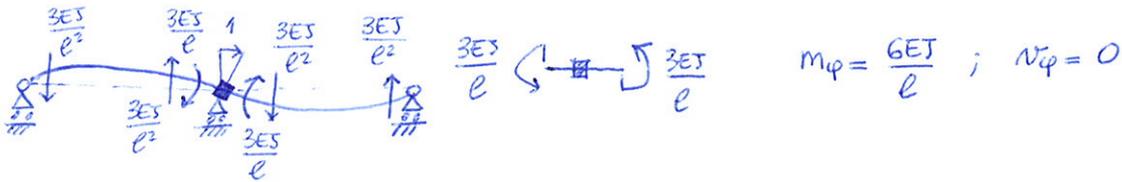
$$\bar{\eta} = \frac{qL^4}{12EJ}$$

$$\frac{\alpha \Delta T}{H} = \frac{qL^3}{EJ}$$

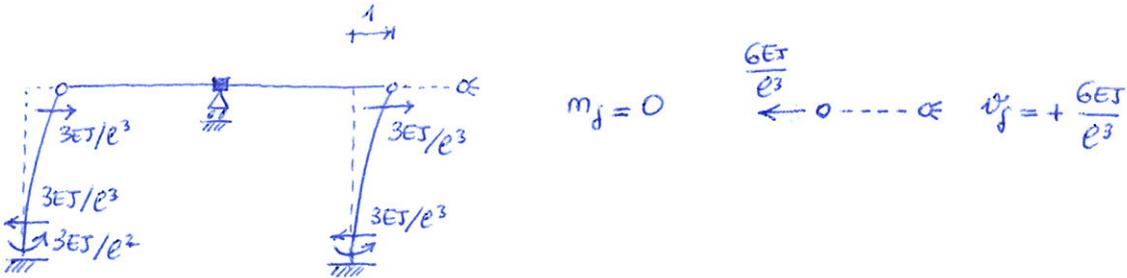


2 incognite nodali: φ_c, δ_D

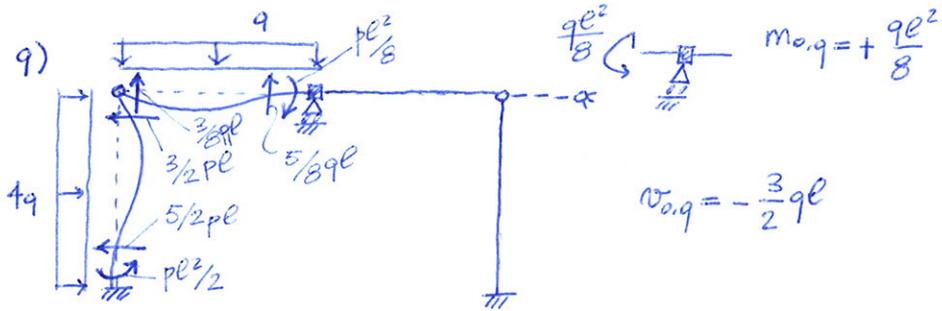
φ_c)



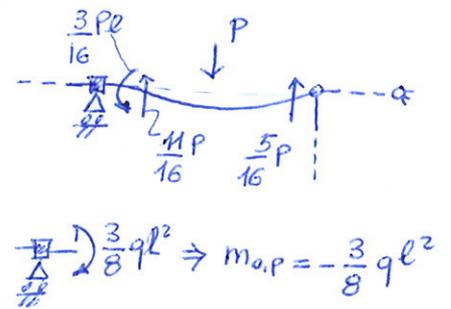
δ_D)



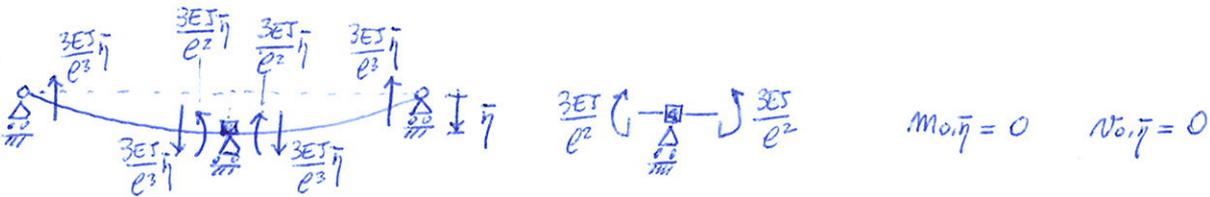
q)



P)



$\bar{\eta}$)

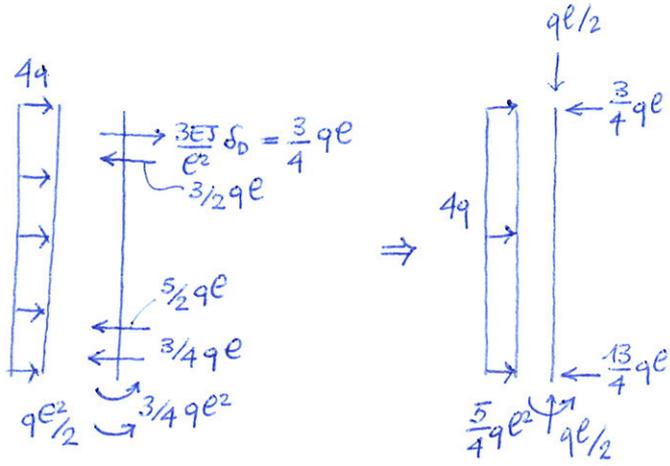


SISTEMA RISOLVENTE

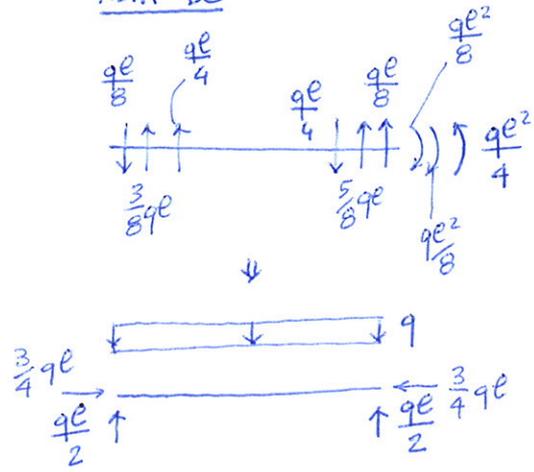
$$\frac{6EJ}{l} \varphi_c = -\frac{9e^2}{8} + \frac{3}{8} ql^2 \Rightarrow \frac{6EJ}{l} \varphi_c = \frac{9e^2}{4} \Rightarrow \varphi_c = \frac{9e^2}{24EJ}$$

$$\frac{6EJ}{e^3} \delta_D = \frac{3}{2} ql \Rightarrow \delta_D = \frac{9e^2}{4EJ}$$

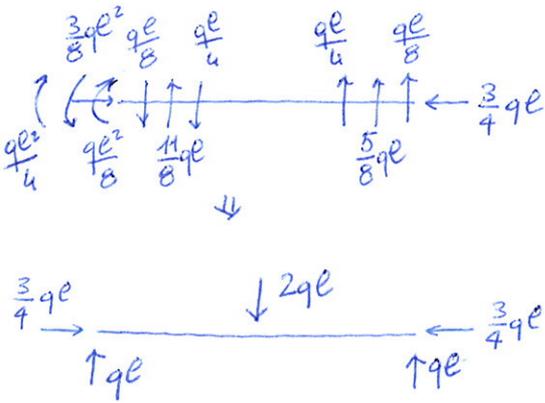
ASTA AB



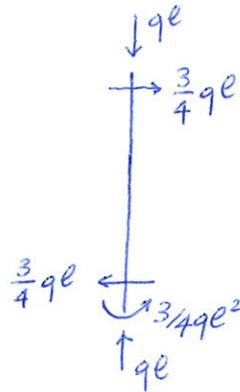
ASTA BC



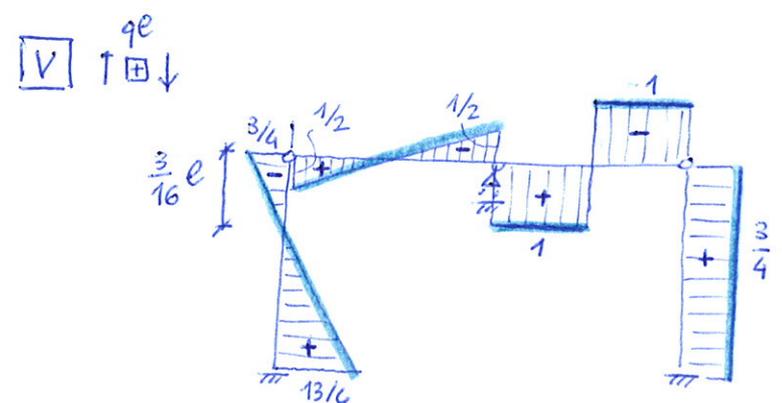
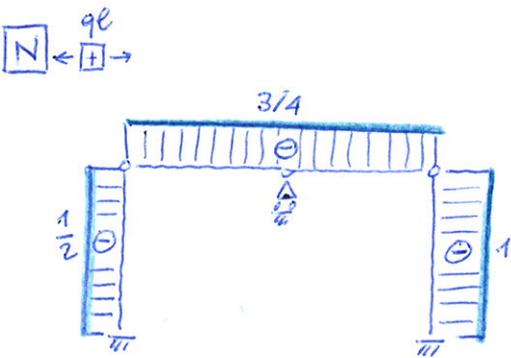
ASTA CD



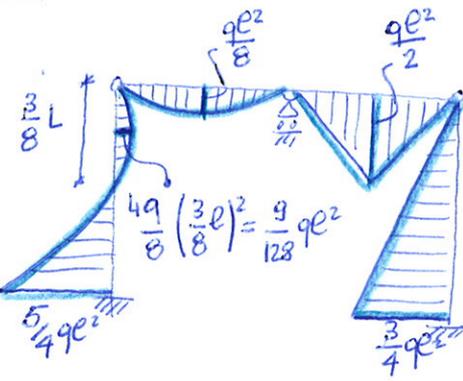
ASTA DE



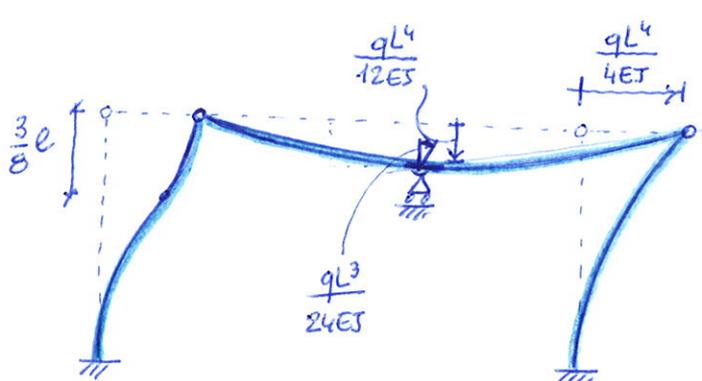
DIAGRAMMI AZIONI INTERNE



M



DEFORMATA QUALITATIVA



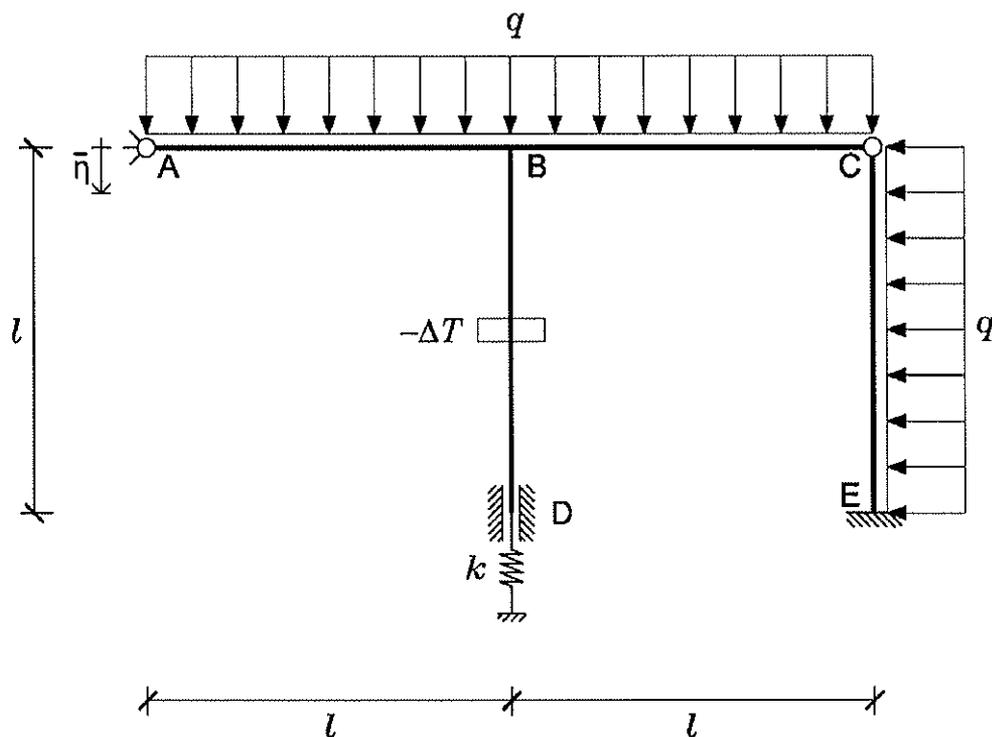
TECNICA DELLE COSTRUZIONI

APPELLO DEL 18 GIUGNO 2012

DOCENTE: PROF. GIOVANNI PLIZZARI

ESERCITATORE: ING. FAUSTO MINELLI

Esercizio



$$k = 4 \frac{EJ}{L^3}$$

$$\bar{\eta} = 2 \frac{qL^4}{EJ}$$

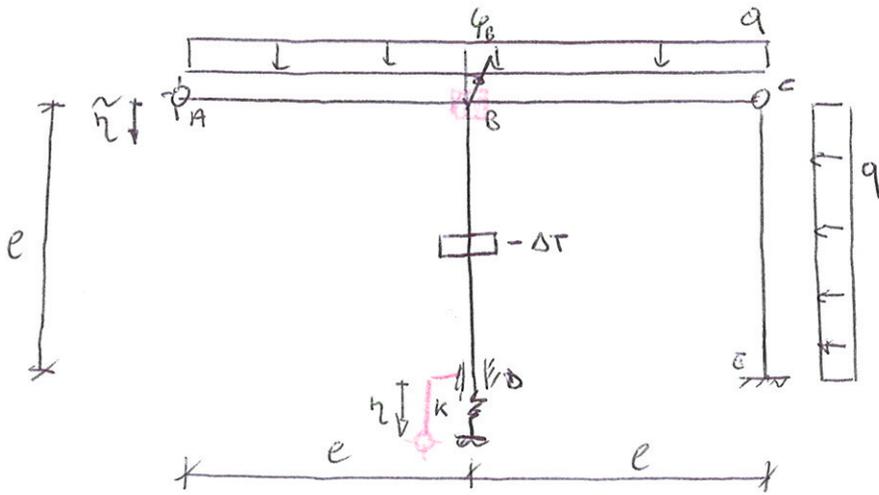
$$\alpha \Delta t = \frac{ql^3}{EJ}$$

Dato il telaio in figura

Si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.

T. E. 18/06/2012



$$K = 4 \frac{EI}{e^3}$$

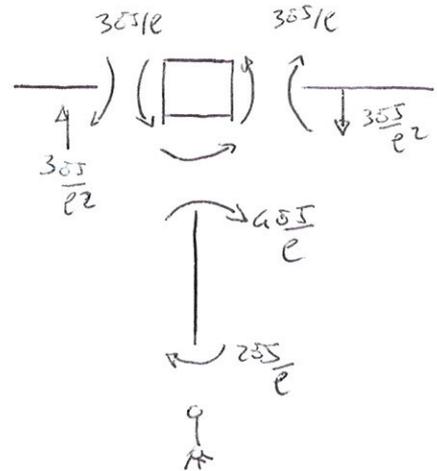
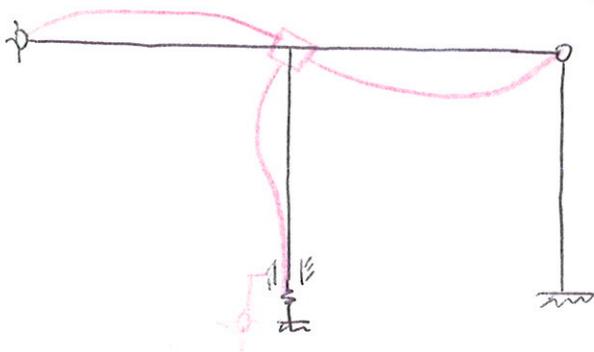
$$\tilde{h} = 2 \frac{qe^4}{EI}$$

$$\alpha \Delta T = \frac{qe^3}{EI}$$

+)

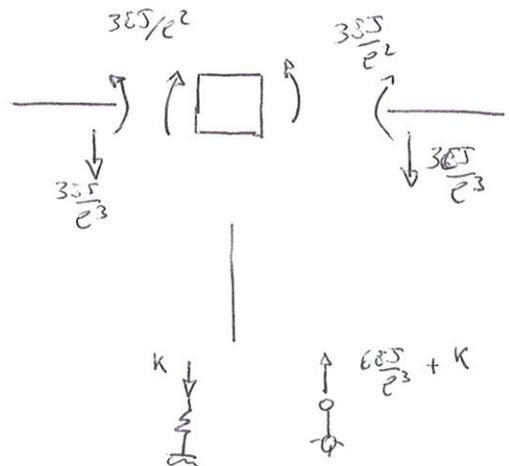
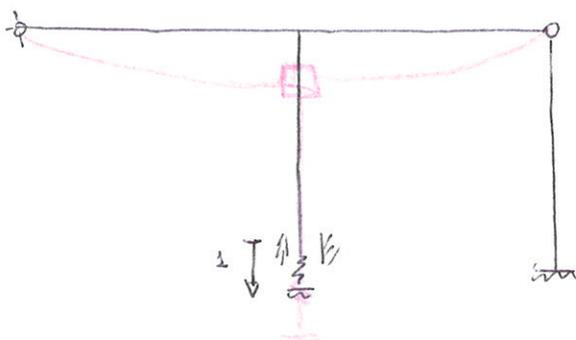
← 0 + 0 →

⊙ $\varphi_B = 1$ $h = q = \Delta T = \tilde{h} = 0$



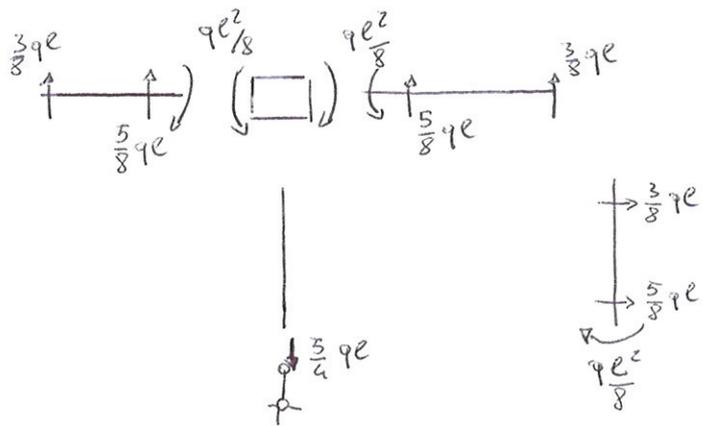
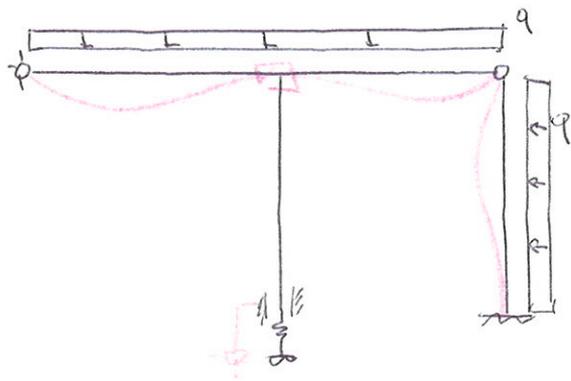
$$\begin{cases} m_\varphi = \frac{305}{e} + \frac{305}{e} + \frac{405}{e} = \frac{1005}{e} \\ h_h = \frac{305}{e^2} - \frac{305}{e^2} = 0 \end{cases}$$

⊙ $h = 1$ $\varphi_B = q = \Delta T = \tilde{h} = 0$



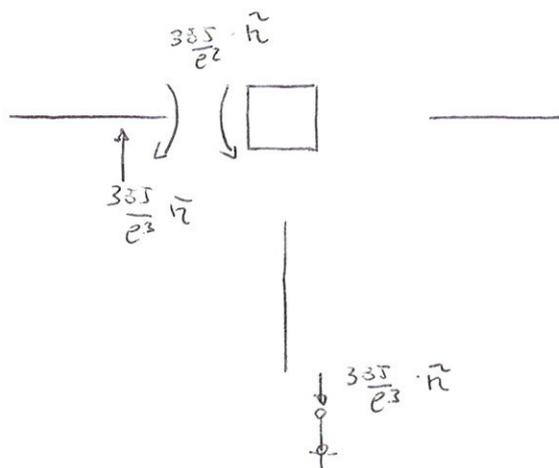
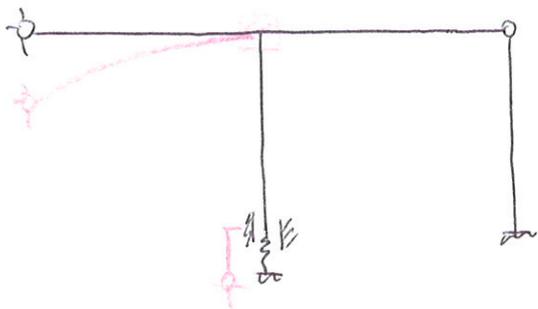
$$\begin{cases} m_h = 0 \\ h_h = \frac{305}{e^2} + \frac{305}{e^2} + K = \frac{605}{e^2} + \frac{405}{e^2} = \frac{1005}{e^2} \end{cases}$$

$\odot q \neq 0 \quad \varphi_B = \eta = \Delta T = \tilde{h} = 0$



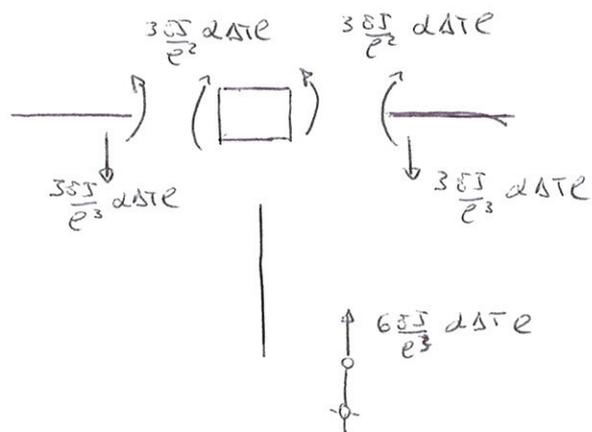
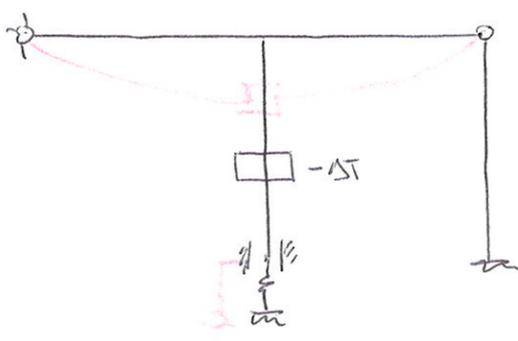
$$\begin{cases} m_{0,q} = \frac{qe^2}{8} - \frac{qe^2}{8} = 0 \\ h_{0,q} = -\frac{5}{8}qe - \frac{5}{8}qe = -\frac{5}{4}qe \end{cases}$$

$\odot \tilde{h} \neq 0 \quad \varphi_B = \eta = \Delta T = q = 0$



$$\begin{cases} m_{0,\tilde{h}} = \frac{335}{e^2} \cdot \tilde{h} = \frac{335}{e^2} \cdot \frac{2qe^4}{65} = 6qe^2 \\ h_{0,\tilde{h}} = -\frac{335}{e^3} \cdot \tilde{h} = -\frac{335}{e^3} \cdot \frac{2qe^4}{65} = -6qe \end{cases}$$

$\odot \Delta T \neq 0 \quad \varphi_B = \eta = q = \tilde{h} = 0$



$$\begin{cases} m_{0,\Delta T} = \frac{335}{e^2} \Delta T e - \frac{335}{e^2} \Delta T e = 0 \\ h_{0,\Delta T} = \frac{335}{e^3} \Delta T e + \frac{335}{e^3} \Delta T e = \frac{635}{e^3} \cdot \frac{qe^3}{65} \cdot e = 6qe \end{cases}$$

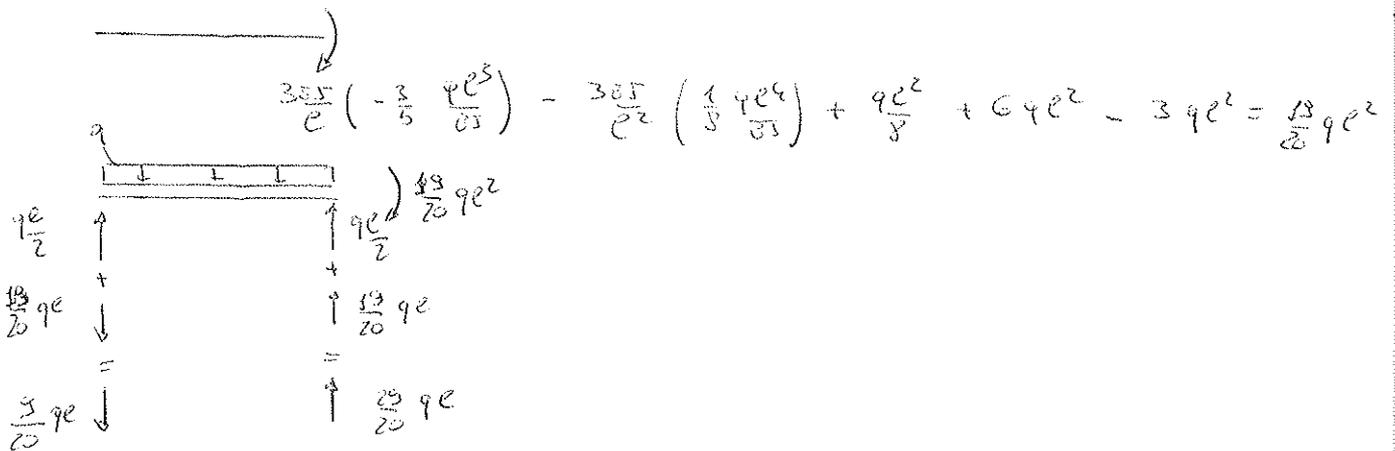
Se sistema risolvibile annulla;

$$\begin{cases} 10 \frac{85}{e} \cdot \varphi_B + 0 \eta + 6 q e^2 = 0 \\ 0 \varphi_B + 10 \frac{85}{e^2} \eta - \frac{5}{2} q e - 6 q e + 6 q e = 0 \end{cases}$$

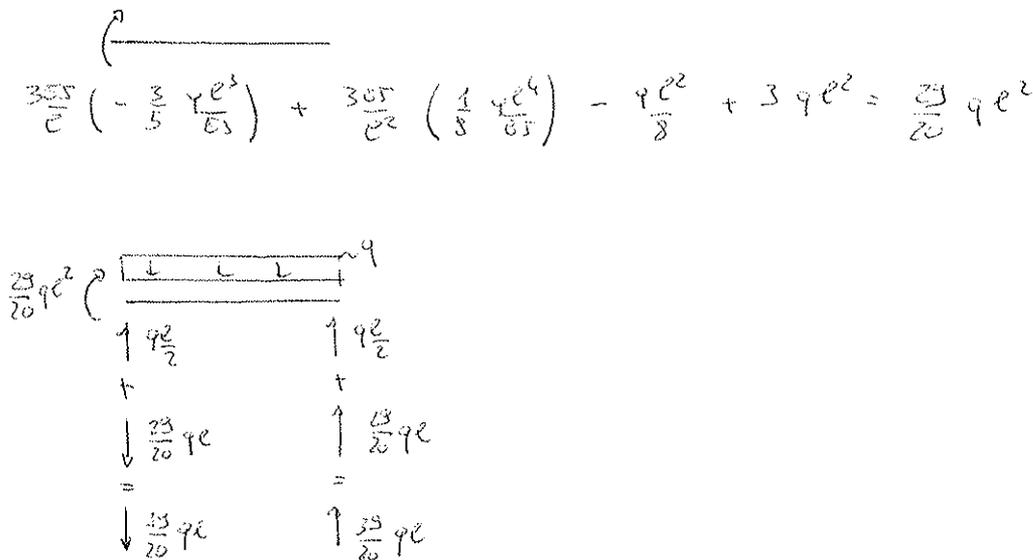
Risolvendo si ottiene:

$$\begin{cases} \varphi_B = - \frac{3}{5} \frac{q e^3}{85} \\ \eta = \frac{1}{8} \frac{q e^4}{85} \end{cases}$$

o Aste AB



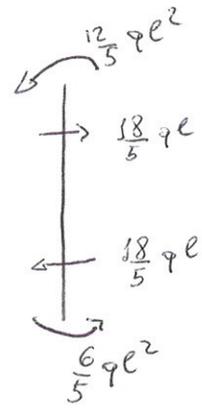
o Aste BC



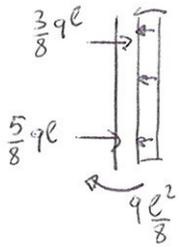
⊙ Asta BD

$$\frac{485}{e} \left(-\frac{3}{5} \frac{ql^3}{85} \right) = -\frac{12}{5} ql^2$$

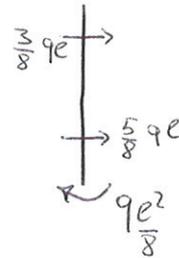
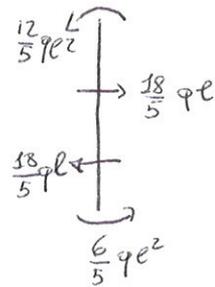
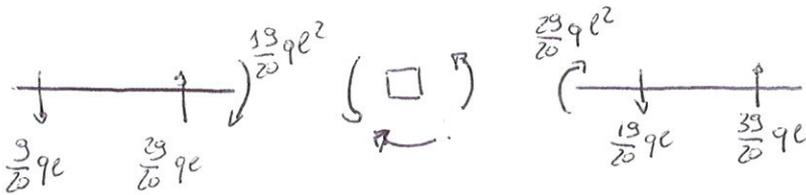
$$\frac{235}{e} \left(-\frac{3}{5} \frac{ql^3}{85} \right) = -\frac{6}{5} ql^2$$



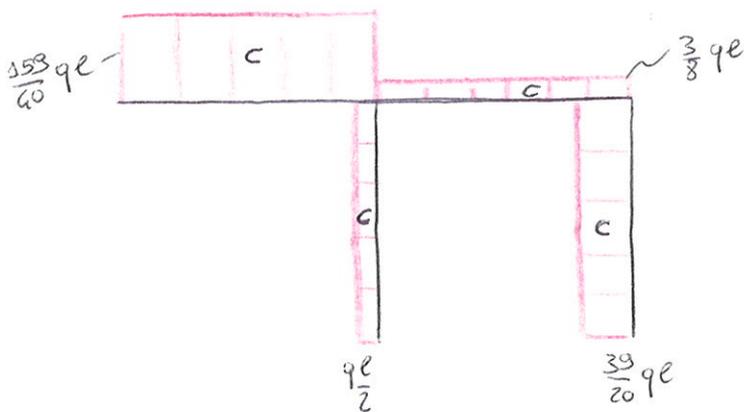
⊙ Asta c5



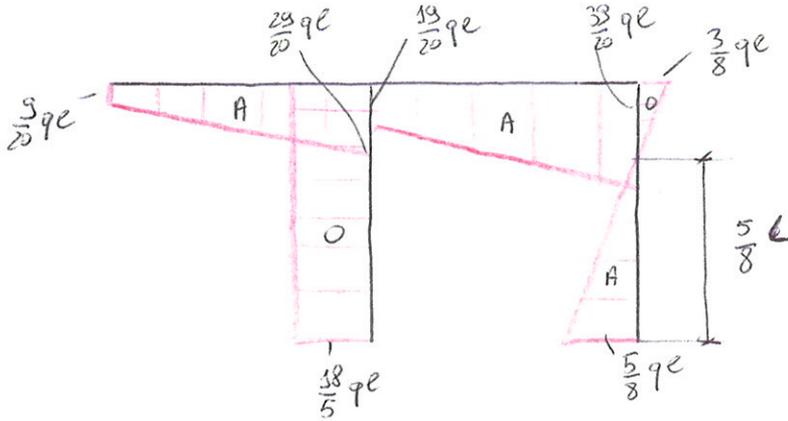
⊙ Asta mendo la singola asta e altre



⊙ Azione assiale N

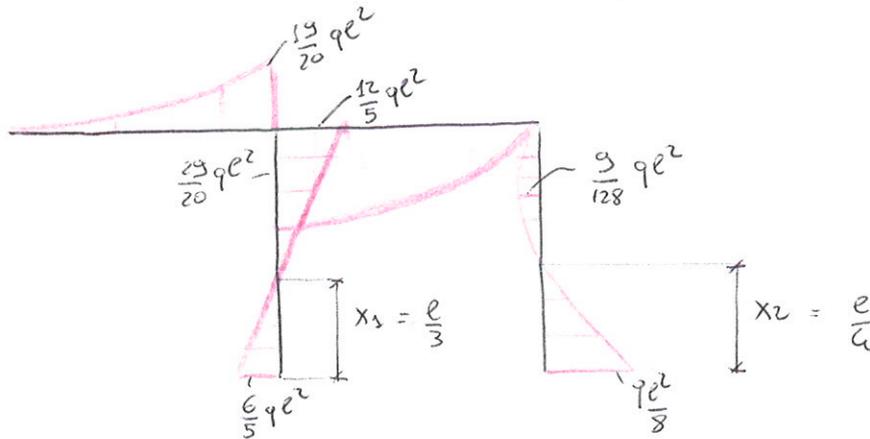


⊙ Reazione ✓



$$\pi\left(\frac{5}{8}e\right) = \frac{5}{8}q\ell \cdot \frac{5}{8}e - \frac{q\ell^2}{8} - q \cdot \frac{5}{8}e \cdot \frac{5}{16}e = \frac{9}{128}q\ell^2$$

⊙ Momento

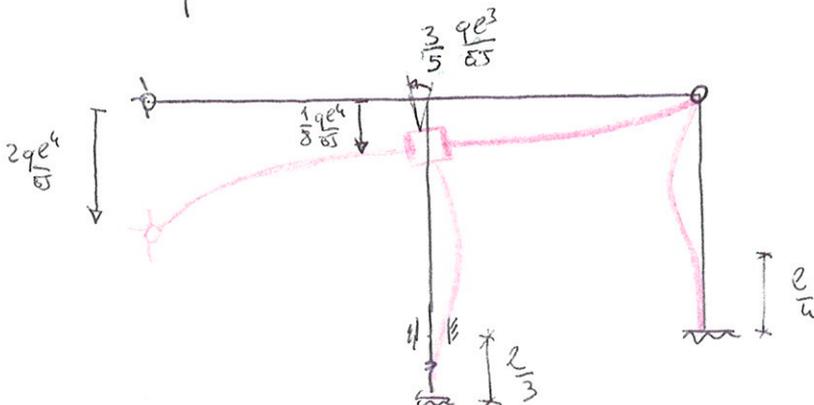


$$\frac{6}{5} : x_1 = \frac{18}{5} : L \Rightarrow x_1 = \frac{L}{3}$$

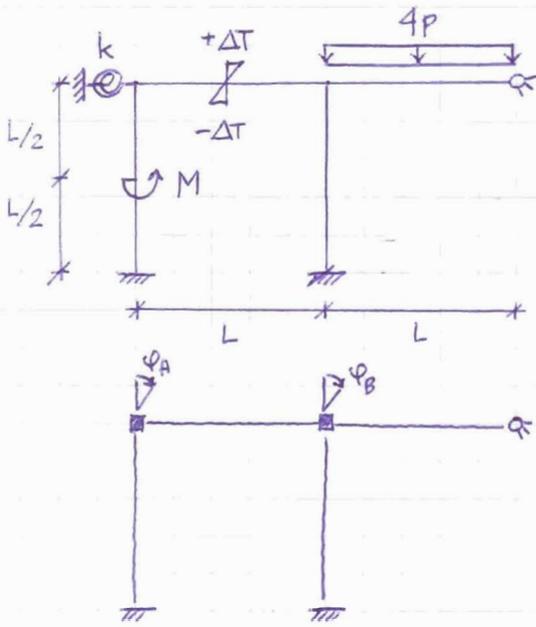
$$q\ell^2 \frac{x_2^2}{8} + q \frac{x_2^2}{2} - \frac{5}{8}q\ell x_2 = 0 ; x_2^2 - \frac{5}{4}\ell x_2 + \frac{\ell^2}{4} = 0$$

$$x_2 = \frac{\frac{5}{4}\ell \pm \sqrt{\frac{25}{16}\ell^2 - 1}}{2} \cdot \ell = \begin{cases} x_{2,1} = \ell & \text{no} \\ x_{2,2} = \frac{\ell}{4} & \text{si} \end{cases}$$

⊙ Deformate qual tecnica



TELAIO TECNICA delle COSTRUZIONI

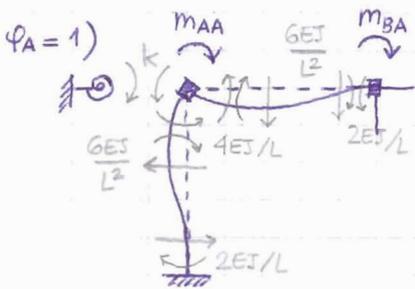


$$\frac{\alpha \Delta T E J}{h} = p l^2$$

$$M = 4 p l^2$$

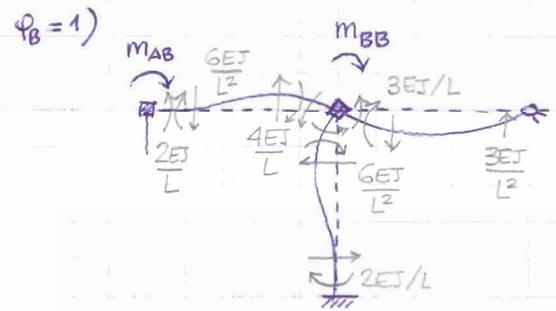
$$k = \frac{4 E J}{L}$$

Telaio 2 volte iperstatico.
 ⇒ BLOCCO I NODI A e B



$$m_{AA} = \frac{8EJ}{L} + k = \frac{12EJ}{L}$$

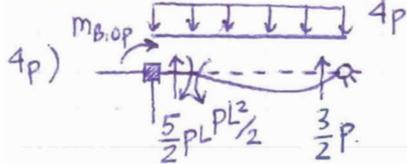
$$m_{BA} = \frac{2EJ}{L}$$



$$m_{AB} = \frac{2EJ}{L}$$

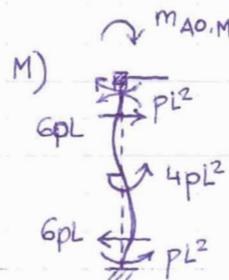
$$m_{BB} = \frac{11EJ}{L}$$

CARICHI:



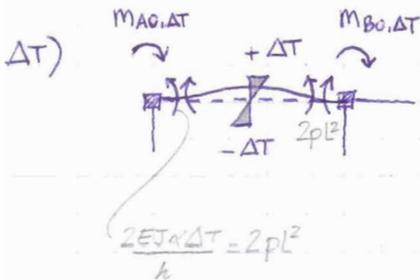
$$m_{A0,p} = 0$$

$$m_{B0,p} = -\frac{pL^2}{2}$$



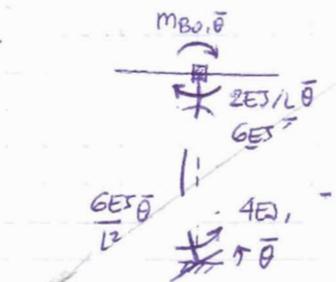
$$m_{A0,M} = -pL^2$$

$$m_{B0,M} = 0$$



$$m_{A0,\Delta T} = 2pL^2$$

$$m_{B0,\Delta T} = -2pL^2$$



$$m_{A0,\theta} = 0$$

$$m_{B0,\theta} = -4pL^2$$

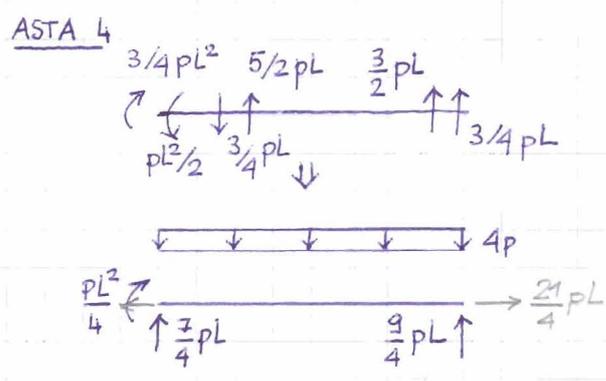
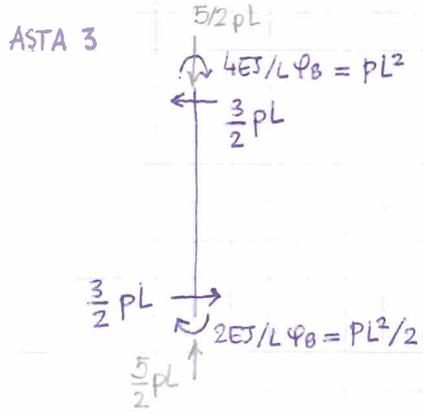
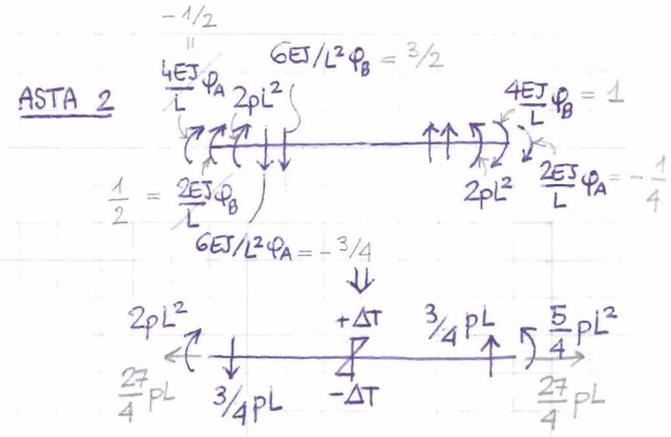
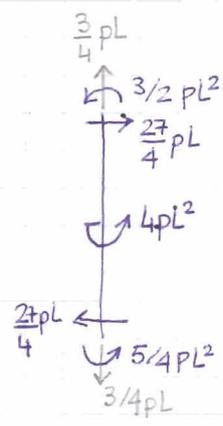
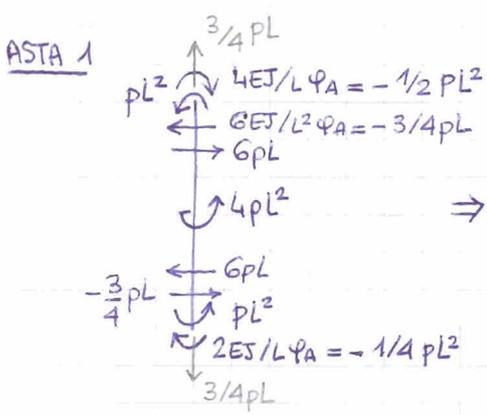
SISTEMA RISOLVENTE

$$\begin{bmatrix} \frac{12EJ}{L} & \frac{2EJ}{L} \\ \frac{2EJ}{L} & \frac{11EJ}{L} \end{bmatrix} \begin{bmatrix} \varphi_A \\ \varphi_B \end{bmatrix} = \begin{bmatrix} -pL^2 \\ +\frac{5}{2}pL^2 \end{bmatrix} \Rightarrow \varphi_B = -6\varphi_A - \frac{pL^3}{2EJ}$$

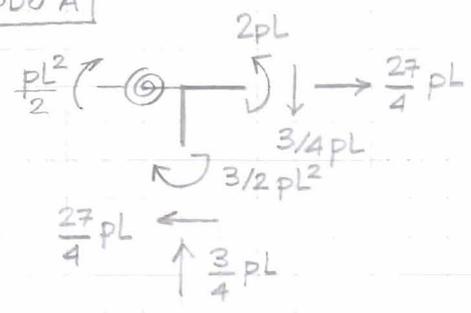
$$+ \frac{2EJ}{L} \varphi_A - 6 \frac{EJ}{L} \varphi_A - \frac{11}{2} pL^2 = \frac{5}{2} pL^2 \Rightarrow -6 \frac{EJ}{L} \varphi_A = 8 pL^2 \Rightarrow \varphi_A = -\frac{1}{8} \cdot \frac{pL^3}{EJ}$$

$$\varphi_B = \frac{1}{4} \cdot \frac{pL^3}{EJ}$$

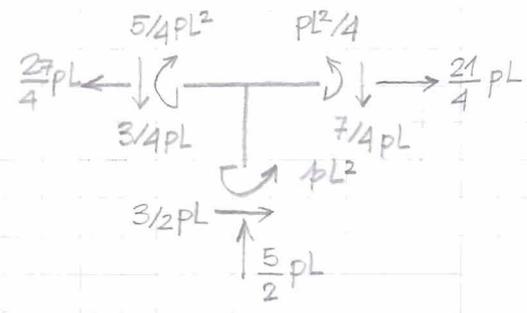
AZIONI INTERNE ASTA PER ASTA



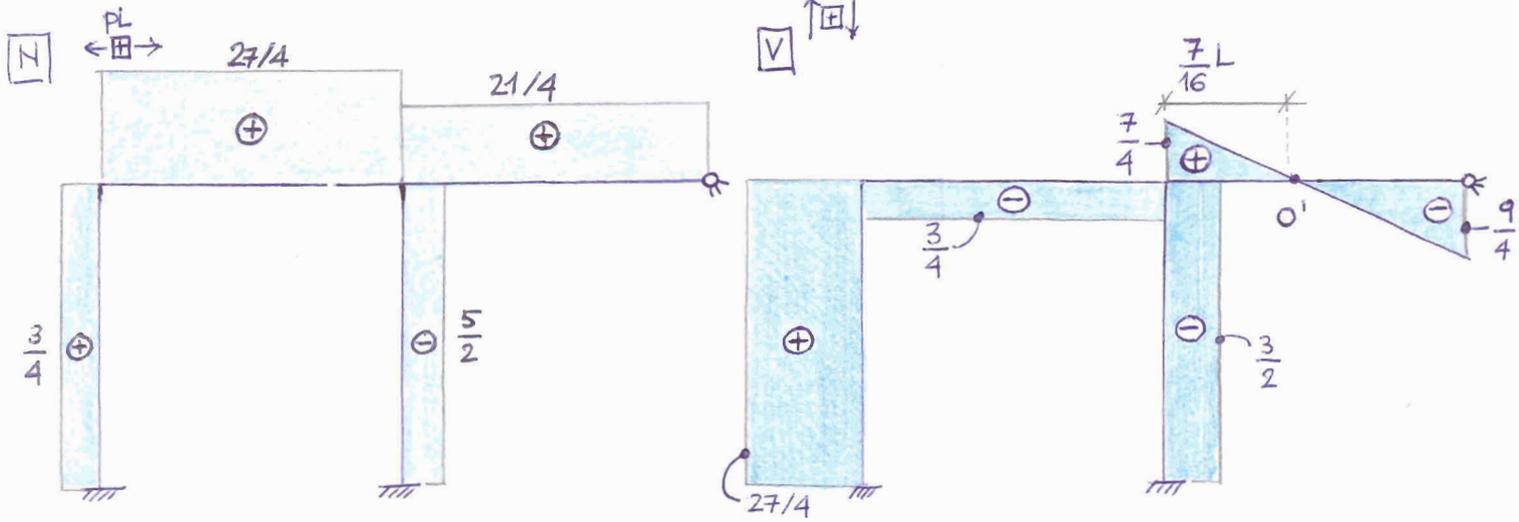
NODO A



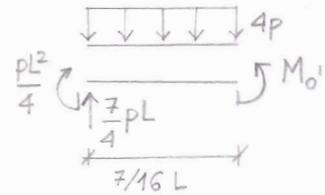
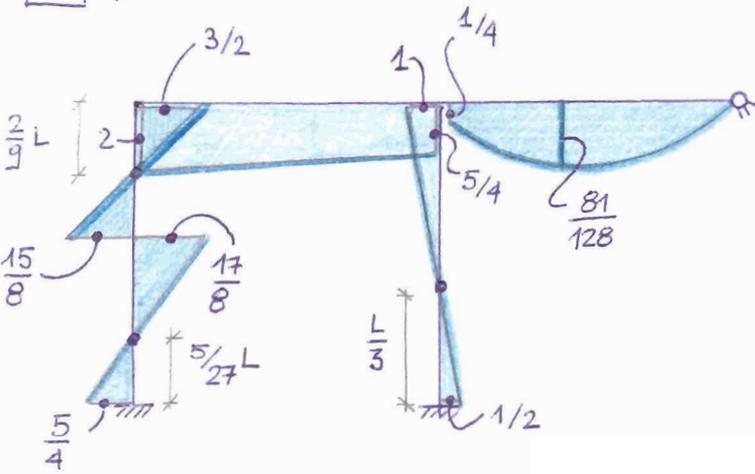
NODO B



DIAGRAMMI DELLE AZIONI INTERNE



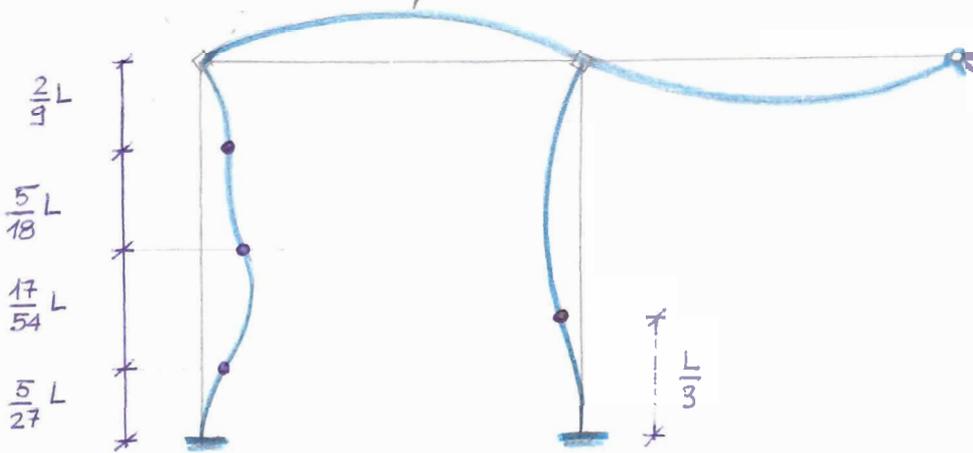
Moment Diagram (M)



$$M_{o'} = \frac{pL^2}{4} + \frac{7}{4} pL \cdot \frac{7L}{16} - 4p \frac{7L}{16} \cdot \frac{7L}{32} = \frac{pL^2}{4} + \frac{49}{64} pL^2 - \frac{49}{128} pL^2 = \frac{81}{128} pL^2$$

DEFORMATA QUALITATIVA

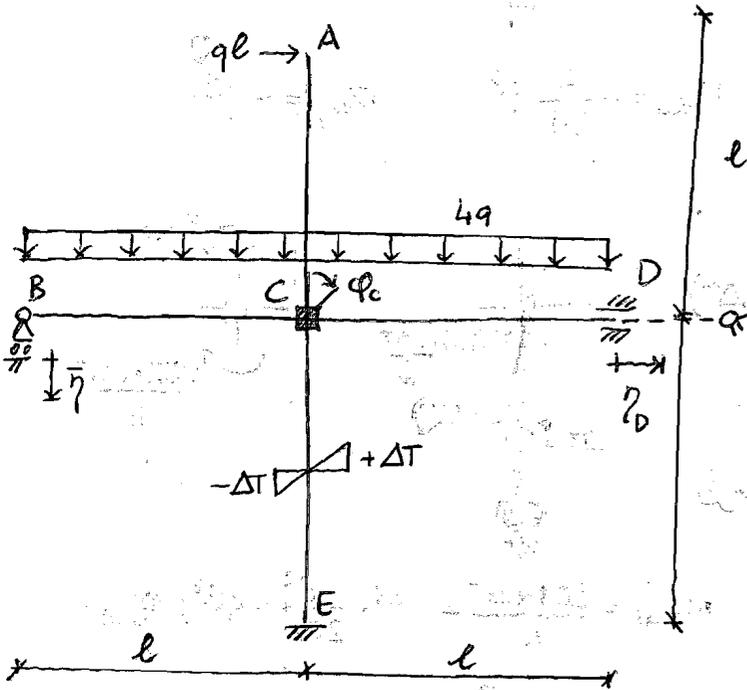
LA CURVATURA È COSÌ A CAUSA DELLA DEFORMAZIONE TERMICA (NONOSTANTE IL MOMENTO TENDA ALLE FIBRE DI SOTTO)



$$y'' = \frac{-M}{EI} + \frac{2\alpha\Delta T}{H}$$

$$-\frac{1}{EI} (-M + 2pL^2) > 0$$

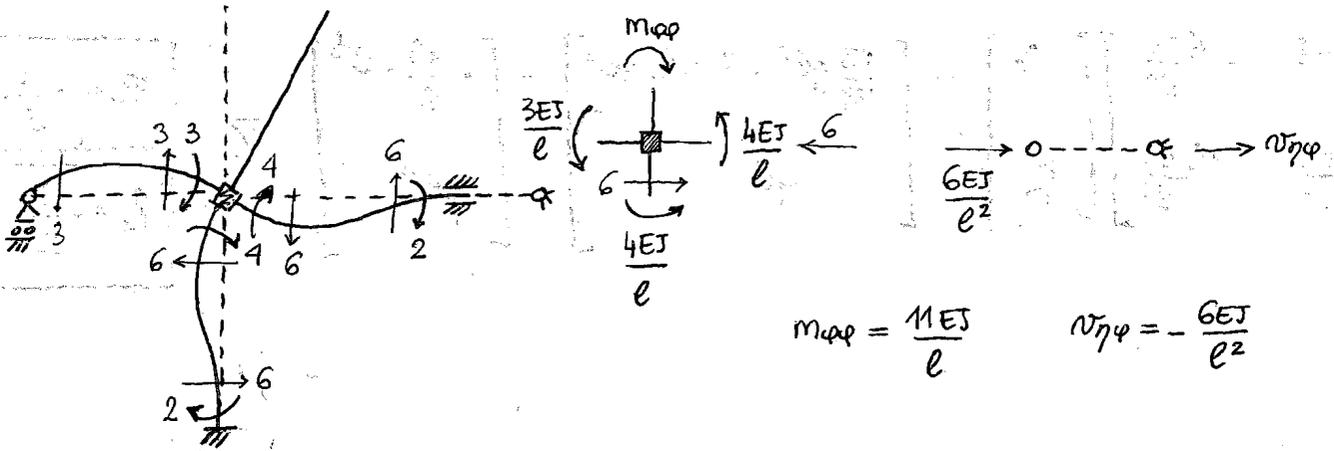
$$\frac{5}{4} pL^2 < M < 2pL^2$$



$$\frac{\alpha \Delta T}{h} = \frac{q e^2}{2EJ}$$

$$\bar{\eta} = \frac{q e^4}{EJ}$$

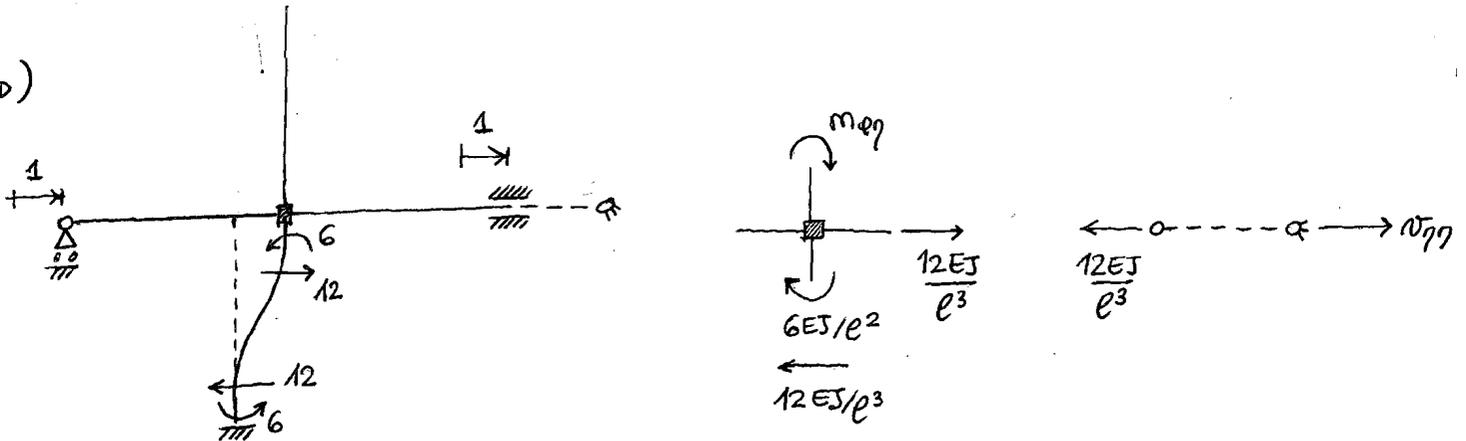
φ_e)



$$m_{\phi e} = \frac{11EJ}{e}$$

$$v_{\eta \phi} = -\frac{6EJ}{e^2}$$

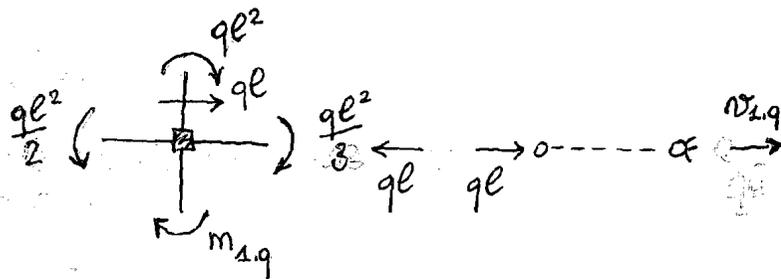
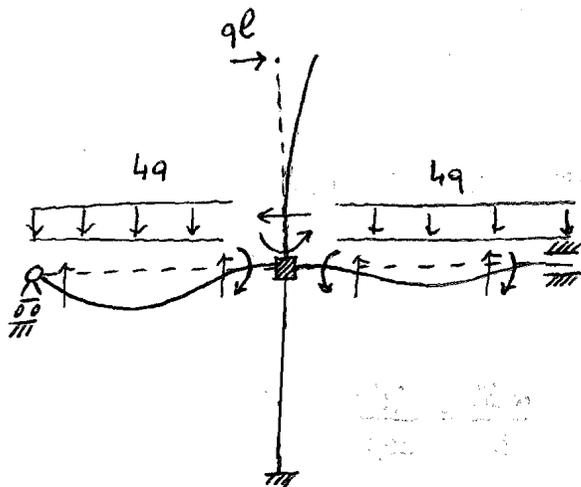
η_D)



$$m_{\phi \eta} = -\frac{6EJ}{e^2}$$

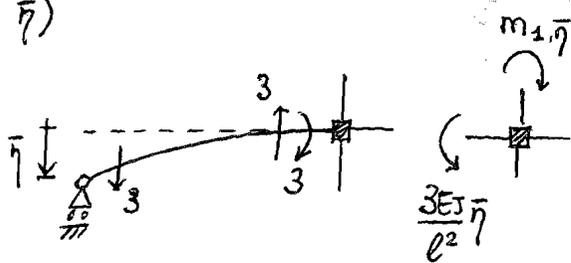
$$v_{\eta \eta} = \frac{12EJ}{e^3}$$

9)



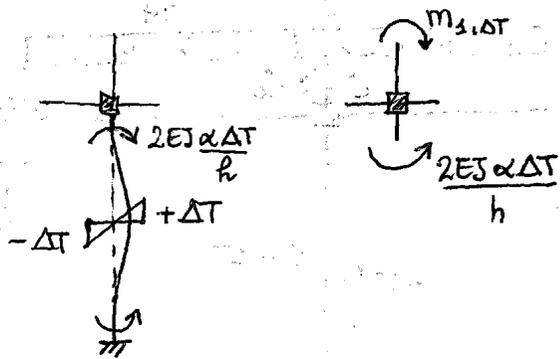
$$m_{1,q} = -\frac{5}{6} q l^2 \quad \sigma_{1,q} = -q l$$

7)



$$m_{1,\bar{\eta}} = \frac{3EJ}{l^2} \cdot \frac{q l^4}{EJ} = 3q l^2 \quad \sigma_{1,\bar{\eta}} = 0$$

ΔT)

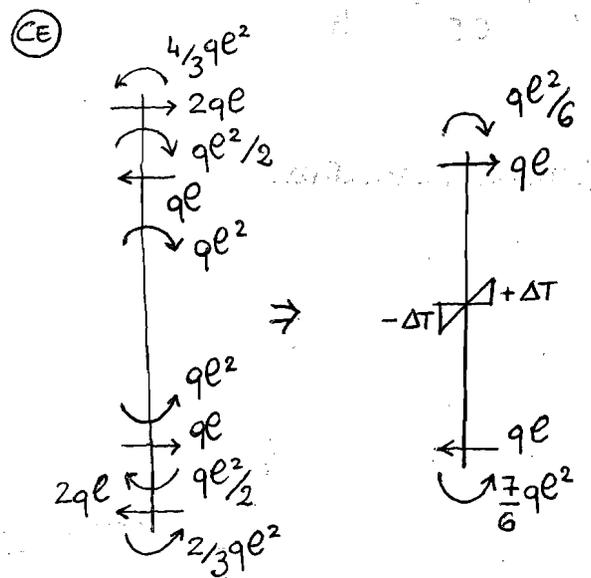
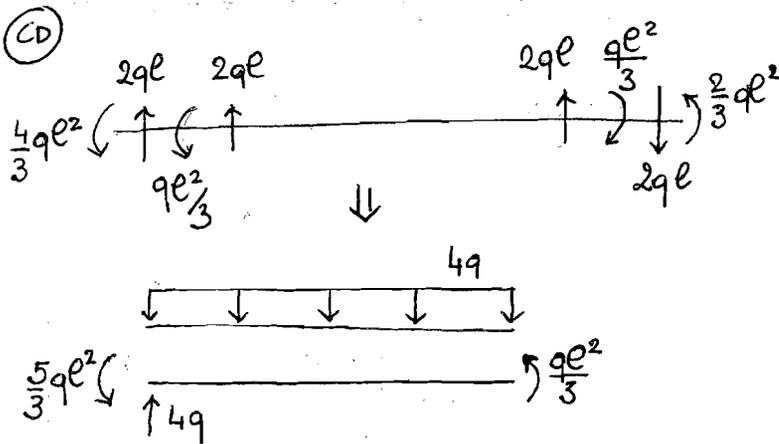
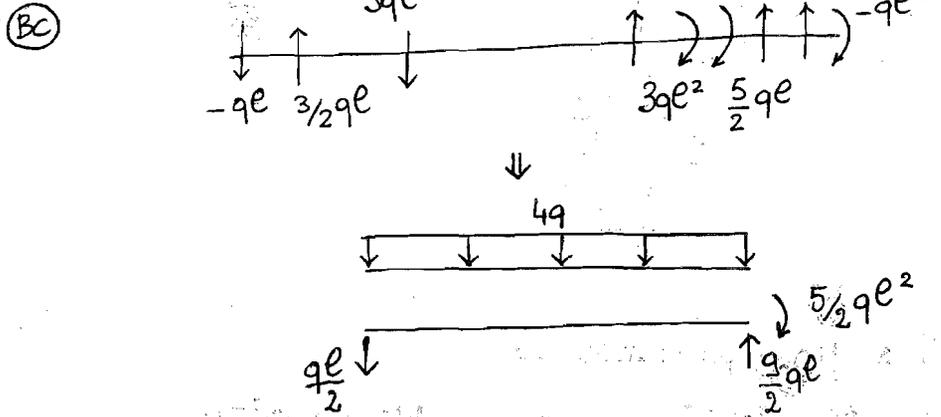
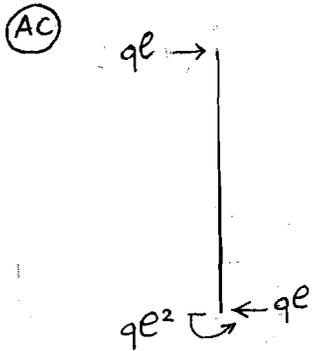


$$m_{1,\Delta T} = \frac{2EJ \alpha \Delta T}{h} = 2EJ \cdot \frac{q l^2}{2EJ} = q l^2; \quad \sigma_{1,\Delta T} = 0$$

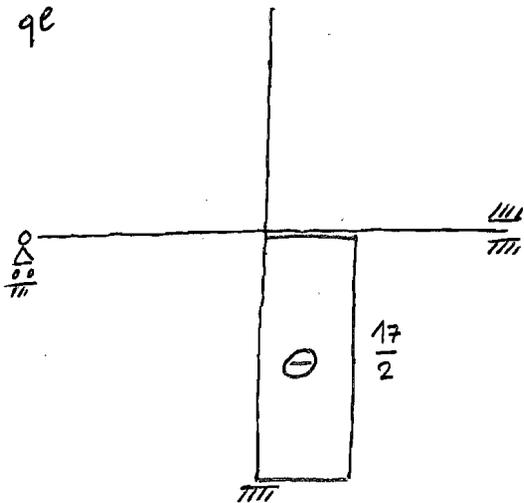
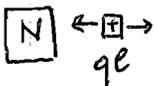
SISTEMA RISOLVENTE

$$\begin{bmatrix} \frac{11 EJ}{l} & -\frac{6 EJ}{l^2} \\ -\frac{6 EJ}{l^2} & \frac{12 EJ}{l^3} \end{bmatrix} \begin{bmatrix} \varphi_c \\ \eta_0 \end{bmatrix} = \begin{bmatrix} \frac{5}{6} q l^2 - 3q l^2 - q l^2 \\ q l \end{bmatrix} = \begin{bmatrix} -\frac{19}{6} q l^2 \\ q l \end{bmatrix} \Rightarrow \begin{cases} \varphi_c = -\frac{1}{3} \cdot \frac{q l^3}{EJ} \\ \eta_0 = -\frac{1}{12} \cdot \frac{q l^4}{EJ} \end{cases}$$

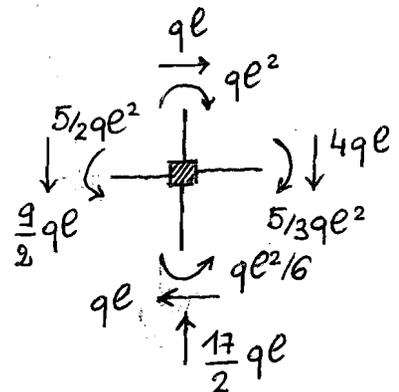
ASTA PER ASTA



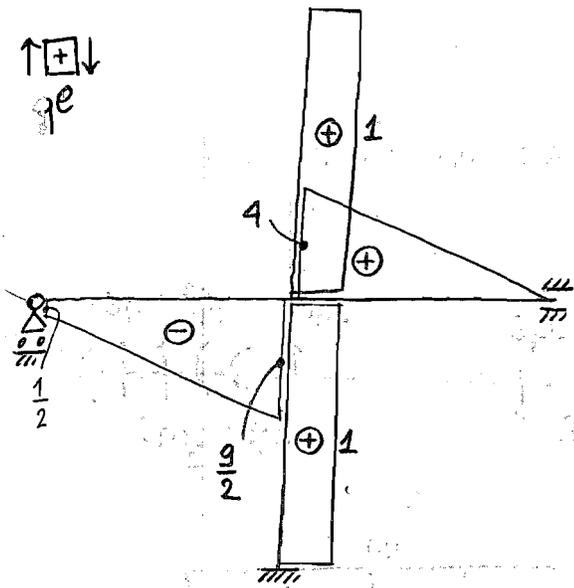
DIAGRAMMI AZIONI INTERNE



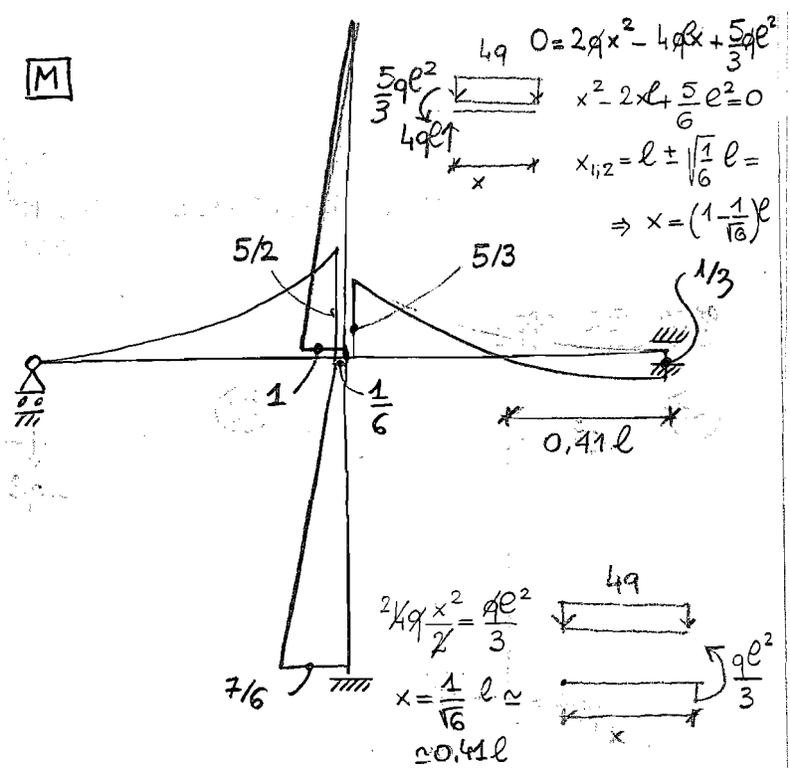
NOTA: EQUILIBRIO AL NODO C



V $\uparrow + \downarrow$
 qe



M



Punti di flesso per l'asta CE

$$y'' = -\frac{M}{ES} + \frac{2\alpha\Delta T}{h} \Rightarrow y''(x) = \frac{-M(x) + qe^2}{ES} = \frac{1}{ES} \left(\frac{qe^2}{6} + qlx - qe^2 \right) = \frac{1}{ES} \left(\frac{5}{6}qe^2 - qlx \right)$$

$$\Rightarrow y'' = 0 \text{ per } x = \frac{5}{6}e$$

Deformata qualitativa.

